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1 **On the Potential for Interim Storage in Dense Phase CO₂ Pipelines**

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13

14 **ABSTRACT**

15 This paper investigates the flexibility that exists within a dense phase carbon dioxide (CO₂) pipeline
16 network to accommodate upset conditions in the Carbon Capture and Storage (CCS) network,
17 primarily due to flow variations or short term operational issues, by utilising the pipeline as storage
18 vessel whilst still maintaining flow into the pipeline. This process is defined in the pipeline industry as
19 “line-packing” and the time available to undertake line-packing is termed the line-packing time. This
20 study investigates the impact of typical CO₂ pipeline design parameters (diameter, wall thickness and
21 length) as well as CO₂ mass flow rate and pipeline inlet and outlet pressure on the available line-
22 packing time and derives relationships between these variables to provide prediction tools that can be
23 used at the pre-design stage to determine the impact of pipeline design and operation on the line-
24 packing capability. It is shown that the line-packing capacity of the pipeline can be increased by
25 increasing the available internal volume of the pipeline, reducing the mass flow rate into the pipeline,
26 increasing the allowable operating stress and managing the inlet pressure and outlet pressures. This
27 work has indicated that, for pipeline dimensions typical of those considered for CCS schemes, line-
28 packing times of only up to 8 hours can be achieved, therefore the pipeline does not represent a long-
29 term storage option. However, if line-packing capability is considered at the design stage then the
30 level of flexibility for the pipeline to act as short-term storage in the network increases.
31

32 **Keywords:** Carbon Capture and Storage; dense phase pure CO₂; line-packing time, pipeline transport;
33 hydraulic analysis
34

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35 **1 INTRODUCTION**

36 Carbon Capture and Storage (CCS) has drawn significant attention in the last decade as one solution
37 to reduce the emissions of Carbon Dioxide (CO₂) into the atmosphere and decelerate, and potentially
38 reverse, the rate of global warming. This is achieved by capturing CO₂ from large sources such as
39 thermal power plants, refineries and other industrial sites and transporting it, predominantly by
40 pipeline, to geological sites for either permanent storage or for use in Enhanced Oil Recovery (EOR)
41 schemes.

42

43 When designing a CCS network, the capture plants, pipeline system and storage sites are selected to
44 comply with specific site and design constraints. As more zero fuel cost renewable energy becomes
45 available to the electricity grid, CO₂ capture plants at power stations will have to operate flexibly to
46 accommodate variable contribution of renewable energy. Operation of the capture plant could then
47 lead to daily and seasonal variations in CO₂ flow being sent through a CCS network, the pipeline
48 network must be designed to accommodate all these variations in flow. The storage site can impose
49 additional variability and constraints on the pipeline, for example, due to maintenance at the injection
50 point or changes in injection rate (Sanchez Fernandez et al., 2016). Therefore, a pipeline network
51 needs to be able to respond to and accommodate all these kind of transient variations in CO₂ flow.

52

53 A considerable body of published literature is dedicated to the identification of transient operation
54 scenarios that could occur during the operating life of a pipeline in a CCS chain. For example, Wiese
55 *et al.*, 2010, Nimtz *et al.*, 2010; Klinkby *et al.*, 2011, ROAD CCS, 2013 discuss supply and demand
56 fluctuations, start-up and shutdown after a planned outage and start-up after a non-planned
57 (emergency) shutdown. The broad objective of this paper is to investigate the flexibility that exists
58 within the pipeline network to accommodate short-term changes in CO₂ flow, primarily due to flow
59 variations or short term operational issues, through the use of pipeline line-packing.

60

61 The term line-packing is most generally used to describe the storage capabilities of natural gas
62 pipelines during times when the pipeline is temporarily used as a storage vessel. In natural gas
63 transportation, line-packing introduces a degree of operational flexibility and offers some variable
64 capacity during possible upsets and supply variations in a system. During line-packing, the flow of
65 fluid out of the pipeline is stopped by closing (or throttling) a downstream valve whilst still allowing
66 fluid to flow into the pipeline upstream. As a result, the fluid contained in the pipeline is compressed
67 (packed) and the pressure of the contained fluid within the pipeline increases until the downstream
68 valve is opened. The amount of line-packing achievable is limited by the Maximum Allowable

69 Operating Pressure (MAOP) of the pipeline[†]. Currently there is no methodology for assessing the
70 line-packing characteristics of dense phase CO₂ pipelines. Hence, the focus of this work is to
71 determine the relevance of line-packing as a strategy for dense phase CO₂ pipelines and to assess
72 whether the pipeline is effectively able to accommodate variations in the upstream and downstream
73 constraints on the system.

74

75 In this study, the flexibility to line-pack a pipeline is assessed by determining the time available for an
76 operator to store dense phase CO₂ in the pipeline before having to shut down the pipeline and
77 potentially vent CO₂ at the capture plant. This time period is termed the “line-packing time”. Natural
78 gas benefits from a significantly higher compressibility factor; therefore line-packing is an established
79 and proven tool in natural gas applications. The available line-packing time for natural gas pipelines is
80 therefore not an operational concern and the open literature on this subject is mainly devoted to
81 optimisation and management of the line-packing in natural gas pipeline networks (Carter and
82 Rachford, 2003; Krishnaswami *et al.*, 2004; Borraz-Sanchez, 2010; Rios-Mercado and Borraz-
83 Sanchez, 2015). However, in the dense phase, CO₂ has a relatively low compressibility compared
84 with gaseous phase CO₂ or natural gas, and therefore during the line-packing of a dense phase CO₂
85 pipeline, the pressure will more rapidly approach the pipeline’s MAOP and the line-packing time
86 needs to be carefully considered.

87

88 The current study will investigate the impact of pipeline design parameters (diameter, wall thickness
89 and length) as well as CO₂ mass flow rate and pipeline inlet and outlet pressure on the available line-
90 packing time.

91

92 Pipeline transportation systems have traditionally been designed using steady-state analysis, as this
93 was found to be sufficient for the design optimisation of relatively stable supply and demand
94 scenarios. The same philosophy applies to CO₂ pipelines operating in the United States where a
95 relatively constant supply and demand scenario exists (Seevam, 2010). In CCS situations, where
96 sources of CO₂ are predominantly from power plants and industrial sources, the pipelines will have to
97 accommodate a more transient flow of CO₂ which will vary with the power plant load cycle or
98 industrial site operating regime. In order to accommodate both steady-state and transient aspects in the
99 CO₂ pipeline design, the modelling in this study has been conducted in two stages; a steady state
100 pipeline analysis (Section 2) to identify a set of viable pipeline geometries (Section 2.2), followed by
101 a transient analysis (Section 2.3) to study the effects of key input variables on line-packing time. All
102 of the modelling in this paper has been conducted assuming 100% pure CO₂. It has been shown that

[†] The MAOP is the maximum pressure at which a system can be operated continuously under normal conditions at any point along the pipeline PD8010-1 (2015).

103 the addition of common impurities into the CO₂ stream decreases the density of the stream (Wetenhall
104 *et al.*, 2014b) and therefore the pure CO₂ case represents the worst case scenario.

105

106 **2 STEADY STATE PIPELINE DESIGN**

107 Steady state hydraulic modelling is primarily used for facility selection, such as compression or
108 pumping distances and pipeline sizing, and is carried out by analysing flow rates, pressure drops,
109 pipeline capacity and corresponding diameter requirements (Mohitpour *et al.*, 2007). In this work the
110 steady state analysis has been conducted to select a range of pipeline geometries that satisfy the dual
111 criteria of stress based design and single phase hydraulic flow for a realistic range of input conditions.
112 The methodology adopted and the input data assumed is described in detail in the following sections.

113

114 **2.1 Selection of Flow Rate**

115 The baseline mass flow rate for this study is based on capturing 90% of the emissions of a reference
116 emitter. The selected reference emitter, described in detail in (Sanchez Fernandez *et al.*, 2014) is an
117 advanced supercritical pulverized coal (ASC PC) power plant with specific emissions of
118 771.9kg/MWh_{net} and a power output of 819MW gross. A CO₂ capture unit based on mono-ethanol
119 amine (MEA) technology is integrated downstream of the power plant and is designed to capture 90%
120 of the CO₂ present in flue gas. At full power plant load after capture, the CO₂ mass flow into the
121 pipeline is calculated to be 150kg/s. For base load operation, the power plant is designed to operate
122 continuously at full load with only major shut downs for maintenance, which results in 7500 operation
123 hours per year and a CO₂ flow to pipeline of 4 MtCO₂ per year.

