



Wrocław University of Technology



The Future Circular Collider (FCC) project and its cryogenic challenges

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The Cryogenics Society of Europe

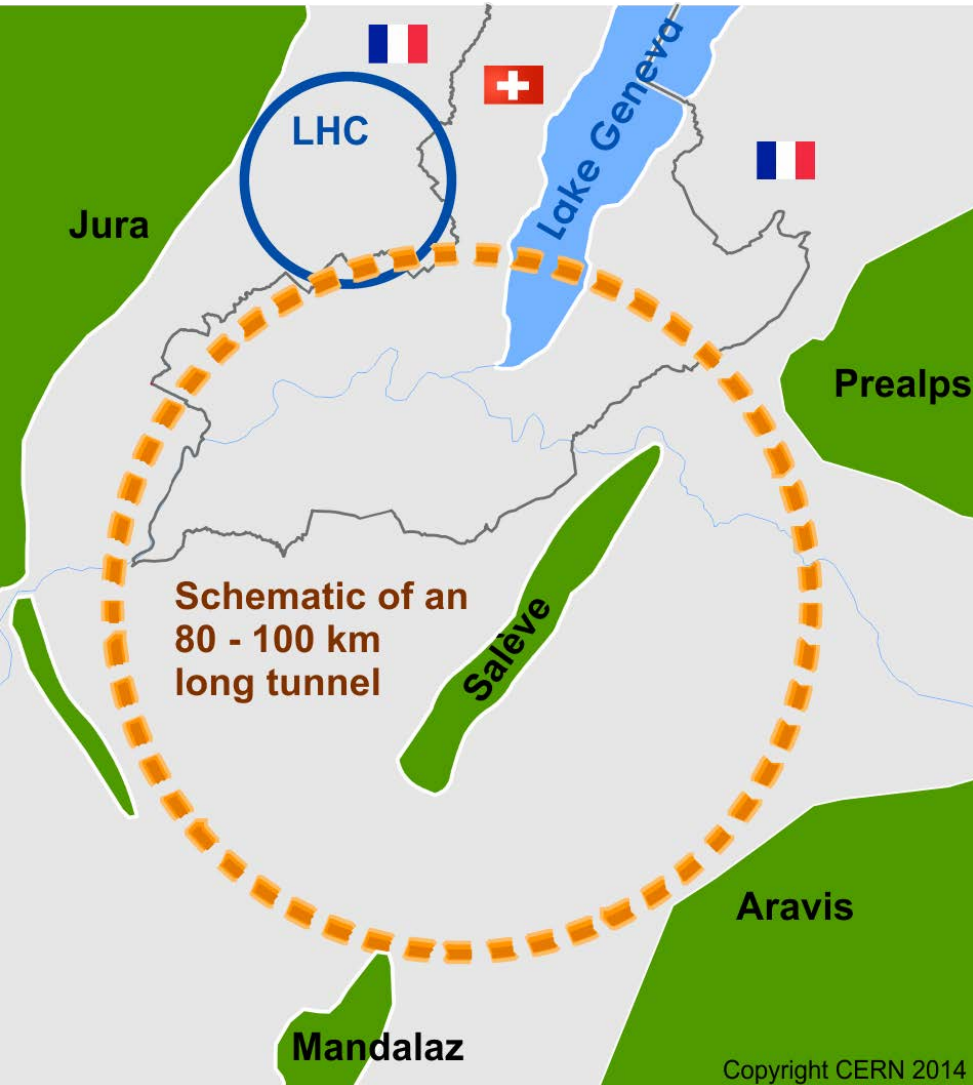




Content

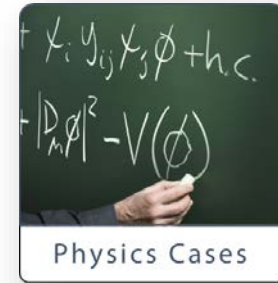


- Introduction: Scope of the FCC study
- FCC-hh tunnel cryogenics and user heat loads
- FCC-hh cryogenics layout and architecture
- FCC-hh cool-down and nominal operation
- FCC-hh electrical consumption and helium inventory
- Conclusion

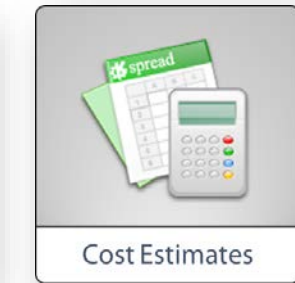


International FCC collaboration (CERN as host lab) to study:

- ***pp*-collider (*FCC-hh*)**
→ main emphasis, defining infrastructure requirements
~16 T ⇒ 100 TeV *pp* in 100 km
- **~100 km tunnel infrastructure** in Geneva area, site specific
- ***e⁺e⁻* collider (*FCC-ee*)**, as potential first step
- ***p-e* (*FCC-he*) option**, integration one IP, e from ERL
- **HE-LHC with *FCC-hh* technology**
- **CDR for end 2018**

