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1 **The structure and entrainment characteristics of partially-confined gravity currents**

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8 **Key Points:**

- 9 • In partially-confined settings, channel depth is a key control on the height of a gravity current's velocity
10 maximum.
- 11 • Both streamwise and overbank discharge rates can rapidly adjust downstream, with evidence of flow
12 tuning and equilibration.
- 13 • The entrainment coefficient of a partially-confined flow is similar to that of a fully-confined flow with the
14 same Richardson number.

15 **Abstract:**

16 Seafloor channels are the main conduit for turbidity currents transporting sediment to the deep ocean and they
17 can extend for thousands of kilometres along the ocean floor. Although it is common for channel-traversing
18 turbidity currents to spill onto levees and other out-of-channel areas, the associated flow development and
19 channel-current interaction remain poorly understood; much of our knowledge of turbidity current dynamics
20 comes from studies of fully-confined scenarios. Here we investigate the role that partial lateral confinement
21 may play in affecting turbidity current dynamics. We report on laboratory experiments of partially-confined,
22 dilute saline flows of variable flux rate traversing fixed, straight channels with cross-sectional profiles
23 representative of morphologies found in the field. Complementary numerical experiments, validated against
24 high-resolution laboratory velocity data, extend the scope of the analysis. The experiments show that partial
25 confinement exerts a first order control on flow structure. Overbank and downstream discharges rapidly adjust
26 over short length-scales, providing a mechanism via which currents of varying sizes can be tuned by a channel
27 and conform to a given channel geometry. Across a wide range of flow magnitudes and states of flow
28 equilibration to the channel, a high-velocity core remains confined within the channel with a constant ratio of
29 velocity maximum height to channel depth. Ongoing overbank flow prevents any flow thickening due to ambient
30 entrainment, allowing stable downstream flow evolution. Despite dynamical differences, the entrainment rates
31 of partially-confined and fully-confined flows remain comparable for a given Richardson number.

32 **1 Introduction:**

33 Seafloor channels are the main conduits through which turbidity currents transport sediment from the
34 continental shelf to the deep ocean [Meiburg and Kneller, 2010; Peakall and Sumner, 2015]. The submarine
35 fans that they form are some of the largest sedimentary accumulations on Earth [Curry et al., 2002; Talling
36 et al., 2007]. Due to the inherent challenges the deep-water environment poses, only recently have direct field
37 measurements become more widespread [Khipounoff et al., 2003; Xu, 2010; Sumner et al., 2013; 2014;
38 Talling et al., 2013; Dorrell et al., 2014; 2016; Azpiroz-Zabala et al., 2017]. In comparison there has been a
39 long history of model development based on laboratory experiments [e.g. Ellison and Turner, 1957; Middleton,
40 1966; Garcia and Parker, 1983; Bonnetcaze et al., 1993; Buckee et al., 2001; Keevil et al., 2006; Straub et al.,
41 2008; Islam and Imran, 2010; Sequeiros et al., 2010] and numerical simulations [e.g. Eidsvik and Brørs, 1989;
42 Imran et al., 2004; Huang et al., 2005; Cantero et al., 2009; Abd El-Gawad et al., 2011; Giorgio Serchi et al.,
43 2011; Dorrell et al., 2014; Kneller et al., 2016].

44 The majority of these studies were conducted within fully-confined channels. Yet the partially-confined channel-

45 levee component of natural systems usually extends much further than the fully-confined canyons that feed
46 them [Normark and Damuth, 1997; Klaucke et al., 1998; Meiburg and Kneller, 2010; Nakajima and Kneller,
47 2013]. Those studies that do consider unconfined/partially-confined settings have been run over erodible beds
48 [Mohrig and Buttles, 2007; Straub et al., 2008; De Leeuw et al., 2016] and tend to focus on morphological
49 evolution and channel inception rather than flow dynamics. While such studies increase knowledge of channel
50 and system development, the evolving channel geometries limit the consistency of flow data measured from
51 successive currents.

52 The dynamics and behaviour of partially-confined flows, where the current can overspill onto the levees, are
53 arguably far more complex and difficult to predict than for fully confined flows. Differing levels of confinement
54 lead to changes in the ratios of ambient entrainment and overbank losses, but a systematic review of the flow
55 field under a range of confinements is lacking. Mohrig and Buttles [2007] defined channelised, quasi-
56 channelised and unconfined regimes based on the advancement of the flow front, but without presentation of
57 detailed flow velocity or density data.

58 To date, it is fully-confined studies that have been widely used to explain and predict the structure and
59 properties of gravity currents. Parker et al. [1987] conducted straight channel experiments and reviewed
60 previous experimental data to find a Richardson number dependent expression for the entrainment coefficient
61 of a flow,

$$62 \quad e_w = \frac{0.075}{\sqrt{1 + 718Ri^{2.4}}}. \quad (1)$$

63 The rate at which a flow entrains ambient fluid is a key factor in both its spatial and temporal development and
64 could help to provide an explanation as to why turbidity currents can travel for thousands of km [Meiburg and
65 Kneller, 2010]. Kneller et al. [2016] used numerical simulations to show that, under certain conditions, turbidity
66 currents can have a stably stratified upper shear layer (Figure 1) with little mixing and low velocity gradients,
67 resulting in a reduction in ambient entrainment; when predicting flow characteristics the use of bulk variables
68 to approximate local variables was also questioned (such as using the bulk Richardson number as a proxy for
69 the gradient Richardson number, a measure of stratification stability). In another fully-confined experiment,
70 Sequeiros et al. [2010] observed a dependence of the velocity structure of the flow on the Richardson number,
71 attributed to changes in stratification stability. The velocity profiles of subcritical flows ($Ri > 1$) exhibited a velocity
72 maximum close to the top of the flow, although a large bed roughness is likely to have caused this. This is in
73 contrast to previously observed profiles where the outer shear layer is 5-10 times thicker than the inner layer

74 [Meiburg and Kneller, 2010]. Additionally, Sequeiros [2012] suggested that channel morphology can be used
 75 to predict Richardson or Froude numbers and subsequently flow conditions. However, this approach has
 76 limitations for erosional or bypassing flows as it does not take into account Reynolds-dependent turbulent
 77 effects in the lower boundary [Imran et al., 2016]. Also, high velocity maximum heights were not replicated in
 78 the simulations of Kneller et al. [2016], despite the stably stratified layer, nor in further experiments of subcritical
 79 flows which found limited dependence on Richardson number [Stagnaro and Pittaluga, 2014].

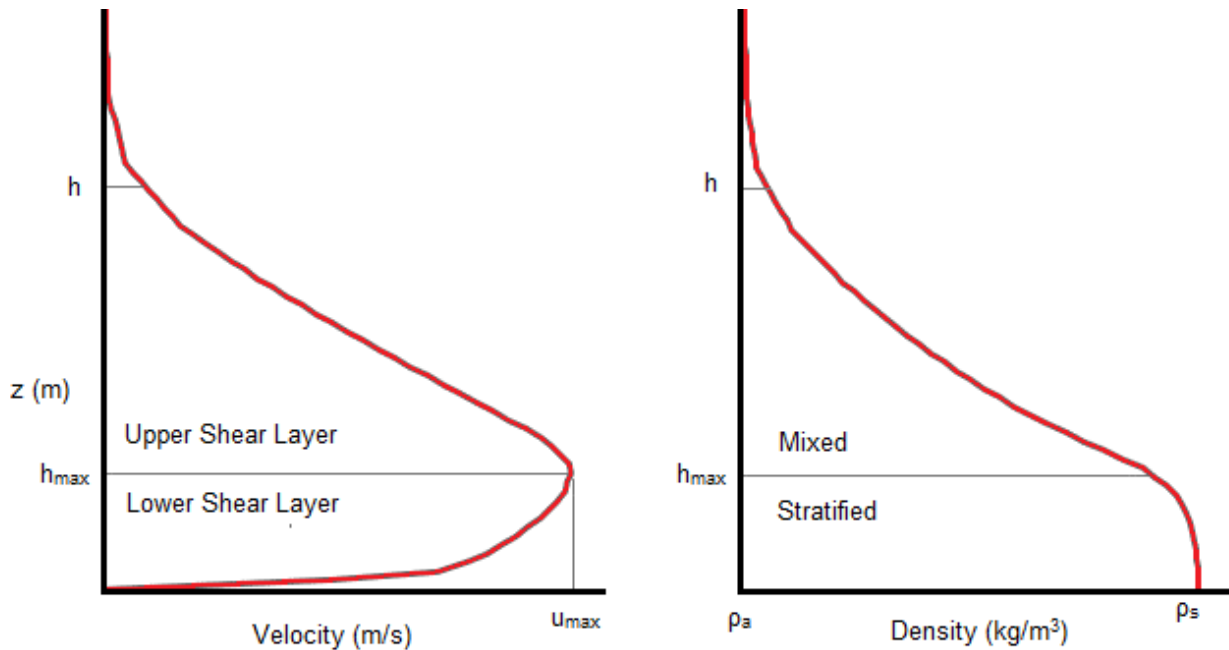
80 Regardless of the debate over confined-flow structure, the kinematics of a partially-confined flow must be
 81 fundamentally different due to the occurrence of overspill. Here, saline flow experiments have been conducted
 82 in a straight fixed channel with a channel-levee profile designed to be a realistic representation of morphology
 83 found in the field. Velocity data for a range of flow magnitudes has been captured (Table 2) with the aim of
 84 analysing partially-confined flow dynamics, entrainment characteristics and flow evolution.