124

125 Another important aspect is the minimum CO₂ flow that can be safely sent to the pipeline without
126 shutting down the CO₂ compressor. In the reference emitter selected, the use of integrally geared
127 centrifugal compressors with inlet guide vane systems was considered. The inlet guide vane system
128 (IGV) manipulates the angle between the inlet flow and the compressor impeller and, therefore, the
129 relative speed of the inlet gas. This system is used to control compressor performance when the inlet
130 conditions change. The part load operation of these compressors for CO₂ capture has been studied by
131 Sanchez Fernandez and *et al.* (2016), providing the performance curves for varying input mass flow.
132 The authors concluded that the IGV system can provide a constant discharge pressure of 110 bar for
133 an actual mass flow inlet to the compressor of at least 76% of the design flow. The compressor
134 isothermal efficiency varies between 80% and 77% for this flow range. For this work, three identical
135 compressors with a maximum mass flow capacity of 50kg/s were assumed to work in parallel and the
136 performance curves provided by Sanchez Fernandez *et al.*, (2016) were used to determine discharge
137 conditions for different inlet mass flows. Each compressor has a minimum actual mass flow of 35 kg/s
138 before reaching surge conditions and shut-down.

139

140 The maximum inlet mass flow rate to the pipeline was therefore taken as a uniform flow of 150kg/s
141 (4Mt/year). However, three part-load conditions of 110kg/s (3.47Mt/year), 70kg/s (2.21Mt/year) and
142 35kg/s (1.10Mt/year), were also studied as explained in more detail in Section 3.2.

143

144 **2.2 Selection of Pipeline Dimensions**

145 Pipeline lengths of 50km, 100km and 150km were chosen for the study as these were considered to be
146 relevant lengths for an onshore CCS pipeline network in the UK. No elevation change was considered
147 in the steady state analysis. Five outside diameters (457mm, 508mm, 559mm, 610mm and 914mm)
148 were selected using available pipeline sizes from ISO 4200 (1991), taking due consideration of the
149 pipeline lengths and flow rates that had been chosen for the study. Selecting different diameters
150 allows the impact of oversizing pipelines on network flexibility and line-packing to be studied. The
151 pipeline sizes identified are also within the range of pipeline diameters that have been considered for
152 onshore CO₂ pipelines in the UK (IEAGHG, 2013).

153

154 The required wall thickness for each pipeline diameter was determined using the stress based design
155 criterion outlined in PD8010-1 (2015). In this approach the hoop stress, σ_h (in MPa) is calculated for
156 thin wall pipe using:

$$\sigma_h = \frac{pD_o}{2wt} \leq e \cdot a \cdot \sigma_{SMYS} \quad (1)$$

157

158 where, p is the internal pressure, D_o is the outer diameter (OD), wt is the wall thickness, e is the weld
159 factor (assumed to be 1), a is the design factor and σ_{SMYS} is the Specified Minimum Yield Stress
160 (SMYS) of the pipeline steel in MPa. In this study the design factor was set to be 0.72 (Wetenhall *et*
161 *al.*, 2014a). Consequently, the maximum stress in the pipeline was limited to 72% SMYS. The
162 material of construction of the pipeline has been assumed as EN ISO 3183 (2012) L450 carbon steel,
163 having an SMYS of 450MPa.

164

165 An inlet pressure to the pipeline system of 110bara has been selected, which is considered to be
166 appropriate given the scale of distances that could be faced in the UK in future developments of CCS
167 networks and has also been used in similar studies (Sanchez Fernandez *et al*, 2014). Using Equation
168 (1) it is therefore possible to calculate the minimum wall thickness required to satisfy this stress based
169 design condition. Although EN ISO 3183 (2012) does not specify discrete wall thicknesses, the
170 approach that has been adopted here is to select the standardised pipeline sizes specified in ISO 4200
171 (1991). Therefore, once the minimum wall thickness has been calculated, the next available increased
172 wall thickness is chosen. For example, for $D_o = 457\text{mm}$, the minimum wall thickness would be
173 calculated to be 7.76mm, using the data outlined above, and therefore the next standardised pipeline
174 size of 8.0mm was selected.

175

176 Once the wall thickness had been calculated for each of the selected pipeline external diameters, the
177 next pipeline wall thicknesses for the given external diameter were selected from ISO 4200 (1991) for
178 inclusion in the study. The number of additional wall thicknesses selected was dependent on the
179 hydraulic constraints to avoid two phase flow detailed in Section 2.3. Increasing the wall thickness
180 allows the Maximum Allowable Operating Pressure (MAOP), determined through the rearrangement
181 of Equation (1) shown in Equation (2), to be increased and therefore increases the capacity for line-
182 packing in the pipeline. This approach also takes into consideration situations where other design
183 constraints, such as the requirement to prevent ductile fracture propagation (Race *et al.*, 2012), may
184 result in an increase in wall thickness above that required for a stress based design.

$$MAOP(p) = \frac{2wt\sigma_h}{D_0} \quad (2)$$

185 where $\sigma_h = e. a. \sigma_{SMYS}$.

186

187 **2.3 Steady State Hydraulic Modelling Methodology**

188 The hydraulic modelling package PIPESIM (Schlumberger, 2012) was used to conduct the steady
189 state analysis. The numerical procedure employed in PIPESIM is based on the method of finite
190 differences. The modelling approach followed the practice outlined by co-authors in Wetenhall *et al.*
191 (2014a). The calculation of steady state fluid flow in pipelines requires the simultaneous solution of
192 the equations for conservation of mass, momentum and energy. For a given inlet mass flow rate,
193 internal pressure, pipeline length and internal diameter, the outlet pressure was calculated to ensure
194 single phase flow in the pipeline. The single component Equation of State (EOS) due to Span and
195 Wagner (1996) was selected to provide a relationship between the thermodynamic variables of the
196 system (*e.g.* temperature, pressure and volume) and to describe the state of the system under a given
197 set of conditions. The other models that were selected in this study include the Pedersen viscosity
198 model (Pedersen *et al.*, 1984) and the Beggs and Brill flow model with the Moody friction factor as
199 the flow equation (Wetenhall *et al.*, 2014a).

200

201 The conditions that were used in the modelling are listed in Table 1. During the simulation, the
202 pressure and temperature drop along the pipeline were checked to ensure that the outlet pressure in the
203 system remains above the critical pressure of CO₂ (74.1 bara), with a safety margin of 10%, for all of
204 the model cases considered (*i.e.* the outlet pressure, P_o , should remain above 81.5 bara). This
205 condition was set to ensure that two phase flow was not encountered during steady state operation.
206 The resultant pipeline geometries, selected using the stress based design approach outlined in Section
207 2.2 were then checked using the hydraulic design criterion described above *i.e.* if the selected wall

208 thicknesses resulted in an outlet pressure below 81.5 bara, then the external diameter was increased in
209 order to achieve single phase flow for all of the pipeline wall thicknesses considered.

210

211 **2.4 Steady State Analysis Summary**

212 Using the approach outlined in the preceding sections, a set of 75 pipelines were designed with a
213 range of outside diameter, length, wall thickness and flow rate as presented in Table 2. The outlet
214 pressure and MAOP are also shown for each of the pipelines considered in the study to demonstrate
215 the application of the stress based and hydraulic criteria. It is highlighted that the smallest diameter
216 pipeline chosen (457mm) with the largest wall thickness (11mm), just satisfies the hydraulic single
217 phase flow condition ($P_o = 87.0$ bar) for the longest pipeline length and the maximum flow rate and
218 therefore there is little spare capacity in this pipeline.

219

220 **2.5 Effect of Inlet Pressure**

221 In addition to the pipelines designed in Table 2, a further 13 pipelines were included in the
222 investigation, to study the effects of inlet pressure on the line-packing time. For these pipelines
223 (detailed in Table 3), the design criteria were slightly different from those described previously. In
224 order to investigate the effect of varying inlet pressure, the outlet pressure from the pipeline was set at
225 90 bara (pipeline numbers 76-81 in Table 3). The inlet pressure was determined using the hydraulic
226 analysis methodology described in Section 2.3 with a criterion that it must not exceed the MAOP of
227 the pipeline, given by Equation (2).

228

229 **3 LINE-PACKING STUDY**

230 **3.1 Line-packing Methodology**

231 The study of line-packing requires a transient analysis approach in order that the impact of valve
232 closure and the corresponding increase in system pressure with time can be investigated. The transient
233 flow package OLGA (Schlumberger, 2014) was utilised for this study, incorporating the single-
234 component, two-phase (liquid and gas) CO₂ module with the Span and Wagner EOS (de Koeijer et al.,
235 2011; Clausen et al., 2012; Aursand et al., 2013). OLGA is a two-fluid model, as described by
236 Aursand *et al.*, (2013b), which solves the conservation equations for mass, momentum and energy for
237 the gas, liquid droplet and liquid film phases at discrete time and distance intervals. The numerical
238 procedure utilises the finite difference method such that the pipeline is divided into a number of
239 segments and a solution is sought at the centre of each segment.