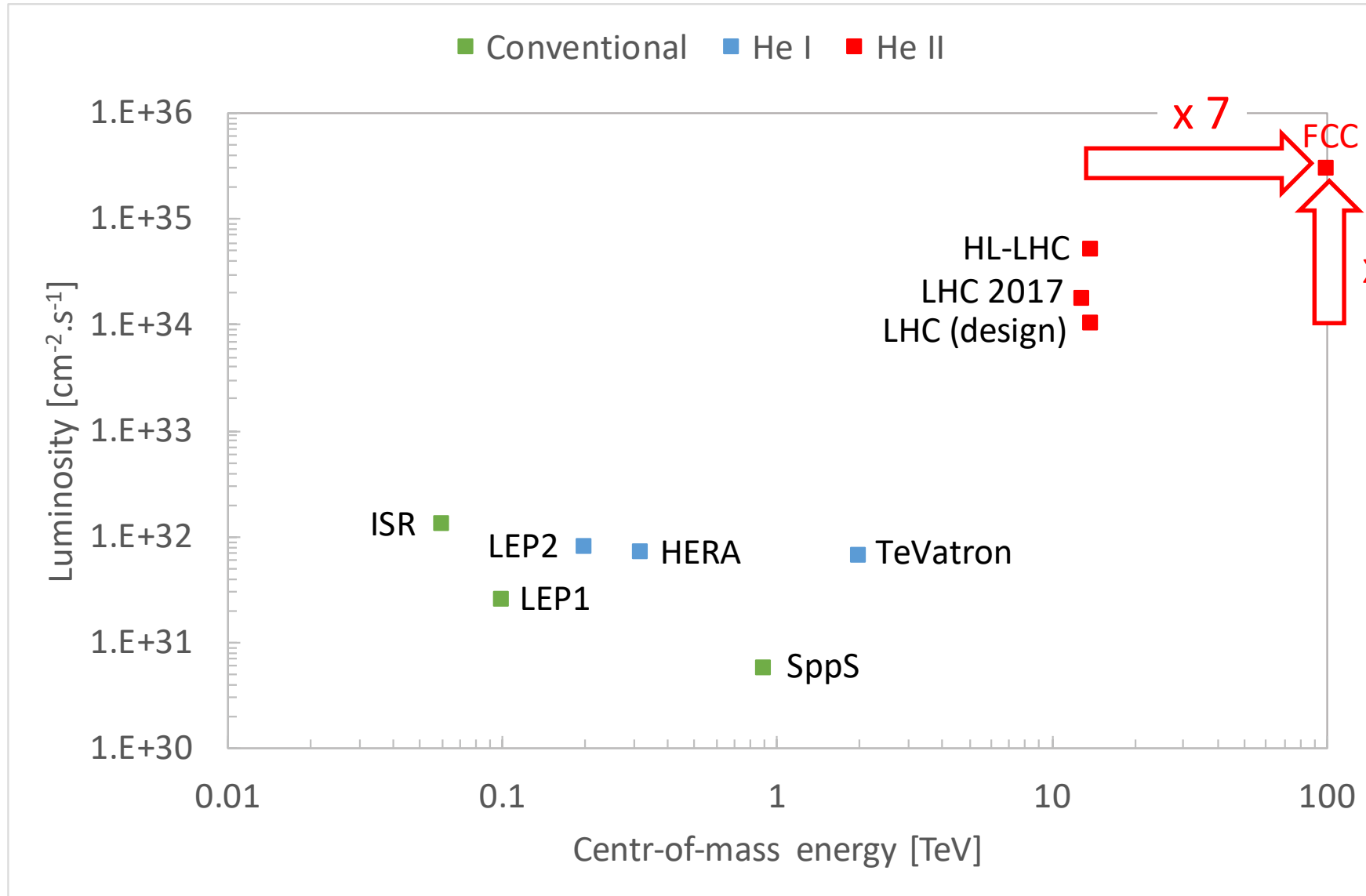


FCC-hh is the most challenging from cryogenics point-of-view



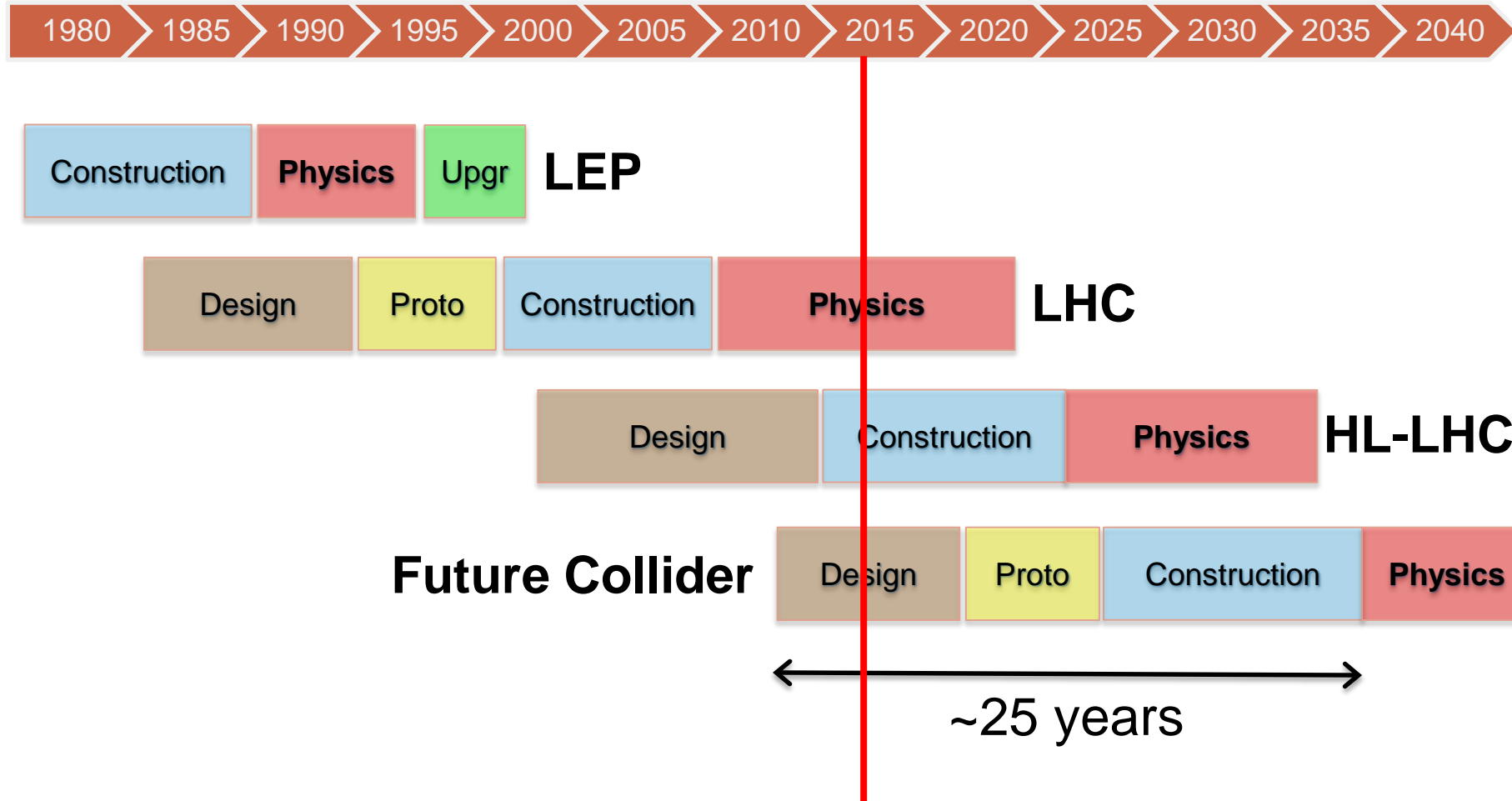


Luminosity vs energy of colliders



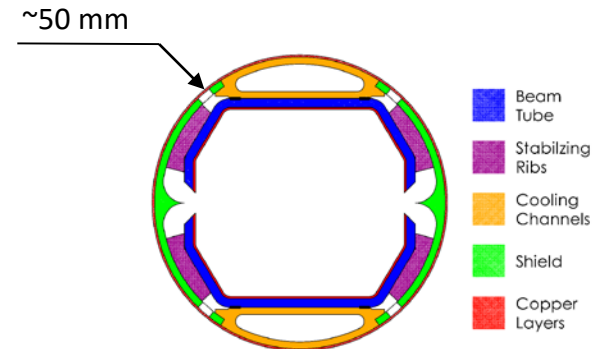


CERN Collider plan



Parameter	LHC	HL-LHC	FCC-hh
c.m. energy [TeV]	14		100
dipole magnet field [T]	8.33		16
circumference [km]	26.7		100
luminosity [$10^{34} \text{ cm}^{-2} \cdot \text{s}^{-1}$]	1	5	5 \rightarrow 29
bunch spacing [ns]	25		25
event / bunch crossing	27	135	170
bunch population [10^{11}]	1.15	2.2	1
norm. transverse emittance [μm]	3.75	2.5	2.2
IP beta-function [m]	0.55	0.15	1.1
IP beam size [μm]	16.7	7.1	6.8
synchrotron rad. [W/m/aperture]	0.17	0.33	28
critical energy [keV]	0.044		4.3
total syn. rad. power [MW]	0.0072	0.0146	4.8
longitudinal damping time [h]	12.9		0.54

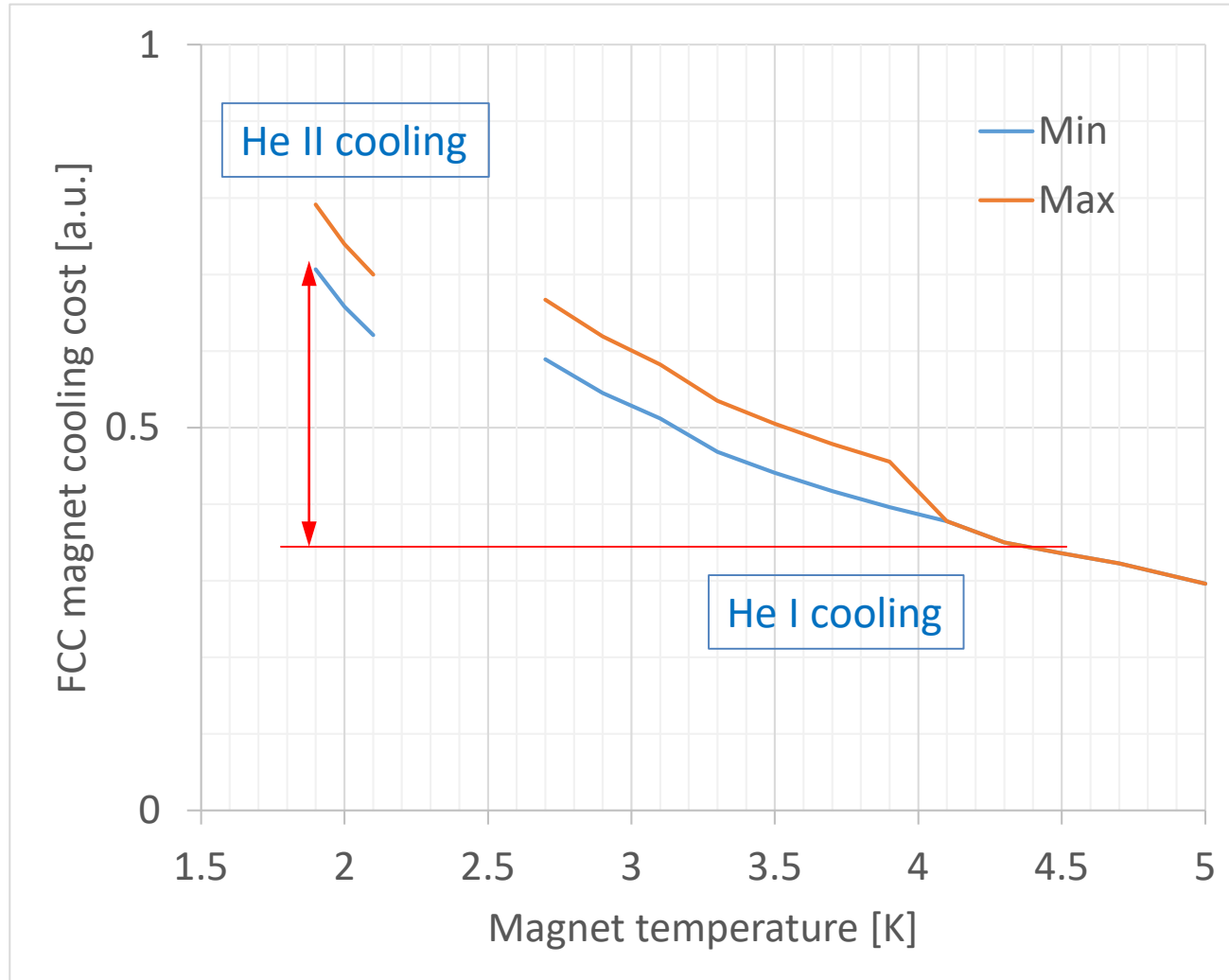
Nb₃Sn superconducting magnets cooled at 1.9 K



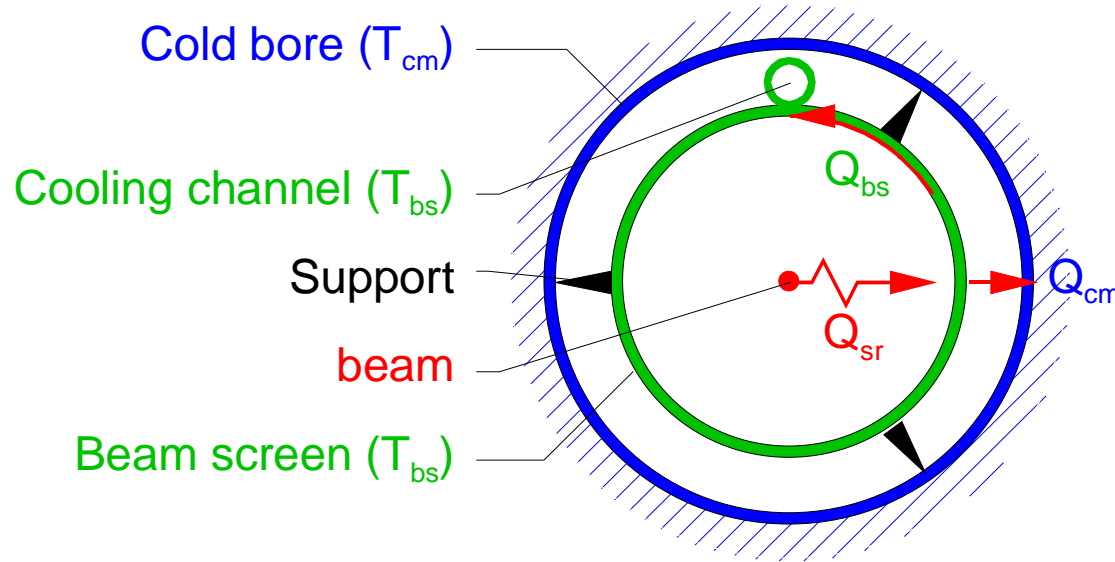
5 MW dissipated in cryogenic environment
 \rightarrow beam screens are mandatory
 \rightarrow Cooling temperature 40-60 K



Magnet cooling cost including 10 years of operation



Magnet cooling at 1.9 K vs 4.5 K:
About a factor 2 on the magnet cooling cost largely compensated by the saving on superconducting material → $T_{\text{magnet}} = 1.9 \text{ K}$



T_a : Ambient temperature

Energy balance:

$$Q_{bs} = Q_{sr} - Q_{cm}$$

- Exergy load ΔE = measure of (ideal) refrigeration duty :

$$\Delta E = \Delta E_{cm} + \Delta E_{bs}$$

$$\Delta E = Q_{cm} \cdot (T_a/T_{cm} - 1) + Q_{bs} \cdot (T_a/T_{bs} - 1)$$

- Real electrical power to refrigerator: $P_{ref} = \Delta E / \eta(T)$

with $\eta(T)$ = efficiency w.r. to Carnot = $COP_{Carnot} / COP_{Real}$

$$P_{ref} = Q_{cm} \cdot (T_a/T_{cm} - 1) / \eta(T_{cm}) + Q_{bs} \cdot (T_a/T_{bs} - 1) / \eta(T_{bs})$$

