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32 **Evaluation of gas explosion overpressures at configurations with**  
33 **irregular-arranged obstacles**

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43 **Abstract**

44 Rapid analytical methods for the calculation of gas explosion overpressures in confined  
45 and congested regions are of great value where a benchmark value is sought rather than  
46 a time consuming detailed analysis obtainable by Computational Fluid Dynamics  
47 (CFD). While earlier correlations have been compared directly to experiments, the  
48 geometries used were often simplistic and displayed homogeneity in confinement and  
49 congestion. Realistic geometries typically display a high degree of inhomogeneity in  
50 confinement and congestion. Here we examine geometries where the confinement and  
51 congestion were deliberately varied such that some of the geometries possessed  
52 inhomogeneity of both parameters. Little experimental data exists for such  
53 configurations and hence we examine these configurations using CFD. The CFD  
54 overpressure predictions at various target locations for 400 scenarios are compared with  
55 the results from a newly derived correlation and the correlation of the Guidance for the

56 Application of the Multi-Energy method (GAME). It is found that the overpressure  
57 predictions obtained using the correlation still better agrees with the CFD modelling  
58 results compared with the GAME correlation suggesting. To show the importance of  
59 increased accuracy in these cases, a structural damage level evaluation process is used  
60 to place the damage levels for 4 monitor points on a p-i curve and the results show that  
61 often these damage levels are near critical, demonstrating the need for improved  
62 accuracy.

63 **Keywords:** obstacles, VCEs, gas explosion, irregularity, overpressure, turbulence

64

## 65 **Introduction**

66 Numerous vapour cloud explosions (VCE), and dust explosions occur each year world-  
67 wide. The vapour cloud explosion, is defined as “an explosion resulting from an ignition  
68 of a premixed cloud of flammable vapour, gas or spray with air, in which flames  
69 accelerate to sufficiently high velocities to produce significant overpressure” (Mercx &  
70 van den Berg, 2005). These represent one of the most significant hazards in the  
71 chemical process industry. Due to the large overpressures generated from the VCEs, it  
72 can result in potential environmental damage and enormous financial loss in addition  
73 to injury and loss of life. As a result, it is of great importance to assess risk at major  
74 hazard facilities accurately.

75

76 Deflagration is a combustion wave propagating at subsonic velocities relative to the  
77 unburned gas immediately ahead of the flame. Detonation is defined as a supersonic  
78 combustion wave (i.e. the detonation front propagates into unburned gas at a velocity  
79 higher than the speed of sound in front of the wave) (Bjerketvedt et al., 1997). In this  
80 paper we consider the MERGE and EMERGE projects (EMEG, 1997; Harris &

81 Wickens, 1989; Mercx et al., 1995; Schumann et al., 1993; Wingerden, 1988, 1989),  
82 which were conducted to investigate the mechanism of the gas explosions. Using  
83 experiments to evaluate risk for each industrial facility is impractically expensive due  
84 to the numerous variations of geometry detail, size and inventory composition and size  
85 in industrial explosion scenarios. Cost constraints mean that experiments performed so  
86 far have been scaled down in size and simplifications were applied. The scaling factor  
87 may result in inherent uncertainties for experimental results and it is sometimes difficult  
88 to even quantify the impact of the simplifications used in these experiments.

89

90 Based on experiments, some theoretical methods were developed, such as the widely  
91 used approach TNO Multi-Energy Method (MEM)(Vandenberg, 1985). MEM is a  
92 simple phenomenological approach to estimate overpressures from approximated  
93 vapour cloud explosion scenarios. However, MEM has some clear limitations. Firstly,  
94 MEM was derived based on limited scale experiments which results in uncertainties in  
95 the prediction of pressures for large-scale explosion scenarios. Secondly, the directional  
96 effects for explosions due to localised confinement and congestion are not accounted  
97 for, as the results output by MEM are radial in nature. Finally and importantly, the near-  
98 field gas explosion overpressure cannot be predicted via the multi-energy approach with  
99 any reasonable accuracy and it relies on an input estimate of the strength of the  
100 explosion which can be either significantly underestimated or significantly  
101 overestimated: both leading to unsatisfactory results.

102

103 An improvement on MEM is the GAME approach (Eggen, 1995). Specifically, several  
104 parameters regarding the directional effects and gas properties, such as the degree of  
105 geometry size, congestion, gas mixture and the laminar flame speed, among others, are

106 investigated in the GAME approach, however, the derivation of the GAME correlations  
107 are based on the phenomenological analysis of the experimental programs which were  
108 arranged with regular obstacles. When it comes to cases with inhomogeneous  
109 congestion and confinement, the accuracy of the GAME correlations has not been  
110 adequately tested against a standard.

111

112 Consequently, at the present time, many of the vapour cloud explosion analyses are  
113 increasingly being carried out using the Computational Fluid Dynamics (CFD) tools  
114 (Marangon et al., 2007). Because it agrees with experiments to a greater degree than  
115 analytical studies, the CFD approach is considered a robust numerical tool based on  
116 finite volume solutions and the ‘physical’ models of combustion process to predict gas  
117 explosion overpressure. In particular, some CFD solvers can capture the flame  
118 acceleration and venting of the overpressure build-up for gas clouds in irregularly  
119 patterned obstacles which have significant effects on overpressures.

120

121 However CFD is time consuming and expensive and in addition requires a degree of  
122 expertise in its application for meaningful results and there is still significant need for  
123 rapid approximate methods for benchmarking such events that can be later targeted, if  
124 necessary with detailed CFD analysis. In this paper we used the detailed CFD  
125 methodology as a benchmark to further investigate a previously suggested rapid  
126 solution - a confinement specific correlation (CSC) (Li et al., 2014).

127

128 Here the highly validated commercial CFD software FLACS (GexCon, 2011) was  
129 utilized in the evaluation, of a benchmarking correlation previously proposed (Li et al.,  
130 2014). The software was used to test the robustness of the correlation particularly its

131 ability to predict overpressures for cases with variation of a few fundamental  
132 parameters including confinement and congestion driven flame propagation, a range of  
133 practical modules with irregularly arranged obstacles and confinement ratios were  
134 assessed by means of the previous proposed correlation (Li et al., 2014). After the  
135 evaluation of the overpressure, the data of the pressure-impulse ( $p-i$ ) was also analysed  
136 in this study which is able to be used for structural damage prediction.