240

241 At the start of the simulation, steady state flow is established in the pipeline and then the outlet valve
242 is closed. The shutdown time for the valve is assumed to be 5 sec (Nimtz *et al.*, 2010). Once the outlet
243 valve is closed, the internal pressure in the pipeline starts to increase. The simulations were stopped at

244 the time when the internal pressure reached the MAOP for the pipeline. This time is defined as the
245 line-packing time in this paper.

246

247 The calculated line-packing time is dependent on the choice of segmentation length of the pipeline
248 and the numerical time step. In this study, the discretisation of the solution domain has been
249 conducted with a segment length of 1.3m. At this resolution, the sensitivity of the line-packing time
250 to the discretisation length was calculated to be less than 1%. The time step is limited by the Courant-
251 Friedrichs-Lewy (CFL) condition, $C=U\Delta x/\Delta t$, where C is the Courant number, U is flow velocity,
252 Δx is the width of the pipeline segment and Δt is the numerical time step. Courant numbers less
253 than 1 will assure the stability of the numerical solution (Anderson, 1995). For this study, the width of
254 pipeline segment (Δx) is 1.3 m and setting the numerical time step to the order of 0.01s gives
255 Courant numbers ranging from 0.5 to 0.7 for the scenarios studied.

256

257 The input parameters used in the transient analysis are the same as those selected for the steady state
258 analysis, unless otherwise stated, and are presented in Table 1.

259

260 **3.2 Line-packing Results**

261 The results of the line-packing study for every pipeline are presented in Table 2 and Table 3. For the
262 scenarios studied, it can be seen that the line-packing time varies between 127 seconds and 27718
263 seconds (7.7 hours) depending on the combination of pipeline dimensions, flow rate and pressure
264 conditions selected. The next sections discuss the simulation results and draw some intermediate
265 observations regarding options for increasing the line-packing time of a CO₂ pipeline.

266

267 **3.2.1 Impact of Pipeline Characteristics**

268 Fig. 1 shows how line-packing time varies with %SMYS for a given mass flow rate of 150kg/s at a
269 constant inlet pressure of 110 bara. Once the outlet valve is closed, the pressure in the pipeline rises
270 from the initial inlet value of 110 bara and approaches the MAOP (calculated at a design stress of
271 72%SMYS using Equation (2)). Consequently, Pipelines 22, 23 and 24, which have initial operating
272 stresses of 70.6%SMYS, show the shortest line-packing times. As the %SMYS is reduced (for
273 example, by increasing the wall thickness), the line-packing times increase. The increase is not linear
274 due to the concurrent changes in internal diameter and outlet pressure.

275

276 The relationship between pipeline stress and line-packing time for the conditions modelled can be
277 represented by a second order polynomial of the form:

$$t = a(\%SMYS)^2 + b(\%SMYS) + c \quad (3)$$

278 where t is the line-packing time in seconds (s), ($\%SMYS$) is the stress in the pipeline expressed as a
 279 percentage of the materials SMYS and a , b and c are coefficients. This trendline has been fitted to the
 280 data in Fig. 1 and the relevant coefficients are provided in Table 4. As would be expected, the largest
 281 impact on line-packing times is seen for the longest pipelines at the largest diameters and lowest
 282 values of $\%SMYS$, where a decrease in $\%SMYS$ of 8% (from 64% to 56% SMYS) can increase the
 283 line-packing time by 225%.

284 3.2.2 Impact of Mass Flow Rate

285 It would be expected that, as the mass flow rate increases the line-packing time should decrease due to
 286 the increased amount of fluid entering the pipeline. Fig. 2 shows the effect of varying mass flow rate
 287 on line-packing time for fixed pipeline lengths, outer diameters and wall thicknesses at an inlet
 288 pressure of 110 bara. It was found that the relationship between mass flow rate and line-packing time
 289 can be fitted to a relationship of the form:

$$t = y \cdot \dot{m}^{-x} \quad (4)$$

290 where t is the line-packing time in seconds (s), \dot{m} is the mass flow rate in kg/s, and y and x are
 291 coefficients. As shown in Fig. 2, the line-packing time increases with the length of the pipeline and
 292 also with the internal area of the pipeline. Therefore relationships were sought between the internal
 293 volume of the pipeline and the coefficients y and x by using non-linear regression analysis.

$$y = 2x10^{-5}V^2 + 4.3069V + 29426 \quad (5)$$

$$x = 1x10^{-11}V^2 + 6x10^{-7}V + 0.744 \quad (6)$$

294 where V is the internal volume of the pipeline in m^3 . Along with Equation (4) it is therefore possible
 295 to predict the line-packing time for any particular flow rate. The results of the predictions are
 296 presented in Fig. 3. The mean %error on this equation is 9% for the 36 data points in this study.

297

298 It is highlighted that the relationships developed in equations (4) to (6) are only applicable to the
 299 modelled case studies. However, they do illustrate the possibilities for flexible operation of the
 300 pipeline and the timescales available. For example, in the event of an outage at the injection site, such
 301 that the downstream valve had to be closed, such relationships would enable a pipeline operator to
 302 reduce the flow rate to a level related to the expected timescales to resolve the issue. Specifically, for
 303 the 150km pipeline shown in Fig. 3c, reducing the flow rate could achieve a line-pack time of 5 hours
 304 which could mean that the operator can avoid having to shut-in the pipeline.

3.2.3 *Impact of Pressure Management at Boundaries*

The effect of changing the outlet pressure of the pipeline on line-packing time was also investigated. The results of this analysis are presented in Table 3. If these are combined with other relevant and comparative simulations from Pipelines 10-12 and 19-21, which were all conducted at an inlet pressure of 110 bara, then the line-packing flexibility due to changes in pressure management can be studied. To illustrate this effect, the results for a 457mm OD pipeline with a wall thickness of 11mm, are plotted in Fig. 4 for a range of pipeline lengths, mass flow rates and pressures. From this figure, it can be seen that the biggest effect of changing the pressure at the inlet and outlet is observed at lower flow rates. At the lower flow rates, changing the outlet pressure condition increases the line-packing time by approximately 70% for all pipeline lengths. If a combined strategy of managing the outlet pressure and lowering the flow rate is possible then the line-packing times can be increased by factors of up to five times depending on pipeline length (*i.e.* the relative difference between the shortest and longest line-packing times).

318

319

4 DEVELOPMENT OF AN ARTIFICIAL NEURAL NETWORK FOR LINE-PACKING TIME PREDICTIONS

Although the previous analysis indicates that individual relationships could be identified between key input parameters, the integration of all the significant input parameters could not be achieved using simple regression analysis techniques. In particular, as a result of the methodology adopted for this study, it was not possible to separate out the effects of individual variables *e.g.* changing the wall thickness of the pipeline will change the operating stress (through Equation (1)) but will also change the internal diameter of the pipeline (as the outer diameter remains constant) and therefore the hydraulic characteristics. In order to achieve one of the aims of this work and develop a relationship between the pipeline geometrical and operational characteristics and the line-packing time, an Artificial Neural Network (ANN) has been developed.

331

An ANN is a statistical machine learning methodology that performs multifactorial analysis on a series of inputs to predict an output. ANNs find particular application in the analysis of problems that have a large number of inputs with a complex relationship to each other and the output. As with other artificial learning methodologies, ANNs ‘learn’ to weight the connections between inputs and output by being presented with a training dataset. Once the ANN has been trained and tested, it can be used to predict an output given a set of input data within the range of the training data set. The function of the ANN developed in this work was to predict line-packing time for a CO₂ pipeline, given information about the size and operating conditions of the pipeline.

340

341 The ANN model is constructed from several layers of neurons; an input layer, hidden layer(s) and an
 342 output layer. The number of hidden layers determines the complexity of the network and has a
 343 significant influence on the performance of the network. Every neuron in the layer is connected to
 344 every neuron in the next layer and the inputs are weighted to give precedence to some inputs over
 345 others. The more weight that is given to a particular input the more effect that that input has on the
 346 overall output of the neural network. An activation function is applied to the sum of the weighted
 347 inputs to get the desired output. This architecture is represented schematically in Fig. 5 in which
 348 connections with higher weights are represented with bolder lines. The weights are determined
 349 through training of the network with a proportion of the dataset. The relationship between the inputs,
 350 x_j , where j varies from 1 to N and N is the number of neurons in layer j , and the output, y_p , of an
 351 individual neuron at layer p , where $p=j+1$, can be expressed as:

$$y_p = \varphi \left(\sum_{j=1}^N w_{pj} x_j + b_p \right) \quad (7)$$

352
 353 where w_{pj} is the respective weight of neuron p from neuron j (shown with w in Fig. 5) b_p is the bias
 354 and $\varphi(w,x,b)$ is the activation function. A bias can be applied to the input signal to ensure that the
 355 output from the network represent known trends and experience. For example, for this application, the
 356 line-packing time cannot be negative. It is the matrix of weights and the transfer function for each
 357 layer that determines the relationship between the input vector and output vector.

