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## The Effect of Normal Contact Forces on the Stress in Shear Rate Invariant Particle Suspensions

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### Abstract

We present a tensorial theory for the microstructure and the stress in shear rate invariant particle suspensions, that includes hydrodynamic and normal but not tangential hard sphere interaction forces. The theory predicts, that hydrodynamic forces produce a negligible first normal stress difference, while contact forces produce a positive first normal stress difference. The theory thereby provides a rationale for seemingly contradicting experimental observations in the literature. In addition, the theory captures experimentally observed time dependence of the shear stress after shear reversal.

#### 8 I. INTRODUCTION

Particle suspensions occur ubiquitously in nature, and their mechanical stress  $\Sigma$  is gov-<sup>10</sup> erned by the particle interaction forces, which can be classified into hydrodynamic, and non-<sup>11</sup> hydrodynamic. The nature of the hydrodynamic forces depends on the particle Reynolds <sup>12</sup> number  $\operatorname{Re}_p = \dot{\gamma} a^2 / \nu$ , where *a* is the particle radius,  $\nu = \eta / \rho$  is the solvent kinematic vis-<sup>13</sup> cosity,  $\eta$  is the solvent dynamic viscosity,  $\rho$  is the solvent mass density, and  $\dot{\gamma}$  is the shear <sup>14</sup> rate. When:  $\operatorname{Re}_p \ll 1$ , flow inertia is negligible, and the hydrodynamic forces are governed <sup>15</sup> by the linear Stokes equations.

The Stokes equations predict that particles make no physical contacts in a fluid, since 17 the lubrication force diverges at contact [1]. With increasing volume fraction  $\phi$  however, 18 the lubrication films become progressively thinner, and when their thickness approaches the 19 atomic length scale, the films disintegrate, resulting in physical contacts.

This work addresses the effect of hard and frictionless contact forces on the particle stress. <sup>21</sup> Hard contacts do not introduce a force scale F into the system, and the non-dimensional <sup>22</sup> suspension viscosity:  $\Sigma_{12}/\eta\dot{\gamma}$ , depends therefore only on the particle volume fraction  $\phi$  and <sup>23</sup> not on the shear rate  $\dot{\gamma}$ , as this can not be non-dimensionalised into  $a^2\eta\dot{\gamma}/F$ , due to the <sup>24</sup> absence of F. This study is therefore restricted to shear rate invariant suspensions.

Experimental data on the suspension stress  $\Sigma$  are mainly concerned with shear flow, where:  $\boldsymbol{L} = \nabla \boldsymbol{U}^T = \dot{\gamma} \boldsymbol{\delta}_1 \boldsymbol{\delta}_2$  is the velocity gradient tensor,  $\dot{\gamma} = \sqrt{2\boldsymbol{E}:\boldsymbol{E}}$  is the shear rate,  $\boldsymbol{Z}_T \boldsymbol{E} = \frac{1}{2} \left( \boldsymbol{L} + \boldsymbol{L}^T \right)$  is the strain rate tensor,  $\boldsymbol{U}$  is the velocity vector, and 1, 2 and 3 are the flow direction, the gradient direction and the vorticity direction, respectively

Fig. 1 summarises experimental data on the relative, first and second normal stress of differences in shear rate invariant suspensions. These quantities are defined as:  $\zeta_1 = (\Sigma_{11} - \zeta_{11} - \Sigma_{22})/\Sigma_{12}$ , and:  $\zeta_2 = (\Sigma_{22} - \Sigma_{33})/\Sigma_{12}$ , respectively. While  $\zeta_2$  has always been observed to be negative,  $\zeta_1$  has been observed to be both negative and small (compared to  $\zeta_2$ ) [5–7], as well as positive [2, 4]. It is noted that a positive  $\zeta_1$  has also been observed in shear thickening suspensions [8–10], which supports the hypothesis that particle contact forces are responsible for  $\zeta_1 > 0$  [11].

