Wind turbine lifetime extension decision-making based on structural health monitoring

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Abstract

In this work, structural health monitoring data is applied to underpin a long-term wind farm lifetime extension strategy. Based on the outcome of the technical analysis, the case for an extended lifetime of 15 years is argued. Having established the lifetime extension strategy, the single wind turbine investigated within a wind farm is subjected to a bespoke economic lifetime extension case study. In this case study, the local wind resource is taken into consideration, paired with central, optimistic, and pessimistic operational cost assumptions. Besides a deterministic approach, a stochastic analysis is carried out based on Monte Carlo simulations of selected scenarios. Findings reveal the economic potential to operate profitably in a subsidy-free environment with a P90 levelised cost of energy of £25.02 if no component replacement is required within the nacelle and £42.53 for a complete replacement of blades, generator, and gearbox.

Keywords: structural health monitoring; wind turbine; lifetime extension; fatigue analysis; remaining useful lifetime; levelised cost of energy

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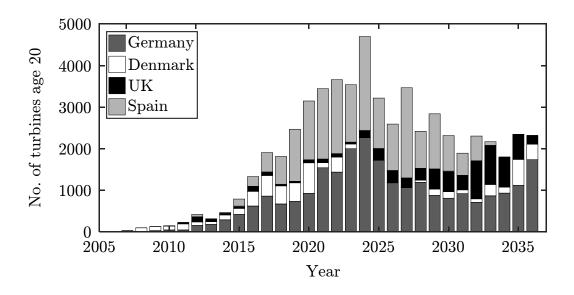


Figure 1: Turbines reaching end of design life by year [1].

1 1. Introduction

As highlighted by Ziegler et al. [1] in Figure 1, an increasing number of 2 wind turbine generators (WTG) are reaching their end of design life. For 3 this growing share of WTGs, a justification for lifetime extension may be 4 based on different operational metrics such as: (i) the site classification, for 5 example a turbine designed for a class II site but operated in a class III, 6 (ii) the level of downtime, (iii) the lifetime energy production, (iv) sufficient 7 design reserves, (v) if components are replaced during the design lifetime, 8 and (vi) any combination of the above. 9

The main advantages of lifetime extension are: (i) the ability to increase the return on investment, with significantly less resources than required in repowering scenarios, (ii) utilise assets until the end of life cycle, thus preventing premature dismantling as well as (iii) using readily available local infrastructure (grid connection, access routes, community ties).

It has been proposed that structural health monitoring (SHM) may play
an important role in supporting the process of lifetime extension (LTE)
decision-making in order to reduce uncertainty of a turbine's site specific
loading or if components are considered critical based on inspections [1, 2, 3].
Therefore, this paper applies SHM data from an operational wind turbine to develop an LTE strategy. Subsequently, the proposed strategy is con-

sidered jointly with operational data and subjected to an economic decisionmaking methodology developed by Rubert et al. [4, 5]. In order to consider
uncertainties in the bespoke wind turbine economic model, uncertainty bands
are applied in cost and mean annual energy production. In addition a Monte
Carlo simulation is executed for selected scenarios.

The remainder of this paper is structured as follows. Section 2 compares 26 SHM activities with other forms of analysis to support the lifetime exten-27 sion decision-making, presents a review of wind turbine tower and founda-28 tion SHM research, and the results from the SHM measurement campaign. 20 Section 3 presents the applied LTE decision-making methodology whilst In 30 Section 4, the case study is presented where a strategy is derived and eco-31 nomic input parameters presented. Results of the case study are presented in 32 Section 5, followed by a discussion of the key findings in Section 6. Finally, 33 conclusions outlining the key findings are presented in Section 7. 34

35 2. SHM for Lifetime Extension

Lifetime extension decision-making can be based on (i) data analysis, 36 (ii) inspections, (iii) aero-elastic simulations, and (iv) gathered data through 37 SHM systems. Inspections generate an in-depth assessment of structure's 38 early failure indicators. However, inspections are only valid for a certain 39 period. As such, frequent assessment is necessary in either 6 or 12 months 40 intervals, thus lacking the ability to support the long-term business case 41 evaluation. Data analysis using SCADA is observed with caution, as the 42 information is often lacking temporally detailed operational history. Aero-43 elastic simulations may generate a detailed analysis; however, simulations 44 require operational data that might have significant uncertainties, if e.g., 45 taken from SCADA data. Additionally, aero-elastic simulations are generally 46 costly to carry out. 47

SHM concepts have the ability to provide long-term and in-depth data
that can be applied to generate the long-term business case, while delivering
a reduced uncertainty in the evaluation.

2.1. Literature Review of SHM Concepts for Wind Turbine Towers and Foundations

With regards to tower sensor installation and data assessment practices, the reader is referred to Smarsly et al. [6] for a 500 kW wind turbine, Rebelo et al. [7, 8] for a 2.1 MW wind turbine, Loraux and Brühwiler [9] for a 2 MW wind turbine, and Botz et al. [10] for a 3 MW hybrid turbine consisting of a concrete and steel tower section. The 2 MW wind turbine tower fatigue analysis results in a remaining useful lifetime (RUL) of 135 years in a low mean wind speed region (5.9 m/s) [9].