85 Additionally, numerical simulations using a RANS (Reynolds-averaged Navier-Stokes) model have been used
 86 both to extend the range of flow conditions that are possible in the laboratory and to produce data for the whole
 87 flow field.

Variable	Expression
Flow depth	$h = \frac{(\int_0^\infty u dz)^2}{\int_0^\infty u ^2 dz}, \text{ where } u = \sqrt{u^2 + v^2}$
Depth-averaged velocity	$U = \frac{\int_0^\infty u dz}{h}, V = \frac{\int_0^\infty v dz}{h}, U = \frac{\int_0^\infty u dz}{h}$
Reynolds number	$Re = \frac{ U h}{\nu}$
Froude number	$Fr = \frac{ U }{\sqrt{g'h}}$
Richardson number	$Ri = \frac{g'h}{ U ^2}$

Reduced gravity	$g' = g \frac{\bar{\rho} - \rho_{ambient}}{\rho_{ambient}}, \text{ where } \bar{\rho} = \frac{\int_0^{\infty} \rho dz}{h}$
Gradient Richardson number	$Ri_g = \frac{-g \frac{\partial \rho}{\partial z}}{\rho \left(\frac{\partial u }{\partial z} \right)^2}$

88 **Table 1:** Variable and notation definitions



89
90 **Figure 1:** Velocity and density profiles for a gravity current generated by the release of a saline solution into
91 an ambient fluid (water), as depicted in Figure 2. These are characterised by two shear layers separated by a
92 velocity maximum. The lower shear layer is generated by basal drag and is stratified in nature, whereas the
93 upper shear layer is a result of drag with the ambient fluid and is subsequently more mixed. h is the height of
94 the current defined by the Ellison and Turner [1959] method in Table 1, ρ_a and ρ_s are the densities of the
95 ambient and saline fluid respectively, and h_{max} and u_{max} are the height and magnitude of the velocity
96 maximum.

97 **2 Method:**

98 **2.1 Laboratory Setup:**

99 A series of continuous release saline gravity current experiments were conducted in the Sorby Environmental

100 Fluid Dynamics Laboratory at the University of Leeds. While saline currents do not allow for the study of
101 particulate settling, they do provide a good dynamical model of turbulent and stratification effects in turbidity
102 currents [Kneller & Buckee, 2000; Islam & Imran, 2010; Cossu & Wells, 2012]. The flume used measured 1.7
103 m x 1.7 m and had a water depth of 1.5 m. An additional 1 m long inlet channel, along which the currents
104 developed, was centred on one side wall. The entire flume was inclined at an angle of 2° downstream. A
105 fibreglass channel model was placed on a suspended floor 0.4 m above the tank base, with the area
106 underneath acting as a sump to collect denser than ambient fluid.

107 The channel model is 0.22 m wide and extended the entire length of the inlet channel and 1.5 m into the main
108 flume. The channel-levee profile was designed specifically to create an environment that might replicate
109 morphology found in the field. The channel itself was 0.0275 m deep, giving an aspect ratio of 8, and the
110 channel profile took the form of a sine curve to give a maximum slope of 22° on the channel sides. Channel
111 size and width/depth ratio were chosen to balance the need for deep enough flows to be fully turbulent, while
112 achieving a low aspect ratio as is often seen in the field [Clark et al., 1992; Kenyon et al., 1995]. The channel
113 is bounded by a 22 cm wide levee on either side. The outer part of the levee profile is determined by the
114 relationship $z = H(L/Y)^{-B}$, where z is the height of the levee, H is the channel depth, L is the distance from
115 the channel thalweg, Y is half the channel width, and $B = 0.5535S^{0.662}$, where S is the slope. This was found
116 to be give the best fit to channel levees on slopes $>0.6^\circ$ by Nakajima and Kneller [2013]. Although this
117 relationship works well for the far field architecture it fails to capture the morphology near the crest. Therefore,
118 the inner third of the levee profile was determined using data from previous gravity current experiments
119 conducted over an erodible bed [Straub et al., 2008].

120 The gravity currents were created by preparing a saline solution of 1025 kg/m³ density (2.5% excess density).
121 The solution was pumped into the tank and controlled by an electromagnetic flow meter to minimise variation
122 in the input flow rate. Before entering the tank, the fluid passed through a momentum diffuser, manufactured
123 by capping the input pipe and drilling a series of holes in the pipe wall; this pipe was placed within a further
124 inlet pipe which fed an inlet box modelled to fit the channel profile. This ensured that a buoyancy driven flow
125 developed, rather than a dynamically different wall jet driven by inherited momentum and pressure (see
126 supplementary material). Fluid was also pumped out from the base of the tank at an equal rate to ensure a
127 constant water depth. Three flow rates were investigated: 0.2, 1 and 2 l/s (Table 2). The 0.2 l/s flow rate was
128 chosen to give a near bank-full current. The 1 l/s flow rate was chosen to ensure a large enough quantity of
129 overbank spill to measure with the ADVs (see below). The 2 l/s flow rate was chosen as the largest achievable
130 rate for which an appropriate flow duration could be achieved (4 minutes) without over-filling the sump.

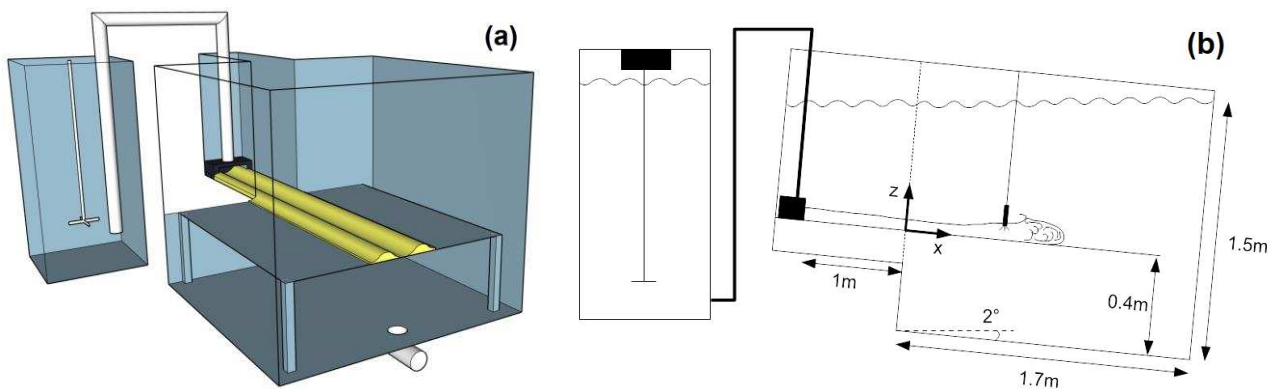
131 Hereafter these will be referred to respectively as bank-full, equilibrium, and oversize currents (Table 2).

132 Instantaneous three-component velocities were captured with a profiling Nortek Vectrino II acoustic Doppler
133 velocimeter (ADV) sampling at 100 Hz. Vertical resolution of the data is 1 mm with each profile extending 30
134 mm above the model base. Velocities were recorded both at the channel thalweg and the channel crest.

135 Ultrasonic Doppler velocity profiling (UDVP) was used at the channel thalweg to capture larger velocity profiles.
136 The ADV velocity profiles were extended with the UDVP data for the purposes of calculating bulk flow
137 properties.

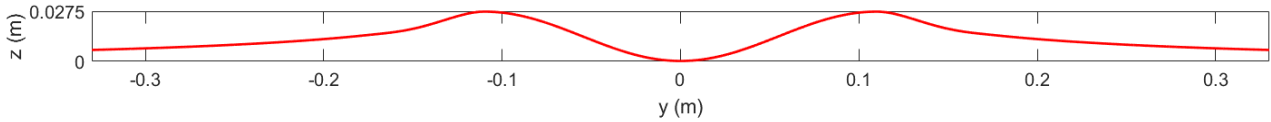
Input Flow Rate (l/s)	0.2 (bank-full)	1 (equilibrium)	2 (oversize)
h (cm)	3.17	4.75	5.33
U (m/s)	0.111	0.153	0.174
Re	3550	7250	9250
Fr/Ri	1.50/0.44	1.65/0.37	1.77/0.32
Flow duration	8 minutes	4 minutes	4 minutes

138 **Table 2:** Bulk flow properties of the three laboratory flows calculated from channel thalweg ADV/UDVP data,
139 1 m downstream from the main tank inlet.



140

141 **Figure 2:** (a) A 3D visualisation (channel profile not to scale) and (b) a cross-sectional schematic of the setup
 142 employed in the Sorby Laboratory. Saline was pumped from a large mixing tank via a momentum diffuser into
 143 the main tank which was inclined at 2°. A 1 m long confined inlet channel allowed the flow to develop. The
 144 channel was elevated on a false floor to allow fluid to collect in a sump underneath. The frame of reference is
 145 defined relative to the channel, with the origin positioned on the channel thalweg at the entrance to the main
 146 tank.