In addition to normal stresses, effects of contact forces are also reflected by a stress discontinuity upon the reversal of shear flow. In the absence of contacts, the Stokes equation dictates, that the stress is linear in the velocity. This means, that, when the flow velocity



FIG. 1. Measured, steady, relative, first and second normal stress differences:  $\zeta_1 = (\Sigma_{11} - \Sigma_{22})/\Sigma_{12}$ (open markers), and:  $\zeta_2 = (\Sigma_{22} - \Sigma_{33})/\Sigma_{12}$  (filled markers), under shear rate invariant conditions, and as functions of the particle volume fraction  $\phi$ .  $\triangle$ :  $a = 20 \ \mu\text{m}$ ,  $\nabla$ :  $a = 70 \ \mu\text{m}$  polystyrene in water, UCON oil and zinc bromide [2];  $\Box$ :  $a = 35 \ \mu\text{m}$ ,  $\Diamond$ :  $a = 70 \ \mu\text{m}$  polystyrene in poly(ethylene glycol-ran-propylene glycol) monobutylether [3];  $\triangleleft$ :  $a = 5 \ \mu\text{m}$  poly (methyl methacrylate) (PMMA) in Triton X-100, anhydrous zinc chloride, and water (TZW) [4];  $\oplus a = 98 \ \mu\text{m}$  PMMA in TZW [5];  $\bigcirc$ :  $a = 22 \ \mu\text{m}$  glass in corn syrup and glycerin [6];  $\overleftrightarrow a = 20 \ \mu\text{m}$  polystyrene in silicone fluid [7]. The lines are drawn to guide the eye, and the lower line represents the empirical relation [Eq. (16)].

<sup>39</sup> is instantaneously reversed, the stress is instantaneously reversed too, as observed experi-<sup>40</sup> mentally for small  $\phi$  [12]. For large  $\phi$ , particles may experience contacts, and since contact <sup>41</sup> forces are not reversed upon flow reversal, there is a discontinuity in the (absolute value of <sup>42</sup> the) particle stress upon flow reversal [12–14].

In this work we provide a micro-structural explanation for the above mentioned experi-44 mental observations, regarding normal stresses in steady shear flow, and stress discontinuity 45 after shear reversal. To this end we include hard and frictionless contact forces into a 46 previously proposed tensorial theory for the suspension microstructure and stress [15].

#### 47 II. DERIVATION OF THE THEORY

#### 48 A. Hydrodynamic Forces

<sup>49</sup> First we summarise the theory in the absence of contact forces. For a full derivation, the <sup>50</sup> reader is referred to Ref. [15]. In the two-body approximation the stress is given by [16–19]:

$$\boldsymbol{\Sigma} = -\frac{1}{V} \sum_{\alpha > \beta} \boldsymbol{F}_{\alpha,\beta} \boldsymbol{r}_{\alpha,\beta} = -n \langle \boldsymbol{F} \boldsymbol{r} \rangle.$$
(1)

<sup>51</sup> Here: n = N/V, is the particle number density, N is the number of particles inside the <sup>52</sup> averaging volume V,  $\mathbf{F}_{\alpha,\beta}$  is the interaction force  $\mathbf{F}$  between particles  $\alpha$  and  $\beta$ , and  $\mathbf{r}_{\alpha,\beta}$ <sup>53</sup> is the corresponding particle pair separation vector  $\mathbf{r} = \mathbf{p}r$ , where  $\mathbf{p}$  is the particle pair <sup>54</sup> orientation unit vector, and  $r = |\mathbf{r}|$  is the particle pair separation. The stress is dominated <sup>55</sup> by particles with small gaps:

$$\boldsymbol{r} = 2a\boldsymbol{p}.\tag{2}$$

56 The interaction force F:

$$\boldsymbol{F} = \boldsymbol{F}_h + \boldsymbol{F}_c, \tag{3}$$

<sup>57</sup> is the sum of the hydrodynamic force  $F_h$ , and the contact force  $F_c$ , which is assumed zero, <sup>58</sup> for the moment. The pair separation vector evolves as:

$$\dot{\boldsymbol{r}} = c_1 \boldsymbol{L} : \boldsymbol{r} \boldsymbol{p} \boldsymbol{p} + \boldsymbol{L} \cdot \boldsymbol{r} \cdot (\boldsymbol{\delta} - \boldsymbol{p} \boldsymbol{p}), \qquad (4)$$

<sup>59</sup> and the corresponding lubrication force is to leading order:

$$\boldsymbol{F}_h = -a^2 \eta c_2 \boldsymbol{E} : \boldsymbol{p} \boldsymbol{p} \boldsymbol{p}. \tag{5}$$