137

## 138 **Simulation Methodology**

139

### 140 **The FLACS Software**

141 In order to extend the range of conditions for the correlation of (Li et al., 2014), the  
142 results for overpressure are compared with the results using the commercial software  
143 FLACS (GexCon, 2011) for conditions not previously considered. FLACS (GexCon,  
144 2011) is a finite volume solver that solves the Reynolds averaged mass, momentum  
145 and energy balance equations, with special schemes for supersonic flows and a database  
146 of chemical kinetics. The mathematical model of FLACS (GexCon, 2011) is given in  
147 (Arntzen, 1998; Ferrara et al., 2006; Hjertager, 1984, 1993).

148

149 For a general variable, the differential equation, which is based on Reynolds averaged  
150 mass, momentum and energy balance equations, may be expressed as follows using  
151 standard symbols:

$$152 \quad \frac{\partial}{\partial t}(\rho\varphi) + \frac{\partial}{\partial x_j}(\rho u_j \varphi) - \frac{\partial}{\partial x_j} \left( \Gamma_\varphi \frac{\partial \varphi}{\partial x_j} \right) = S_\varphi ; \Gamma_\varphi = \frac{\mu_{eff}}{\sigma_\varphi} \quad (1)$$

153 where  $\mathfrak{f}$  denotes a general variable,  $\rho$  is the gas mixture density,  $x_j$  is the coordinate  
 154 in  $j$ -direction,  $u_j$  is the velocity component in  $j$ -direction,  $\Gamma_\varphi$  is the effective (turbulent)  
 155 diffusion coefficient,  $\mu_{eff}$  is the effective turbulence viscosity and  $S_\varphi$  is a source term.

156

157 A summary of all the governing equations needed for a typical reactive gas dynamic  
 158 calculation are presented below.

159 The state equation of an ideal gas:

$$160 \quad pW = \rho RT \quad (2)$$

161 where  $p$  is the pressure,  $R$  is the universal gas coefficient  $T$  is temperature and  $W$  is the  
 162 molar weight of the gas mixture.

163

164 The continuity equation:

$$165 \quad \frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_j} (\rho u_j) = 0 \quad (3)$$

166 The momentum balance equation:

$$167 \quad \frac{\partial}{\partial t} (\rho u_i) + \frac{\partial}{\partial x_j} (\rho u_j u_i) = -\frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j} (\sigma_{ij})$$

(4)

168

169 The energy balance equation:

$$170 \quad \frac{\partial}{\partial t} (\rho h) + \frac{\partial}{\partial x_j} (\rho u_j h) = \frac{\partial}{\partial x_j} \left( \Gamma_h \frac{\partial h}{\partial x_j} \right) + \frac{\partial p}{\partial t} + u_j \frac{\partial p}{\partial x_j} \quad (5)$$

171 where  $\sigma_{ij}$  is the flux of momentum and  $h$  is the enthalpy.

172

173 The solver accounts for dissipation of turbulent kinetic energy with a modified k- $\epsilon$   
 174 model (Arntzen, 1998; Hjertager, 1993).

175 The equation for turbulent kinetic energy:

176 
$$\frac{\partial}{\partial t}(\rho k) + \frac{\partial}{\partial x_j}(\rho u_j k) = \frac{\partial}{\partial x_j} \left( \frac{\mu_{\text{eff}}}{\sigma_k} \frac{\partial k}{\partial x_j} \right) + G - \rho \varepsilon; \quad G = \sigma_{ij} \frac{\partial u_j}{\partial x_i} \quad (6)$$

177 The equation for dissipation of turbulent kinetic energy:

178 
$$\frac{\partial}{\partial t}(\rho \varepsilon) + \frac{\partial}{\partial x_j}(\rho u_j \varepsilon) = \frac{\partial}{\partial x_j} \left( \frac{\mu_{\text{eff}}}{\sigma_\varepsilon} \frac{\partial \varepsilon}{\partial x_j} \right) + 1.44 \frac{\varepsilon}{k} G - 1.79 \rho \frac{\varepsilon^2}{k} \quad (7)$$

179 where  $G$  is the generation rate of turbulence.

180

181 The combustion process is treated as a single step irreversible reaction with finite  
 182 reaction rate between fuel and oxidant. The reaction scheme results in mixture  
 183 composition being determined by solving for only two variables, namely mass fraction  
 184 of fuel  $m_{fu}$ , and the mixture fraction  $f$  (Hjertager, 1984):

185 
$$\frac{\partial}{\partial t}(\rho m_{fu}) + \frac{\partial}{\partial x_j}(\rho u_j m_{fu}) = \frac{\partial}{\partial x_j} \left( J_{fu,j} \frac{\partial \varepsilon}{\partial x_j} \right) + R_{fu} \quad (8)$$

186 
$$\frac{\partial}{\partial t}(\rho f) + \frac{\partial}{\partial x_j}(\rho u_j f) = - \frac{\partial}{\partial x_j} (J_{f,j}) \quad (9)$$

187

188 where  $R_{fu}$  is the time mean rate of combustion of fuel,  $J_{fu,j}$  and  $J_{f,j}$  are the diffusive fluxes  
 189 in the  $x_j$ -direction.

190 FLACS (GexCon, 2011) solves the equations above such that the overpressures from  
 191 previous time step, the momentum equation gives a velocity field, which will be  
 192 corrected along with the updated pressure and density field by implementing a pressure  
 193 correction algorithm (Patankar, 1980).

194



195 The factors of the fuel density, the flame radius, the initial laminar flame speed of fuel  
196 play important roles in the combustion of an explosion, thereby resulting in the  
197 development of the overpressure.

198

199 Overall, influence of all parameters on the formation of explosion pressures including  
200 the mechanism of turbulent reactive gas dynamics, combustion processes and the  
201 geometry of the configurations are taken into account in the methodology of the CFD-  
202 based solver – FLACS (GexCon, 2011).