147
 148 **Figure 3:** Cross-sectional view of the channel model. The channel measures 0.22 m wide and 0.0275 m deep
 149 with an aspect ratio of 8. The profile is that of a sine curve which results in a maximum steepness of 22°. The
 150 levee profile was determined using a combination of laboratory data [Straub et al., 2008] and field data
 151 [Nakajima and Kneller, 2013].

152 2.2 Numerical Model:

153 Numerical simulations of the laboratory flows and additional flow conditions were performed with a Reynolds-
 154 averaged Navier-Stokes (RANS) model, solved using the software ANSYS CFX. This is governed by the
 155 Reynolds-averaged mass and momentum conservation equations,

$$156 \quad \frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0, \quad (2)$$

$$157 \quad \frac{\partial \rho u_i}{\partial t} + \frac{\partial \rho u_i u_j}{\partial x_j} = -\frac{\partial P}{\partial x_i} + \frac{\partial}{\partial x_j} \left(\mu \frac{\partial u_i}{\partial x_j} - \rho \overline{u'_i u'_j} \right) + \rho f_i, \quad (3)$$

158 where the velocity terms have been separated into Reynolds-averaged components, u_i , and fluctuating
 159 components, u'_i . Reynolds-averaged external forces and pressure are denoted by f_i and P respectively.

160 A shear stress transport (SST) turbulence closure has been used to model the Reynolds stresses, $-\rho \overline{u'_i u'_j}$.
 161 This combines the free-stream capability of the popular $k-\epsilon$ model with the explicit wall resolution of the $k-\omega$
 162 model, and was found to perform better when compared with the laboratory data. It is still a two-equation eddy
 163 viscosity model, with transport equations for k , the turbulent kinetic energy and ω , the turbulence frequency.
 164 However, blending functions are utilised in order to exploit the near-wall treatment of the $k-\omega$ model and the

165 free-stream capability of the $k-\epsilon$ model [Menter, 1994]. A more detailed description can be found in the
 166 supplementary material.

167 To model variations in flow density, a mixture model was employed. This requires the solving of one
 168 conservation of mass equation (2) and one conservation of momentum equation (3) for the mixture. In this
 169 case, the mixture comprises water and saline with densities $\rho_w = 1000 \text{ kg/m}^3$ and $\rho_s = 1025 \text{ kg/m}^3$,
 170 respectively. The density of the mixture is defined by $\frac{1}{\rho} = \frac{1-\alpha}{\rho_w} + \frac{\alpha}{\rho_s}$, where α is the saline mass fraction. This
 171 variable density is used in all terms of the model, including that of gravity. Additionally, a transport equation is
 172 solved for the saline mass fraction,

$$173 \quad \frac{\partial \alpha \rho}{\partial t} + \nabla \cdot (\alpha \rho \mathbf{u}) = -\nabla \cdot (\overline{\alpha' \rho \mathbf{u}'}), \quad (4)$$

174 where the Reynolds flux term is modelled using the eddy diffusion hypothesis as,

$$175 \quad -\overline{\alpha' \rho u_j'} = \frac{\mu_t}{\sigma_t} \frac{\partial \alpha}{\partial x_j}, \quad (5)$$

176 and μ_t , and $\sigma_t = 1$, are the eddy viscosity and turbulent Schmidt number respectively. Flow conditions and
 177 channel morphology were kept identical to laboratory values. Two larger flows with flow rates of 3 and 4 l/s,
 178 higher than was possible in the laboratory, were also simulated. Moreover, to investigate the role of Reynolds
 179 number, a set of flows were simulated in a channel 4 times larger than in the laboratory. Flow rates were scaled
 180 upwards by a factor of 16 to ensure the same flow rate per unit area. Table 3 shows the bulk quantities of
 181 these flows.

182 Use and validation of this modelling approach is extensive both in this field [e.g. Imran et al., 2004, 2007;
 183 Giorgio Serchi et al., 2011] and related fields [e.g. Gauer et al., 2005; Doronzo, 2013]. Additionally, the
 184 numerical model has been compared to the experimental data in this study (Section 3.1).

Input Flow Rate (l/s)	0.2	1	2	3	4	3.2	16	32	48	64
h (cm)	3.05	4.36	5.01	5.43	5.69	11.7	15.1	17.2	18.5	19.4

U (m/s)	0.111	0.151	0.175	0.194	0.212	0.179	0.294	0.325	0.353	0.377
Re	3390	6580	8770	10500	12100	20900	44300	55900	65100	73300
Fr	1.64	1.84	1.96	2.03	2.15	1.77	1.89	1.89	1.97	2.06
Ri	0.372	0.295	0.260	0.243	0.216	0.321	0.251	0.281	0.257	0.235

185 **Table 3:** Bulk flow properties of the numerically simulated flows calculated from channel thalweg data, 1 m
186 downstream from the main tank inlet. Flows in the left column traverse the laboratory scale channel, with the
187 flows in the right column traversing a channel scaled 4 times larger.

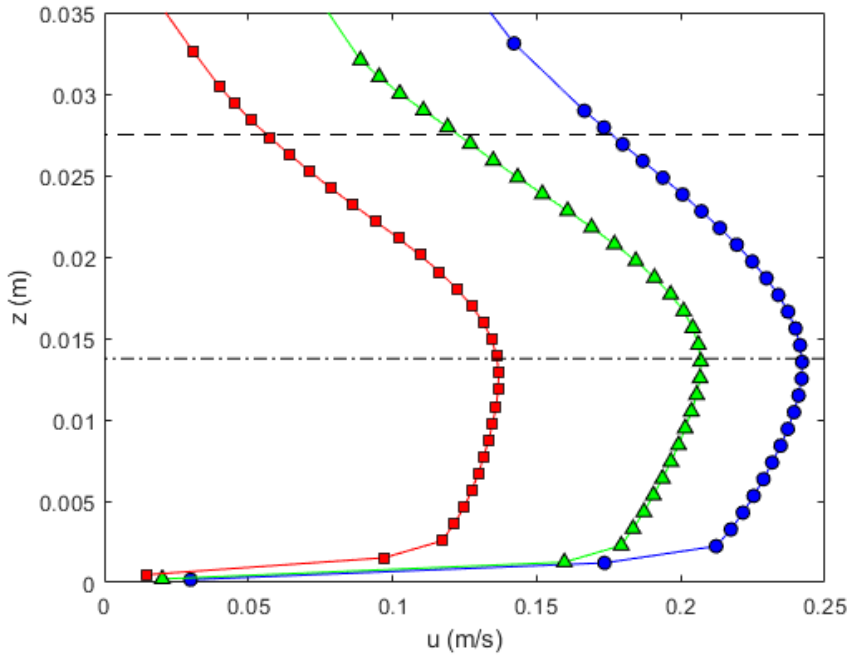
188 **3 Results:**

189 **3.1 Velocity and density structure:**

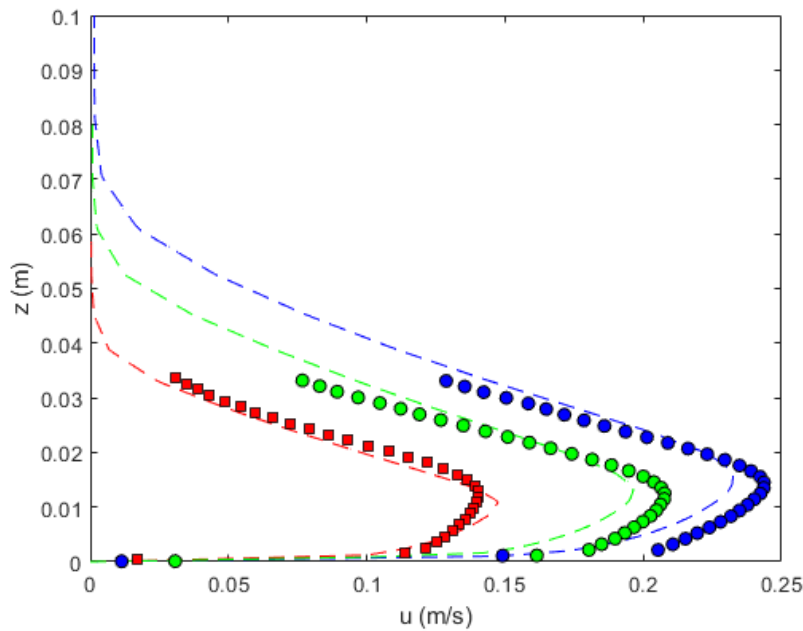
190 The velocity profiles of the three laboratory flows are shown in Figure 4. These were captured with an ADV 1
191 m downstream of the main tank inlet to allow the flows to develop. As has been observed in many previous
192 studies [e.g. Ellison and Turner, 1959; Garcia and Parker, 1993; Islam and Imran, 2010] all profiles exhibit a
193 lower shear layer caused by basal drag and an upper shear layer caused by drag and subsequent mixing with
194 the ambient fluid. These are separated by a velocity maximum. Here, the height of the velocity maximum
195 remains almost constant for all flows at a height equal to half the channel depth. This is despite the changes
196 in flow height, discharge, and Richardson number, suggesting that channel depth is a key control on partially-
197 confined flow development.