Related SHM concepts of onshore wind turbine foundations are available 60 by Currie et al. [11, 12] aimed at monitoring the displacement between the 61 tower and foundation. Based upon this work, Bai et al. [13] evaluate sensors 62 embedded in concrete blocks, to monitor the displacement and crack devel-63 opment at the bottom of the inserted can flange that area is prone to failure 64 initiation. In this project, empty steel tubes are further vertically inserted 65 in the foundation, facilitating horizontal ultrasonic testing, to identify the 66 structural integrity with height. In addition, Perry et al. [14] and McAlorum 67 et al. [15] present a short and long term crack monitoring solution of wind 68 turbine foundations, whereas Rubert et al. [16] demonstrate a field case 60 study of embedding optical strain gauges in reinforced concrete foundations. 70 The interested reader is referred to [17, 18, 19] for a general review of 71 SHM opportunities, failures, and inspection practices of wind turbines. 72

73 2.2. SHM Campaign

The WTG of focus is a multi-MW, individual pitch regulated, onshore 74 generator located in Scotland. Due to confidentiality reasons, the type, man-75 ufacturer, and rated power are not disclosed. In addition, all presented data 76 is normalized or the axis labels and tics are removed. The overall SHM 77 installation process, characterisation, temperature compensation, and valid-78 ation is detailed in Ref. [20]. In comparison to other tower RUL assessments, 79 in this work, a turbine with a greater mean wind speed (> 7 m/s) and greater 80 rated power (> 3 MW) undergoes a load measurement campaign using op-81 tical strain gauges at the tower base sampled at a high frequency (> 50 Hz). 82 The overall procedure of the fatigue analysis is taken from available and 83 previously mentioned publications; however, the novelty is to apply SHM in-84 formation to derive and evaluate the long-term strategic LTE business case 85 for a specific wind farm. 86

87 2.2.1. Tower SHM

Ideally, strain gauges are installed at the locations on the tower situated in the prevailing wind direction. However, the installation of tower sensors might not be feasible in all areas; access restrictions and risk of damage due to maintenance processes can limit the available positioning of sensors (e.g.

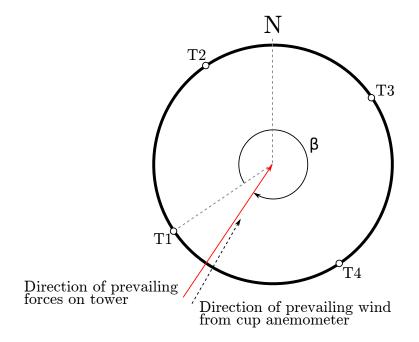


Figure 2: Schematic of tower sensor positions with respect to prevailing wind direction.

in proximity to the foundation-tower bolts that require servicing). Such constraints were encountered in this work; however, as explored below, the problem of imperfect positioning of sensors has not been of serious consequence
to the adopted methodology.

The locations of the tower base strain gauges (T1-T4) with respect to 96 north is illustrated in Figure 2. The normalised strain data, paired with 97 30 minute average supervisory control and data acquisition (SCADA) wind 98 speed data (in the respective directional corridor $\pm 10^{\circ}$) is illustrated in 99 Figure 3 for T1 and in Figure 4 for the 90° rotated tower strain T2, re-100 spectively. Overall, the measurements are well in agreement with the yaw 101 reference SCADA data, allowing confidence in the nacelle sensor calibration. 102 Based on the measurement campaign, as expected due to access con-103 straints, the sensors are not aligned with the prevailing wind direction. This 104 was confirmed (i) based on the mean SCADA nacelle direction and (ii) since 105 the operational SCADA period of T1's inflow corridor ($\pm 10^{\circ}$) over the total 106 recorded time covered 7.5% and 3.2% for T2, respectively. 107

¹⁰⁸ In order to evaluate a component's total lifetime based on measured or

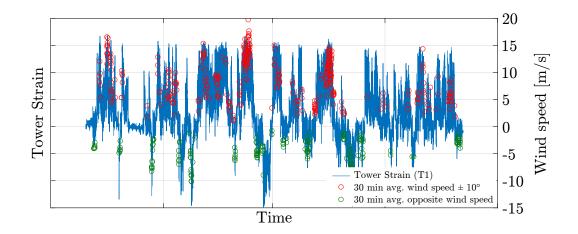


Figure 3: Strain data of base tower measurement (T1). The data is paired with recorded SCADA wind speed measurements on the right y-axis.

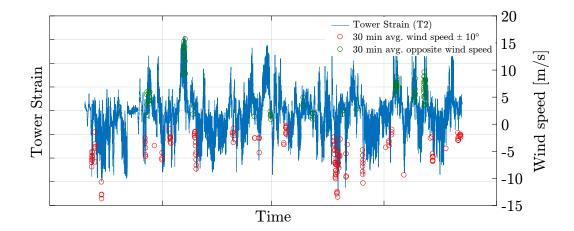


Figure 4: Strain data of 90° rotated base tower measurement (T2). The data is paired with recorded SCADA wind speed measurements on the right y-axis.

simulated data, the recorded signal is decomposed in defined discrete cycle ranges and each range's total number of occurrence is counted through a process referred to as rainflow counting [21]. Since, the rainflow counting algorithm is highly sensitive to changes in the maximum strain as well as in the frequency of occurrence of each range [9], the actual prevailing wind direction requires evaluation.

Given that the tower is radially symmetrical and the component's material (S355 steel) is designed to operate in its elastic limit, the stress across the circumference of the tower can be found as a vector sum of the stresses from the sensors. The two sensor strain measurements $v_{T1}(t)$ and $v_{T2}(t)$ respectively from T1 and T2, being positioned on the tower at 90° from each other allows calculation of the magnitude of the resulting vector, |v(t)|, and angle, $\gamma(t)$, by:

$$|v(t)| = \sqrt{v_{T1}(t)^2 + v_{T2}(t)^2} \tag{1}$$

$$\gamma(t) = \tan^{-1} \left(\frac{v_{T1}(t)}{v_{T2}(t)} \right).$$
(2)

The direction of the prevailing forces on the tower (which in turn is dictated by the prevailing wind direction), is identified counting the number of occurrences in the angle $\gamma(t)$ using a moving window of 5° as illustrated in Figure 5. The prevailing wind direction β with respect to T1 is then identified as the angle with the maximum number of occurrences.