203

#### 204 **Geometry model**

205 The cases examined in this paper are analysed using CSC and also modelled using  
206 FLACS (GexCon, 2011). These are cases of large-scale geometries at scales  
207 encountered in industrial scenarios in process safety. Examples are artificial and  
208 realistic models in Fig. 1 with sizes of 90x45x15(m) and 80x50x50(m), respectively.  
209 The artificial geometries in this study were modelled with mixed obstacle arrangement  
210 patterns, obstacle diameters and confinement ratios and one realistic module truncated  
211 from a LNG (Liquefied Natural Gas) train (Fig. 1 (b) ) was also investigated.

212

213 Both propane and methane VCEs were modelled in this paper. The ambient temperature  
214 and pressure were set as 26°C and 101 kPa, respectively. Eulerian boundary conditions  
215 of the domain were used and the BC pressure was set to be equal to the ambient pressure.

216

217 Walls and decks were assumed to be unyielding during the entire explosion, i.e. rigid  
218 walls remain in place even for the largest explosion loads. FLACS (GexCon, 2011) is  
219 based on several subgrid models which require careful observation of some best

220 practice guidelines. These include the use of cubical grid cells in the combustion region  
221 were applied in order to diminish the deviations of flame propagation and pressures;  
222 the aspect ratio of the grid is controlled to within 20% and grid cells smaller than 5cm  
223 were avoided to ensure the accurate results.

224

225 For purpose of extracting the pressures, monitor points were defined at specific  
226 locations in the simulation domain where variables including volume blockage ratio  
227 (VBR), the distance of flame propagation, the characteristic average obstacle diameter  
228 are to be monitored. For instance, as shown in Fig. 1 (a), the gas cloud was ignited at  
229 the edge centre of the configuration; the monitor points were then placed along the  
230 direction of flame propagation to obtain the pressures at the increasing of the flame  
231 propagation distance. And for each simulation in this paper, more than 30 monitor  
232 points were assigned according to the grid arrangement.

233

### 234 **Evaluation of the irregular-arranged configurations subjected to gas explosion**

235 The confinement specific correlation (CSC) derived in previous work (Li et al., 2014)  
236 is used to independently predict the overpressures for similar cases with irregular  
237 arrangement of obstacles. The dimensionless and confinement specific correlation  
238 regarding the parameters of confinement, volume blockage ratio, the average obstacle,  
239 laminar flame velocity and gas density is given by:

$$240 \quad \frac{\Delta P_o}{P_{air}} = 0.037 \cdot e^{8.5C_m} \cdot [1.6 \ln(VBR_t) + 6] \cdot \left(\frac{L_{fd}}{H}\right)^{2.2} \cdot \left(\frac{D}{H}\right)^{-1.5} \cdot \left(\frac{\rho_{gas}}{\rho_{air}}\right)^{0.5} \cdot \left(\frac{S_L}{S_s}\right)^2 \quad (10)$$

241 where:

242  $\Delta P_o$  = the escalation overpressure [barg],

243  $P_{air}$  = 1 standard atmospheric pressure 101.325kPa [1 barg],

244  $D$  = the average obstacle diameter [m],  
245  $L_{fd}$  = the direct distance from the ignition location to the target point[m],  
246  $S_l$  = the laminar flame speed of the flammable gas [m/s],  
247  $S_s$  = the speed of sound [m/s],  
248  $C_m$  = the confinement ratio,  
249  $VBR_t$  = the volume blockage ratio of configuration region from the ignition point  
250 to the target,  
251  $\rho_{gas}$  = mass density of gas ( $\text{kg/m}^3$ ) (the gas density is assumed ideally under one  
252 standard atmosphere pressure at normal temperature 26 degrees in this study),  
253  $\rho_{air}$  = mass density of air ( $\text{kg/m}^3$ ),  
254  $H$  = the height of the configuration (m).

255

### 256 **Definition of regularity and irregularity of Confinement and Congestion**

257 In this study, we examined regular and irregular arrangements of congestion and  
258 confinement. This subsection describes both types of geometries.

259

260 In terms of the congestion, the artificial module in Fig. 1 (a) features uniform obstacle  
261 diameter and a regular pattern of obstacles. By contrast, the module 1 and module 4 in  
262 Fig. 3 were modelled with irregularities. And more importantly, unlike the previous  
263 study (Li et al., 2014) where the simulations are modules extracted from an existing  
264 LNG (Liquefied Natural Gas) train; they are composed of realistic layouts of structural  
265 components with random irregularities. The geometry displayed in Fig. 3 of this paper  
266 are artificial modules with controllable irregularities, for example, from module 1 to 4,  
267 they are intentionally organized with increasing obstacle diameters, equidistant

268 separation distances and mixed intersecting obstacle arrangements, etc. Additionally,  
269 those artificially arranged irregular modules in this paper are large-scale modules whilst  
270 those artificial ones in the previous study (Li et al., 2014) are in small-scale.

271

272 Using the definition of confinement in the previously proposed paper (Li et al., 2014),  
273 all simulations were conducted under the configurations with the parallel plates in semi-  
274 3D overpressure expansion; the confinement ratio is characterized as the ratio of the  
275 blocked area on the top and bottom plates over the total area of the top and bottom  
276 surfaces. Therefore, a configuration covered with two solid top and bottom plates, such  
277 as the module in Fig. 2 (a), is considered to be fully confined in the z- direction; and  
278 the one without top plate is defined as open in the +z-direction, as seen Fig. 2 (c). In  
279 this study, the partial confinement between the open air and the full confinement is used  
280 to test the correlations under conditions of irregular confinement.

281

### 282 **Application of the CSC to the irregular-arranged modules**

283 By using the CSC, overpressures were estimated for configurations with congestion of  
284 an irregular arrangement subjected to vapour cloud explosions and the results are  
285 described in this section. As seen in Fig. 3, four modules with inhomogeneous obstacles  
286 plus one realistic module were modelled here to simulate 400 new explosions for this  
287 study. Four of the modules are of highly confined configurations. In the explosion  
288 models, a stoichiometric flammable gas cloud was used to fill the obstacle  
289 configurations; methane and propane are both used as fuels in this study. The  
290 parameters are shown in Table 1.

291

292 Fig. 4 shows the correlation pressure predictions on the x- axis against the pressures  
293 calculated with FLACS (GexCon, 2011) on the y- axis. The R-squared ( $R^2$ ) value is  
294 extracted for each of these cases. As seen in Fig. 4, the R-squared value for each  
295 simulation model is between 0.66 and 0.90, which shows the CSC correlation applies  
296 to practical geometries of greatly varying confinement ratios as well as irregular pattern  
297 of VBR and varying obstacle diameters in the configurations. The results from the CSC  
298 correlation were also compared to results from the Guidance for the Application of the  
299 Multi-Energy method (GAME) correlation (Eggen, 1995).