<sup>60</sup> Here  $c_1$  and  $c_2$  are non-dimensional functions of r/a and  $\phi$ . Combining Eqs. (1, 2, 3, 5) and <sup>61</sup> using that  $\phi \sim na^3$  gives the following particle stress tensor:

$$\boldsymbol{\Sigma} = \alpha \eta \boldsymbol{E} : \langle \boldsymbol{p} \boldsymbol{p} \boldsymbol{p} \boldsymbol{p} \rangle. \tag{6}$$

<sup>62</sup> Here  $\alpha = \tilde{c}_2 \phi$  is the lubrication parameter, and  $\tilde{c}_2$  is the effective  $c_2$ , which is averaged over <sup>63</sup> the distribution of pair configurations, and which diverges when  $\phi$  approaches maximum <sup>64</sup> packing. The average  $\langle \cdots \rangle$  in Eq. (6) is expressed as an integral over the probability distribution function  $\Psi(\mathbf{r})$  of the particle pair separation vector  $\mathbf{r}$ :

$$\langle \cdots \rangle = \int_{|\boldsymbol{r}|=2a}^{|\boldsymbol{r}|=2a+\delta r} \Psi(\boldsymbol{r}) \cdots d^{3}\boldsymbol{r},$$
(7)

<sup>67</sup> where the integration is restricted to the so-called 'interaction shell', where particle pairs <sup>68</sup> have small gaps:  $0 < r - 2a < \delta r$ . The evolution of  $\Psi(\mathbf{r})$  is governed by the Smoluchowski <sup>69</sup> equation:

$$\partial_t \Psi + \partial_k \left( \dot{r}_k \Psi \right) = 0. \tag{8}$$

Since computing  $\Psi(\mathbf{r})$  is costly, we compute instead its second order orientation moment  $\mathbf{a} = \langle \mathbf{pp} \rangle$ , referred to as the microstructure. The evolution equation for  $\mathbf{a}$  is derived, by inserting Eq. (4) into Eq. (8), multiplying the result by  $\mathbf{pp}$ , and integrating the result from r = 2a to  $r = 2a + \delta r$ ; see Ref. [15]:

$$\partial_t \langle \boldsymbol{p} \boldsymbol{p} \rangle = \boldsymbol{L} \cdot \langle \boldsymbol{p} \boldsymbol{p} \rangle + \langle \boldsymbol{p} \boldsymbol{p} \rangle \cdot \boldsymbol{L}^T - 2\boldsymbol{L} : \langle \boldsymbol{p} \boldsymbol{p} \boldsymbol{p} \boldsymbol{p} \rangle - \beta \left[ \boldsymbol{E}_e : \langle \boldsymbol{p} \boldsymbol{p} \boldsymbol{p} \boldsymbol{p} \rangle + \frac{1}{15} \left( 2\boldsymbol{E}_c + \operatorname{Tr}(\boldsymbol{E}_c) \boldsymbol{\delta} \right) \right]. \quad (9)$$

The first line of Eq. (9) described rotation of rigid dumbbells, i.e. fixed pair separations. The r1 second line accounts for changes in the separation, which correspond to an orientation flux r2 between the interaction shell and the exterior. This term is interpreted as the association and r3 dissociation of interacting particle pairs, by the action of the compressive and the extensional r4 parts of the rate of strain tensor:  $E_c$  and  $E_e$ , respectively, which pushes particles together r5 and pulls them apart, respectively. These effects are controlled by the pair association rate r6  $\beta$ , which is an increasing function of  $\phi$ .

To close the theory a relation is needed to express the fourth order moment  $\langle pppp \rangle$  in terms of the second order moment  $\langle pp \rangle$ . Here we use the linear closure, that was proposed in Ref. [20], which is accurate when the distribution is close to isotropy, such that  $\Psi(p)$ is well captured by a linear expansion in the anisotropy tensor  $\boldsymbol{a} - \boldsymbol{\delta}/3$ , i.e.:  $\Psi(\boldsymbol{p}) =$  $(4\pi)^{-1} \left[1 + \frac{15}{2} (\boldsymbol{a} - \boldsymbol{\delta}/3) : \boldsymbol{pp}\right]$ .