358 359 **4.1 ANN Development**

360 The results analysis described in Section 3.2 indicate that the relationship between the input
 361 parameters and the line-packing time is non-linear. Therefore, a feed forward, multi-layer network
 362 with one hidden layer was chosen for this application because the architecture of this type of network
 363 allows the non-linearity of the relationship between inputs and outputs to be taken into account. The
 364 log-sigmoid transfer function was selected for the network. The transfer function is applied to the
 365 input data to produce an output result which is similar to the output data produced in the dataset from
 366 the OLGA simulations.

367
 368 The weights and biases in the model were determined iteratively in order to achieve the optimum
 369 performance of the network. Network performance was measured by calculating the mean-squared
 370 error (MSE) and R value of the output predictions. The target was to achieve an MSE close to zero
 371 and an R value close to one to attain the most accurate predictions from the model. Initially random,
 372 arbitrary values were assigned to the weights and biases. These initial values were then updated using
 373 a training algorithm to minimise the MSE and maximise the R value. The training algorithm selected
 374 was the Levenberg-Marquardt (LM) back propagation algorithm. The Bayesian training algorithm

375 was also tested and gave comparable results, however, the LM algorithm was chosen due to its
376 broader acceptance in the literature. The number of neurons in the hidden layer was also determined
377 for each network to give the lowest MSE. The optimum number of neurons in the hidden layer was
378 found to be between 10 and 15, for all networks tested.

379
380 For this work, the neural network toolbox in MATLAB was used to create, train and optimise a
381 customised ANN model. The dataset of 81 pipelines (Table 2 and Table 3) was randomly divided into
382 three subsets; 70% of the data was used for training, 15% for validation, and 15% for testing. To
383 remove the influence of case sequence during the training process, a dividerand function in MATLAB
384 was used to arbitrarily divide the data into the subsets. Two different ANNs were developed with
385 different input data sets. The details of these data sets are provided in Table 5. The motivation behind
386 developing this series of ANNs was to investigate the sensitivity of the network to the type and
387 number of input variables. ANN1 uses all of the available pipeline design data; however, it requires a
388 steady state hydraulic analysis to be conducted for every design combination to determine the outlet
389 pressure and therefore has less practical use. ANN2 was therefore also investigated as it does not
390 require the input of the outlet pressure. The ANN1 and ANN2 networks were selected from 10,000
391 networks of each type (built using the procedure outlined above) based on having the highest R value
392 and lowest MSE for the validation data.

393

394 **4.2 ANN Results**

395 The MSE results from the two ANNs developed are shown in Table 5. From these results it can be
396 seen that ANN1 gives the least error in the predicted line-packing time when compared against the
397 results of the OLGA analysis. However, the analysis indicates that the difference between the MSE
398 results for the two networks is very small in real terms.

399

400 **4.2.1 Sensitivity Analysis**

401 To conduct a sensitivity analysis of the line-packing time to changes in the input variables, noise was
402 added to the input data by adding a random normal distribution using the same seed for each set of
403 input data to ensure that the same set of random numbers were generated. The mean of the input
404 distribution was taken to be 0.2% of the average of each input and the standard deviation was fixed at
405 0.5 to ensure that the input data was still physically coherent *i.e.* no negative pipeline dimensions were
406 generated. The predictions of the ANN models using the ‘noisy’ data as input for all 81 pipelines were
407 compared against those generated using the standard input data. In order to ascertain the importance
408 of the variables, the mean squared errors between the noisy and original predictions were compared
409 for each input variable. These results are shown in Table 6. From Table 6 it can be seen that the wall
410 thickness has the largest effect on line-packing time and that all other inputs have a similar effect.

411 This is because increasing wall thickness increases the MAOP and in turn the line packing time; and
412 also the wall thickness affects the volume of the pipeline.

413

414 **4.2.2 Case Study Scenario**

415 In order to demonstrate the application of the ANN tool for line-packing analysis, a case study
416 scenario is presented. This scenario illustrates the effect of implementing different design and
417 operational strategies on the line-packing time available if a problem were to occur in the network that
418 required the pipeline to be line-packed. The pipeline considered is an 80km, Grade EN10208 L450
419 pipeline with an outside diameter of 914mm. The baseline mass flow rate into the pipeline is 2Mt/year
420 (65kg/s) and the inlet pressure is 110 bar. Consider now a case where the pipeline has been designed
421 with a maximum stress of 70% SMYS by setting the pipeline wall thickness to 16mm. As discussed in
422 this paper, one of the options open to the operator is to reduce the flow rate into the pipeline when the
423 outlet valve is closed.

424

425 ANN2 was chosen to conduct the further analysis in the case study as this network does not require a
426 static hydraulic analysis to be conducted in order to calculate the outlet pressure as an input variable
427 to the network. This makes this ANN more versatile as a preliminary design tool. Table 7 shows the
428 predictions from the ANN for this pipeline for changes in flow rate between -75% and +100% of the
429 baseline flow rate (65kg/s). It can be seen that for this scenario, a line-packing time of between 0.3
430 and 5.4 hours could be achieved in the pipeline through manipulation of the flow rate. However,
431 consider now the case where an operator includes line-packing as a design parameter and increases
432 the wall thickness of the pipeline by 20% to 20mm. The operating stress for this pipeline is
433 56%SMYS. The results in Table 7 show that, at the baseline flow rate, the line-packing time can be
434 doubled by changing the wall thickness by 20%. The difference is even higher at higher flow rates,
435 although not as marked at lower flow rates. Using the ANN as a design tool in this way allows the
436 pipeline operator to make decisions on the benefits of variation in input values.

437

438 Through this case study, it has been shown how an ANN provides a convenient tool for pipeline
439 designers to use when considering the effect of different parameters during the preliminary design
440 phase of a CO₂ pipeline. However, once the design has been finalised, it is always recommended that
441 a full static and transient hydraulic analysis is undertaken using appropriate hydraulic simulation
442 software.

443

444 **5 DISCUSSION AND CONCLUSIONS**

445 One of the main conclusions of this work is that, whilst line-packing time can be increased during
446 operation of the pipeline, through the modification of the mass flow rates and inlet pressures, the
447 ability of the pipeline to be act as a short-term storage option within the network should also be

448 considered at the pipeline design stage. In this paper, it has been demonstrated that, as would be
449 expected, the line-packing capacity of the pipeline can be increased by increasing the available
450 internal volume of the pipeline, reducing the mass flow rate into the pipeline, increasing the allowable
451 operating stress and managing the inlet pressure and outlet pressures. This work has indicated that, for
452 pipeline dimensions typical of those considered for CCS schemes, line-packing times of upto 8 hours
453 would be feasible for dense phase CO₂ pipelines. Whilst this could be useful as a short term storage
454 option, which may allow operational issues elsewhere in the network to be addressed, it will not
455 provide a solution to a major planned or unplanned outage at the capture or injection site. However, it
456 may allow for short-term maintenance activities (*e.g.* at compressor and pump stations) to be
457 undertaken whilst maintaining the output from the capture plant.

458

459 If flexibility of the pipeline system is considered at the design phase then the capacity for line-packing
460 could be increased. This work has demonstrated that the variable that has the most impact on the line-
461 packing capacity of a pipeline is the wall thickness. Although increasing the wall thickness reduces
462 the internal volume of the pipeline, for a given fixed outside diameter, the effect that the wall
463 thickness has on increasing the allowable stress in the pipeline outweighs this effect. The selection of
464 wall thickness obviously has to be considered at the design stage and will have a concomitant impact
465 on the cost of the pipeline and the inlet and outlet pressure. In pipeline design, the wall thickness is
466 generally selected to satisfy stress based design criterion, although for CO₂ pipelines containing
467 impurities, in particular, it has been shown that increasing the wall thickness of the pipe is a key factor
468 in controlling fracture propagation (Race *et al.*, 2012). This work has shown that the effect of line-
469 packing should also be considered at the design stage if the flexibility of the network is a key
470 consideration.

471

472 It has been shown that the relationships between the key variables in determining the line-packing
473 time are inter-related and non-linear. Consequently, it was found that the most appropriate method for
474 investigating the effects of input variables on the line-packing time was through a multi-variate
475 analysis or machine learning methodology, such as ANNs. Through this work, it has been
476 demonstrated that an ANN can be used to develop a tool for evaluation of the available options for
477 increasing the line-packing times for a CO₂ pipeline. However, as with all statistical analytical tools,
478 the ANN can only be used within the bounds of the data on which it has been trained. Therefore, the
479 tool is only applicable for pipelines carrying pure CO₂ on flat terrain and within the data limits for the
480 variables shown in Table 8.