300

301 The GAME correlation below was used to determine the gas explosion overpressure  
302 for the modules in Fig. 3 with confinement between parallel plates.

$$303 \quad \Delta P_o = 3.38 \cdot \left( \frac{VBR \cdot L_f}{D} \right)^{2.25} S_l^{2.7} \cdot D^{0.7} \quad (11)$$

304 where VBR is the volume blockage ratio defined as the ratio of the total volume of the  
305 obstacles inside an obstructed region,  $L_f$  is the maximum distance of flame propagation  
306 obtained by assuming  $L_f$  equal to the radius of a hemisphere with a volume equal to the  
307 volume of the configuration,  $D$  is the averaged obstacle diameter based on the entire  
308 configuration,  $S_l$  is the laminar flame speed of the flammable gas

309 The GAME correlation is seen to be generally, but not always conservative in the  
310 determination of the overpressure for cases with artificially homogenous congestion.

311 When applied to geometries (Fig. 3) with irregularities of confinement and congestion,  
312 the overall comparison results, seen in Fig. 5, give a poor agreement with the FLACS  
313 results, specifically, the data obtained by means of the GAME correlation tend to  
314 overestimate the overpressure significantly whereas the CSC correlation result agrees  
315 well with FLACS simulations, Fig. 5.

316

317 The GAME correlation was derived from MERGE experiments (EMEG, 1997; Harris  
318 & Wickens, 1989; Merx et al., 1995; Schumann et al., 1993; Wingerden, 1988, 1989)  
319 which possesses the idealized obstacles with average diameter and homogeneously  
320 distributed in the configuration, the volume blockage ratio and confinement ratio are  
321 regularly patterned. In this study, we examine the performance of the GAME  
322 correlation for cases where the irregularities of the obstacles as well as high degrees of  
323 confinement are characteristics of geometry. This has not been adequately tested using  
324 GAME correlation up till this point. The CSC correlation is derived based on the CFD  
325 coded software – FLACS (GexCon, 2011), the parameters regarding the geometrical  
326 detail and the turbulent reactive gas dynamics mechanism are accounted for, hence this  
327 approach better models the inhomogeneous configurations where the turbulence  
328 generation/degeneration and the burning velocity acceleration/ deceleration are key  
329 factors in the variation of the congestion and confinement.

330

### 331 **Rapid prediction of structural damage**

332 The CSC correlation has undergone validation (Li et al., 2014) with very good  
333 agreement with pressures predicted using CFD modelling. In this study we also add a  
334 rapid structural damage level prediction process; two different simulation  
335 configurations with 8 well-located monitor points were numerically modelled using  
336 using FLACS (GexCon, 2011) as the case studies shown below, the pressure vs. time  
337 history data was obtained for the specific structure members at those monitor points.

338

339 As seen in Fig. 7, the overpressure figures are observed from the fully congested  
340 configuration Fig. 6 (a) and the configuration with a sufficient separation distance Fig.

341 6 (b). For both configurations, the explosion occurs from the centre of the left module  
342 as illustrated in Fig. 1 (a), the flame propagates through the fuel away from the ignition  
343 point till the fuel exhausted, the monitor points 1 to 4 are place in the centre along the  
344 flame propagation direction from left to right. It is noted in Fig. 7 (a) that the magnitude  
345 of the maximum overpressure increases from 125 kPa to 230 kPa as the flame path from  
346 the ignition through congestion increases, the maximum overpressure is seen at monitor  
347 point 4.

348

349 The phenomenon observed above is attributed to flame acceleration which is described  
350 in (Bjerketvedt et al., 1997; Eggen, 1995; Li et al., 2014), the geometry of the gas  
351 explosion scenario and flame propagation distance both contribute the development of  
352 the flame acceleration and overpressure. In a gas explosion scenario, turbulence is  
353 generated when the flame interacts with the obstacles, which results in the flame  
354 acceleration and the generation of more turbulence as the flame propagates further in  
355 the congested area: a self-feeding mechanism increasing flame speed and thereby  
356 increasing the overpressure. This is in contrast to an explosion pressure field from a  
357 scenario using explosives where the maximum blast load is seen at the minimum stand-  
358 off distance decreasing with distance from ignition point.

359

360 However, if a flame propagates in a premixed air-fuel cloud in an uncongested open  
361 space, as seen in Fig. 7 (b), the phenomenon of flame acceleration does not continue in  
362 the open uncongested space. The separation space in Fig. 6 (b) reduces the congestion  
363 and intensity of turbulence which results in the decrease of the overpressure. An  
364 explosion generated with explosives is not affected by a separation space in the same

365 manner and hence the determination of TNT explosion overpressure is only a function  
366 of stand-off distance in the space.

367

368 For gas explosions, the pressure time history is typically a triangular shaped wave with  
369 an extremely short time period, (Fig. 7). For each monitor point, the impulse vs. time  
370 data obtained by means of integration of the pressure time history and this seen in Fig.  
371 8. The maximum impulse is observed after the peak of the overpressure and the steady  
372 state of the impulse is seen after the pressure attenuates to 0kPa.

373

374 By applying the data above to the structural members, the calculation of the final states  
375 of damage, which is of major concern can be assessed. Specifically, a structural member  
376 in an offshore module subjected to gas explosion is simplified as a Single Degree Of  
377 Freedom (SDOF) equivalent structural model to assess its structural response behaviour.  
378 The maximum deflection rather than the detailed deflection-time history of the structure  
379 determines the failure criterion of the structure.