BS – CM thermodynamics

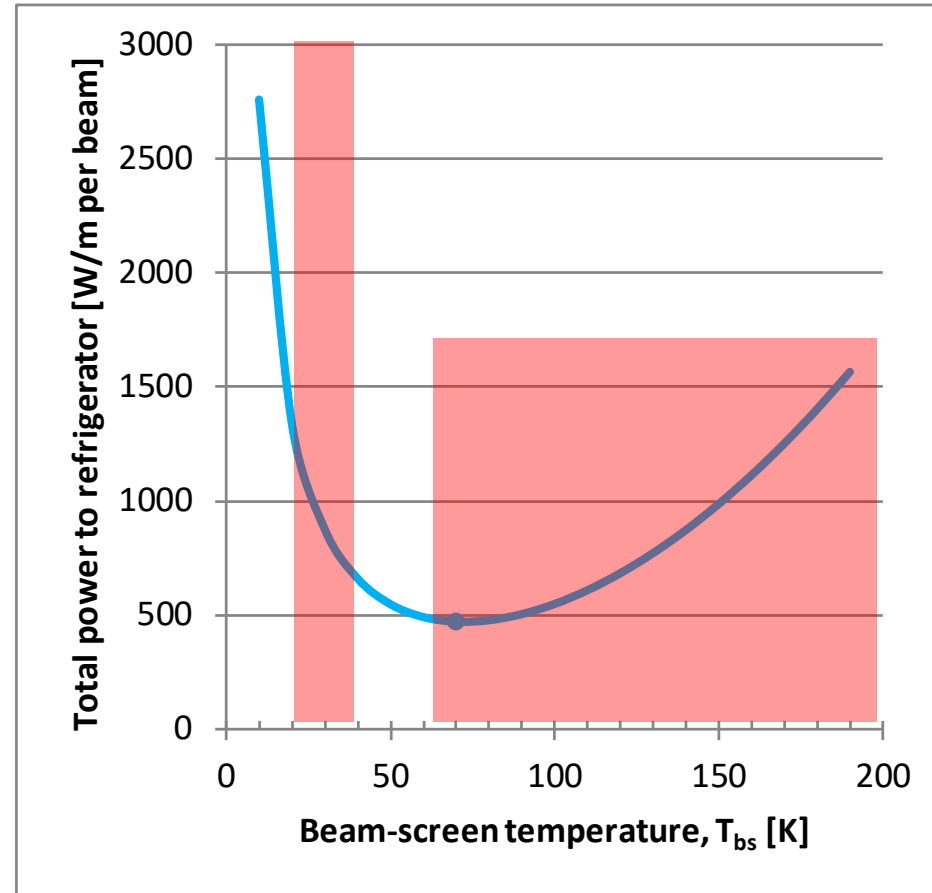
Numerical application

Total electrical power to refrigerator P_{ref} considering:

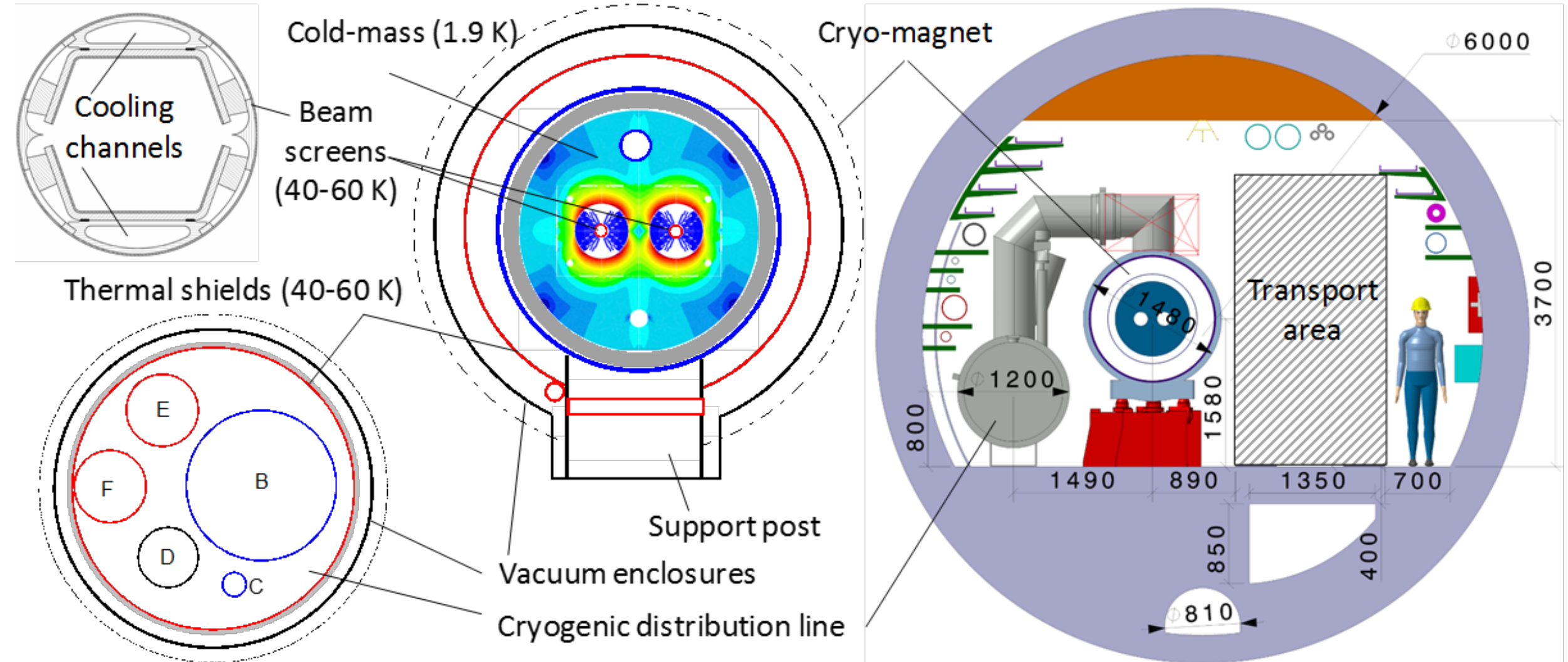
- a beam screen similar to that of the LHC
- refrigerator efficiencies identical to those of the LHC.

Optimum for $T_{\text{bs}} = \sim 70$ K

Temperature range 40-60 K retained



Forbidden by vacuum and/or by surface impedance



Main distribution based on INVAR® technology

Contribution of WUST

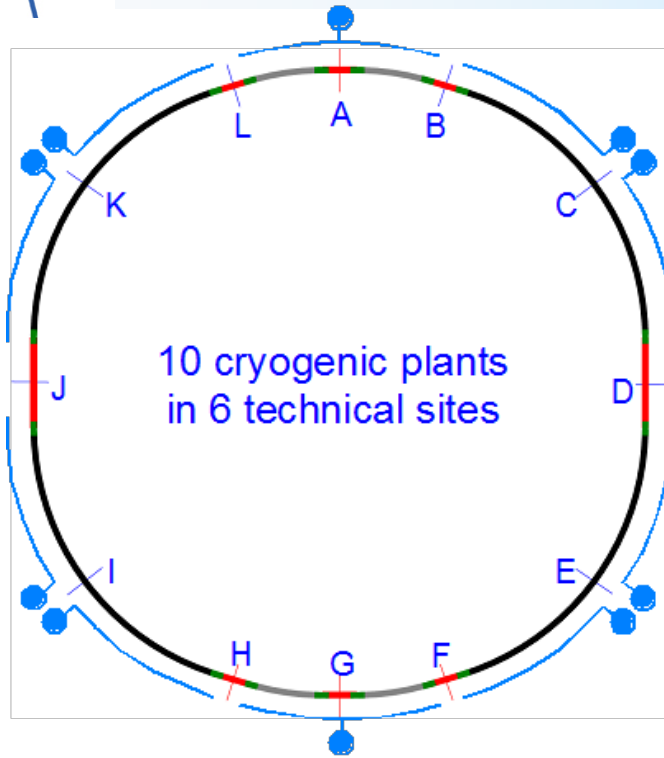


Specific heat loads at different temperature levels



Temperature level		40-60 K	1.9 K	4 K VLP
Static heat in-leaks [W/m]	CM supporting system	2	0.13	
	Radiative insulation		0.13	
	Thermal shield	3.1		
	Feedthrough & vacuum barrier	0.2	0.1	
	Beam screen		0.12	
	Distribution	4	0.1	0.24
	Total static	9.3	0.58	0.24
Dynamic heat loads [W/m]	Synchrotron radiation	57	0.08	
	Image current	3.4		
	Resistive heating in splices		0.3	
	Beam-gas scattering		0.45	
	Total dynamic	60	0.83	
Total [W/m]		70	1.4	0.24
Dynamic range [-]		8	2.5	1