481

482 The dataset developed for this work has been derived through a detailed process of static and
483 hydraulic analysis to ensure that constraints on stress design and hydraulic performance are
484 maintained. However, it is recommended that when utilising this method for calculating line-packing

485 times, a static analysis is conducted to ensure that the stress based and hydraulic design criteria are
486 both met for the pipeline input conditions selected.

487

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496

497 **REFERENCES**

- 498 Anderson, J., 1995. Computational Fluid Dynamics. McGraw-Hill.
- 499 Aursand, E., Aursand, P., Berstad, T., Dorum, C., Hammer, M., Munkejord, S.T. Nordhagen, H.O.,
500 2013a. CO₂ pipeline integrity: A coupled fluid-structure model using a reference equation of
501 state for CO₂. Energy Procedia, 37, 3113-3122.
- 502 Aursand, E., Dorum, C., Hammer, M., Morin, A., Munkejord, S.T. Nordhagen, H.O., 2014. CO₂
503 pipeline Integrity: Comparison of a coupled fluid-structure model and uncoupled two-curve
504 methods. Energy Procedia, 51, 382-391.
- 505 Aursand, P., Hammer, M., Munkejord, S.T. Wilhelmsen, O., 2013b. Pipeline transport of CO₂
506 mixtures: Models for transient simulation. International Journal of Greenhouse Gas Control, 15,
507 174-185.
- 508 EN ISO 3183, Petroleum and natural gas industries - Steel pipe for pipeline transportation systems
509 (ISO 3183:2012) CEN European Committee for Standardisation.
- 510 Carter R.G., Rachford JR H.H., 2003. Optimizing line-pack management to hedge against future load
511 uncertainty. In: Proceedings of the 35th PSIG annual meeting, Bern, Switzerland. paper 0306.
- 512 Clausen, S., Munkejord, S.T., 2012. Depressurization of CO₂ – a Numerical benchmark study. Energy
513 Procedia, 23, 266-273.
- 514 Clausen, S., Oosterkamp, A., Strom, K. L. 2012. Depressurization of a 50km Long 24 inches CO₂
515 pipeline. Energy Procedia, 23, 256-265.
- 516 De Koeijer, G., Borch, H.J., Jakobsen, J., Drescher, M., 2009. Experiments and modeling of two-
517 phase transient flow during CO₂ pipeline depressurization. Energy Procedia, 1, 1683-1689.
- 518 De Koeijer, G., Borch, H.J., Drescher, M., Li, H., Wilhelmsen, Ø., Jakobsen, J., 2011. CO₂ transport-
519 depressurization, heat transfer and impurities. Energy Procedia, 4, 3008-3015.
- 520 Havelrud, M., 2012. Improved and verified models for flow of CO₂ in pipelines. In: The 3rd
521 International Forum on the Transportation of CO₂ by Pipeline, Clarion Technical Conferences,
522 Newcastle, UK.
- 523 IEAGHG, 2013. "UK FEED studies 2011- a summary", 2013/12, October 2013.
- 524 ISO 4200 (1991). Plain end steel tubes, welded and seamless -- General tables of dimensions and
525 masses per unit length, International Standards Organisation, Geneva, Switzerland.

- 526 Klinkby, L., Nielsen, C.M.L., Krogh, E., Smith, I.E., Palm, B. Bernstone, C., 2011. Simulating rapidly
527 fluctuating CO₂ flow into the Vedsted CO₂ pipeline, injection well and reservoir. *Energy*
528 *Procedia*, 4, 4291-4298.
- 529 Krishnaswami, P., Chapman, K.S., Abbaspour, M., 2004. Compressor station optimization for line-
530 pack maintenance. In: *Proceedings of the 36th PSIG annual meeting, Palm Springs.*
- 531 Mohitpour, M., Golshan, H., Murray, A., 2007. *Pipeline Design and Construction*. 3rd Edition, ASME
532 publication, USA.
- 533 Nimtz, M., Klatt, M., Wiese, B., Kuhn, M. Krautz J.H., 2010. Modelling of the CO₂ process- and
534 transport chain in CCS systems—Examination of transport and storage processes. *Chemie der*
535 *Erde - Geochemistry*, 70, 185-192.
- 536 Pedersen, K.S., Fredenslund, A., Christensen, P.L., Thomassen, P., 1984. Viscosity of crude oils.
537 *Chemical Engineering Science* 39, 5.
- 538 PD8010-1, (2015) Code of practice for pipelines, Part 1: Steel pipelines on land. British Standard
539 Institution.
- 540 Race, J.M., Seevam, P., Downie, M.J., 2007. Challenges for offshore transport of anthropogenic
541 carbon dioxide. 26th International Conference on Offshore Mechanics and Arctic Engineering,
542 OMAE2007. San Diego, California, USA: ASME.
- 543 Race, J.M., Wetenhall, B., Seevam, P.N., Downie, M.J., 2012. Towards a CO₂ Pipeline Specification:
544 Defining Tolerance Limits for Impurities. *Journal of Pipeline Engineering* 11 (3), 173.
- 545 Rios-Mercado, R.M., Borrás-Sánchez, C., 2015. Optimization problems in natural gas transportation
546 systems: A state-of-the-art review. *Applied Energy*, 147, 536-555.
- 547 Sanchez Fernandez, E., Goetheer, E.L.V., Manzoloni, G., Machhi, E., Rezvani, S., Vlugt, T.J.H., 2014.
548 Thermodynamic assessment of amine based CO₂ capture technologies in power plants based on
549 European Benchmarking Task Force methodology. *Fuel*, 129, 318-329.
- 550 Sanchez Fernandez, E., Sanchez del Rio, M., Chalmers, H., Khakharia, P., Goetheer, E.L.V., Gibbins,
551 J., and Lucquiaud, M., 2016. Operational flexibility options in power plants with integrated
552 post-combustion capture. *International Journal of Greenhouse Gas Control*, (accepted for
553 publication)
- 554 Sanchez Fernandez, E., Naylor, M., Lucquiaud, M., Wetenhall, B., Aghajani, H., Race, J., Chalmers,
555 H., 2016. Impacts of geological store uncertainties on the design and operation of flexible CCS
556 offshore pipeline infrastructure. *International Journal of Greenhouse Gas Control* 52, 139-154.
- 557 SCHLUMBERGER, 2012. PIPESIM software version, 2012.1.
- 558 SCHLUMBERGER, 2014. OLGAs software version, 7.3.4.
- 559 Seevam, P., 2010. Transporting the next generation of CO₂ for carbon capture and storage. Ph.D.,
560 Newcastle University.
- 561 Seevam, P.N., Race, J.M., Downie, M.J., 2010. Infrastructure and pipeline technology for carbon
562 dioxide (CO₂) transport. Chapter 13 in Maroto-Valer, M. M., 2010. *Developments and*
563 *innovation in carbon dioxide (CO₂) capture and storage technology; Volume 1: Carbon dioxide*
564 *(CO₂) capture, transport and industrial applications. Woodhead Publishing Series in Energy:*
565 *Number 8. Page 422.*
- 566 Span R. & Wagner, W. 1996. A new equation of state for carbon dioxide covering the fluid region
567 from the triple-point temperature to 1100 K at pressures up to 800 MPa. *J. Phys. Chem. Ref.*
568 *Data*, 25, 1509-1596.
- 569 Uilenreef, J. Kombrink, M. 2013. ROAD – Special Report ‘Flow Assurance & Control Philosophy’.

- 570 Wetenhall, B., Race, J.M., Downie, M. 2014a. The effect of CO₂ purity on the development of
571 pipeline networks for carbon capture and storage schemes. *International Journal of Greenhouse*
572 *Gas Control*, 30, p. 197-211.
- 573 Wetenhall, B., Aghajani, H., Chalmers, H., Benson, S.D., Ferrari, M-C., LI, J., Race, J.M., Singh, P.,
574 and Davison, J., 2014b. Impact of impurity on CO₂ compression, liquefaction and
575 transportation. *Energy Procedia*, 63, 2764-2778.
- 576 Wiese, B., Nimtz, M., Klatt, M., Kuhn, M., 2010. Sensitivities of injection rates for single well CO₂
577 injection into saline aquifers. *Chemie der Erde - Geochemistry*, 70, 165-172.
- 578 Zhang, Z.X., Wang, G.X., Massarotto, P. Rudolph, V., 2006. Optimization of pipeline transport for
579 CO₂ sequestration. *Energy Conversion and Management*, 47, 702-715.