380

381 In order to evaluate the structural damage level, a pressure-impulse ( $p$ - $i$ ) diagram of the  
382 equivalent SDOF structural model (Mays & Smith, 1995; Smith & Hetherington, 1994)  
383 was developed as shown in Fig. 9. Once the critical deflection (maximum allowable  
384 deflection)  $y_c$  of the structure is specified, a curve was obtained, as the dashed line  
385 shows in Fig. 9, which indicates various combinations of the non-dimensional initial  
386 peak overpressure  $p$  and the impulse  $i$  of the external load that will cause the same  
387 deflection of the structure. The non-dimensional pressure and impulse are defined as

388 
$$p = P_o A / (k y_c / 2) \text{ and } i = I_o / y_c \sqrt{k m_{se}} .$$



389

390 The impulsive asymptote of the curve is  $i=1.0$  and the quasi-static asymptote is  $p=1.0$ .  
391  $P_o$  is the initial peak pressure of the blast load and  $I_o$  is the impulse of the blast load as  
392 shown in Fig. 8,  $A$  is the cross-sectional area of the SDOF structural,  $m_{se}$  is the  
393 equivalent mass of the equivalent SDOF structure and  $k$  is its stiffness. In this study,  
394 we take the gas explosion scenarios at the four monitor points in the congested  
395 configuration as examples, the steel material was used to simulate the offshore  
396 structural members which are modelled as simply supported beams, the cross-sectional  
397 area, the equivalent mass and the stiffness were set as  $1\text{m}^2$ ,  $1\text{kg}$  and  $3\times 10^6\text{ N/m}$ .  
398 Therefore, the  $p-i$  combinations of the gas explosion blast load were determined; the  
399 four points indicated in Fig. 9 represent the blast load results obtained in Fig. 8. For the  
400 four monitor points, any data below the dashed curve (overpressure and impulse at point  
401 1 and point 2) will not result in any damage of the structure while those above the curve  
402 (overpressure and impulse at point 3 and point 4) will induce failure of the structure.

403

#### 404 **Conclusion**

405 This paper examined 400 scenarios in geometries similar to the MERGE experiments  
406 on which the GAME correlation is based, with one important distinction: The  
407 confinement and congestion were deliberately varied such that some of the geometries  
408 possessed inhomogeneity of both parameters. Little experimental data exists for such  
409 configurations and hence the cases were modelled here with the commercial CFD  
410 software FLACS (GexCon, 2011). A realistic model was also examined and modelled  
411 using the commercial code. Realistic geometries also typically display a high degree of  
412 inhomogeneity in confinement and congestion.

413

414 The overpressure predictions using FLACS (GexCon, 2011) at various target locations  
415 were compared with the results from a newly derived correlation by (Li et al., 2014)  
416 and the GAME correlation. It is found that the CSC correlation better agrees with the  
417 overpressure predictions obtained using CFD when compared with the GAME  
418 correlation. The results further demonstrate that the correlation by the CSC is suitable  
419 for the modelling of realistic geometries.

420

421 The numerically calculated pressure and impulse vs. time results were related to  
422 damage level by simplifying the offshore structural component as an SDOF equivalent  
423 model, the structural damage level was determined within the  $p-i$  diagram. The results  
424 show that the cases examined are ones that require an increased level of accuracy as  
425 they are very close to cases that may cause permanent damage to structural members.

426

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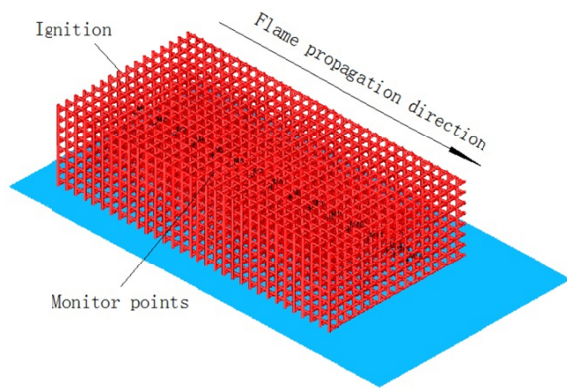
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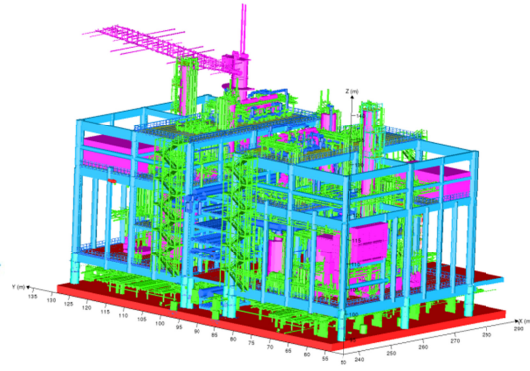
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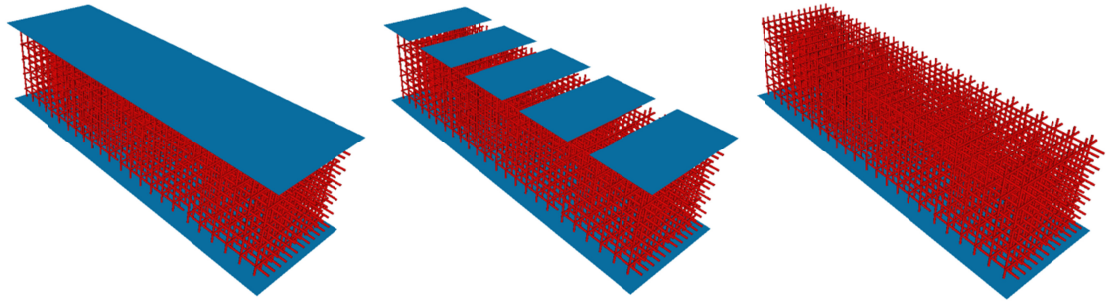


**(a) Artificial Model**



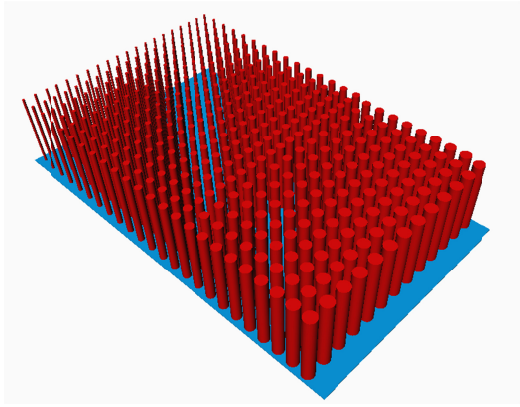
**(b) Realistic Model**

**Fig. 1 FLACS simulation models**

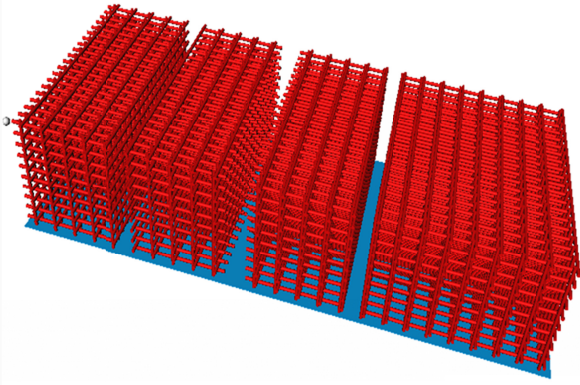


(a) Fully confined module      (b) partially confined module      (c) Open in +z-direction