FCC-hh cryogenic layout and architecture

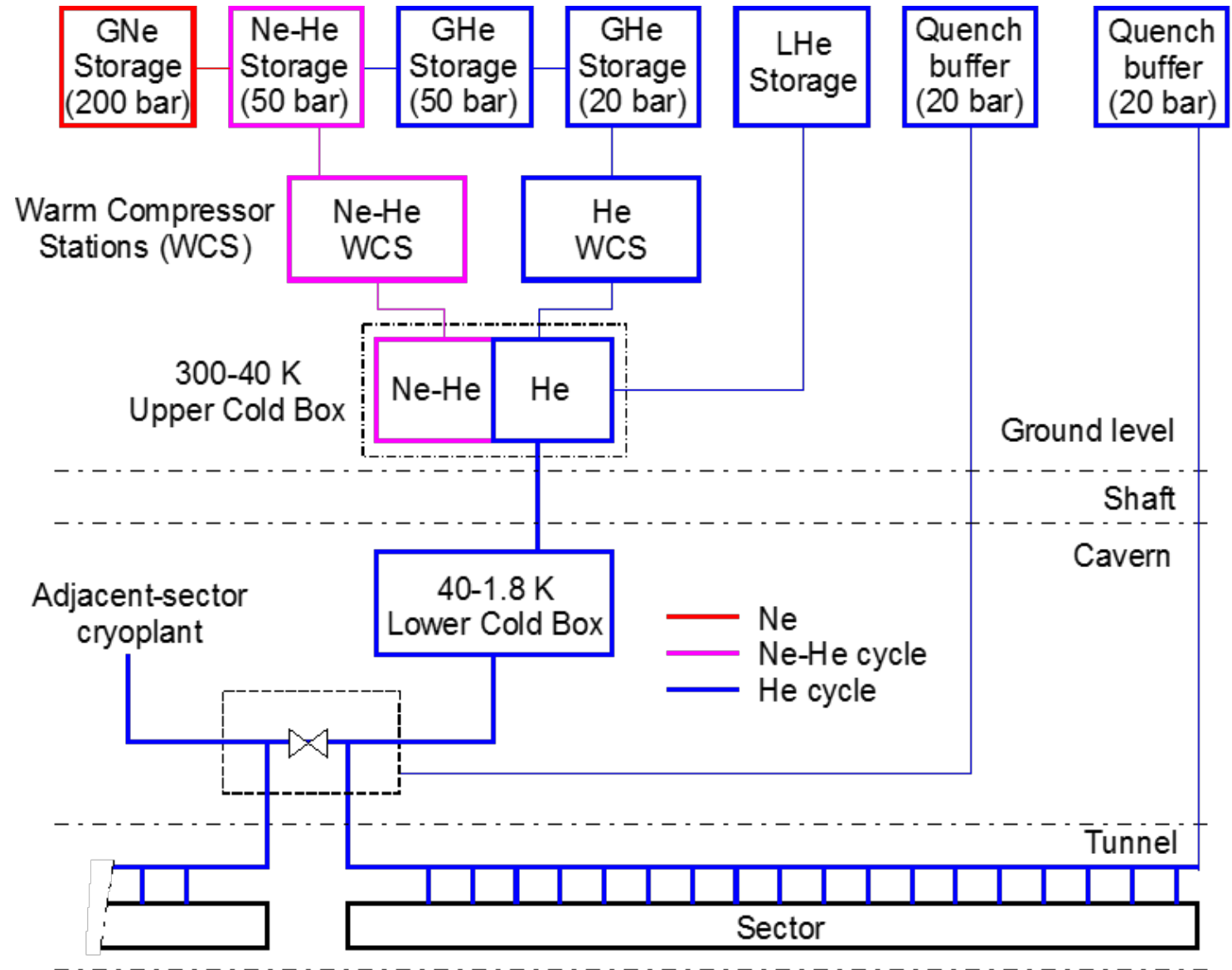


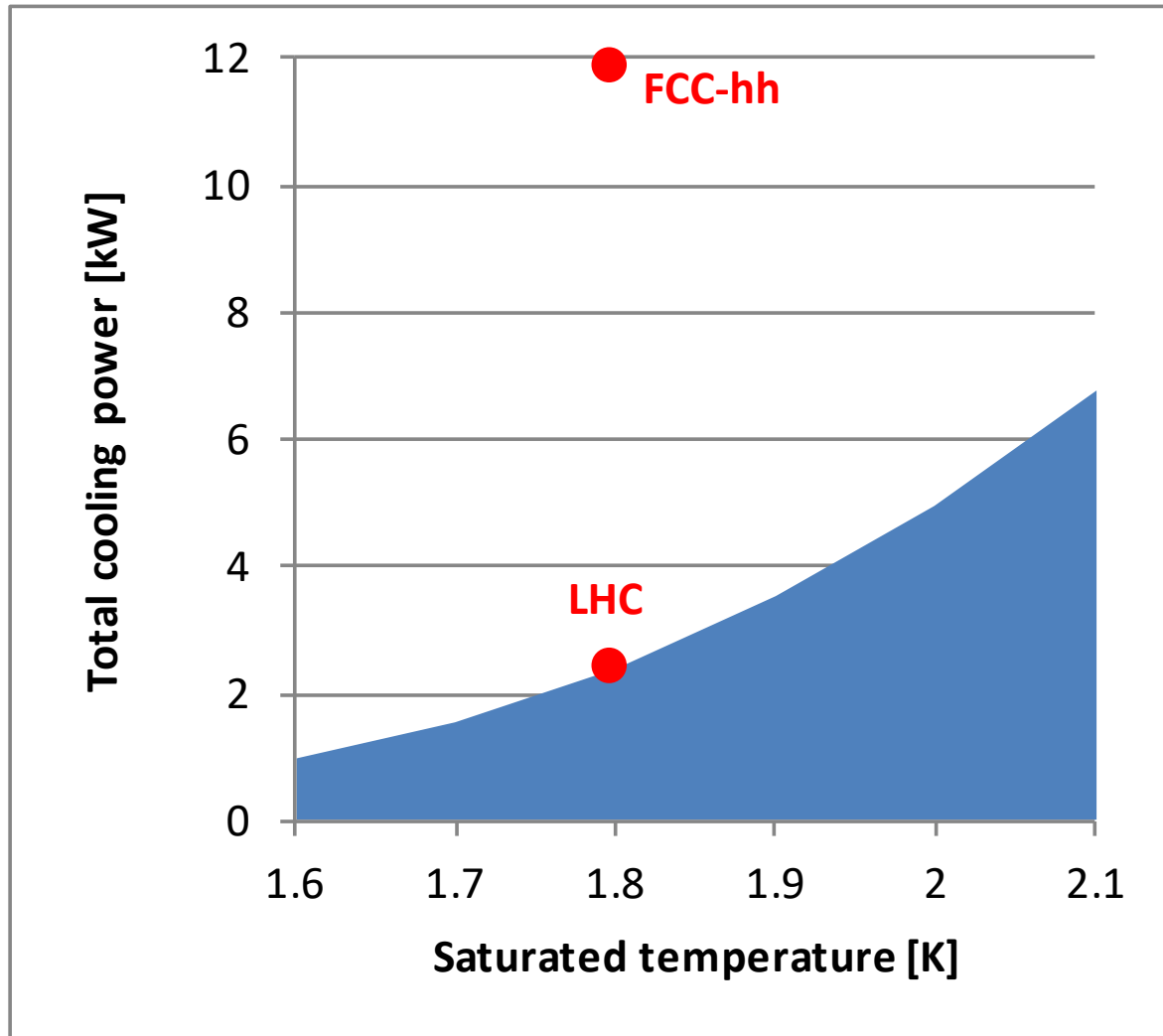
● Cryogenic plant
— Cryogenic distribution

40-60 K [kW]	1.9 K [kW]	4 K VLP [kW]	40-300 K [g/s]
580	12*	2	85

Without operational margins as the working conditions has to be considered as ultimate

*: Outside State-of-the-Art





Increase by a factor 5 on the cooling power, i.e with respect to the present technology:

→ Impeller diameter from 350 mm to 700 mm (factor 2)

→ Shaft power from 10 kW to 30 kW (factor 3)

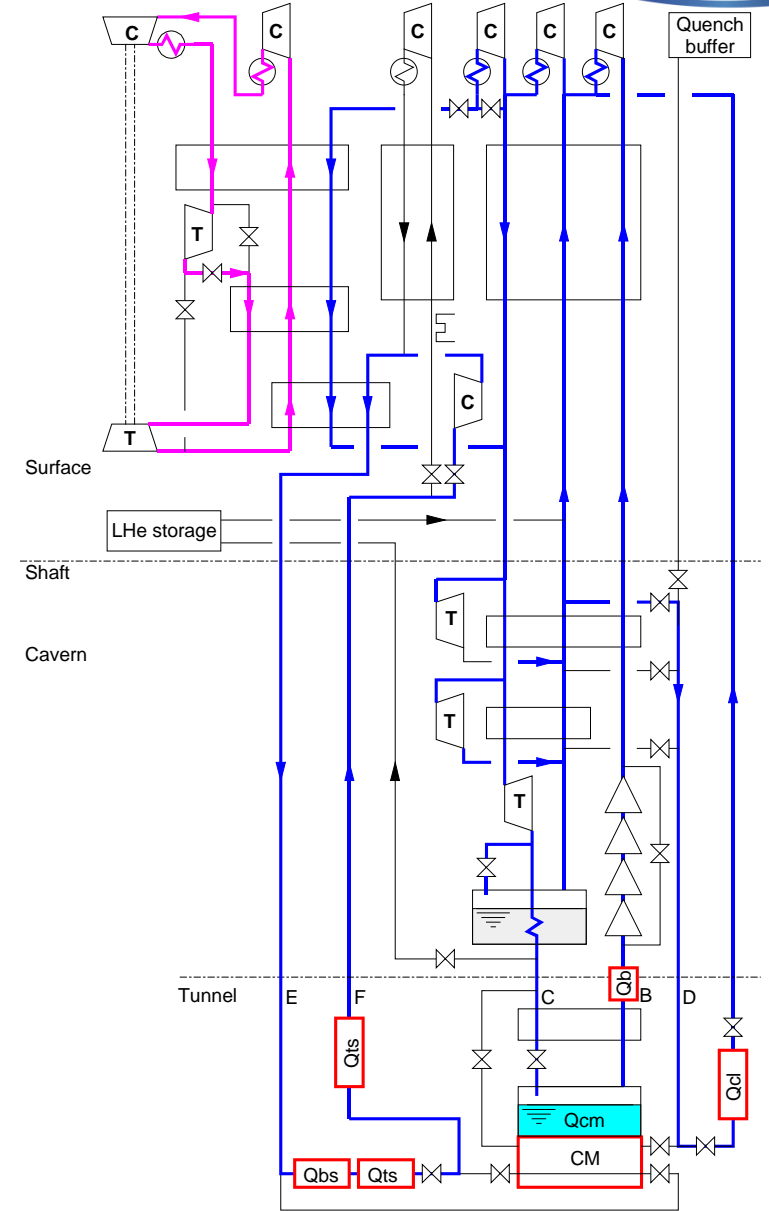
**Ne-He
300-40 K
cryoplant**

- Beam screen (40-60 K)
- Thermal shield (40-60 K)
- Current leads (40-300 K)
- Precooling of 1.9 K cryoplant

**He 1.8 K
cryoplant**

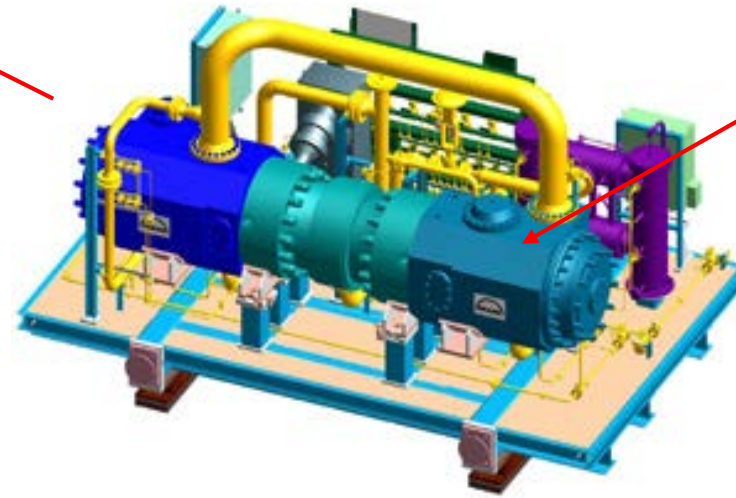
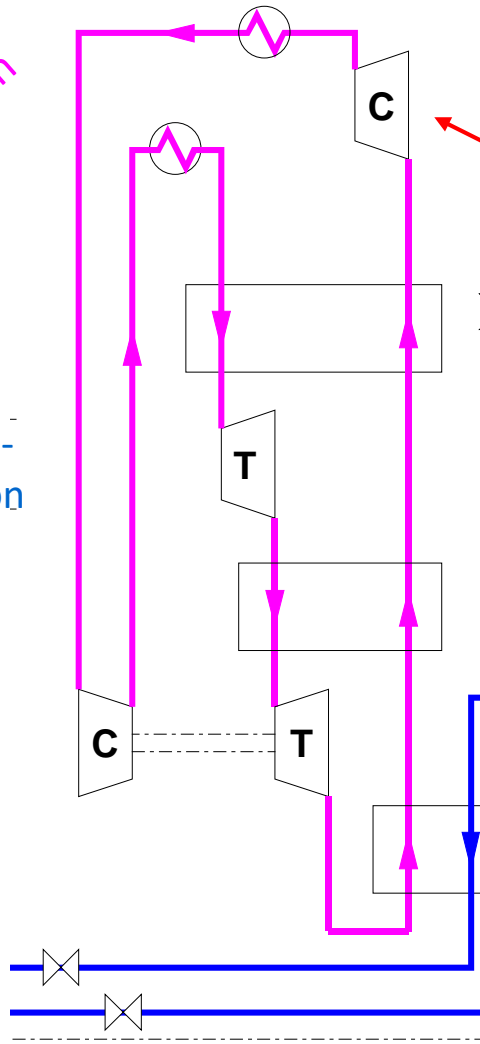
- SC magnet cold mass

Contributions of TU Dresden and CEA/SBT



TU Dresden

Turbo-Brayton cycle



Courtesy of MAN Diesel & Turbo



Hermetically sealed centrifugal compressors:

- No dry gas seals, no lube-oil system and no gearbox
- Use of high speed induction motor (up to 200 Hz) and active magnetic bearings. The motor is cooled by process gas and directly coupled to the barrel type compressor.