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Table 1: Initial conditions considered for the onshore transportation of the dense phase CO₂

| PARAMETER | VALUE | UNIT |
|---|-----------------|---------------------|
| Horizontal Distance | 50, 100 and 150 | km |
| Roughness | 0.0457 | mm |
| Ambient Temperature | 5 | °C |
| Inlet Pressure | 110 | bara |
| Internal Diameter | Table.2 | mm |
| Wall Thickness | Table.2 | mm |
| Inlet Temperature | 30 | °C |
| Burial depth | 1.1 | m |
| Specific heat † | 490 | J/kg-C |
| Steel Heat Transfer Coefficient | 60.55 | W/m ² /K |
| Soil Heat Transfer Coefficient [§] | 2.595 | W/m ² /K |

582

† For carbon steel

§ Assumed to constant over the whole pipeline length

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584

Table 2: Results of steady state and transient analysis for all pipeline designs considered to study the effects of pipeline dimensions and flow rate on line-packing times

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586

| Inlet conditions | | | | | Steady state analysis | | Transient analysis |
|------------------|------------------------------|----------------------------|------------|------------------|------------------------------|-----------------------------------|---------------------|
| | | | | | Stress criterion <72%SMYS | Hydraulic criterion > 81.5bara | |
| Pipeline no. | Outer diameter (D_o) /mm | Wall thickness (wt)/mm | Length /km | Flow rate (kg/s) | %SMYS | Outlet pressure (P_o) /bara | Linepacking time /s |
| 1 | 457 | 8 | 50 | 150 | 69.8 | 105.6 | 135 |
| 2 | 457 | 8 | 100 | 150 | 69.8 | 101.3 | 335 |
| 3 | 457 | 8 | 150 | 150 | 69.8 | 97.5 | 557 |
| 4 | 457 | 8.8 | 50 | 150 | 63.5 | 102.3 | 509 |
| 5 | 457 | 8.8 | 100 | 150 | 63.5 | 95.0 | 1020 |
| 6 | 457 | 8.8 | 150 | 150 | 63.5 | 87.8 | 1559 |
| 7 | 457 | 10 | 50 | 150 | 55.9 | 102.1 | 1007 |
| 8 | 457 | 10 | 100 | 150 | 55.9 | 94.6 | 1938 |
| 9 | 457 | 10 | 150 | 150 | 55.9 | 87.2 | 2853 |
| 10 | 457 | 11 | 50 | 150 | 50.8 | 102.1 | 1402 |
| 11 | 457 | 11 | 100 | 150 | 50.8 | 94.6 | 2665 |
| 12 | 457 | 11 | 150 | 150 | 50.8 | 87.0 | 3885 |
| 13 | 457 | 11 | 50 | 110 | 50.8 | 105.7 | 1604 |
| 14 | 457 | 11 | 100 | 110 | 50.8 | 101.7 | 2976 |
| 15 | 457 | 11 | 150 | 110 | 50.8 | 97.7 | 4320 |
| 16 | 457 | 11 | 50 | 70 | 50.8 | 108.3 | 2314 |
| 17 | 457 | 11 | 100 | 70 | 50.8 | 106.6 | 4155 |
| 18 | 457 | 11 | 150 | 70 | 50.8 | 105.0 | 5887 |
| 19 | 457 | 11 | 50 | 35 | 50.8 | 109.5 | 4320 |
| 20 | 457 | 11 | 100 | 35 | 50.8 | 109.1 | 7294 |
| 21 | 457 | 11 | 150 | 35 | 50.8 | 108.7 | 10522 |

| Inlet conditions | | | | | Steady state analysis | | Transient analysis |
|------------------|------------------------------|----------------------------|------------|------------------|------------------------------|-----------------------------------|---------------------|
| | | | | | Stress criterion <72%SMYS | Hydraulic criterion > 81.5bara | |
| Pipeline no. | Outer diameter (D_o) /mm | Wall thickness (wt)/mm | Length /km | Flow rate (kg/s) | %SMYS | Outlet pressure (P_o) /bara | Linepacking time /s |
| 22 | 508 | 8.8 | 50 | 150 | 70.6 | 105.2 | 127 |
| 23 | 508 | 8.8 | 100 | 150 | 70.6 | 100.9 | 265 |
| 24 | 508 | 8.8 | 150 | 150 | 70.6 | 96.6 | 458 |
| 25 | 508 | 10 | 50 | 150 | 62.1 | 105.2 | 704 |
| 26 | 508 | 10 | 100 | 150 | 62.1 | 100.7 | 1328 |
| 27 | 508 | 10 | 150 | 150 | 62.1 | 96.3 | 1932 |
| 28 | 508 | 11 | 50 | 150 | 56.4 | 105.0 | 1133 |
| 29 | 508 | 11 | 100 | 150 | 56.4 | 100.5 | 2098 |
| 30 | 508 | 11 | 150 | 150 | 56.4 | 95.9 | 3011 |
| 31 | 559 | 10 | 50 | 150 | 68.3 | 107.1 | 314 |
| 32 | 559 | 10 | 100 | 150 | 68.3 | 104.4 | 566 |
| 33 | 559 | 10 | 150 | 150 | 68.3 | 101.7 | 838 |
| 34 | 559 | 11 | 50 | 150 | 62.1 | 107.0 | 812 |
| 35 | 559 | 11 | 100 | 150 | 62.1 | 104.3 | 1491 |
| 36 | 559 | 11 | 150 | 150 | 62.1 | 101.6 | 2128 |
| 37 | 559 | 12.5 | 50 | 150 | 54.7 | 107.0 | 1522 |
| 38 | 559 | 12.5 | 100 | 150 | 54.7 | 104.1 | 2769 |
| 39 | 559 | 12.5 | 150 | 150 | 54.7 | 101.3 | 3900 |
| 40 | 610 | 11 | 50 | 150 | 67.8 | 108.3 | 353 |
| 41 | 610 | 11 | 100 | 150 | 67.8 | 106.6 | 660 |
| 42 | 610 | 11 | 150 | 150 | 67.8 | 105.1 | 964 |
| 43 | 610 | 12.5 | 50 | 150 | 59.6 | 108.2 | 1175 |
| 44 | 610 | 12.5 | 100 | 150 | 59.6 | 106.6 | 2105 |
| 45 | 610 | 12.5 | 150 | 150 | 59.6 | 104.9 | 2942 |
| 46 | 610 | 14.2 | 50 | 150 | 52.5 | 108.2 | 2038 |
| 47 | 610 | 14.2 | 100 | 150 | 52.5 | 106.5 | 3656 |
| 48 | 610 | 14.2 | 150 | 150 | 52.5 | 104.8 | 5099 |

| Inlet conditions | | | | | Steady state analysis | | Transient analysis |
|------------------|------------------------------|----------------------------|------------|------------------|------------------------------|-----------------------------------|---------------------|
| | | | | | Stress criterion <72%SMYS | Hydraulic criterion > 81.5bara | |
| Pipeline no. | Outer diameter (D_o) /mm | Wall thickness (wt)/mm | Length /km | Flow rate (kg/s) | %SMYS | Outlet pressure (P_o) /bara | Linepacking time /s |
| 49 | 610 | 14.2 | 50 | 110 | 52.5 | 109.0 | 2386 |
| 50 | 610 | 14.2 | 100 | 110 | 52.5 | 108.1 | 4280 |
| 51 | 610 | 14.2 | 150 | 110 | 52.5 | 107.2 | 6017 |
| 52 | 610 | 14.2 | 50 | 70 | 52.5 | 109.6 | 3464 |
| 53 | 610 | 14.2 | 100 | 70 | 52.5 | 109.2 | 6147 |
| 54 | 610 | 14.2 | 150 | 70 | 52.5 | 108.8 | 8698 |
| 55 | 610 | 14.2 | 50 | 35 | 52.5 | 109.9 | 6112 |
| 56 | 610 | 14.2 | 100 | 35 | 52.5 | 109.9 | 11091 |
| 57 | 610 | 14.2 | 150 | 35 | 52.5 | 109.7 | 16219 |
| 58 | 914 | 16 | 50 | 150 | 69.8 | 109.8 | 335 |
| 59 | 914 | 16 | 100 | 150 | 69.8 | 109.6 | 600 |
| 60 | 914 | 16 | 150 | 150 | 69.8 | 109.4 | 850 |
| 61 | 914 | 17.5 | 50 | 150 | 63.8 | 109.8 | 1466 |
| 62 | 914 | 17.5 | 100 | 150 | 63.8 | 109.6 | 2548 |
| 63 | 914 | 17.5 | 150 | 150 | 63.8 | 109.4 | 3510 |
| 64 | 914 | 20 | 50 | 150 | 55.9 | 109.8 | 3307 |
| 65 | 914 | 20 | 100 | 150 | 55.9 | 109.6 | 5739 |
| 66 | 914 | 20 | 150 | 150 | 55.9 | 109.3 | 7907 |
| 67 | 914 | 20 | 50 | 110 | 55.9 | 109.9 | 3898 |
| 68 | 914 | 20 | 100 | 110 | 55.9 | 109.8 | 6877 |
| 69 | 914 | 20 | 150 | 110 | 55.9 | 109.6 | 9674 |
| 70 | 914 | 20 | 50 | 70 | 55.9 | 109.9 | 5648 |
| 71 | 914 | 20 | 100 | 70 | 55.9 | 109.9 | 10049 |
| 72 | 914 | 20 | 150 | 70 | 55.9 | 109.8 | 14375 |
| 73 | 914 | 20 | 50 | 35 | 55.9 | 110.0 | 10054 |
| 74 | 914 | 20 | 100 | 35 | 55.9 | 110.0 | 18746 |
| 75 | 914 | 20 | 150 | 35 | 55.9 | 110.0 | 27718 |