$$\langle p_i p_j p_k p_l \rangle = -\frac{1}{35} \langle p_m p_m \rangle \left( \delta_{ij} \delta_{kl} + \delta_{ik} \delta_{jl} + \delta_{il} \delta_{jk} \right) + \frac{1}{7} \left( \delta_{ij} \langle p_k p_l \rangle + \delta_{ik} \langle p_j p_l \rangle + \delta_{il} \langle p_j p_k \rangle + \langle p_i p_j \rangle \delta_{kl} + \langle p_i p_k \rangle \delta_{jl} + \langle p_i p_l \rangle \delta_{jk} \right).$$
(10)

#### 77 B. Contact Forces

Here, we extend the theory with hard and frictionless contact forces. We consider a limiting member of the class of shear rate invariant suspensions in which the interparticle friction coefficient vanishes and tangential friction forces may be ignored; see e.g. Ref. [21], and the microstructure equation [Eq. (9)] is unaffected by the contact forces. It is noted, that under shear thickening conditions, the tangential friction may have an effect on the particle motion, involving a transition from sliding to rolling friction, and these effects are not captured by the present theory.

<sup>85</sup> By definition, the normal contact force  $F_c$  is directed along p, i.e.  $F_c = |F_c|p$  when a <sup>86</sup> particle pair is under compression, while it is zero, when a pair is under extension. The <sup>87</sup> contact force magnitude  $|F_c|$  is therefore assumed to be proportional to the compressive <sup>88</sup> part  $E_c$  of E projected onto p, i.e.  $|F_c| = -c_3 a^2 \eta E_c : pp$ , where  $c_3$  is a non-dimensional <sup>89</sup> function of p, and  $a^2\eta$  is added to make the expression dimensionally correct.

$$\boldsymbol{F}_c = -c_3 a^2 \eta \boldsymbol{E}_c : \boldsymbol{p} \boldsymbol{p} \boldsymbol{p}. \tag{11}$$

<sup>90</sup> Combining Eqs. (1, 2, 3, 5, 11) and using that  $\phi \sim na^3$  we arrive at the following particle <sup>91</sup> stress tensor:

$$\boldsymbol{\Sigma} = \eta \left( \alpha \boldsymbol{E} + \chi \boldsymbol{E}_c \right) : \langle \boldsymbol{p} \boldsymbol{p} \boldsymbol{p} \boldsymbol{p} \rangle, \tag{12}$$

<sup>92</sup> where  $\chi = \tilde{c}_3 \phi$  is the contact parameter, and  $\tilde{c}_3$  is the effective  $c_3$ , which is averaged over <sup>93</sup> the distribution of pair configurations, and which diverges when  $\phi$  approaches maximum <sup>94</sup> packing.

#### 95 III. THEORETICAL PREDICTIONS

#### 96 A. Steady Shear

In shear flow Eqs. (9, 10) predict that particle pairs associate in the compressive quadrants:  $a_{12} < 0$ , rotate towards  $x_2$ , and dissociate in the extensional quadrants  $a_{12} > 0$ . For:  $\beta > 3$ , the association and dissociation dominate the rotation. The resulting distribution aligns in the compressive quadrant:  $a_{12} < 0$ , with a slight tilt towards  $x_2$ , i.e.:  $a_{22} > a_{11}$ . For:  $\beta < 3$ , on the other hand, the pair rotation dominates the association and dissociation. Starting from isotropy, the resulting distribution oscillates and dampens towards a preferred



FIG. 2. (a) Modelled [Eqs. (9, 10)] microstructure  $\boldsymbol{a}$  as a function of the pair association rate  $\beta$ . (b) Volume fraction  $\phi$  dependence of the planar microstructure  $\boldsymbol{a}^{2D}$ , measured in Ref. [22] [Eq. (14), markers] and modelled [Eqs. (13, 15, 16), lines].

alignment in the  $x_1$  direction, corresponding to  $a_{11} > a_{22}$  and  $a_{12} > 0$ . As these oscillations have not been observed in experiments, we restrict the following analysis to  $\beta > 3$ . The corresponding analytical solution to Eqs. (9, 10):

$$\boldsymbol{a} = (6240 + 810\beta + 135\beta^2)^{-1} \\ \times \begin{pmatrix} 3256 - 374\beta + 129\beta^2 & -252\beta + 84\beta^2 & 0 \\ -252\beta + 84\beta^2 & 904 + 410\beta + 129\beta^2 & 0 \\ 0 & 0 & 820 + 564\beta + 87\beta^2 \end{pmatrix}, \quad (13)$$

 $_{97}$  is plotted as a function of  $\beta$  in Fig. 2a.