Figure 5 indicates that the actual prevailing wind direction does not coincide with any sensor positions as it is not a multiple of 90°. In fact, the actual prevailing wind direction is shifted by 22° counterclockwise with respect to T1, which is also closely in agreement with the nacelle's mean SCADA direction with a difference of 3° as illustrated in Figure 2.

Further, it is necessary to determine if the strain is positive or negative for the rainflow counting as the range (tension and compression) dictates fatigue cycles. Therefore, the difference between angles is calculated:

$$\alpha(t) = \beta - \gamma(t) \tag{3}$$

and the strain variation over time in the prevailing wind direction, denoted as A(t) is calculated by:

$$A(t) = \cos(\alpha(t)) \cdot |v(t)|.$$
(4)

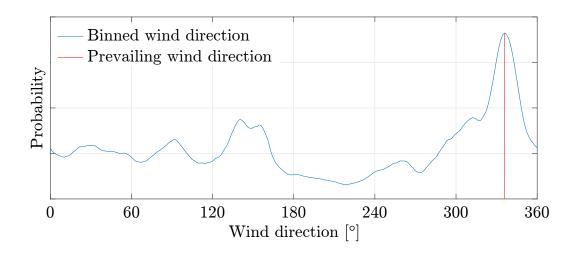


Figure 5: Identification of prevailing wind direction, β based on $\gamma(t)$ binning. The data is derived from tower strain sensor T1 & T2.

And for the perpendicular direction as:

$$B(t) = \sin(\alpha(t)) \cdot |v(t)|.$$
(5)

¹²⁵ It was further verified that of this new set of axes, the higher frequented ¹²⁶ component is selected.

Figure 6 displays the calculated strain in the prevailing wind direction. The strain profile is in agreement with the wind speed measurements from the SCADA data. Also, the SCADA data shows that, in the operational corridor considered, the turbine was operational for 23% of the total recorded time. This corroborates the above analysis.

The tower is usually made from hot-rolled steel, welded together circumferentially and longitudinally [22], with welded flanges at either tower end. As such the S-N curve assumption is dependent on the weld type [23]. The rainflow counting algorithm was applied according to the ASTM standard where half cycles are conservatively treated as full cycles [21, 24]. The S-N curve for the tower is used with the following parameters. The endurance limit at 2 million cycles, $\Delta \sigma_C = 80$ MPa [25, 23], the constant amplitude fatigue limit at 5 million cycles, $\Delta \sigma_D = 59$ MPa, and the cut-off limit, $\Delta \sigma_L = 32$ MPa according to EN 1993–1–9 [23]. With the established S-N curve, Miner's damage calculation was applied, after the strain was transformed into a stress (Young's Modulus, E = 200 GPa). The cumulative

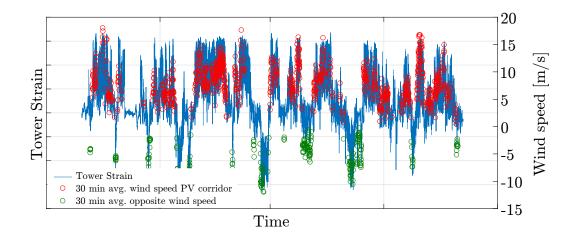


Figure 6: Strain data of derived prevailing wind direction. The data is paired with recorded SCADA wind speed measurements on the right y-axis.

fatigue damage D_{tot} is:.

$$D_{tot} = \sum D_i \tag{6}$$

where D_i is the partial damage in each discretised rainflow counting bin *i*. D_i is thus:

$$D_i = S_m^{-m} \sum_i^N n_i \sigma_i^m \tag{7}$$

where S_m as well as m are material constants, and σ the stress amplitude with 132 n numbers of observed occurrences for the respective bin i. If $\Delta \sigma_i > \Delta \sigma_D$, 133 m = 3 and if $\Delta \sigma_L < \Delta \sigma_i < \Delta \sigma_D$, m = 5. Otherwise, $D_i = 0$. The total 134 fatigue damage D_{tot} is thus calculated. The binning width of the rainflow 135 counting algorithm and sampling frequency determine the accuracy of the 136 lifetime prediction; however, a high sampling frequency in combination with 137 a small binning width, significantly increase processing requirements. As 138 such, the appropriate binning width of 0.2 MPa was identified as illustrated 139 in Figure 7 while an appropriate minimum sampling frequency is identified 140 as 100 times the first tower mode as illustrated in Figure 8. 141

The total tower lifetime, based on the recorded measurement data T1was thus estimated to be 248 years and for T2 339 years, respectively. In the prevailing wind direction, the derived and more frequented corridor β , the lifetime analysis resulted in a reduced lifetime of roughly 23 years with a total

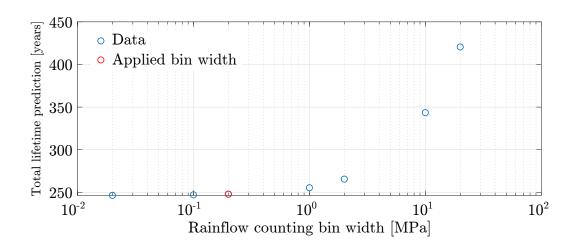


Figure 7: Impact of binning width on lifetime prediction. Applied frequency is 380 times the first tower mode.