Difficult to get high compression ratio and high compression efficiency with pure helium (light mono-atomic gas):

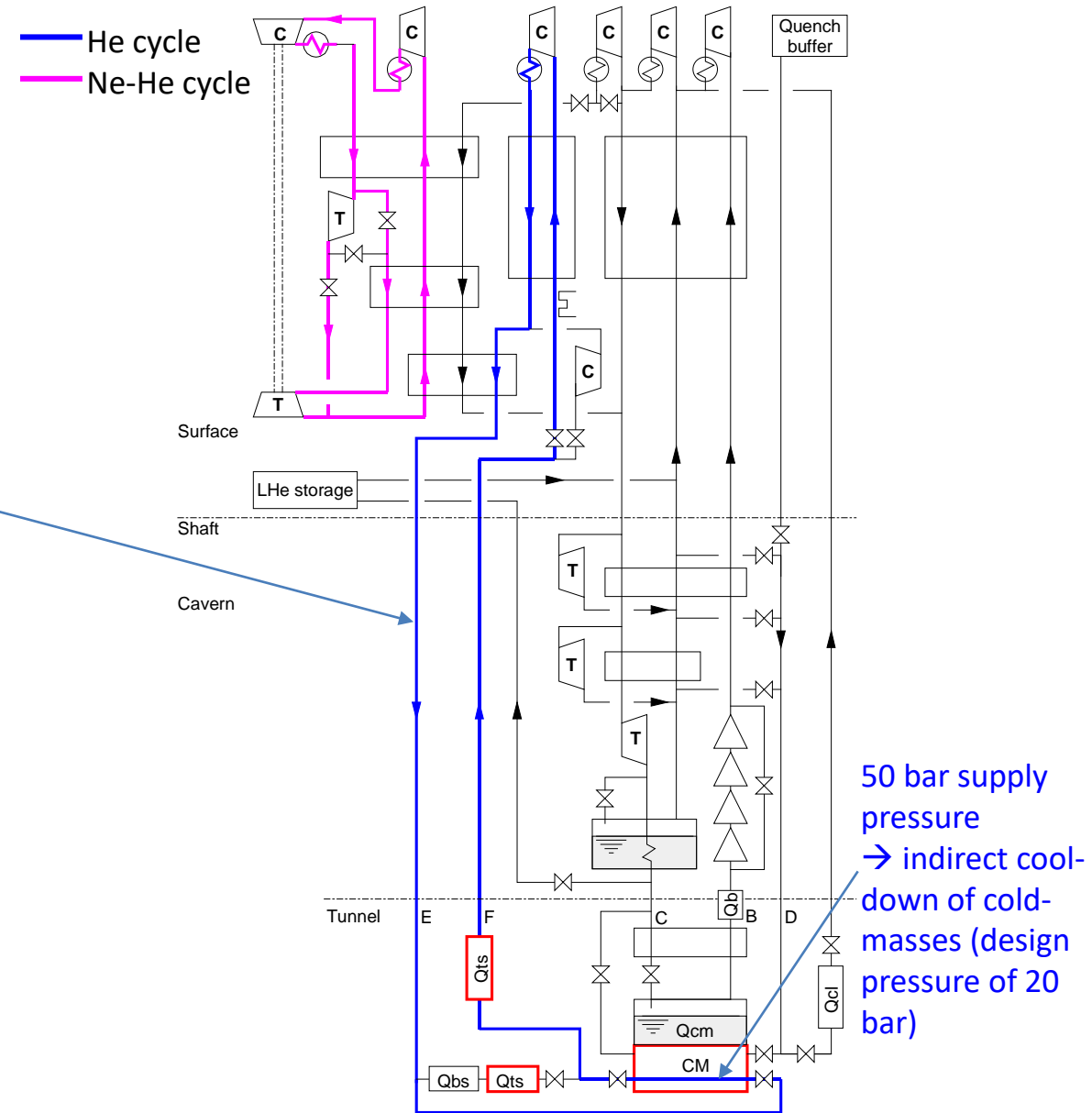
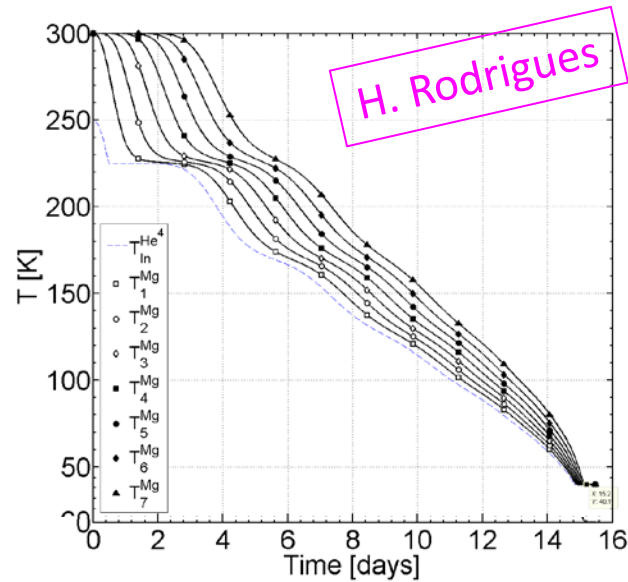
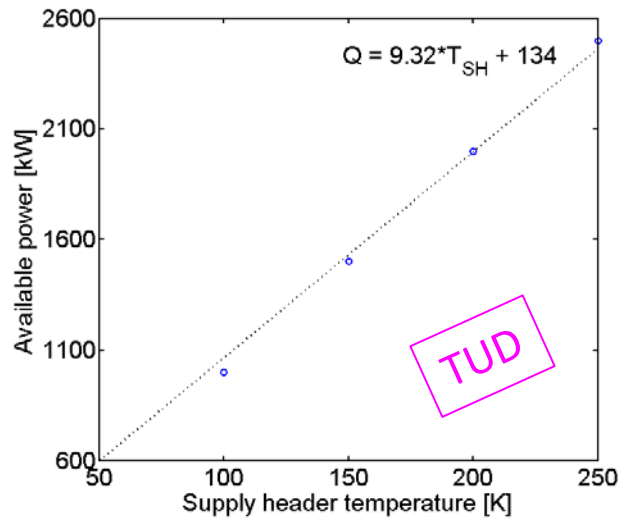
- Compression of a mixture of helium and neon (~75-25 %) (OK with neon as refrigeration $T > 40$ K)
- The warm compression efficiency is improved
- Expected global efficiency with respect to Carnot → **42 %**

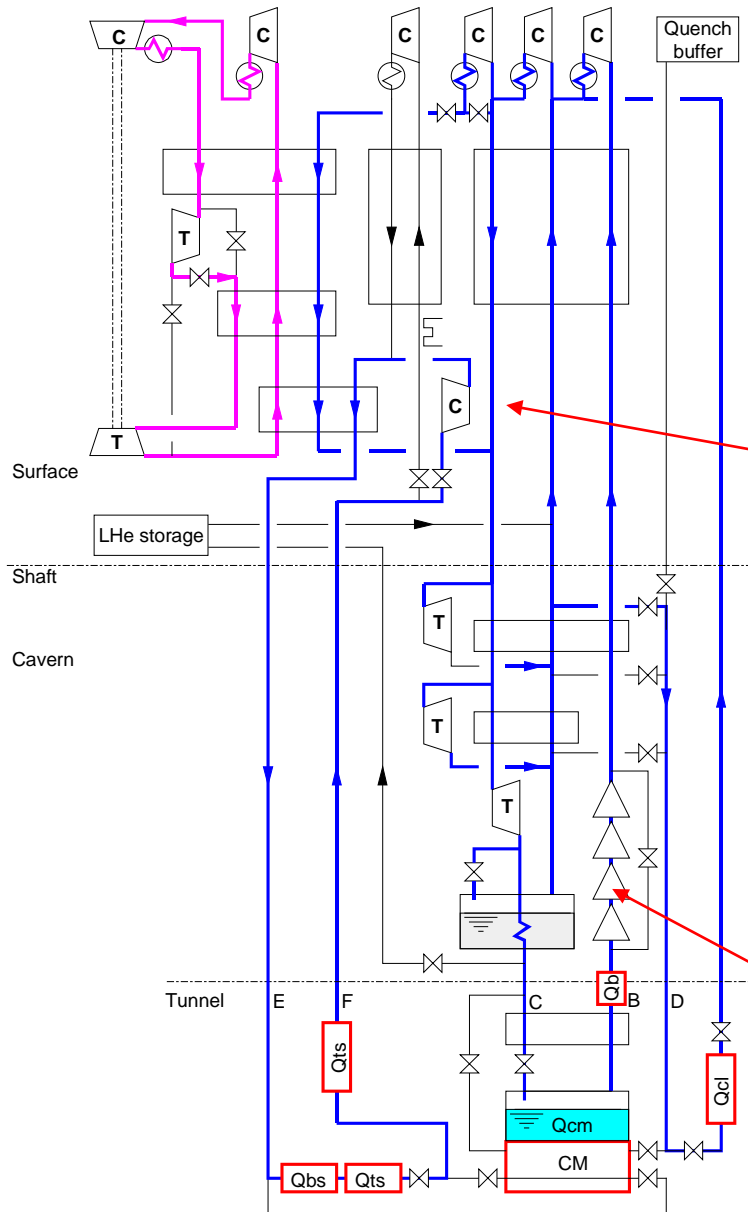
Cool-down capacity produced by the Ne-He cycle:
No need of LN2 cool-down unit with its huge LN2 storage and logistics

Cold mass: 2.8 t/m i.e.
23 kt per sector
230 kt for FCC

15 days of cool-down time from 300 to 40 K (10 days (on paper) with LN2).