In Fig. 2b, we compare the modelled microstructure to experimental data from Ref. [22], 99 reporting the planar, pair distribution function  $\Psi^{2D}$  in the  $(r_1, r_2)$  - plane. In Fig. 2b the 100 markers indicate the corresponding, measured, planar moments  $a^{2D}$ :

$$\boldsymbol{a}^{\text{2D}} = \int_{|\boldsymbol{r}|=1.7a}^{|\boldsymbol{r}|=2.3a} \Psi^{\text{2D}}(\boldsymbol{r}) \boldsymbol{p} \boldsymbol{p} d^2 \boldsymbol{r}.$$
 (14)

<sup>101</sup> These measurement data show a weak departure from isotropy over the entire  $\phi$ -range, which <sup>102</sup> supports the validity of the linear closure [Eq. (10)]. To compare our theory [Eq. (13)] to



FIG. 3. Modelled [Eqs. (9, 10, 12)] relative, first normal stress difference:  $\zeta_1 = (\Sigma_{11} - \Sigma_{22})/\Sigma_{12}$ (dashed lines), and second normal stress difference:  $\zeta_2 = (\Sigma_{22} - \Sigma_{33})/\Sigma_{12}$  (solid lines), as functions of the pair association rate  $\beta$  for systems dominated by (a) hydrodynamic forces:  $(\alpha, \chi) = (1, 0)$ and (b) contact forces  $(\alpha, \chi) = (0, 1)$ .

<sup>103</sup> these experimental data, we convert the volumetric moments a into the planar moments <sup>104</sup>  $a^{2D}$  using:

$$a^{\rm 2D} = \frac{a}{a_{11} + a_{22}},\tag{15}$$

<sup>105</sup> and we convert  $\beta$  to  $\phi$  by using the modelled relation between  $\beta$  and  $\zeta_2$  [see Eq. (17) below, <sup>106</sup> assuming  $\chi = 0$ ], and the empirical relation:

$$\zeta_2 = -4\phi^3,\tag{16}$$

<sup>107</sup> which captures the experimental data shown in Fig. 1. In Fig. 2b the resulting modelled <sup>108</sup>  $a^{2D}$  are plotted with the lines. Both experimental data and theory predict that:  $a_{12}^{2D} < 0$ , <sup>109</sup> and:  $a_{22}^{2D} > a_{11}^{2D}$ .

The relative, first and second normal stress differences are obtained by inserting Eq. (13) 111 into Eqs. (10, 12), giving:

$$\begin{pmatrix} \zeta_1 \\ \zeta_2 \end{pmatrix} = \begin{pmatrix} \frac{336(\beta-3)\chi}{\alpha(54\beta^2 - 24\beta + 904) + (63\beta^2 - 120\beta + 452)\chi} \\ -\frac{48\alpha(\beta-3)\beta + (57\beta^2 + 6\beta + 128)\chi}{\alpha(54\beta^2 - 24\beta + 904) + (63\beta^2 - 120\beta + 452)\chi} \end{pmatrix},$$
(17)

<sup>112</sup> which are plotted as functions of  $\beta$  for contactless systems:  $(\alpha, \chi) = (1, 0)$ , in Fig. 3a, and <sup>113</sup> for contact dominated systems:  $(\alpha, \chi) = (0, 1)$ , in Fig. 3b. For systems without contact <sup>114</sup> forces, we find:  $\zeta_1 = 0$  and  $\zeta_2 < 0$  (Fig. 3a), and for systems with contact forces we find: <sup>115</sup>  $\zeta_1 > 0$  and  $\zeta_2 < 0$  (Fig. 3b). These results may rationalise the data shown in Fig. 1, <sup>116</sup> and suggest, that the stress in Refs. [5–7] is dominated by hydrodynamic forces, while <sup>117</sup> that in Refs. [2, 4] is dominated by contact forces. It is further noted that, the positive <sup>118</sup> effect of contact forces on the first normal stress difference in shear invariant suspensions, is <sup>119</sup> supported by two dimensional Stokesian Dynamics simulations in Ref. [23].