198 The numerical simulations predict velocity profiles that compare well with the laboratory data (Figure 5) and
199 model performance is comparable to previous gravity current studies [e.g. Huang et al., 2005; Giorgio Serchi
200 et al., 2011]. Except for the bank-full flow, the constant velocity maximum height is replicated (Figure 6) and
201 the simulations show it remains constant at flow magnitudes larger than were possible in the laboratory. The
202 upper shear layers are captured well, although the numerical simulations predict slightly different magnitudes
203 for the maximum velocity and lower shear layer. In accordance with previous laboratory [e.g. Sequeiros et al.,
204 2010; Islam and Imran, 2010] and numerical studies [e.g. Imran et al., 2004, 2007; Giorgio Serchi et al., 2011;
205 Kneller et al., 2016], the simulations provide density data that show a stratified region below the velocity

206 maximum with an increasingly mixed region above. The collapse of the simulated profiles in the lower shear
207 layer (Figure 6) shows the bank-full flow to be characteristically different to the larger, overspilling flows,
208 suggesting that overspill plays an important role in the development of flow structure.

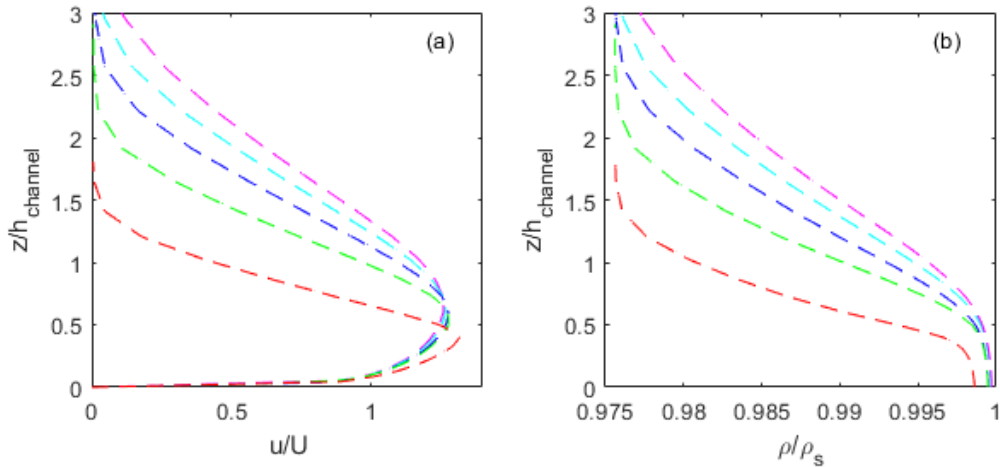


209
210 **Figure 4:** Channel thalweg ADV velocity profiles measured 1 m downstream from the main tank inlet, time-
211 averaged over a 3 minute period. Red squares – 0.2 l/s; Green triangles – 1 l/s; Blue circles – 2 l/s. The dashed
212 lines indicate channel depth and half channel depth. The height of the velocity maximum remains almost
213 constant despite changes in flow rate and depth. This is in contrast to confined flows where velocity maximum
214 height scales with flow depth.



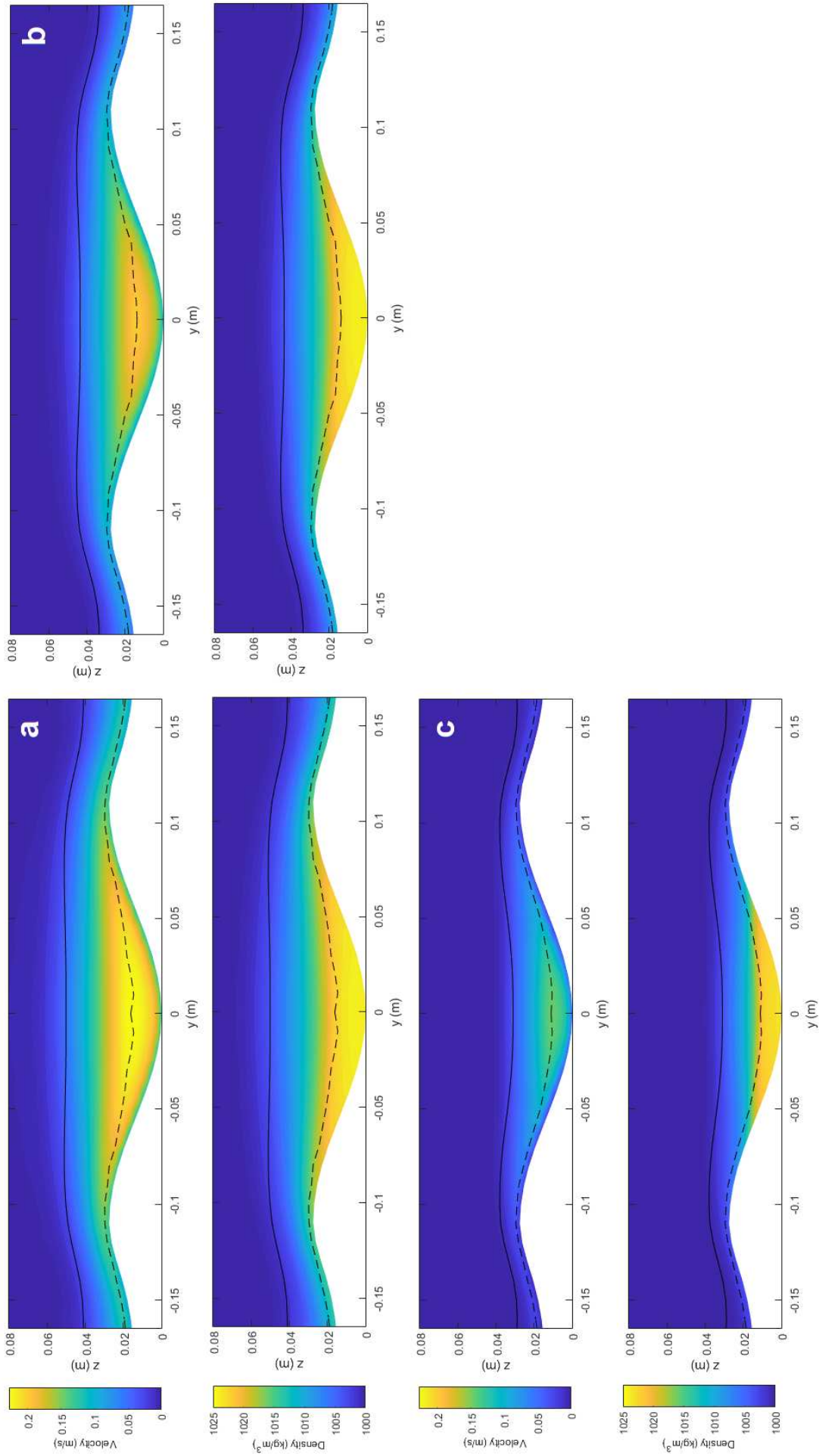
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216 **Figure 5:** Channel thalweg ADV (symbols) and numerical (dashed lines) velocity profiles, measured 1 m
 217 downstream from the main tank inlet and time-averaged over a 3 minute period. Red- 0.2 l/s; Green - 1 l/s;
 218 Blue - 2 l/s. Data is not normalised to explicitly show similarities and differences.



219

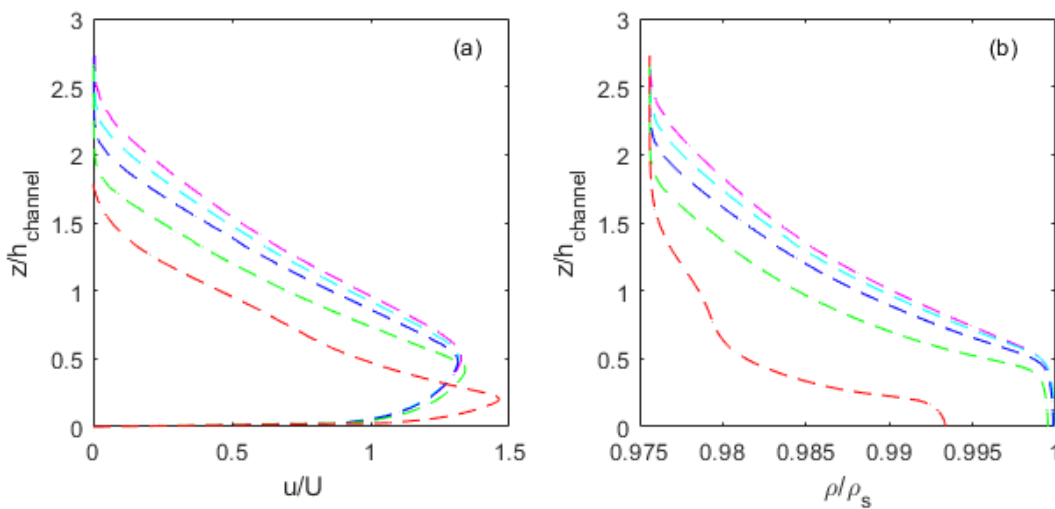
220 **Figure 6:** Channel thalweg numerical velocity (a) and density (b) profiles, normalised with depth averaged
 221 velocity/saline density and channel depth, measured 1 m downstream from the main tank inlet and time-
 222 averaged over a 3 minute period. Red – 0.2 l/s; Green – 1 l/s; Blue 2 l/s; Cyan – 3 l/s; Magenta 4 l/s. Numerical
 223 simulations show a constant velocity maximum height for larger flow rates and heights than could be achieved
 224 in the laboratory. With the exception of the bank-full flow (red trace), both velocity and density profiles collapse
 225 well in the lower shear layer where large levels of stratification are present.