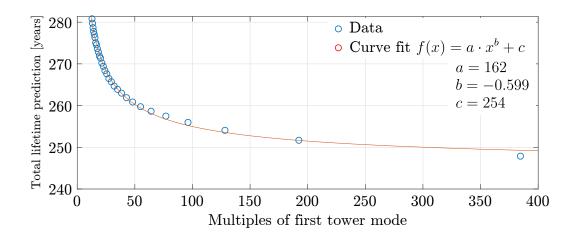


Figure 8: Impact of sampling frequency on lifetime prediction based on a 0.2 MPa binning width.

of 225 years¹. The magnitude of this reduction further allowed confidence in the data processing. In order to verify this result, the lifetime analysis was carried out for varying β (0-180°) as more significant loading, albeit with an overall lower number of occurrence, could have been experienced for wind directions off the prevailing axis. This analysis verified the prevailing wind direction β , identified in Figure 5.

Further, based on findings by Rebelo et al. [7, 8] and Loraux and Brühwiler 152 [26], the maximum tower stress is likely to be experienced at 30-40% of the 153 hub height. At present, the complete tower geometry of the considered wind 154 turbine is unknown. Therefore, a conservatively selected correction factor, 155 derived from the previously mentioned tower monitoring campaigns, is in-156 troduced. The corrected total lifetime at the critical tower height is thus 157 identified as 81.6 years. A further correction is required as the outer shell of 158 the tower has a greater stress, as the inner walls' strains are monitored. Thus 150 this correction leads to a total lifetime of 78.4 years. So far, the carried out 160 stress correction procedure has neglected any reliability aspects. In order to 161 allow for sufficient safety margins, the IEC power production safety factor 162 (1.25) is further applied. With the safety factor included, the total lifetime 163 results in 34.6 years. The overall data processing steps are further illustrated 164 in Table 1. If residual cycles of the rainflow counting process are treated as 165 half cycles, as suggested by the IEC 61400-13 standard [27], the total lifetime 166 is identified as 35.2 years. 167

Overall, from the point of view of the tower, a LTE of 15 years thus appears feasible, given considerate safety margin, as the carried out fatigue analysis reveals a total lifetime of 35 years (turbine design life is 20 years).

171 2.2.2. Foundation SHM

Overall, SHM of wind turbine foundations is a challenging area of re-172 search as highlighted by several studies, since the foundation is mainly in-173 accessible for inspection [13, 14, 20]. Given that wind turbine foundations 174 (i) are designed for a lifetime of 50 years or more, (ii) their design is based 175 on conservative assumptions, and (iii) they are structurally of key import-176 ance, there is little concern to accommodate for LTE. Based on an internal 177 strain analysis of the reinforcement cage by Rubert et al. [20], this is further 178 supported. As a consequence, from an economic lifetime extension decision-179

¹binning width of 0.2 MPa and frequency of 380 times the first tower mode

Analysis	RUL	Comment
	[years]	
T1 (tower base)	248	Sensor 22° to prevailing wind
T2 (tower base)	339	Sensor 112° to prevailing wind
PW (tower base)	225	Derived prevailing wind with Equation 4
PW + HC	81.6	Corrected stress at most critical height
PW + HC + SC	78.4	Corrected for the outer shell
PW + HC + SC + SF	34.6	Added IEC safety margin

Table 1: Process of Data Manipulation. PW: prevailing wind, HC: height correction, SC: section correction, SF: safety margin. Frequency of 380 times the first tower mode.

making perspective, the foundation is not of concern (except when severe 180 cracks are encountered). "Cracking is normal in reinforced concrete struc-181 tures subject to bending, shear, torsion or tension resulting from either direct 182 loading or restraint or imposed deformations" [28]. Although cracking is ex-183 pected to some degree, there is a crack width limit, w_{max} that is governed 184 under the service limit state. The acceptable crack width is dependent on 185 the concrete exposure class and type of reinforcement and can be looked up 186 in design codes and guidelines. Also, if cracks appear, work by Perry et al. 187 [14] and McAlorum et al. [15] may be applied for SHM. Results thus reveal 188 a possibility of an extended WTG operation of greater than 15 years. 189

¹⁹⁰ 3. Lifetime Extension Methodology

The lifetime extension decision-making methodology is schematically illustrated in Figure 9, where the lifetime extension period is treated as a separate investment and calculated based upon levelised cost of energy (LCOE₂). To calculate LCOE₂, the net present value (NPV) of costs is divided by the NPV of the annual energy production (AEP):

$$LCOE_{2} = \frac{NPV_{costs}}{NPE} = \frac{C_{0} + L_{0} + \sum_{n=1}^{T} \frac{F_{n} + O_{n} + V_{n}}{(1+d)^{n}}}{\sum_{n=1}^{T} \frac{E_{n}}{(1+d)^{n}}}$$
(8)

where NPE is the net present energy, C_0 the equity capital expenditure of component replacements (CAPEX_{Replace,E}), L_0 the lifetime extension capital expenditure (CAPEX_{LTE}), n is the period ranging from year 1 after the design lifetime to T the final year of operation (end of extended lifetime), F_n

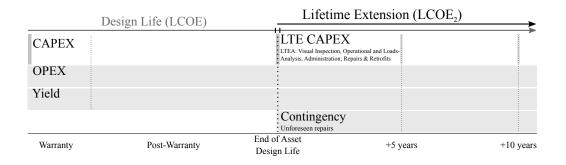


Figure 9: Lifetime extension decision methodology [4].

the constant annuity payment of the component replacement's expenditure debt in period n (CAPEX_{Replace,D}), O_n the fixed operating cost including decommissioning² in period n, V_n the variable operating cost in period n, E_n the energy generated in period n, and d the discount rate.