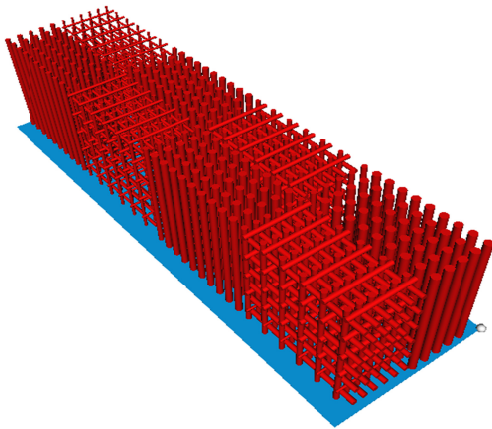
**Fig. 2 Artificial modules with varying confinement**



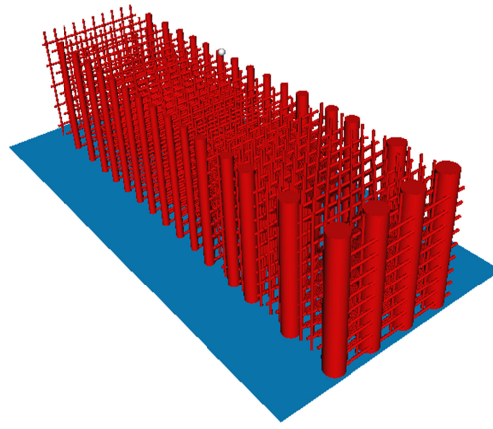
**Module 1 - Irregular-arranged**



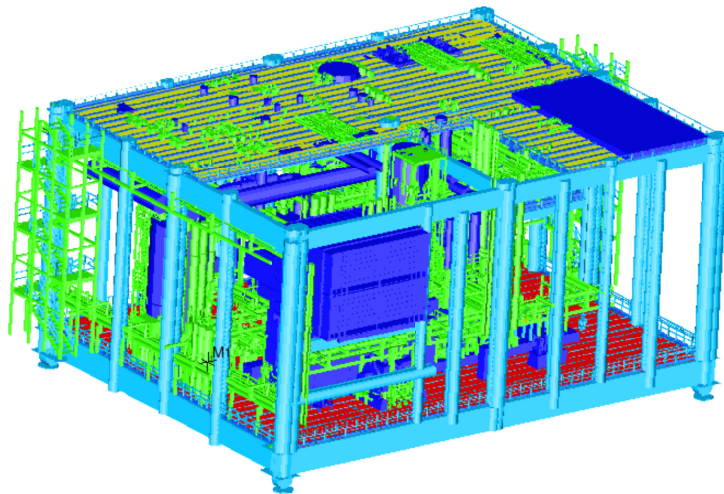
**Module 2 - Irregular-arranged**



**Module 3 - Irregular-arranged**

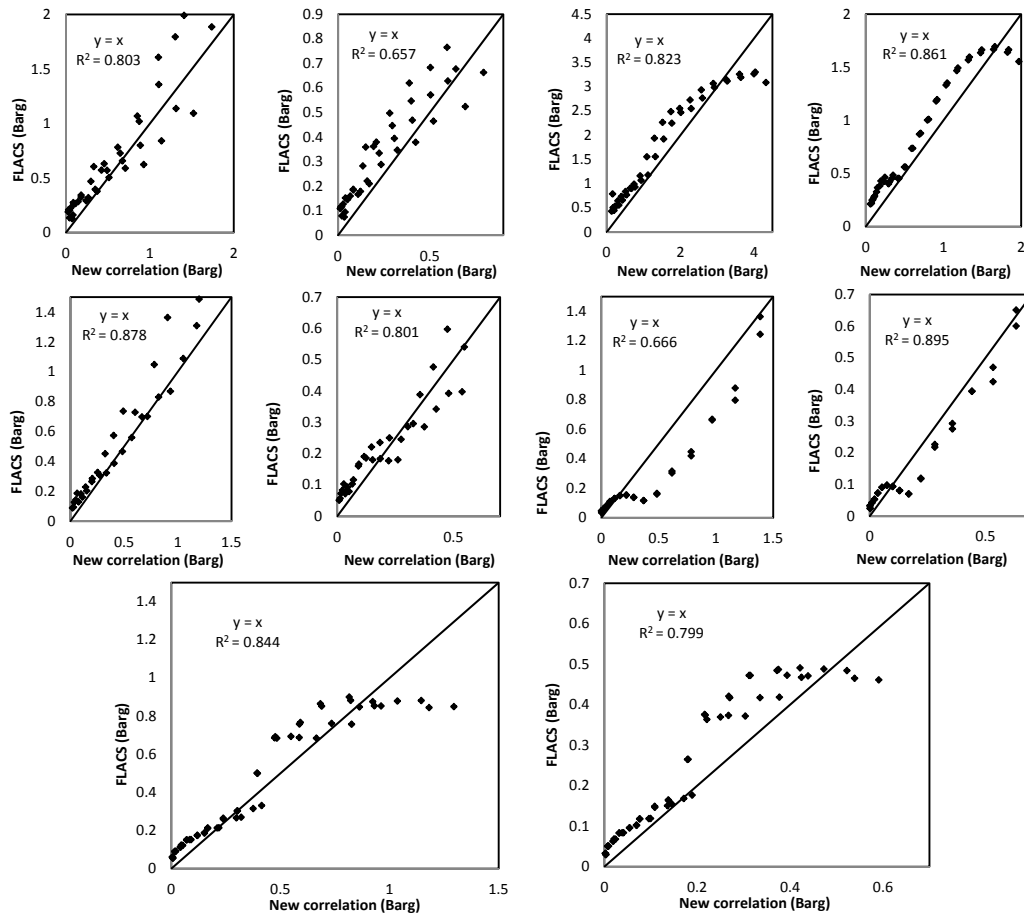


**Module 4 - Irregular-arranged**

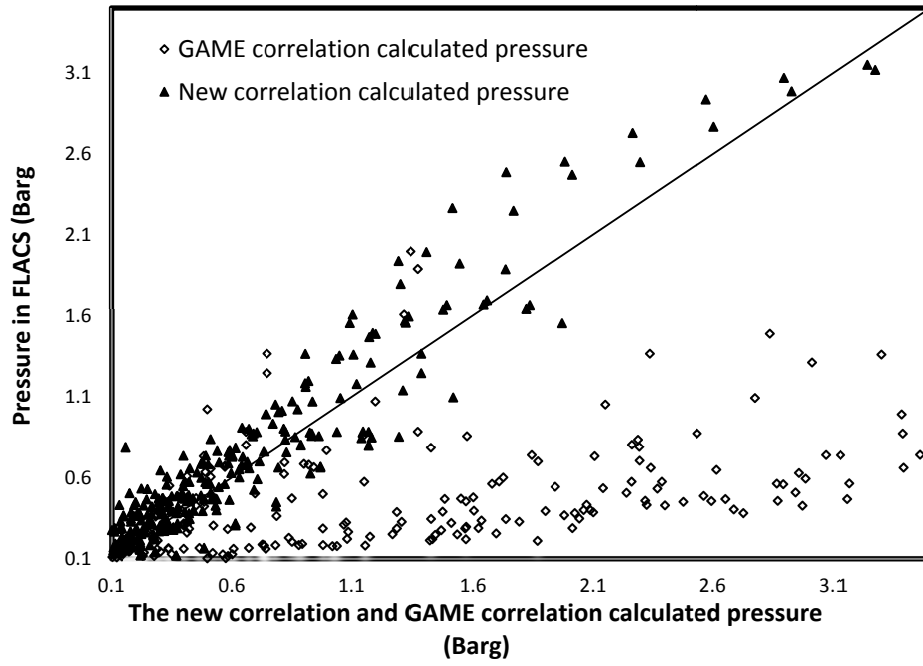