The transition from negligible to significantly positive  $\zeta_1$  is explained as follows. The hydrodynamic part of the particle stress [Eq. (12)] produces a first normal stress difference:

$$\Sigma_{11} - \Sigma_{22} = \alpha \eta \dot{\gamma} \left( \langle p_1 p_1 p_1 p_2 \rangle - \langle p_2 p_2 p_2 p_1 \rangle \right), \tag{18a}$$

<sup>122</sup> which is quadratic in the microstructure anisotropy, and is therefore ignored by the linear <sup>123</sup> closure [Eq. (10)], which predicts  $\zeta_1 = 0$ . The contact part of the particle stress, on the <sup>124</sup> other hand, produces a first normal stress difference:

$$\Sigma_{11} - \Sigma_{22} = \chi \eta \dot{\gamma} \Big[ \frac{1}{2} \langle p_1 p_1 p_1 p_2 \rangle - \frac{1}{2} \langle p_2 p_2 p_2 p_1 \rangle + \frac{1}{4} \langle p_2 p_2 p_2 p_2 \rangle - \frac{1}{4} \langle p_1 p_1 p_1 p_1 \rangle \Big],$$
(18b)

<sup>125</sup> which is first order in the microstructure anisotropy, and according to the linear closure [Eq. <sup>126</sup> (10)]:

$$\Sigma_{11} - \Sigma_{22} = \frac{3}{14} \chi \eta \dot{\gamma} \left( a_{22} - a_{11} \right), \qquad (18c)$$

which is positive, since  $a_{22} > a_{11}$ ; see Fig. 2b.

#### 128 B. Shear Reversal

Finally we consider the case of shear reversal. We use the Euler forward integration scheme with a time step of:  $\Delta t = 0.01/\dot{\gamma}$ , to compute the time dependent microstructure and stress after shear reversal, using various values for  $\beta$ ,  $\alpha$  and  $\chi$ . In the computation, the initially isotropic suspension:  $\boldsymbol{a} = \boldsymbol{\delta}/3$ , is sheared until a steady state is reached, after which the flow direction is reversed from negative to positive, at which instant we define:  $t_{134} t = 0$ . The reversal induces a reorganisation of the microstructure and the attainment of a mess steady state.



FIG. 4. (a) Modelled [Eqs. (9, 10, 12)] suspension shear stress, scaled with the steady value:  $\Sigma_{12}/\Sigma_{12,\infty}$ , as a function of the strain  $\dot{\gamma}t$ , after shear reversal, using various  $\beta, \alpha, \chi$ . (b) Measured  $\Sigma_{12}/\Sigma_{12,\infty}$  as a function of  $\dot{\gamma}t$ , after shear reversal, using various volume fractions  $\phi$  [12].

The modelled shear stress, scaled with the steady value  $\Sigma_{12}/\Sigma_{12,\infty}$  is plotted as a function <sup>137</sup> of the strain  $\dot{\gamma}t$  in Fig. 4a. As expected, the stress in the contactless theory:  $(\alpha, \chi) = (1, 0)$ , <sup>138</sup> is conserved, upon shear reversal, followed by a decrease and subsequent recovery to the <sup>139</sup> steady value. In the contact dominated theory:  $(\alpha, \chi) = (0, 1)$ , on the other hand, the shear <sup>140</sup> stress is not conserved upon shear reversal, i.e., there is a discontinuous drop, followed by <sup>141</sup> a recovery, in qualitative agreement with experimental data from literature [12], which are <sup>142</sup> plotted in Fig. 4b. The qualitatively correct prediction of the stress discontinuity, which <sup>143</sup> is related to the contact forces, further validates the physical significance of the proposed <sup>144</sup> constitutive equations [Eqs. (9, 10, 12)].

#### 145 IV. CONCLUSION

We propose a tensorial theory for suspension microstructure and stress, that includes hydrodynamic and hard sphere interaction forces.

The theory assumes hard and frictionless contact forces, which is a reasonable assumption for shear rate invariant suspensions, but may not be valid for shear thickening suspensions. The theory furthermore assumes a linear relationship between the stress and the microstruc<sup>151</sup> ture anisotropy [Eq. (10)], which is supported by experimental data in the literature [22], <sup>152</sup> as illustrated in Fig. 2b.

The theory predicts that hydrodynamic forces produce a negligible first normal stress difference  $\zeta_1$ , while contact forces produce a positive  $\zeta_1$ . These results may provide a rationale for seemingly contradicting experimental observations in the literature, as illustrated in Fig. 156 1.

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<sup>209</sup> Fluid Mech. **814**, 614–617 (2017).