227 **Figure 7:** a) 2 l/s; b) 1 l/s; c) 0.2 l/s. Numerical velocity and density contours. The solid line shows the flow
228 height and the dashed line shows the velocity maximum height. A non-mixed, stratified region below the
229 velocity maximum height is evident in all flows. The channel also appears to maintain a confined 'high-velocity
230 core'.

231 3.2 High Reynolds number simulations

232 In order to investigate the effect of Reynolds number, flows were simulated in a channel scaled four times
233 larger than the laboratory geometry. To compare to the laboratory scale flows, flow rates were scaled upwards
234 by a factor of 16 to keep the same flow rates per unit area. The resultant flows had Reynolds numbers between
235 20,900 and 73,300 (Table 3). The thalweg velocity and density profiles are shown in Figure 8. Similarly to the
236 laboratory scale flows, the height of the velocity maximum of these larger remains fixed at around half the
237 channel depth. The smallest, bank-full flow shows distinctly different characteristics with a relatively faster,
238 more mixed core.



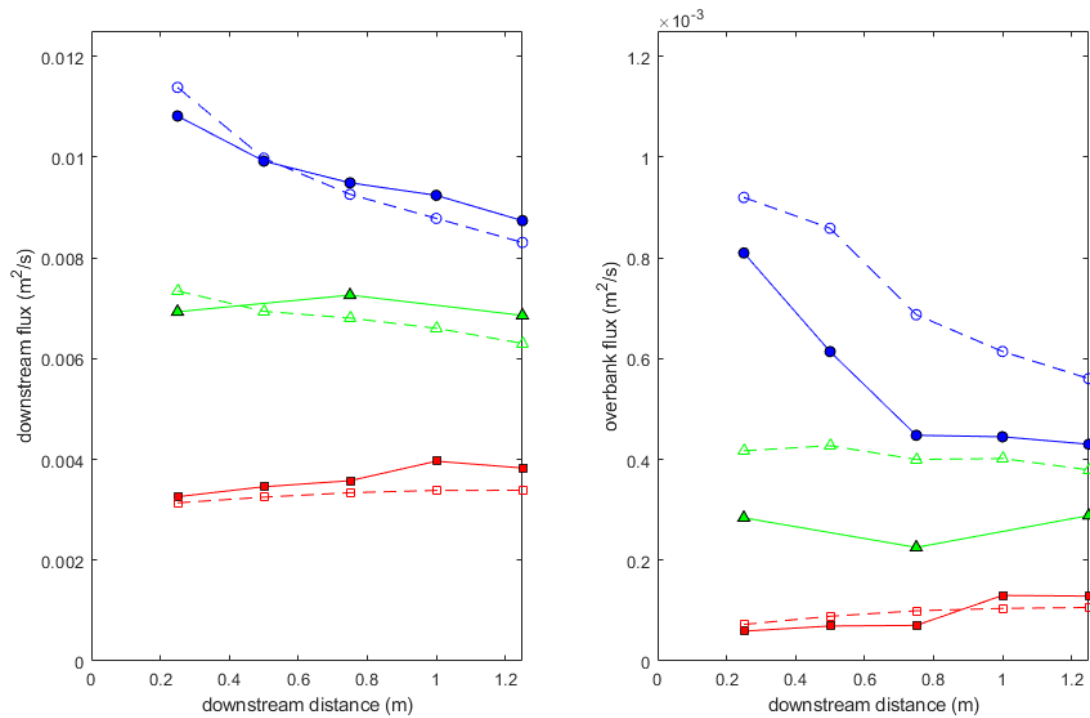
239 **Figure 8:** Channel thalweg numerical velocity (left) and density (right) profiles for the higher Reynolds number
240 flows traversing the scaled-up channel. Profiles are normalised with depth averaged velocity/saline density
241 and channel depth, measured 4 m downstream from the main tank inlet. Red – 3.2 l/s; Green – 16 l/s; Blue 32
242 l/s; Cyan – 48 l/s; Magenta 64 l/s.

244 3.3 Flow evolution and overspill:

245 Total streamwise and overbank discharges are shown in Figure 9 using both the laboratory and numerical
246 data. The simulations predict the downstream discharge well, showing close agreement with both the

247 magnitudes and the spatial evolution. The downstream evolution of the overbank losses is also predicted well,
248 although magnitudes for the two larger flows were over-predicted by 13-73%.

249 The three currents clearly interact with the channel in different ways. The bank-full current is dominated by
250 ambient entrainment and as a result the streamwise discharge increases downstream. Overbank losses
251 subsequently also increase as the current inflates and overspills the confinement of the channel. Both the
252 streamwise discharge and overbank losses of the equilibrium current remain fairly constant, suggesting a
253 balance between entrainment and overspill. The oversize current exhibits large initial overbank losses which
254 result in a reduction in streamwise discharge. Overspill rates reduce rapidly downstream however as the
255 current size reduces. These are examples of the two main ways - inflation vs. deflation - in which a current can
256 evolve and be 'tuned' to equilibrium by a channel.



257

258 **Figure 9:** Downstream evolution of streamwise and overbank discharges from laboratory data (solid) and
259 numerical simulations (dashed). Red – 0.2 l/s; Green – 1 l/s; Blue 2 l/s. The simulations predict the spatial
260 evolution well, although they overestimate the magnitude of overspill for the two larger flows. Flow tuning is
261 evident in the different ways each flow evolves. Both the streamwise and overbank discharge of the 0.2 l/s
262 flow increase downstream as ambient fluid is entrained and the flow inflates. The discharges of the 1 l/s flow
263 remain relatively constant indicating a close-to-equilibrium balance between overbank losses and ambient
264 entrainment. The discharge of the 2 l/s flow changes rapidly with large initial overbank losses. The streamwise

265 discharge continues to reduce downstream, despite ambient entrainment.

266 3.4 Entrainment:

267 The entrainment of a flow can be found by a depth integration of the incompressibility equation,

$$268 \quad \frac{\partial}{\partial x} \int_0^\infty u \, dz + \frac{\partial}{\partial y} \int_0^\infty v \, dz + w_\infty = 0, \quad (6)$$

269 where $w_\infty = \partial h / \partial t - w_e$ is a product of the shallow-water approximation [Parker, 1986]. Assuming a temporally
270 stable flow, and using definitions in Table 1, this becomes,

$$271 \quad e_w |U| = \frac{\partial U h}{\partial x} + \frac{\partial V h}{\partial y}, \quad (7)$$

272 where the entrainment velocity, $w_e = e_w |U|$, has been defined as a product of the entrainment coefficient, e_w ,
273 and the depth-averaged velocity magnitude of the flow. The entrainment coefficient describes the ability of a
274 flow to entrain ambient fluid. For fully-confined flows with no cross-stream variation, (7) becomes,

$$275 \quad e_w U = \frac{\partial U h}{\partial x}, \quad (8)$$

276 which is the standard form used for confined laboratory flows [Parker, 1987]. For partially-confined flows in a
277 straight channel, when integrated across the channel from thalweg to crest, (5) becomes,

$$278 \quad \hat{e}_w |\widehat{U}| Y = \frac{\partial \widehat{U} A}{\partial x} + V(Y) h(Y), \quad (9)$$

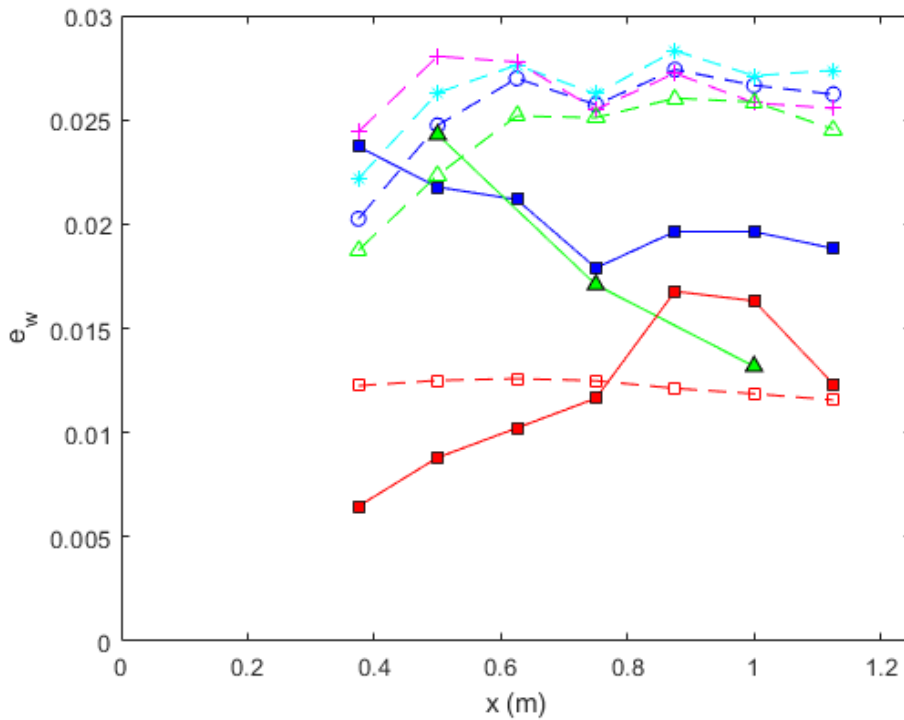
279 where the cross-sectional area of the current is defined as $A = \int_0^Y h \, dy$, channel average velocities as $|\widehat{U}| =$
280 $(\int_0^Y \int_0^h |u| \, dz \, dy) / A$, the channel average entrainment coefficient as $\hat{e}_w = (\int_0^Y e_w |U| \, dy) / |\widehat{U}| Y$, and Y is half
281 the channel width. The values of e_w presented here are all calculated using (9). If (8) is used for an overflowing,
282 partially-confined flow, negative values will be observed if the current is deflating. Such a current is still clearly
283 entraining ambient fluid and shows how overspill must be taken into account when analysing the entrainment
284 characteristics of such flows. A channel-average Richardson number, defined as the mean of the thalweg and
285 crest Richardson number, is also used in order to account for cross-stream variations.