This extended lifetime methodology is equipped with operational data in terms of cost and yield parameters. The prior includes the CAPEX LTE and operational & maintenance (O&M) expenditure and the latter identified through operational knowledge or alternatively the application of a Weibull wind distribution in combination with a turbine's power curve [29]. Of course all variables are ideally based upon the operational design lifetime and may be adjusted depending on; e.g., failure and reliability data.

²⁰⁶ 4. Lifetime Extension Case Study

207 4.1. Strategy

The structural integrity of the foundation and tower is one of the main factors in determining economic lifetime extendibility (high replacement costs) and the high importance in serving as a load-carrying component, their RUL is of significant interest for a given wind turbine. As previously discussed, the foundation design lifetime significantly exceeds other components, provided that the design and construction procedures have been correct. Hence, in

²onshore it is expected that the scrap value equalises decommissioning costs; offshore this is certainly not the case

the great majority of cases, the tower RUL is of greater concern. There-214 fore, knowledge of the site-specific tower RUL will provide argument for the 215 long-term economic business case. 216

The results from the SHM campaign presented above indicate that life-217 time extension of 15 years appears feasible. Therefore, for the LTE business 218 case the strategic extension period is considered to be 15 years. 219

4.2. Input Data 220

The input data for the economic model is a combination of actual and 221 generic data as illustrated in Table 2. Where possible, real input is applied; 222 however, the commercial business case is highly sensitive, thus not all actual 223 data is applied in the model. As such, the economic model generates an 224 academic case scenario that is aligned as best as possible to a potential real 225 scenario. The power curve was reproduced as highlighted by Rubert et al. 226 [4]; however, rather than applying the maximum power coefficient, $C_{p,max}$ to 227 derive the power curve, $C_{\rm p}$ varies with wind speed, $C_{\rm p}(v)$ that was derived 228 based on the manufacturer's data sheet ($\rho = 1.225 \text{ Kg/m}^3$). This enables 229 greater accuracy in the yield modeling as outlined by Carillo et al. [30] and 230 Lydia et al. [31]. As identified by [4, 32], the mean wind speed has the highest 231 magnitude in the impact, thus careful evaluation is necessary. The turbine's 232 mean wind speed was derived using operational SCADA data, accounting 233 for the impact of curtailment (provided by the operator). Curtailment was 234 included in the model by reducing the average wind speed for the specific 235 wind turbine. 236

Given that the foundation and tower are able to facilitate the target 237 lifetime extension period, components along the drive train may require re-238 placement. This is budgeted as CAPEX_{SPARE,D} and CAPEX_{SPARE,E} with a 239 70/30% debt-equity split, the latter budgeted as a constant annuity with the 240 interest rate set as 3.5% [33]. Cost and time assumptions for the necessary 241 crane (1,200 t) and service team for component replacements were evaluated. 242 The time requirement was increased by 50% and the service team number 243 increased by 25% from those from [4]. The overall cost assumptions are sum-244 marised in Table 3 for the central case as well as optimistic and pessimistic 245 scenario, respectively. 246

247

The discount factor is assumed at 7.5%, with inflation set at 1.5% ac-

Table 2: Wind turbine parameters. Actual are real operational parameters for the respective wind turbine, while generic data is applied due to confidentiality in the business case. The resulting capacity factor is a combination as actual and generic data is applied to derive the metric.

Parameter	Value	Actual/Generic Data
Cut-in wind speed	$3 \mathrm{[m/s]}$	Actual
Cut-out wind speed	$25 \mathrm{[m/s]}$	Actual
Rated wind speed	12.5 [m/s]	Actual
Rotor diameter	Not disclosed	Actual
Wind speed	Not disclosed	Actual
Power coefficient	Not disclosed	Actual
Turbulence intensity	0.1	Generic
Availability	97 ~[%]	Generic
Wake & park losses	10 [%]	Generic
Discount factor	7.5~[%]	Generic
Inflation	1.5 [%]	Generic
Weibull shape factor	2	Generic
Resulting capacity factor	Not disclosed	Actual/Generic

²⁴⁸ counted to administration and spare parts of the O&M expenditure³.

Also, for the scenario with no component replacement, an annual performance degradation of 0.3% is modeled based on findings by [35, 36, 5]. In the other scenarios, due to component upgrades the performance degradation is likely significantly smaller and thus neglected.

To get greater confidence limits, a Monte Carlo simulation is further 253 applied based on the application of normal distributions. This allows to 254 account for statistical factors, as component/installation costs and the wind 255 inflow parameters may vary over time. As such, variability in the results 256 are expected⁴. This was carried out for the scenario with no component 257 replacement and the exchange of the entire drive train. The annual wind 258 speed was characterised based on SCADA mean data paired with a standard 259 deviation of 7% [39]. The cost data was modeled with a standard deviation 260 of 25% as illustrated in Table 3. For the component replacement process, if 261

 $^{^{3}\}mathrm{The}$ interested reader is referred to [34] for detailed commentary on LCOE input parameters.

⁴For detailed information of Monte Carlo simulations, the reader is referred to [37, 38].