**Module 5 – Realistic Module**  
**Fig. 3 Modules with irregularities 1-5**

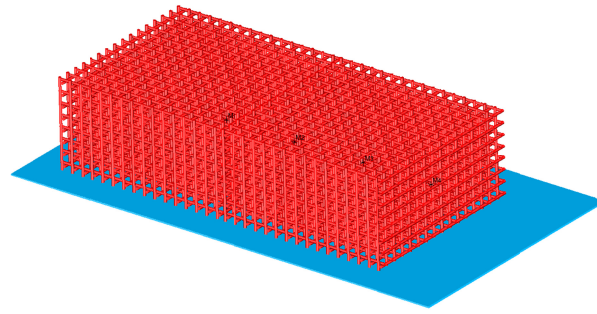




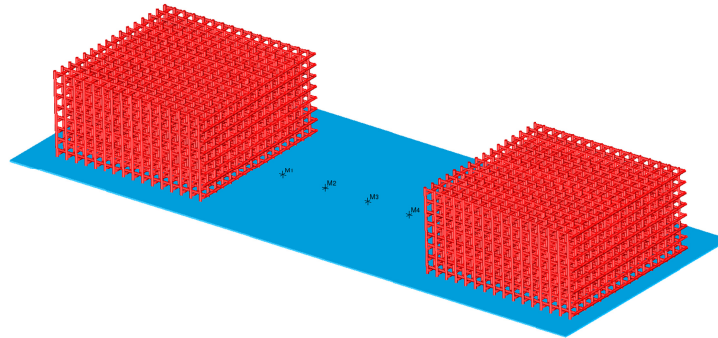
**Fig. 4** the comparison of CSC correlation overpressure data vs. FLACS results for the irregular-patterned configurations subject to methane and propane vapour explosions



**Fig. 5** the comparison of the new correlation and the GAME overpressure data vs. FLACS results for the irregular-patterned configurations subject to methane and propane vapour explosions