286 Both the laboratory and numerical data output entrainment coefficients of the same order of magnitude (Figure
287 10), with the range of simulated values overlapping with the laboratory counterparts. However the simulated

288 values, based on the numerical velocity and density data, largely predict higher values. This is attributed mostly
289 to the overprediction over overbank losses (Figure 9). The difference between simulated and laboratory values
290 is largest for the 1 l/s flow which is attributed to the lower longitudinal resolution in the laboratory data for this
291 flow. A clear difference can be seen between the bank-full and the larger, overflowing flows. The dependence
292 of e_w on Richardson number for fully-confined flows, described by Parker et al., [1987] using (1), still appears
293 to hold for the partially-confined setting. Figure 11 shows how the data presented here fall within the scatter of
294 the previous laboratory data. However, there is also an apparent upper bound on e_w for these partially-confined
295 flows. Neither an increase in flow magnitude, nor a reduction in Richardson number, results in a change in e_w
296 (Figures 10 and 11), perhaps suggesting a limit imposed on the entrainment ability of a current by the channel.

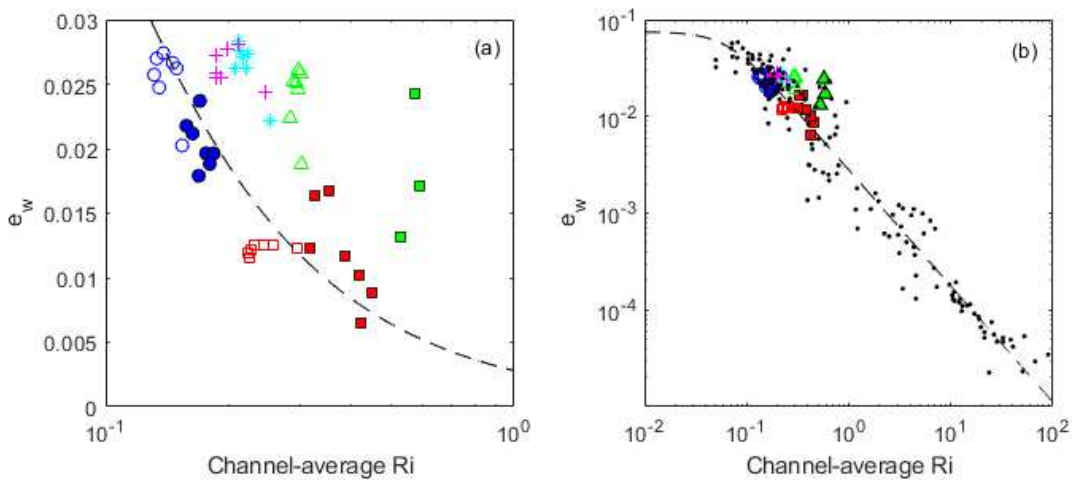
297 Further evidence for the ‘tuning’ effect of the channel described above is displayed in Figure 12. The
298 downstream evolution of the Richardson number shows how each flow approaches an equilibrium. This is
299 particularly evident in the thalweg. Cross-sectional contours of gradient Richardson number in Figure 13,
300 produced using numerical simulation data, show how the stability of the stratification varies throughout each
301 of the flows. The vertical structure is typical of a gravity current [Kneller et al., 2016], with values approaching
302 infinity around the velocity maximum due to the reversal of the velocity gradient while a less stable layer above
303 this that helps to drive entrainment. Here, localised low gradient Richardson regions are seen over the levee
304 crests.

305 A reduction in bulk Richardson number is also seen over the levee crests for all flows. Similar cross-stream
306 variations and magnitudes are found for the gradient Richardson number when depth-averaged over the upper
307 shear layer. The depth-averaging region was defined to be between 0.5 and 2.5 standard deviations above
308 the velocity maximum, found by approximating the upper velocity profile with a Gaussian distribution. This
309 region was chosen to include the entire upper shear layer which is responsible for ambient entrainment while
310 excluding the very high magnitudes found around the velocity maximum. This region also spans above the
311 flow height determined by the Ellison and Turner [1959] definition (Table 1) which is used in the calculation of
312 bulk quantities.



313

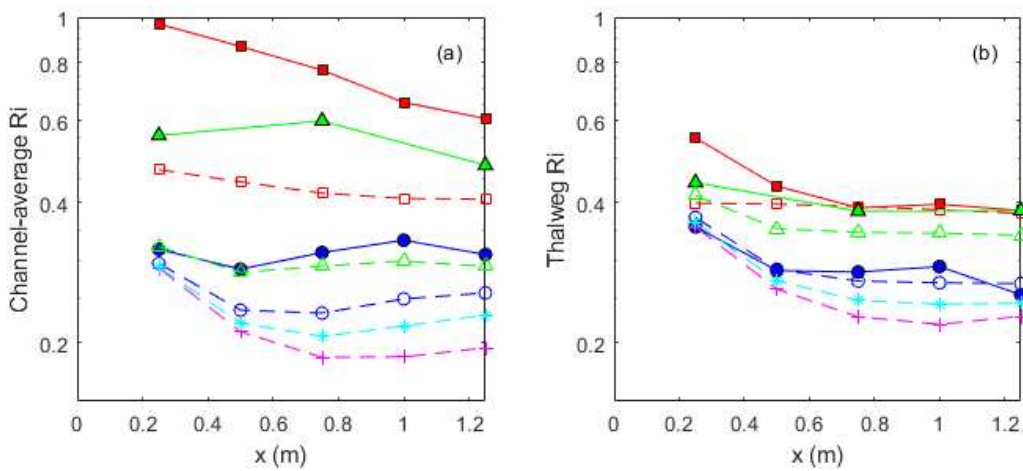
314 **Figure 10.** Downstream evolution of entrainment coefficient. Laboratory – solid; Numerical – dashed. Red –
 315 0.2 l/s; Green – 1 l/s; Blue 2 l/s; Cyan – 3 l/s; Magenta 4 l/s. The magnitudes of the entrainment coefficient
 316 show overlap between the numerical and experimental data, although the simulations largely predict slightly
 317 higher values.



318

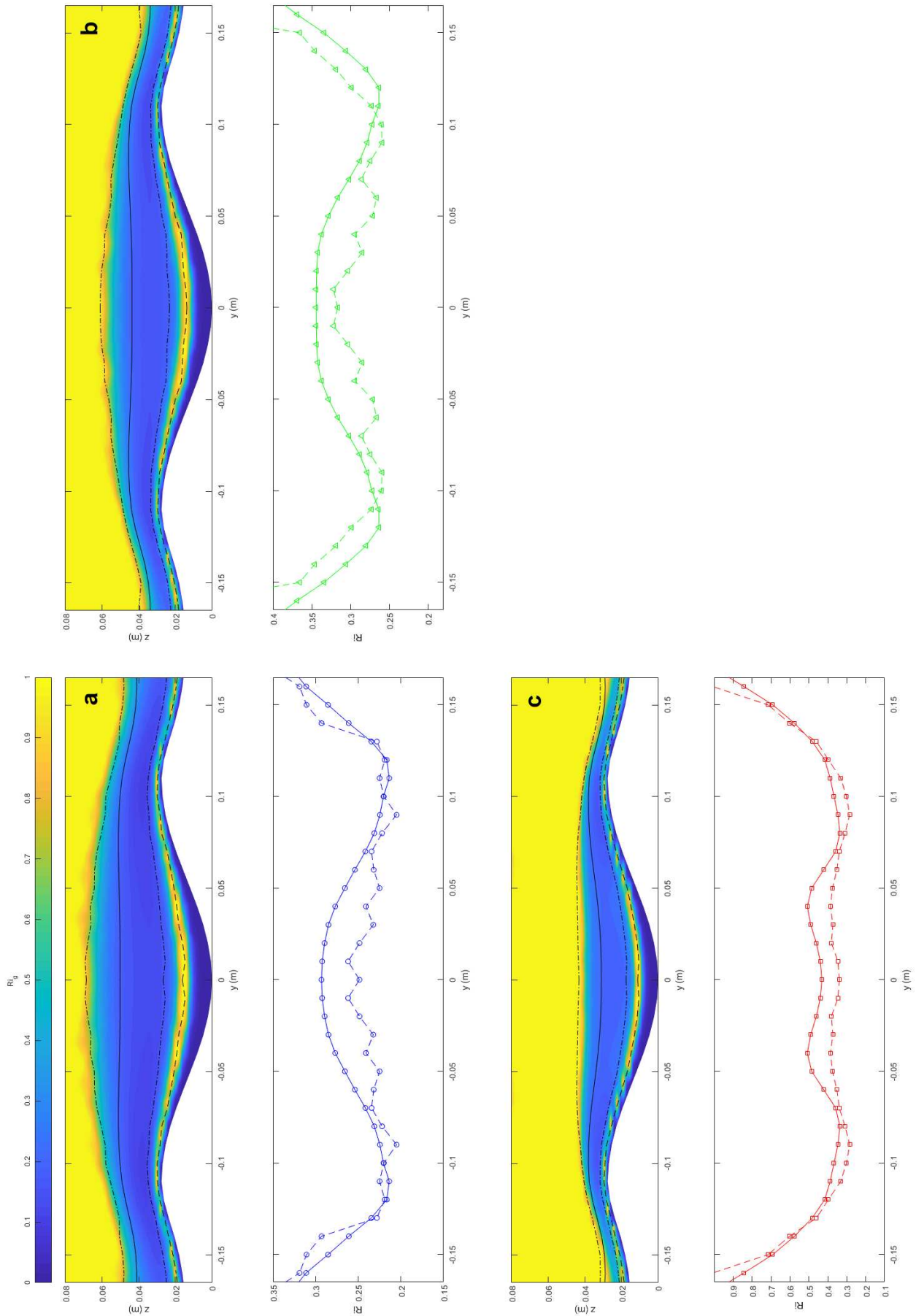
319 **Figure 11.** Entrainment coefficient is dependent on the (channel-average) Richardson number. Laboratory –
 320 filled; Numerical – hollow. Red – 0.2 l/s; Green – 1 l/s; Blue 2 l/s; Cyan – 3 l/s; Magenta 4 l/s. Data shown on
 321 a linear (a) and logarithmic (b) axis. The dashed line indicates the Parker et al. [1987] relationship (1). Previous
 322 experimental data from confined flows, collated by Parker et al., are shown in black on the right [Ellison and
 323 Turner, 1959; Lofquist, 1960; Ashida and Egashira, 1975]. The standard deviation of the entrainment