Table 3: Generic lifetime extension cost estimations for a wind farm [4]. The range inter- is applied in the Monte Carlo simulation, with the central parameter defined as the medi- value.				
	Parameter	Central	Range	Unit

Parameter	Central	Range	\mathbf{Unit}
O&M			
Fixed	$30,\!192$	$22,\!644 \text{-} 37,\!740$	$\pounds/MW/y$
Variable	5.1	3.83 - 6.38	\pounds/MWh
Insurance	2,226	$1,\!669\text{-}2,\!782$	$\pounds/MW/y$
Connection charges	$3,\!810$	2,857-4,762	$\pounds/MW/y$
CAPEX LTE			
Visual inspection	$2,\!689$	2,017-3,361	\pounds/WTG
Loads analysis	$3,\!500$	$2,\!625\text{-}4,\!375$	\pounds/WTG
Operations analysis	2,000	1,750-2,250	\pounds/WTG
Administration	1,000	750 - 1,250	\pounds/WTG
Spare parts			
3 blades	$238,\!560$	178,920-298,200	\pounds/WTG
Gearbox	$147,\!680$	110,760-184,600	£/WTG
Generator	$93,\!152$	69,864-116,440	£/WTG
Installation expenditure			
Crane Mob/Dmob	20,000	$15,\!000\text{-}25,\!000$	$\pounds/Wind Farm$
Crane operation	2,000	1,500-2,500	\pounds/day
Service personal	58	43.1-71.9	£/h

the wind speed is above a certain wind speed threshold, components cannot be lifted. Therefore, the required crane and service hours were applied based on the minimum expected time and a normally distributed time component added to account for wind related delays. Based on the procedure detailed by Vose [37], the number of required iterations n was identified as 50,000 based on a standard error of 3% and a 90% confidence interval.

268 5. Results

When operating a wind farm, each turbine can be characterised differ-269 ently; i.e., some turbines have greater average wind speeds than others, de-270 pending on the local terrain, wake effects, and operational parameters. With 271 regards to LCOE calculations, the mean wind speed has the greatest im-272 pact [32, 4]. When pairing the mean wind speed with operational know-273 ledge (downtime, degradation, curtailment, etc.) the AEP or capacity factor 274 can be derived. Therefore, when operating a wind farm that is reaching its 275 end of design lifetime with fewer revenues or when directly exposed to the 276 spot-market electricity price, some turbines might be less profitable in their 277 continued operation than others. As a consequence, a LTE decision-making 278 requires turbine specific evaluation. 279

The lifetime extension LCOE₂ of the bespoke economic turbine model based on operational wind conditions are illustrated in Figure 10 under the assumption of (i) no retrofit and (ii) the exchange of the entire drive train; in Figure 11 for a single retrofit of a drive train component; and in Figure 12 for any retrofit combination of drive train components. As mentioned before, each scenario has an assumed extended lifetime of 15 years.

The error bands are based on the cost variation illustrated in Table 3. A wind farm usually consists of several individual turbines, with varying degree of loading and electricity production, thus when it comes to lifetime extension, not necessary all turbines are economically suitable to keep in operation. Knowing that the annual wind speed and hence AEP has the greatest impact on LCOE, the wind speed is varied in order to determine profitability of the different cases.

With turbines mostly being exposed to the subsidy-free spot market electricity price, a threshold is defined to determine individual turbine suitability. This is defined as 10% below the average UK's spot market price of the past 5 years [4].

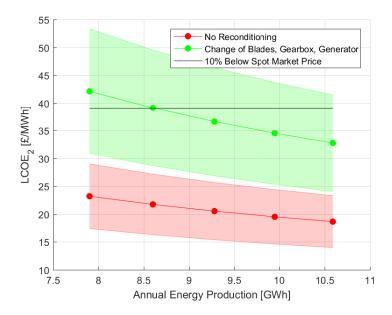


Figure 10: $LCOE_2$ of lifetime extension period with annual energy production (no retrofitting and drive train exchange).

Overall, without any component replacement, the LCOE₂ is significantly below the defined subsidy-free threshold (£39), hence LTE is supported for any of the modeled AEP cases. Alternatively, if the entire drive train requires replacement (blades, gearbox, and generator), this would only be economically viable if the annual energy production is above 8.6 GWh/WTG. The complete range is illustrated in Figure 10.

For any single component exchange (blades, gearbox, and generator), all medium cost estimates are below the threshold; however, for the pessimistic cost scenario, the replacement of blades are economically infeasible and decommissioning is advised as illustrated in Figure 11 when below 8.3 GWh/WTG.

For any two component replacement scenario, the cases including new blades require at least 7.5 GWh/WTG when paired with a generator exchange, and 7.8 GWh/WTG when paired with a gearbox exchange in order to be economically viable as illustrated in Figure 12. The replacement of a gearbox in combination with the generator is feasible in the medium cost scenario; however, in a pessimistic scenario caution is required.

Table 4 further displays the annual available contingency with respect to

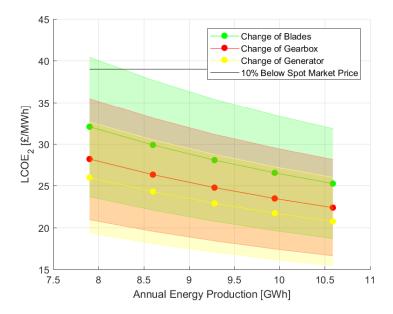


Figure 11: $LCOE_2$ of lifetime extension period with annual energy production (single retrofit).

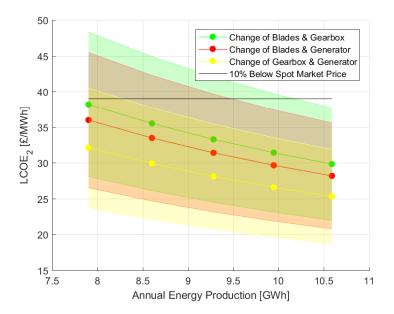


Figure 12: $LCOE_2$ of lifetime extension period with annual energy production (double retrofit).