Cost (only energy)
Full FCC CD → 2.3 MCHF
(To be compared with the cost of a CD using LN2 → 45000 t, i.e. ~4 MCHF)



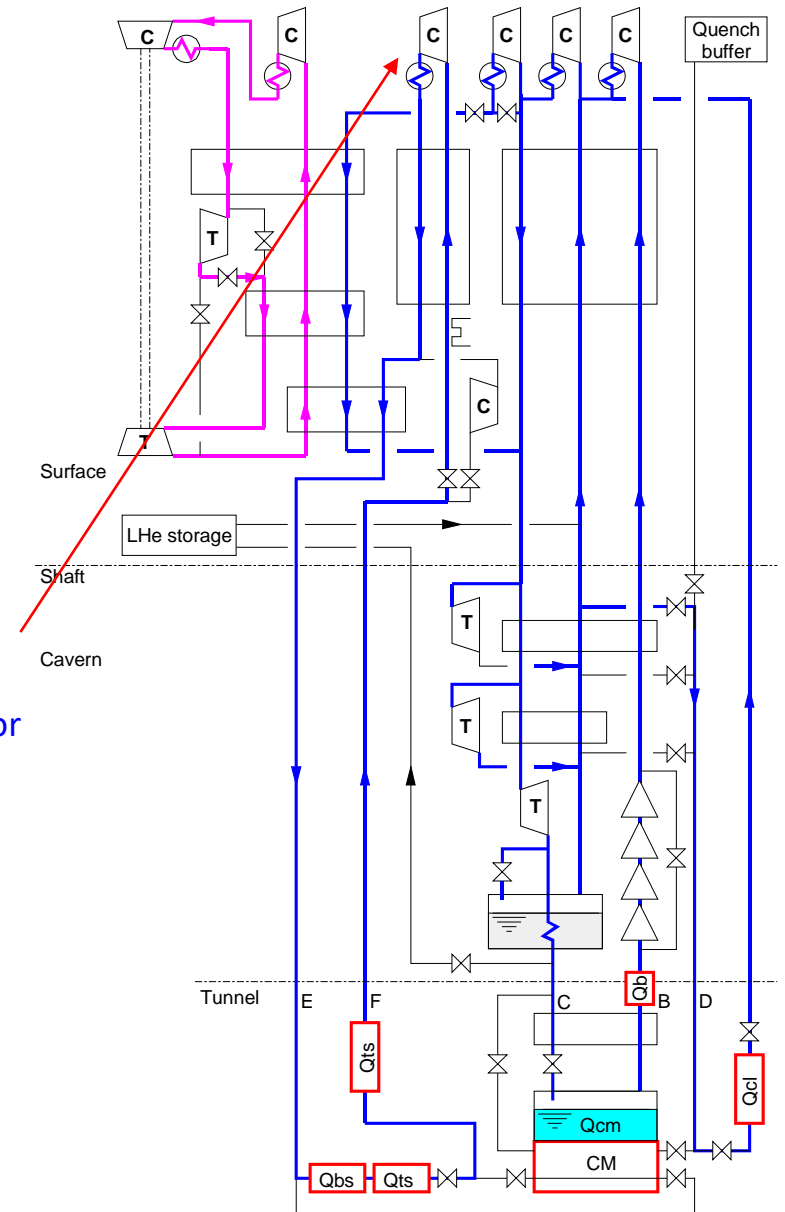


— He cycle
— Ne-He cycle

BS cooling with cold circulator (+130 kW @ 40-60 K) →
Exergetic efficiency of BS cooling loop: 82 %

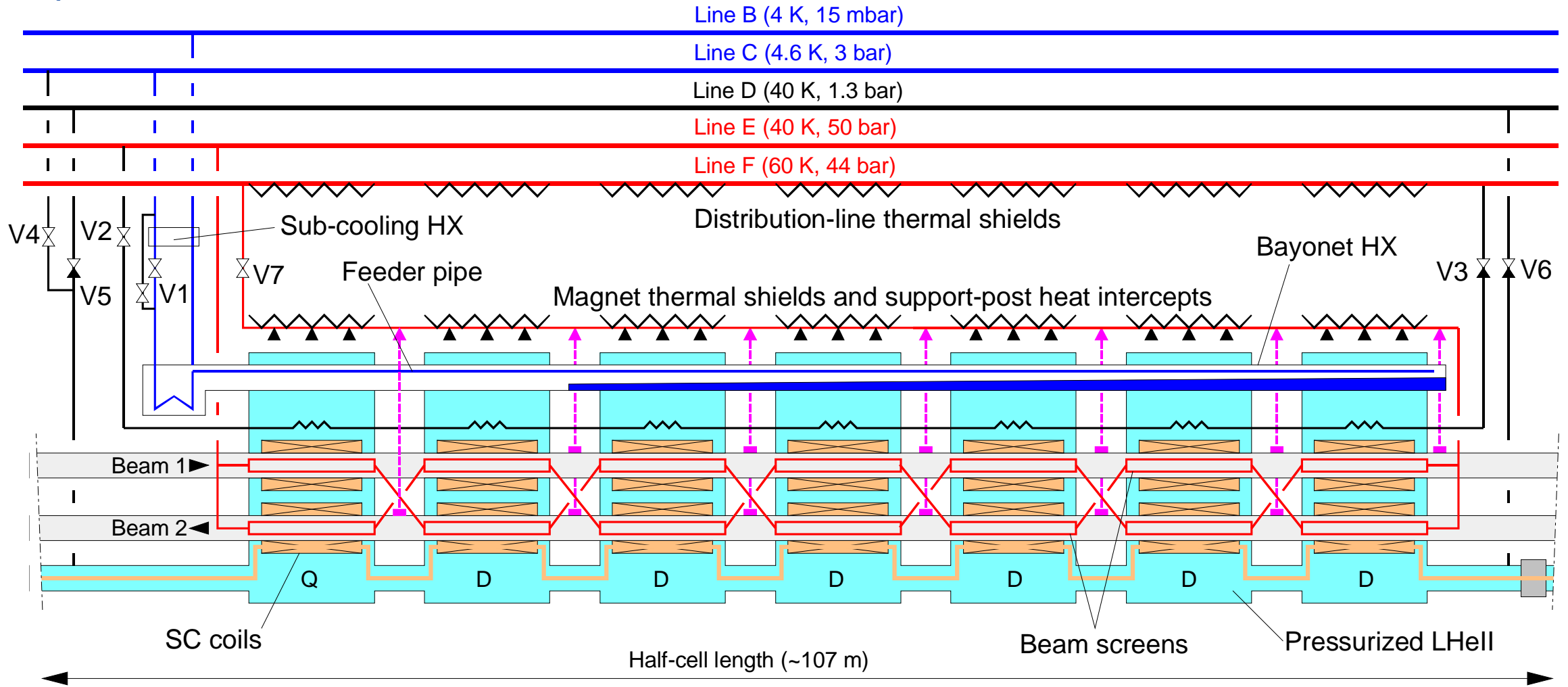
BS cooling with warm circulator needed for cooldown
(+140 kW @ 40-60 K and +500 kW @ 300 K) →
Less efficient (exergetic efficiency of 71 %) but good for
redundancy.

1.8 K refrigeration based on a mixed compression
cycle (cold centrifugal compressors in series with
warm volumetric compressors)

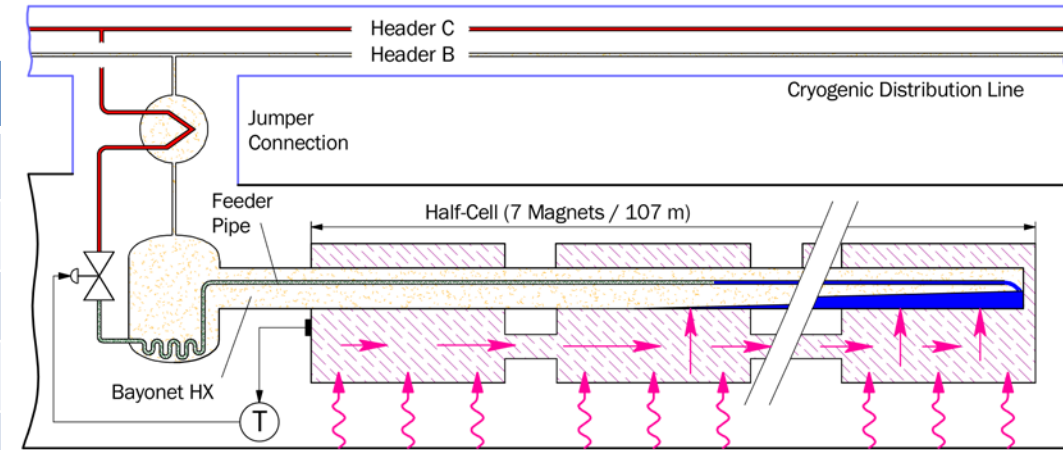




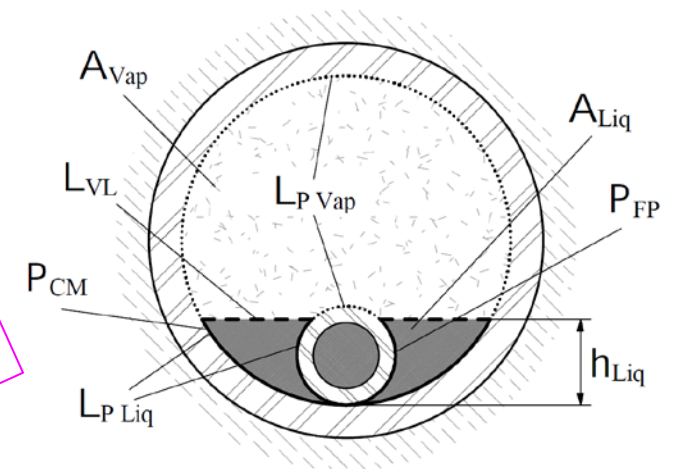
FCC-hh Half-cell cooling loop



Variable	Unit	LHC	FCC
Unit cooling length	m	106.9	107.1
Sector cooling length	m	2900	8400
Average heat load nominal capacity	W/m	0.40	1.38
Bayonet HX inner diameter	mm	53.4	83.1
Feeder pipe inner diameter	mm	10.0	15.0
Thickness bayonet HX pipe wall	mm	2.3	5.0
Joule-Thomson valve inlet temperature	K	2.18	2.18
Free longitudinal cross-section area	cm ²	60	156
DT max Pressurized-saturated Hell	mK	50	50
Cold mass operating pressure	bar	1.3	1.3
Header B diameter	mm	270	630 (500)
Heat load on header B	W/m	0.11	0.24
Pumping pressure at cryoplant interface	mbar	15	15
Maximum cold-mass helium temperature	K	1.9	1.9 (1.98)



- Helium I (4.6 K / 3 bar)
- Helium II vapour saturated or overheated
- Helium I / II non-stratified two-phase flow
- Helium II pressurized bath (1.3 bar)
- Helium II liquid saturated or sub-cooled