324 coefficient from the defined relationship is 0.041 for the previous confined data and 0.015 for the data
325 presented here.



326

327 **Figure 12.** Downstream development of channel-average (a) and thalweg (b) Richardson number. Laboratory
328 – solid; CFD – dashed. Red – 0.2 l/s; Green – 1 l/s; Blue 2 l/s; Cyan – 3 l/s; Magenta 4 l/s. CFD density data
329 are used in the calculation of the laboratory values in the absence of laboratory density data. There is an
330 adjustment period before each flow approaches an equilibrium Richardson number, the distance of which is
331 dependent on flow magnitude.



332

333 **Figure 13:** a) 2 l/s; b) 1 l/s; c) 0.2 l/s. Gradient Richardson contours for each flow rate exhibit regions of

334 decreased magnitudes above the levee crests, and indication of decreased stability and increased mixing.
335 Both the cross-stream variations and magnitudes of the bulk Richardson number (solid line) are comparable
336 with the depth-averaged gradient Richardson number (dashed line). The bulk Richardson number would
337 appear to be a good proxy for the gradient Richardson number in the upper shear layer and a good indication
338 of mixing levels. The depth-average was calculated between 0.5 and 2.5 standard deviations (dash-dot lines)
339 above the velocity maximum (dashed line). The flow height is also shown with a solid line.

340 **4 Discussion:**

341 **4.1 Channel forcing**

342 The occurrence of overspill and associated inherent cross-stream variation mean the dynamics of a partially-
343 confined flow are fundamentally different to those of a fully-confined flow. For a fully-confined flow, the velocity
344 maximum height, h_{max} , is determined solely by the balance between basal and ambient drag [Middleton, 1993];
345 h_{max} scales with height, with values observed between $h_{max}/h = 0.1$ [Buckee et al., 2001] and $h_{max}/h = 0.3$
346 [Kneller et al., 1999]. Variations are to be expected with differences in basal materials, laboratory conditions
347 and the difficulty in defining a current's height. A dependence of h_{max} on both the flow's Richardson number
348 [Sequeiros et al., 2010] and Reynolds number [Stagnaro and Pittaluga, 2014] has also been observed. For the
349 partially-confined flows analysed here, h_{max} remains nearly constant for all the laboratory-scale flows at a
350 height equal to half the channel depth, regardless of flow height or Richardson number. This could suggest an
351 increase in the ratio of ambient to basal drag for larger flows, perhaps due to the increase in overspill and the
352 surface area of the ambient interface. For the upscaled flows, described in Section 3.2, the smaller flows have
353 a relatively lower position of h_{max} . This can be explained by the basal drag remaining constant but ambient
354 drag increasing with Reynolds number. However, half the channel depth remains as an upper limit on h_{max} for
355 the larger flows indicating that, even at large Reynolds numbers, channel depth remains a first-order control
356 on flow structure.

357 It would appear the channel has the ability to maintain a high velocity 'core' (illustrated in Figure 7). A value of
358 h_{max} less than the channel depth allows the current to maintain a highly stratified lower region confined by the
359 base of the channel. This region provides a gravitational driving force that is sustained along the length of the
360 channel and enables the possibility of a stable downstream flow evolution pattern. The forcing on the current
361 exerted by the channel is therefore further confirmed as a key control on the flow dynamics and can be
362 recognised as an important mechanism in sustaining current run-out.

363 It is unclear at what point h_{max} could exceed the channel depth, although this would make a rapid dissipation
364 of the current likely, with the lower region no longer fully restricted and nothing to prevent lateral spreading. In
365 a laboratory study with varying levels of flow confinement, Mohrig and Buttles [2007] defined a threshold of
366 $h/H > 5$, where H is the channel depth to differentiate confined vs. effectively unconfined flow. It was proposed
367 that at this threshold the high velocity core exceeds the confines of the channel, resulting in an unconfined
368 flow, although there was no vertical resolution in the velocity data which were acquired from overhead
369 cameras. The laboratory and simulated flows described here have values in h/H ranging from 1.15 to 3. While
370 none of these flows approach the $h/H > 5$ threshold, the constant height of the velocity maximum suggests
371 any transition would not be gradual.

372 **4.2 Numerical model performance**

373 A numerical RANS model with a shear stress transport turbulence closure has been used to simulate flows
374 with magnitudes too large to produce in this laboratory setup and investigate the role of higher Reynolds
375 numbers. Performance, in terms of agreement with laboratory velocity data, is comparable to those of similar
376 models [e.g. Imran et al., 2007; Giorgio Serchi et al., 2011]. Crucially, the numerical model helps to show how
377 the constraint of the half channel depth on the velocity maximum height is not an artefact of the lower Reynolds
378 numbers found in the laboratory. An increase in Reynolds number (Section 3.2), and the resultant increase in
379 ambient drag, has limited impact on this upper constraint. The comparison between the laboratory and
380 numerical velocity profiles (Figure 5) shows reasonably good agreement, particularly with the velocity gradients
381 in the shear layers. However, there are still clear differences between the simulations and the experiments.
382 While the velocity maximum heights are predicted well for the 1 and 2 l/s flows, the height is underpredicted
383 for the smallest 0.2 l/s flow. There are also discrepancies of up to 7% in the magnitudes of the velocity maxima.
384 The modelling of the stratification and subsequent levels of overbank losses could be one source of these
385 errors, with Figure 9 showing significant overprediction of overbank loss. Furthermore, the time-averaging
386 introduced in RANS modelling could not completely capture the effect of large-scale, transient flow features
387 such as the mixing introduced by Kelvin-Helmholtz instabilities at the ambient interface. Finally, the use of
388 numerical density data in the calculation of laboratory Froude number and entrainment coefficient values
389 means that discrepancies in these areas are introduced solely from the observed differences in velocity data.