Scenario	Pessimistic	Central	Optimistic
No reconditioning	141,363	186,268	231,173
Reconditioning of blades	37,704	104,901	$173,\!560$
Reconditioning of gearbox	76,837	135,610	$195,\!286$
Reconditioning generator	99,283	153,214	207,719
Reconditioning blades, gearbox, & generator	N/A	24,980	116,871
Reconditioning blades & bearbox	N/A	$56,\!138$	138,952
Reconditioning blades & generator	N/A	73,743	$151,\!385$
Reconditioning gearbox & generator	37,221	104,452	173,158

Table 4: Annual Contingency $[\pounds]$ for 15 year LTE under different scenarios. N/A: costs exceed revenue.

(i) the different replacement scenarios and (ii) the expenditure range based on 315 an AEP of 9.3 GWh. As illustrated in Figure 9, this parameter indicates the 316 potential money to spend before the project becomes non-profitable along the 317 life extended period; i.e., when decommissioning is advised. The remaining 318 contingency may be applied to support the operational LTE decision-making 319 as the available budget indicates the risk of an aimed strategic decision. 320 An example would be if the replacement of the drive train is strategically 321 considered matched with central cost estimates, as the remaining annual 322 contingency is $\pounds 24,980/WTG$. In such an event, if severe issues occur (such 323 as a major generator or bearing failure), the project is likely more risky to be 324 profitable than other decisions with a greater annual contingency. This risk 325 can potentially be reduced by in-depth structural analysis and the application 326 of reliability models based on inspection results. 327

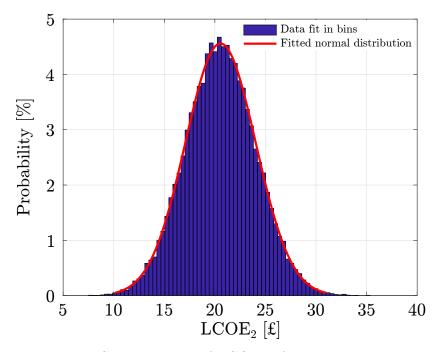


Figure 13: Monte Carlo analysis of $LCOE_2$ of no component replacement.

Results of the Monte Carlo simulation are presented in Figure 13 with no component replacement and in Figure 14 for the replacement of the entire drive train. In addition, Table 5 presents the respective P10/50/90 percentiles.

Table 5: Project expenditure percentiles $[\pounds]$ based on Monte Carlo simulation.

Scenario	P10	P50	P90
No replacement New drive train			$25.02 \\ 42.53$

Overall, there is a 90% probability that the LCOE₂ is below £25.02 with no component replacement, whereas when exchanging the entire drive train, there is a 50% chance that LCOE₂ are above £37.07. With respect to the threshold spot market electricity price, there is a 69% chance to be economically profitable. Of course, results of the Monte Carlo simulation will change with differently encountered mean AEP.

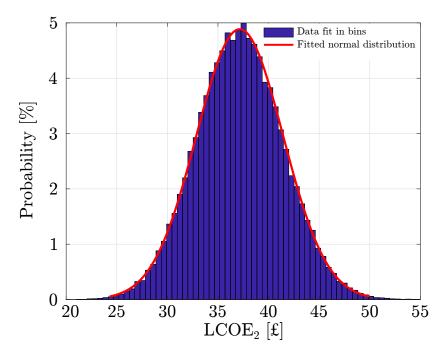


Figure 14: Monte Carlo analysis of LCOE₂ of drive train exchange.

338 6. Discussion and Future Work

Confidence in the SHM measurement campaign increases as a function of 339 the duration of the data monitoring campaign; a longer monitoring period 340 will thus deliver an increase in confidence in the strategic LTE business case. 341 Applying the AEP of each turbine requires closer examination as often 342 turbines are curtailed due to network restrictions. Therefore, besides looking 343 at the AEP in isolation, curtailment information can deliver a more accur-344 ate picture. Also, when having operated a wind farm for 20 years, its grid 345 integration is well understood and thus data readily available. 346

As identified by Tavner [40], Wilson [41], and Reder [42], wind turbine 347 reliability is correlated with environmental conditions. Thus, a turbine's 348 components have an individual and thus varying load profile. Of course, 349 the design of the respective turbine should accommodate for such differences 350 given the IEC classes (IEC 61400-1). The turbine in question was identified 351 based on the highest annual wind speed of the respective wind farm. Nev-352 ertheless, such indicators as turbulence intensity are also important. The 353 O&M costs may therefore fluctuate per turbine and should ideally be taken 354

into consideration in the economic evaluation. In order to accommodate
 fluctuations, the optimistic and pessimistic cost bands are presented.

While local wind conditions may change over the years [43, 44], so in turn would the AEP. Therefore, when extracting the AEP, a period of several years should be considered. Ideally, the entire operational life.

It is further possible to extrapolate tower fatigue findings onto each indi-360 vidual wind turbine in the wind farm by application of a tower finite element 361 model and, ideally, analysis of high frequency SCADA data (if available). 362 This will be considered in future work, in order to determine a wind farm 363 lifetime extension strategy, by clustering turbines into cells with different 364 loading. In this regard, low wind speed and turbulence intensity exposed 365 wind turbines might be selected for turbine removal and the spare parts 366 might be stored or straight away used to replace turbine components with 367 higher mean wind speed and turbulence intensity values. 368

Judging from the cost to carry out a tower measurement campaign (roughly £20,000-30,000), we argue that to gain an accurate LTE strategy, the benefit outweighs the costs of the installation of such a system. Of course, the latter depends on the deployed turbine and wind farm size [5] as well as the SHM system design.