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Table 3: Results of steady state and transient analysis for all pipeline designs considered to study the effects of outlet pressure management on line-packing times.

| Inlet conditions | | | | | | Steady state analysis | | Transient analysis |
|------------------|------------------------------|----------------------------|------------|-----------------|---------------------------------|------------------------------|--------------------------------|---------------------|
| | | | | | | Stress criterion <72%SMYS | Hydraulic criterion <MAOP | |
| Pipeline no. | Outer diameter (D_o) /mm | Wall thickness (wt)/mm | Length /km | Flow rate /kg/s | Outlet pressure (P_o) /bara | %SMYS | Inlet pressure (P_o) /bara | Linepacking time /s |
| 76 | 457 | 11 | 50 | 150 | 90 | 45.2 | 97.9 | 1746 |
| 77 | 457 | 11 | 50 | 35 | 90 | 41.8 | 90.5 | 7486 |
| 78 | 457 | 11 | 100 | 150 | 90 | 48.7 | 105.4 | 2736 |
| 79 | 457 | 11 | 100 | 35 | 90 | 42.0 | 90.9 | 12659 |
| 80 | 457 | 11 | 150 | 150 | 90 | 52.3 | 113.3 | 3310 |
| 81 | 457 | 11 | 150 | 35 | 90 | 42.1 | 91.3 | 17640 |

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Table 4: Coefficients for the polynomial trendlines shown in Equation (3) fitted to the data in Fig. 1 to predict the relationship between %SMYS and line-packing time for a pipeline carrying 150kg/s of CO₂ operating at an inlet pressure of 110bara.

| Pipeline length | OD (mm) | Coefficient a | Coefficient b | Coefficient c |
|-----------------|---------|---------------|---------------|---------------|
| 50km | 914 | 2.9868 | 588.19 | 26842 |
| | 610 | 1.2989 | 266.5 | 12449 |
| | 559 | 1.1398 | 229.02 | 10639 |
| | 508 | 0.5131 | 136.07 | 7175.5 |
| | 457 | 0.6544 | 145.32 | 7092.1 |
| 100km | 914 | 5.3364 | 1038.7 | 47106 |
| | 610 | 2.5908 | 507.76 | 23173 |
| | 559 | 1.7264 | 374.43 | 18080 |
| | 508 | 0.7021 | 218.26 | 12174 |
| | 457 | 1.2325 | 270.8 | 13236 |
| 150km | 914 | 7.6304 | 1464.4 | 65896 |
| | 610 | 3.8479 | 733.53 | 33005 |
| | 559 | 2.3038 | 508.51 | 24822 |
| | 508 | 1.1153 | 321.43 | 17591 |
| | 457 | 1.5485 | 360.94 | 18214 |

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Table 5: Input data combinations used for the development of ANN models

| | ANN1 | ANN2 |
|---------------------------|------|------|
| Outer diameter, D_o | x | x |
| Wall thickness, wt | x | x |
| Length, L | x | x |
| Mass flow rate, \dot{m} | x | x |
| Inlet pressure, P_i | x | x |
| Outlet pressure, P_o | x | |
| MSE ($\times 10^{-5}$) | 0.08 | 2.53 |

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Table 6: MSE values from the sensitivity analysis using ANN1 and ANN2 to determine the variables that had the most significant effect on line-packing time

| | ANN1 | ANN2 |
|---------------------------|--------|--------|
| Inlet pressure, P_i | 0.0073 | 0.0022 |
| Mass flow rate, \dot{m} | 0.0004 | 0.0004 |
| Outer diameter, D_o | 0.0002 | 0.0002 |
| Wall thickness, wt | 0.1174 | 0.1185 |
| Length, L | 0.0001 | 0.0001 |
| Outlet pressure, P_o | 0.0002 | |

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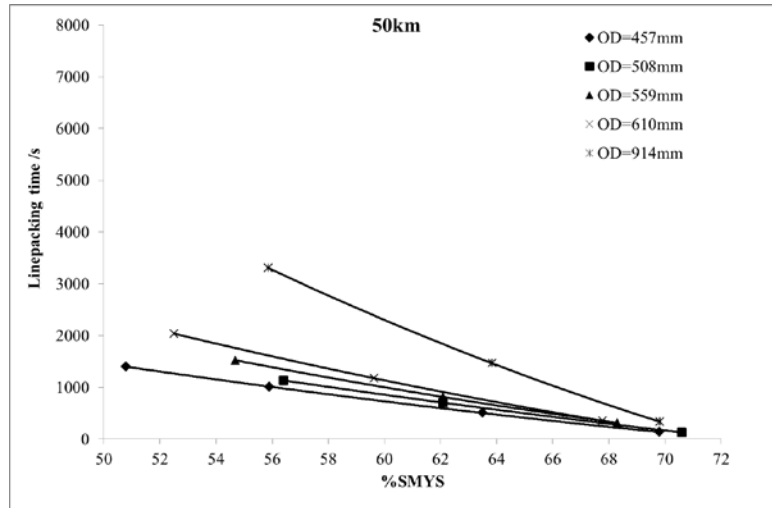
Table 7: Predictions of line-packing time for a case study pipeline (OD = 914mm, Inlet pressure = 110 bar, steel grade = Grade EN10208 L450) using ANN2 at two different wall thicknesses

| Wall thickness = 16mm | | Wall thickness = 20mm | |
|-------------------------|-----------------------------|-------------------------|-----------------------------|
| Mass flow rate (kg/sec) | Estimated line-packing time | Mass flow rate (kg/sec) | Estimated line-packing time |
| 16.25 | 5.37 | 16.25 | 5.99 |
| 32.5 | 3.64 | 32.5 | 4.46 |
| 48.75 | 2.21 | 48.75 | 3.37 |
| 65 | 1.26 | 65 | 2.60 |
| 97.5 | 0.36 | 97.5 | 1.78 |
| 130 | 0.27 | 130 | 1.58 |

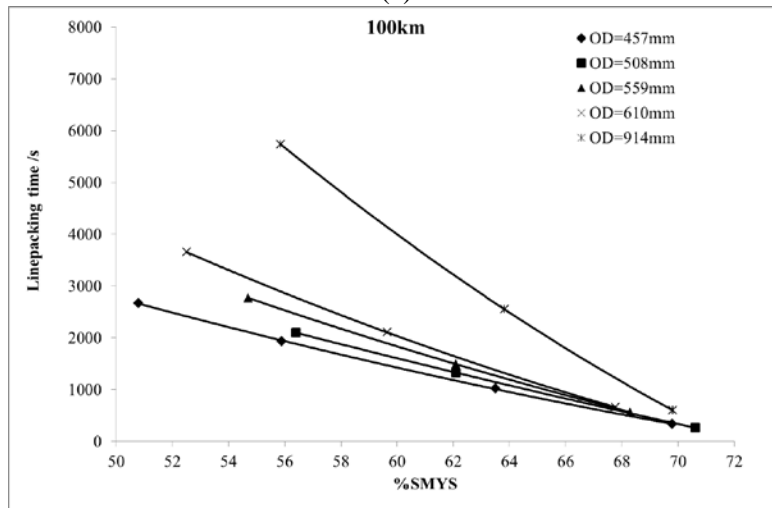
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Table 8: Range of validity of key parameters for the ANN