C. Kotnig



FCC-hh Beam-screen cooling loop parameters



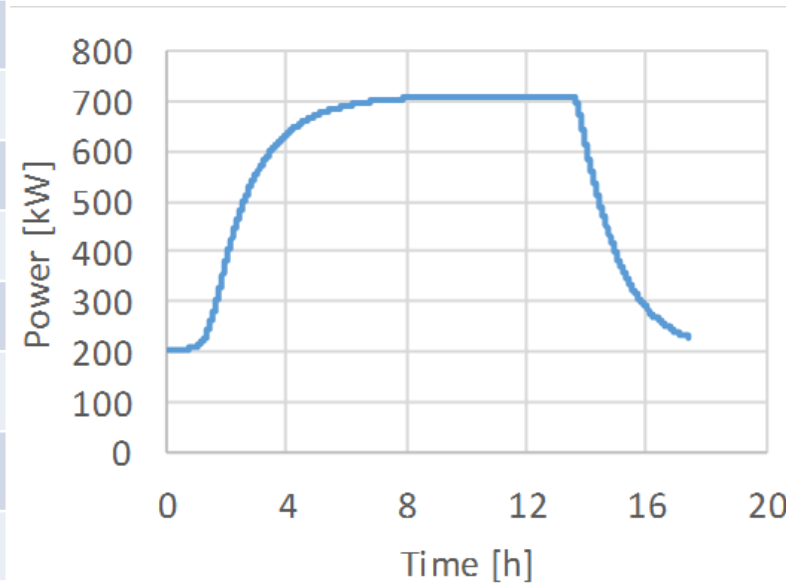
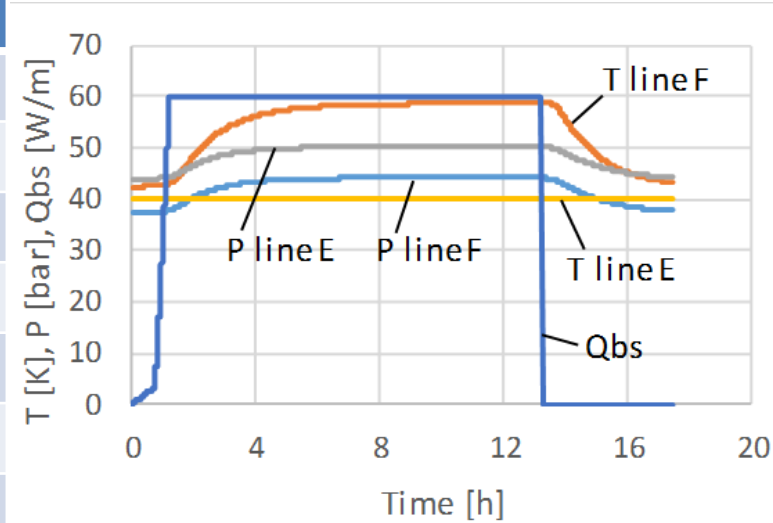
Transient modes

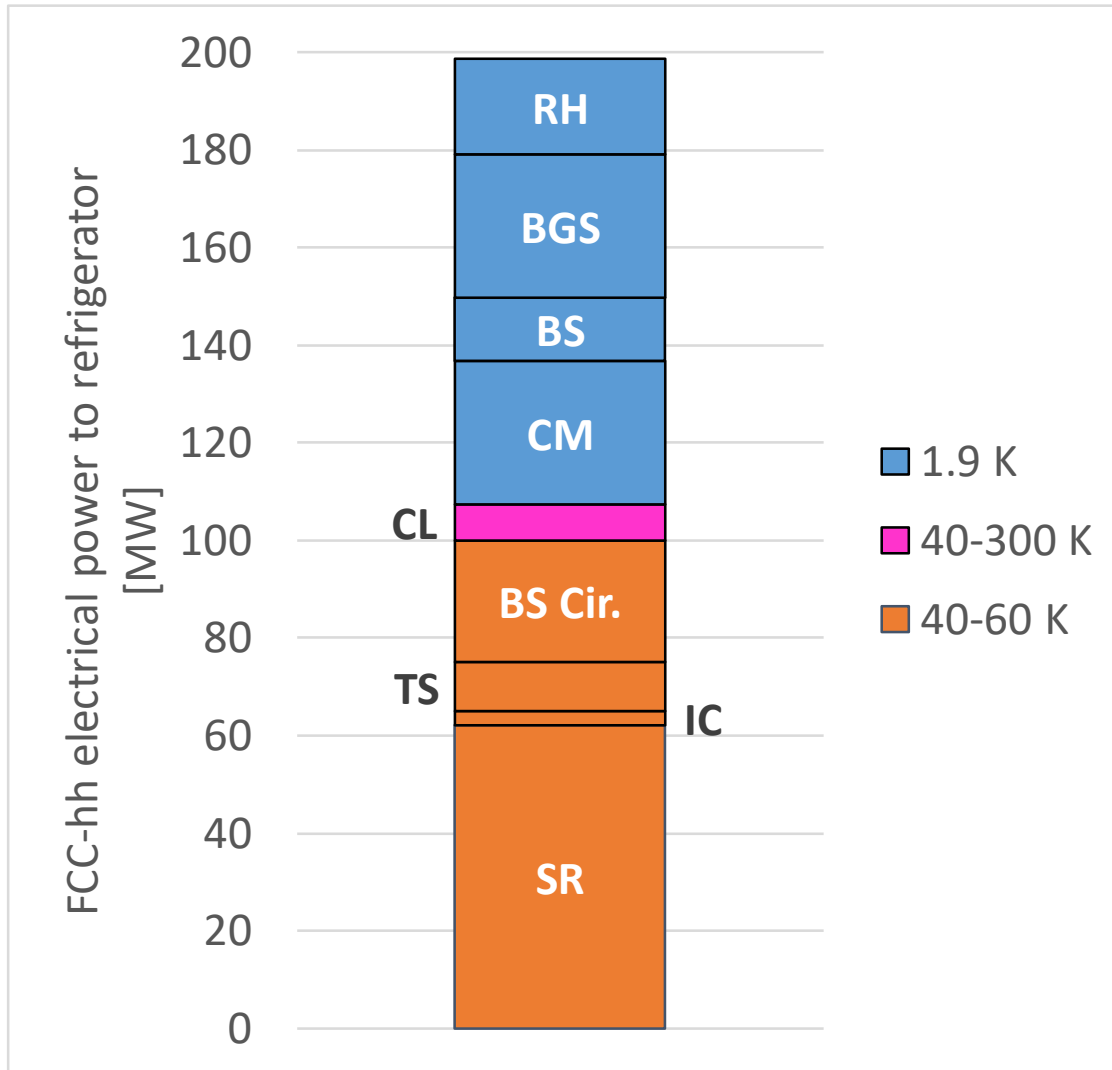
- Working at constant nominal mass-flow to handle the severe transient during energy ramp-up
- Working at constant He inventory to avoid big mass discharge and refill (~6 t) (i.e. pressure increase during energy ramp)

Large inertia of the distribution system
 → time constant of ~ 4 h
 → OK with the capacity adaptation of the cryoplants
 → In high luminosity operation (4 h of stable beams), the cryoplants will be never in steady-state

H. Rodrigues
 C. Kotnig

Main parameter	Unit	LHC	FCC
Unit cooling length	m	53.4	107.1
Sector cooling length	m	2900	8400
Average BS nominal capacity	W/m	1.6	60
Max. supply pressure	bar	3	50
Supply helium temperature	K	5	40
Max. allowed BS temperature	K	20	60
BS helium outlet temperature (nominal)	K	20	57
Minimum BS temperature (nominal)	K	5	43
BS pressure drop (nominal)	bar	0.5	3
ΔP control valve (nominal)	bar	0.8	1
ΔP supply and return header (nominal)	bar	0.4	2
Total cooling loop pressure drop	bar	1.7	6
Supply/return header diameter	mm	100/150	250/250
Exergetic efficiency (distribution only)	%	76	86
Total exergetic eff. (with cold circulator)	%	N/A	82
Total exergetic eff. (with warm circulator)	%	?	71





RH: resistive heating

BGS: beam-gas scattering

BS: beam screen

CM: cold mass heat-inleaks

CL: current lead

BS cir.: Beam screen circulator (warm)

TS: thermal shield

IC: image current

SR: synchrotron radiation

Carnot efficiency:

- Ne-He plants: 40 %

- Helium plants: 28.8 %

Isentropic efficiency

- cold compressors: 75 % per stage

- Warm circulator: 83 %

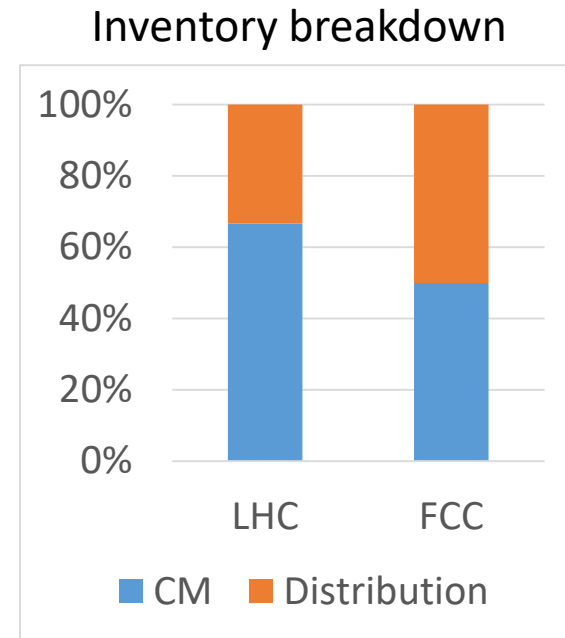
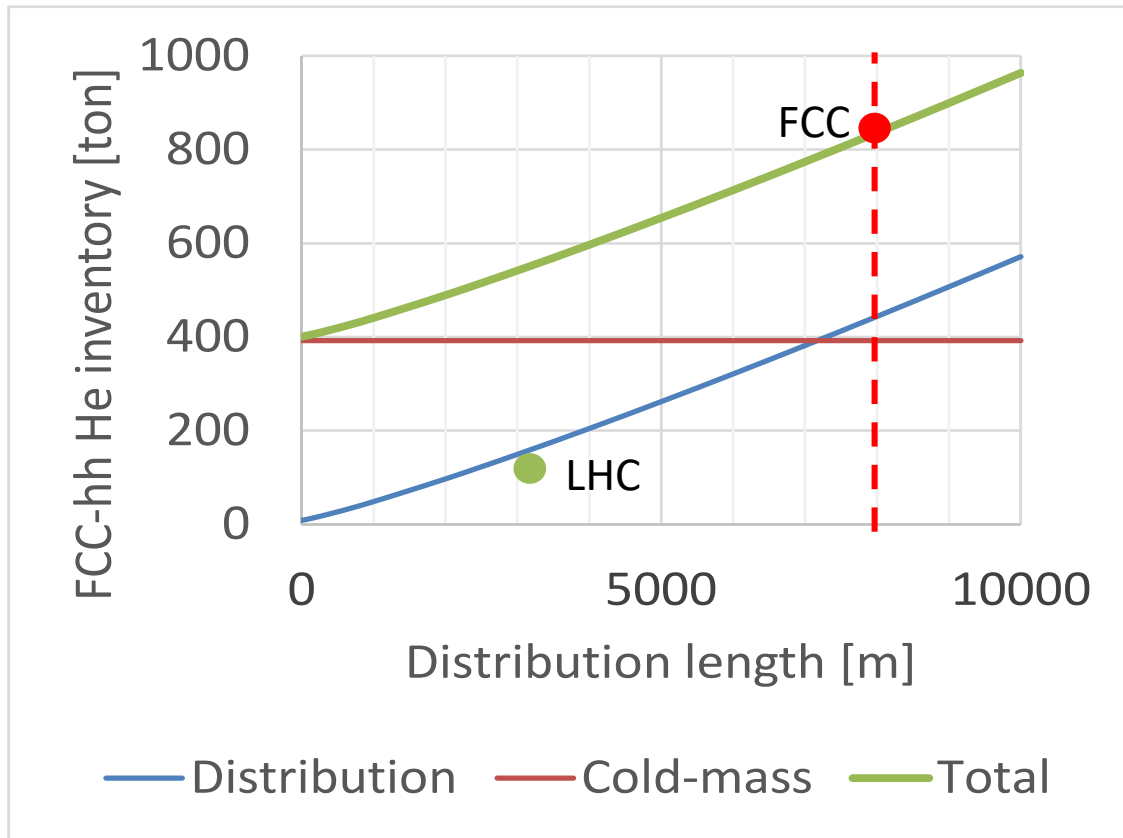