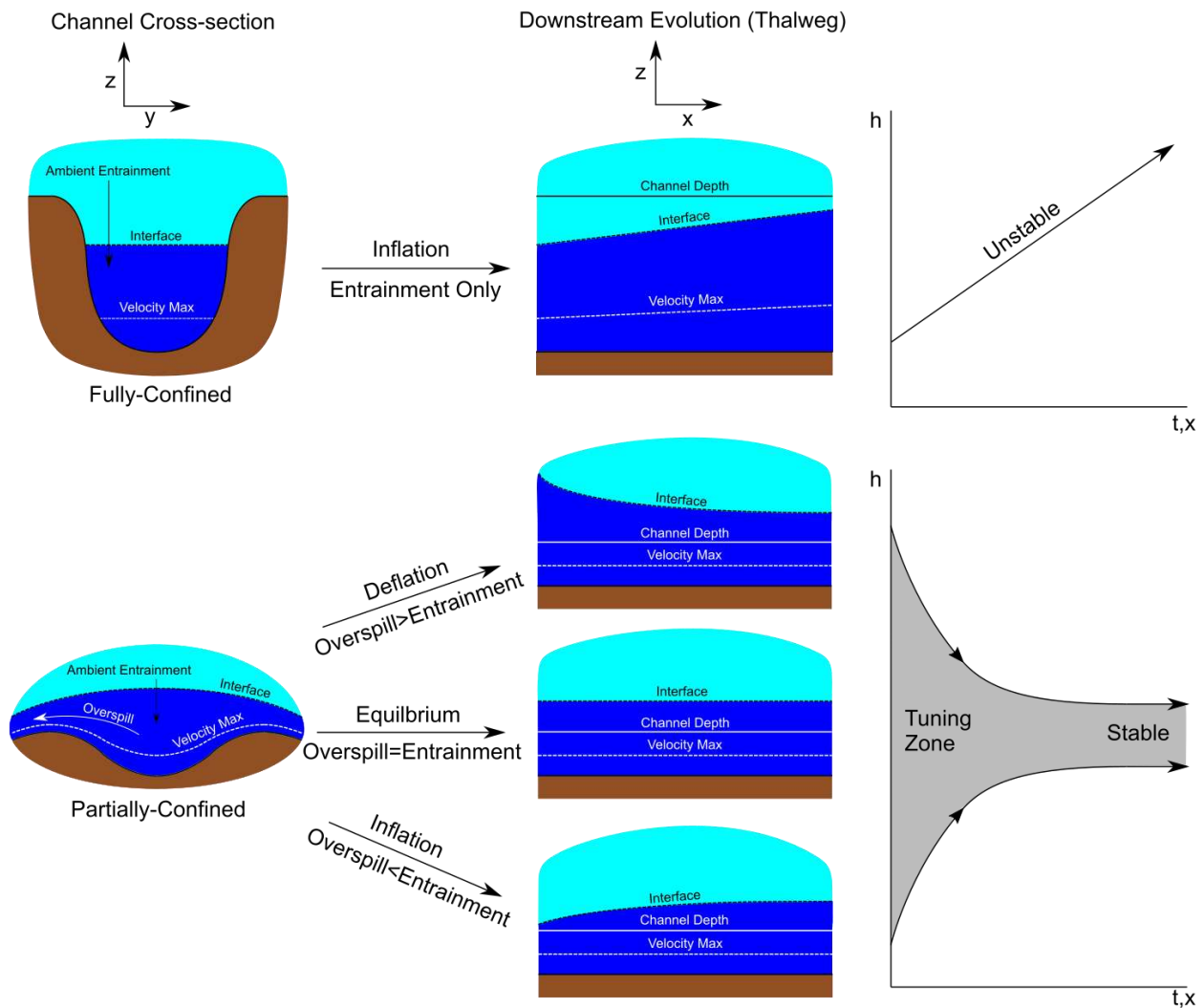
390 **4.3 Flow tuning**

391 A channel is clearly capable of 'tuning' oversize flows via overspill, with deflation and flow stripping occurring
392 here for flows with $h/H > 1.9$. Mohrig and Buttles [2007] also observed this tuning effect, reporting flows with

393 $h/H > 1.3$ undergoing deflation until a constant flow height was reached. At the laboratory scale at least, such
394 oversize flows appear to be unable to propagate in a partially-confined setting. While it is therefore unlikely the
395 $h/H > 5$ threshold would be breached via gradual flow evolution, external factors could trigger this scenario.
396 A current emerging from a canyon system could be disproportionately deep before being stripped or thinned by
397 the channel, analogous to the oversize current described here that experienced significant overspill proximally
398 (Figure 9). A break in slope, as often seen at a channel-lobe transition zone [Wynn et al., 2002; Dorrell et al.,
399 2016], could also cause a sudden thickening of the flow and a subsequent avulsion or transition to
400 unconfinement. Additionally, increasing channel instability, caused by continual deposition, could lead to a
401 channel being unable to provide the necessary degree of confinement to contain the high velocity core [Dorrell
402 et al., 2015]. Here we are considering the dynamics of straight channel confinement; channel sinuosity leads
403 to flow elevation at bend apexes [Keevil et al., 2006; Cossu and Wells, 2010; Dorrell et al., 2013], providing an
404 additional mechanism for flow avulsion.

405 While the size of the flow can be tuned via overspill, ambient entrainment can also lead to the inflation of an
406 undersize flow. This mechanism allows the achievement of an equilibrium whereby a current's overbank losses
407 are balanced with ambient entrainment. In contrast, entrainment is the sole mechanism for fully-confined flow
408 evolution, resulting in continued inflation [Symons et al., 2017]. Here, a quasi-equilibrium current, characterised
409 by $h/H = 1.75$, can be identified in the 1 l/s case. Both streamwise and overbank discharges remain relatively
410 constant along the length of the channel (Figure 9). Further evidence of tuning can be seen in Figure 12. Each
411 flow must propagate for a characteristic length before attaining a constant Richardson number, with the
412 magnitude of this length correlated with the size of the flow. It is unlikely, however, that for a given channel
413 geometry, there exists a unique equilibrium flow condition that all currents evolve towards regardless of input.
414 Rather, a partially-confining channel allows a range of currents to develop a balance between ambient
415 entrainment and overbank losses which allows stable downstream evolution. These mechanisms are
416 illustrated in Figure 14.

417 If a channel has the capability to modify flows along its length, an impact in the overbank deposit record would
418 be expected. Differing levels of overspill near the channel inlet followed by an approach to an equilibrium value
419 would suggest a transformation from heterogeneous overbank deposits proximally to homogenous deposits
420 distally. This is, however, based on the assumption that all overbank flow is of a similar depositional character.
421 Larger overbank flows may bypass the channel-proximal levee, significantly complicating the depositional
422 record in these locations.



423

424 **Figure 14:** Downstream evolution patterns of fully and partially-confined flows. Entraining fully-confined flows
 425 can only inflate in an unstable evolution pattern. Partially-confined flows can either inflate or deflate to approach
 426 a stable equilibrium where overbank losses are balanced by ambient entrainment.

427 **4.4 Entrainment and cross-stream variation**

428 It can be seen from (9) that for a partially-confined flow the overspill term, Vh , has a significant impact on the
 429 entrainment. This is evident in the markedly lower entrainment coefficient values for the bank-full flow (Figure
 430 10). It is also the primary reason for the difference in simulated and laboratory values (Figure 9 shows how the
 431 numerical model over-predicts overspill levels for the larger flows). It is therefore slightly surprising that, for a
 432 given Richardson number, these partially-confined flows exhibit similar entrainment rates to fully-confined flows
 433 (Figure 11), despite the differences in flow dynamics described above, such as the occurrence of overspill. It
 434 should be noted that the calculation of the Richardson numbers for the laboratory flows is dependent on the
 435 numerical density data. Given the relatively low spread of this and previous data, however (see Figure 11), it

436 is unlikely any discrepancies would significantly affect the Richardson number calculations or any conclusions
437 drawn.

438 As is the case with the velocity maximum height, there does appear to be an upper limit on flow entrainment
439 efficiency. Despite an increase in input flow rate and a reduction in thalweg Richardson number (Table 3), the
440 larger 3 and 4 l/s laboratory scale flows do not exhibit higher values of entrainment coefficient. This appears
441 to be driven by a lower Richardson number at levee crests resulting in a lower channel average Richardson
442 number and the corresponding associated average entrainment characteristics. Again, the constraints of the
443 channel morphology and the increasing levels of overspill appear to be a key control on flow dynamics.

444 For all the flows considered it is important to take into account cross-stream variations, as these can be
445 significant, affecting not only calculated entrainment levels but also definitions of Richardson number. The bulk
446 Richardson number is often used as an approximation for the gradient Richardson number (see definitions in
447 Table 1), which can be used to identify regions of increased mixing due to buoyant instability. For partially-
448 confined flows, these regions occur above both levee crests (Figure 13) highlighting how mixing processes at
449 channel boundaries are key to the entrainment process. Using 2D direct numerical simulation of the Navier-
450 Stokes equations, Kneller et al. [2016] found that the bulk Richardson number was not a good measure of the
451 gradient Richardson number, which served as a good indicator to a flow's entrainment behaviour. Here though,
452 the bulk Richardson number, for all flows, appears to be a good proxy for the gradient Richardson number in
453 the upper shear layer (Figure 13). This is the region responsible for ambient entrainment and thus of most
454 interest when examining mixing rates. Both the magnitudes and the cross-stream variations are captured well
455 in the numerical modelling reported here. It is possible that the 2D nature of the simulations reported by Kneller
456 et al. [2016] may have resulted in the artificial dampening of some of the flow's mixing mechanisms.

457 **5 Conclusions:**

458 Both laboratory experiments and numerical simulations show that for a partially-confined gravity current the
459 geometry of the containing channel is a first-order control on the flow dynamics. Here, at the laboratory scale,
460 the height of the velocity maximum for a range of flows was not affected by changes in multiple factors including
461 flow height and Richardson number. The velocity maximum remained fixed at a height equal to half the channel
462 depth, which resulted in the development of a high-velocity core and highly stratified lower shear layer, both
463 confined within the channel. Numerical simulations at larger Reynolds numbers confirm the half channel depth
464 upper limit on the velocity maximum height. The channel form plays a key factor in controlling the downstream
465 evolution of the current. The joint mechanisms of overspill and ambient entrainment allow partially-confined

466 flows to either deflate or inflate towards a quasi-equilibrium state. There are significant cross-stream variations
467 in the Richardson and gradient Richardson numbers of partially-confined flow. Low Richardson number regions
468 observed over the levee crests indicate increased levels of mixing and highlight the importance of overspill in
469 the entrainment process. Despite this, the entrainment coefficients for a given Richardson number are similar
470 to those of fully-confined flows in previous studies.

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