We further suggest to install tower sensor sets (one sensor each side for 374 validation purposes [20]) 90° apart as well as to analyse each wind corridor 375 by varying β in order to cover any eventualities if e.g., the assumed pre-376 vailing wind direction does not match the real prevailing wind corridor as 377 highlighted in Section 2.2. In addition, as the cross sectional moment of 378 inertia and bending moment change with tower height, so does the stress 379 distribution. Ideally, the tower wall thicknesses and sectional diameters are 380 measured to derive the maximum stress location. Nevertheless, in the absence 381 of tower geometry data, correction factors may be applied as highlighted in 382 Section 2.2. Overall, we strongly recommend to measure the tower's geo-383 metry (thickness and diameter with hub height) to identify the most critical 384 stress location. At this location, the fatigue analysis shall be carried out. 385 As such, the application of generic or simplified tower geometries may lead 386 to severe uncertainties and inaccuracies of aero-elastic simulations and thus 387 caution is advised. 388

The SHM monitoring campaign may be tailored for a global analysis aimed at evaluating stresses of critical tower areas, such as along the entrance $_{391}$ door⁵ as well as flanges as discussed by Schedat et al. [46].

With respect to the rainflow counting algorithm, we suggest to use a binning width equal or lower than 0.2 MPa paired with a minimum sampling frequency of 100 times the first tower mode. This allows accurate measurements while maintaining an appropriate accuracy (within 10%). Also, a correction parameter can be applied based on the findings presented in Figure 7 and 8 if data is available at a lower sampling frequency.

SHM data combined with economic findings do not suggest that long-398 term lifetime extensions should be carried out blindly, thus the necessary 390 inspections are key in making sure that the continued operation is safe. For 400 the tower, critical sections are welded and bolted connections as well as areas 401 with corrosion [3, 45]. An inspection guideline published by DNV GL for the 402 tower and foundation is presented in Table 6 of the Appendix. In addition, 403 an inspection guideline is published by Megavind [45]. In critical cases, it is 404 further suggested to reduce the inspection interval or to install tailored SHM 405 hardware. For an example of tower flange cracking, the reader is referred to 406 work developed by Do et al. [47]. To access experimental mechanical and 407 fracture properties of welded S355 steel, work by Mehmanparast et al. [48] is 408 suggested. We also recommend monitoring the first natural frequency as well 409 as damping ratio of the tower as variations can indicate structural changes 410 with little resources spend, if sensors are installed. 411

As illustrated by Helm [49] based on data by the Department for Business, Energy, Industry, and Strategy (BEIS), the electricity price is expected to remain at current prices and then gradually increase from 2020, reaching a high in 2024 before dropping off in the UK. In fact, this requires careful observation and scrutiny in order to define the profitability threshold appropriately.

Uncertainties further origin from the weld assumption; data that is not necessarily shared by turbine manufacturers. Potentially, the weld class might be analysed with ultrasonic wall thickness measurement devices to get confidence in the selection of the appropriate weld classes.

Finally, in comparison to previous findings by Rubert et al. [4], this work derives the strategic lifetime extension case for a significantly greater rated

⁵According to the publication from Megavind, the tower entrance door is not considered as a critical area: "As for tower fatigue, cracks in the door-tower connection may, with low probability, occur when the turbine reaches the design lifetime" [45].

turbine taking the actual structural integrity into consideration as well as
the actual wind speed. As such, the lifetime extension business case appears
in general more positive than the assessment of smaller scale generators.

427 7. Conclusion

This work explores a strategic case specific lifetime extension decisionmaking process, based on information gathered through SHM. The process indicates that if the tower and foundation are in a good condition (acceptable level of corrosion, no cracks for the tower; foundation cracks within acceptable limit), these key turbine components are generally well suited to facilitate lifetime extension decision-making.

Based on the SHM of the wind turbine tower, the total lifetime was identified as 35 years by evaluation of the prevailing wind direction at the most critical tower location, including a load safety margin. In addition, parameters are provided for the analysis to derive the tower's RUL.

Forwarding the structural information to the economic business case, results suggest a P90 LCOE₂ of £25 if no components require reconditioning, paired with a lifetime extension of 15 years. If the blades, gearbox, and generator are exchanged in year 20, the P90 LCOE₂ is identified as £42.50. For this case, the probability to be 10 % below the average spot market price is 69%, thus caution and due diligence is advised or alternatively a lower profit margin shall be defined.

Overall, the results of this study further support the operational knowledge that lifetime extension is highly site specific; however, it is essential to derive a suitable LTE strategy for the continued operation to generate the economic business case. This is especially valid for multi-MW turbines with substantial annual energy production. Besides allowing continued electricity generation and maintaining local O&M jobs, lifetime extension reduces the generation of waste, which is of general interest.

452 8. Appendix

Table 6: Tower & foundation inspection guideline [3]. D is damage, C is cracks, Co is corrosion, Sp is safety sign plates, Ps is prestress, Cf is connection/fitting, and F is function.

Tower Component	Inspection	
Tower structure	D,Co,C,Sp	
Ladder, fall protection	$_{\rm D,Co,F,Sp}$	
Bolted connections	$_{\rm Co,Ps,C}$	
Foundation, embedded section	$_{\rm D,Co,C}$	
Foundation	$^{\rm D,C}$	
Grounding/earthing strip	Cf,D,Co	

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