FCC-hh He inventory



Cold mass He inventory : 33 l/m (scaled from LHC)

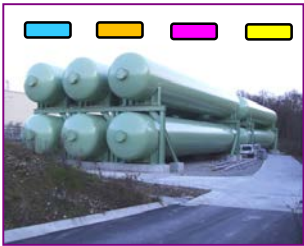
Distribution inventory dominated by the beam-screen supply and return headers











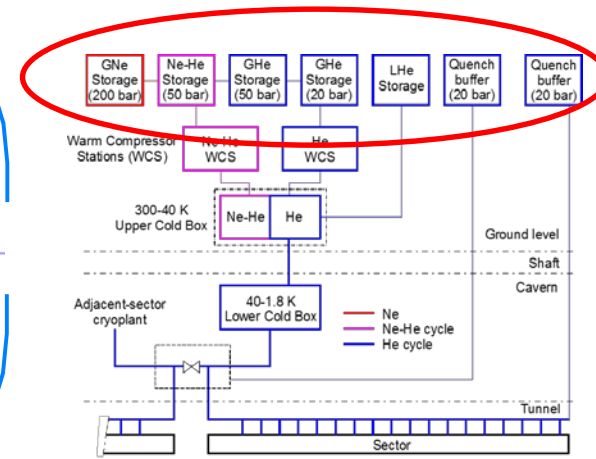
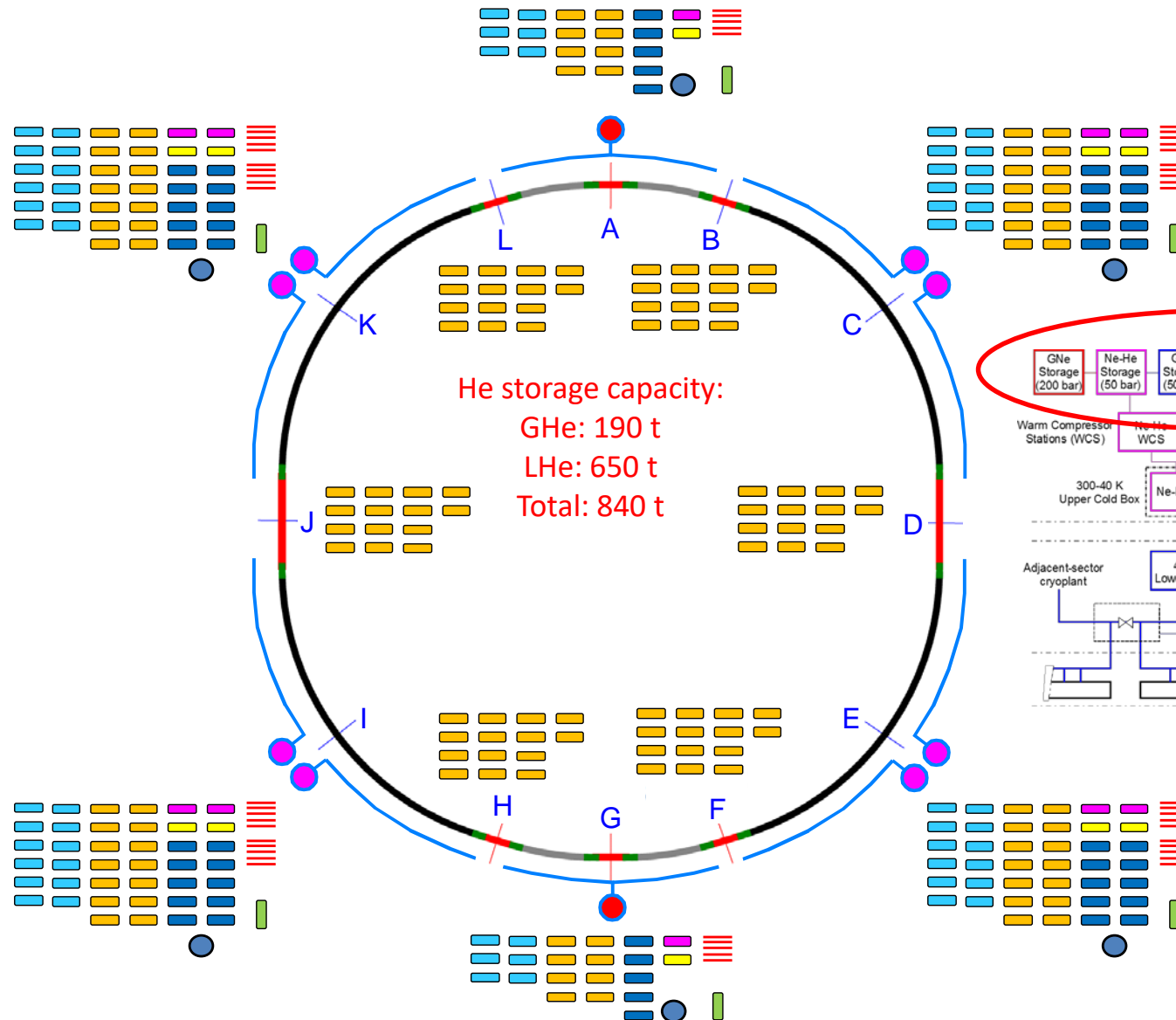
FCC He inventory: ~800 t ! (~6 LHC He inventory)



FCC-hh Storage management

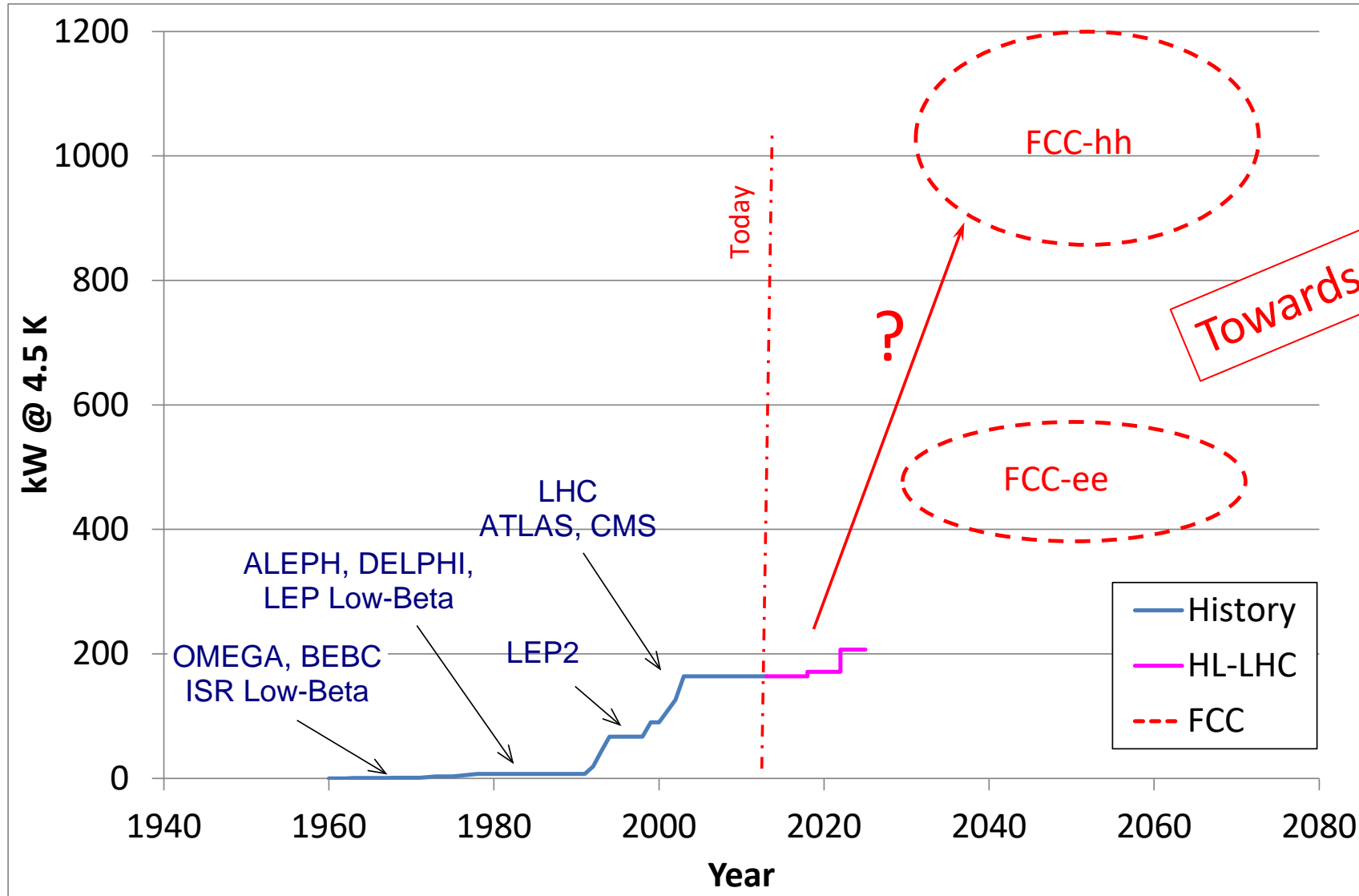


-  GHe storage - 60 (250 m3, 20 bar)
-  Quench buffer - 156 (250 m3, 20 bar)
-  GHe storage - 10 (250 m3, 50 bar)
-  Ne-He storage - 10 (250 m3, 50 bar)
-  GNe cylinders - 10 (10 m3, 200 bar)
-  LHe storage - 50 (120 m3)
-  LHe boil-off liquefier - 6 (150 to 300 l/h)
-  LN2 storage - 6 (50 m3)





Main FCC cryogenics challenges: Cryogenic power





Conclusion



- The conceptual design of the cryogenic systems for a Future Circular Collider is in progress in the framework of an international collaboration (CEA, CERN, TUD, WUST)
- The final Conceptual Design Report (CDR) will be issued by 2018 and then examined by the next European Strategy for high energy particle physics.
- In the case of a positive feedback, the next step will be the studies and developments of the new concepts with the construction of demonstrators and/or prototypes.