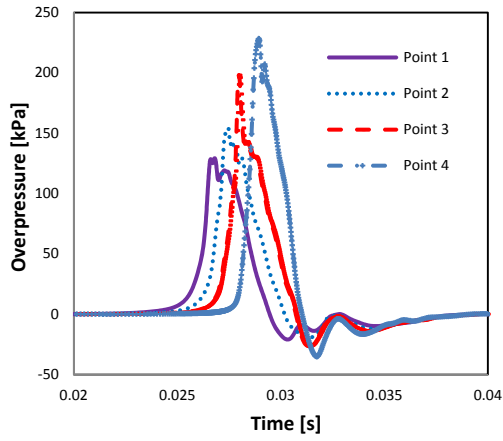


**(a) Monitors within the congestion**

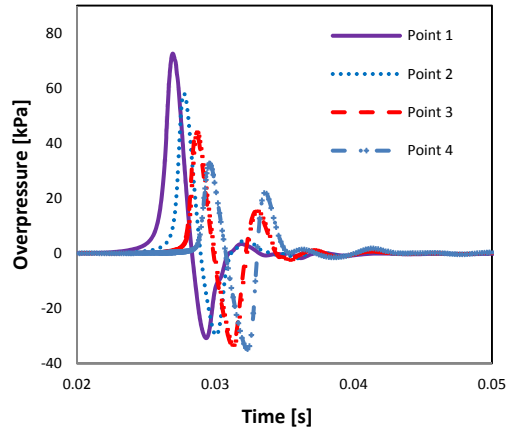


**(b) Monitors in the open space**

**Fig. 6 Specified monitor points at different gas explosion scenarios**

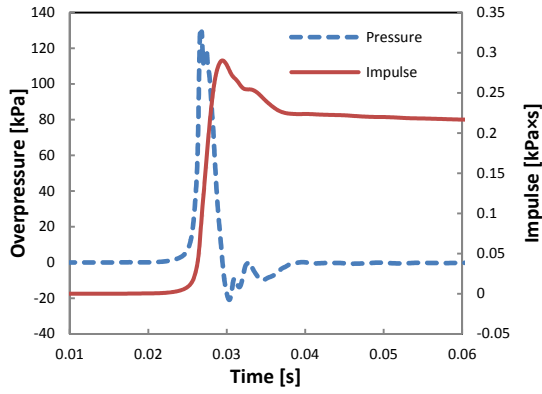


(a) Monitors within the congestion

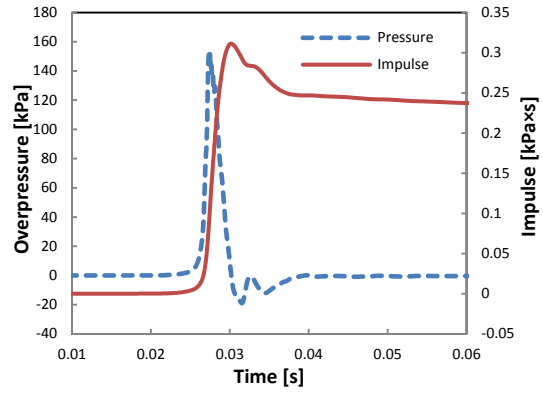


(b) Monitors in the open space

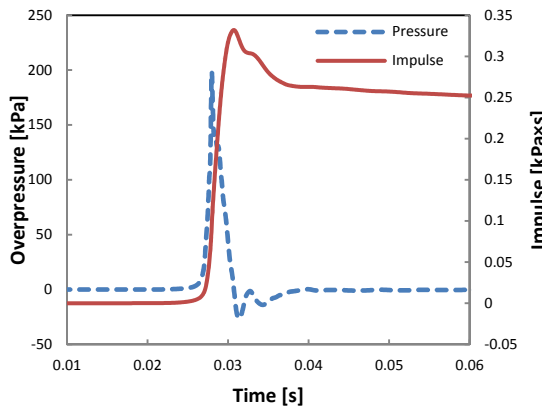
Fig. 7 The overpressure vs. time results for the specified monitor points



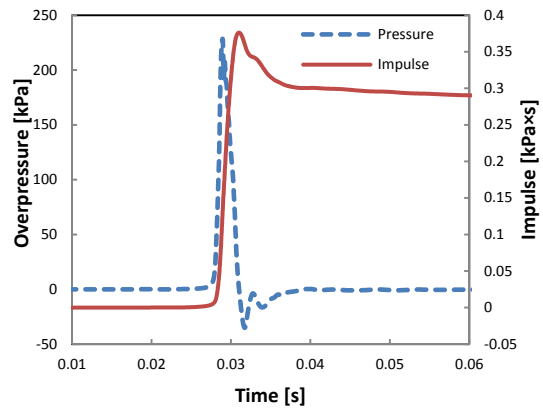
(a) Monitor point 1



(b) Monitor point 2

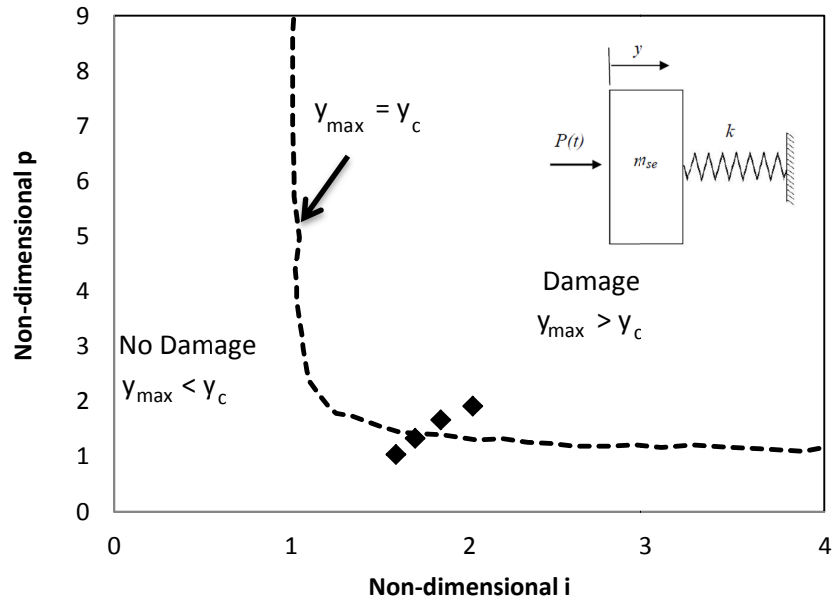


(c) Monitor point 3



(d) Monitor point 4

Fig. 8 The time domain results of overpressure and impulse at different monitor points in the congested space



**Fig. 9 Non-dimensional  $p$ - $i$  diagram of an equivalent SDOF structural model**

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Fig. 9 Non-dimensional p-i diagram of an equivalent SDOF structural model

**Table 1 Parameters in difference modules**

<b>Case No.</b>	<b>Gas composition</b>	<b>D (m)</b>	<b>VBR*</b>	<b>S<sub>l</sub> (m/s)</b>	<b>Gas density (kg/m<sup>3</sup>)</b>	<b>C<sub>m</sub></b>
<b>1. Module 1</b>	Pure Methane	0.37	0.11	0.40	0.65	1.00
<b>2. Module 1</b>	Pure Propane	0.37	0.11	0.46	1.80	1.00
<b>3. Module 2</b>	Pure Methane	0.31	0.14	0.40	0.65	0.96
<b>4. Module 2</b>	Pure Propane	0.31	0.14	0.46	1.80	0.96
<b>5. Module 3</b>	Pure Methane	0.33	0.13	0.40	0.65	0.90
<b>6. Module 3</b>	Pure Propane	0.33	0.13	0.46	1.80	0.90
<b>7. Module 4</b>	Pure Methane	0.21	0.04	0.40	0.65	0.90
<b>8. Module 4</b>	Pure Propane	0.21	0.04	0.46	1.80	0.90
<b>9. Module 5</b>	Pure Methane	0.59	0.12	0.40	0.65	0.76
<b>10. Module 5</b>	Pure Propane	0.59	0.12	0.46	1.80	0.76

\* VBR here is the volume blockage ratio of the entire obstructed region for Module 1 to 5.