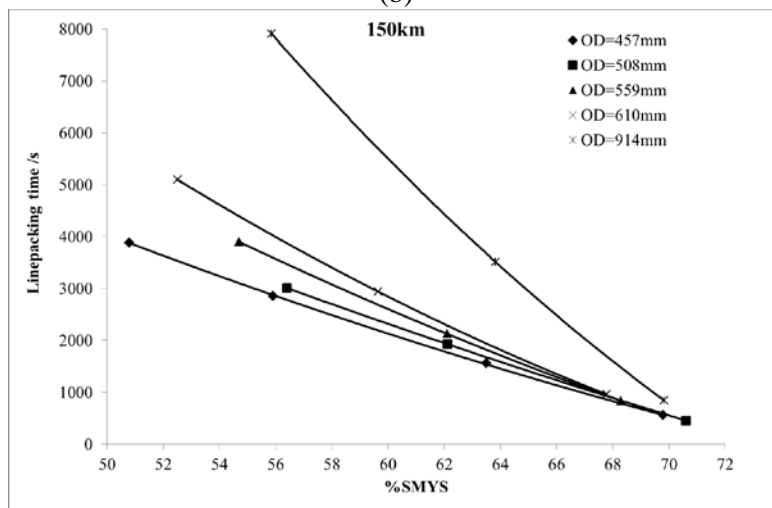
| | Range of validity |
|---------------------------|-------------------|
| Outer diameter, D_o | 457-914mm |
| Wall thickness, wt | 8-20mm |
| SMYS | 50.8-70.6% |
| Length, L | 50-150km |
| Mass flow rate, \dot{m} | 35-150kg/s |
| Inlet pressure, P_i | 90.5-113 bara |
| Inlet pressure, P_o | 87-110 bara |



(a)

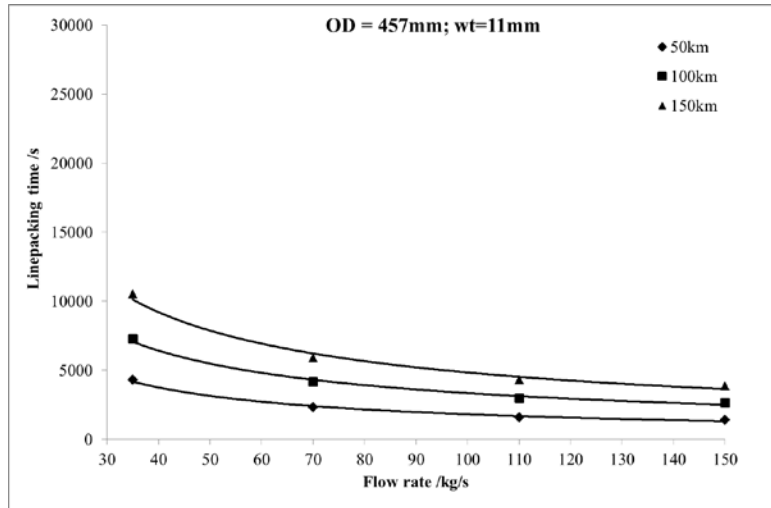


(b)

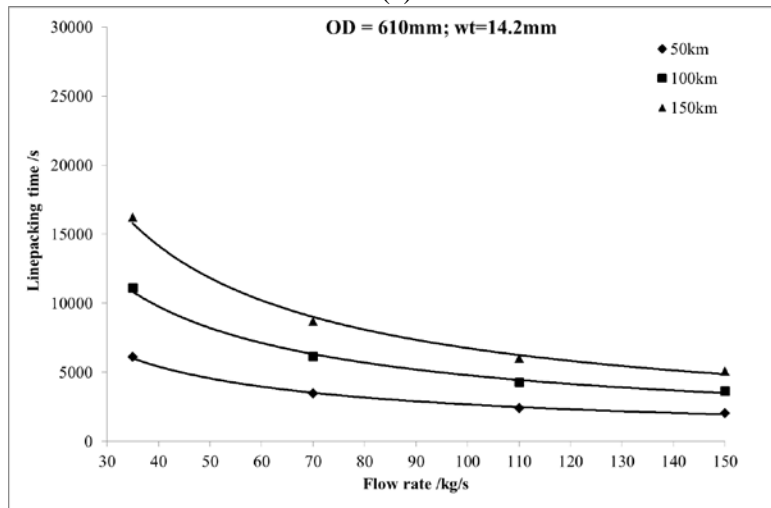


(c)

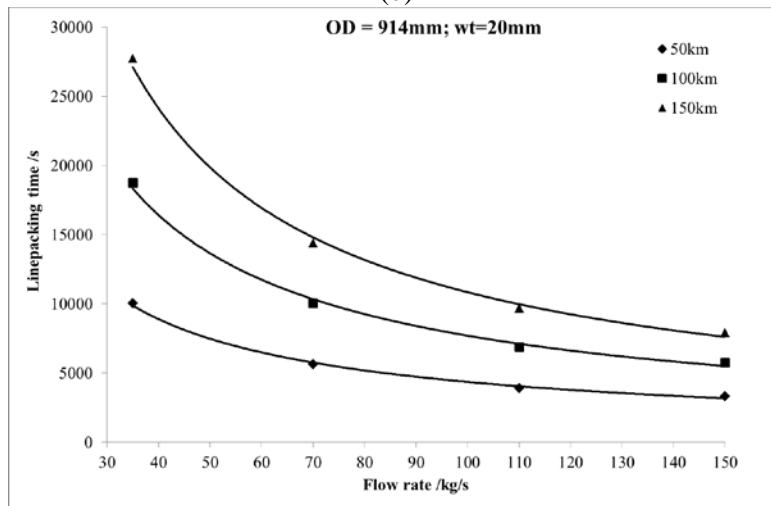
Fig. 1: The effect of stress (%SMYS) on line-packing time for a pipeline carrying 150kg/s of CO₂ operating at an inlet pressure of 110bara with given lengths and outer diameters. (a) Pipeline length = 50km, (b) Pipeline length = 100km, (c) Pipeline length = 150km. A second order polynomial trend line (Equation (3)) has been fitted to the data. The coefficients for the equations are provided in Table 4.



(a)



(b)



(c)

Fig. 2: The effect of flow rate on line-packing time for fixed pipeline lengths, outer diameters and wall thickness operating at a constant inlet pressure of 110 bara. (a) OD = 457mm; wt= 11mm, (b) OD = 610 mm; wt= 14.3mm 100km, (c) = 914 mm; wt= 20mm. A power law trend line (Equation (4)) has been fitted to the data.

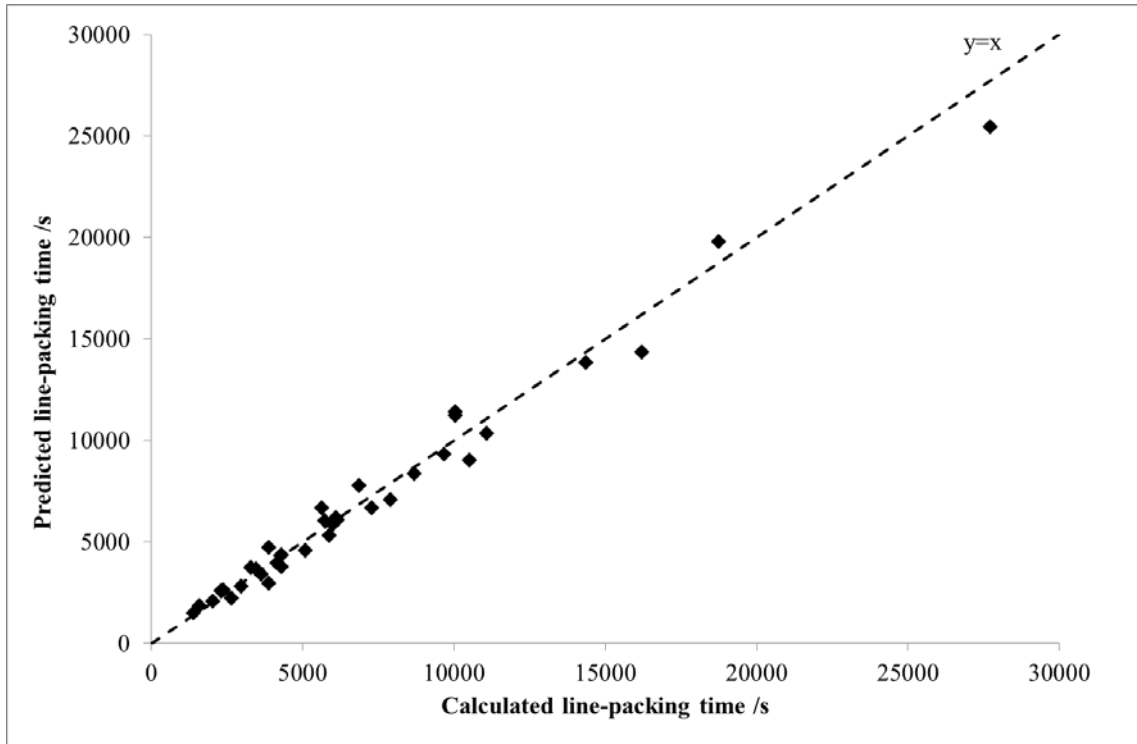


Fig. 3: Relationship between calculated and predicted line-packing times as a function of pipeline internal volume and mass flow rate at a constant inlet pressure of 150bara. The $y=x$ line indicates the position where the calculated and predicted values would be equal.

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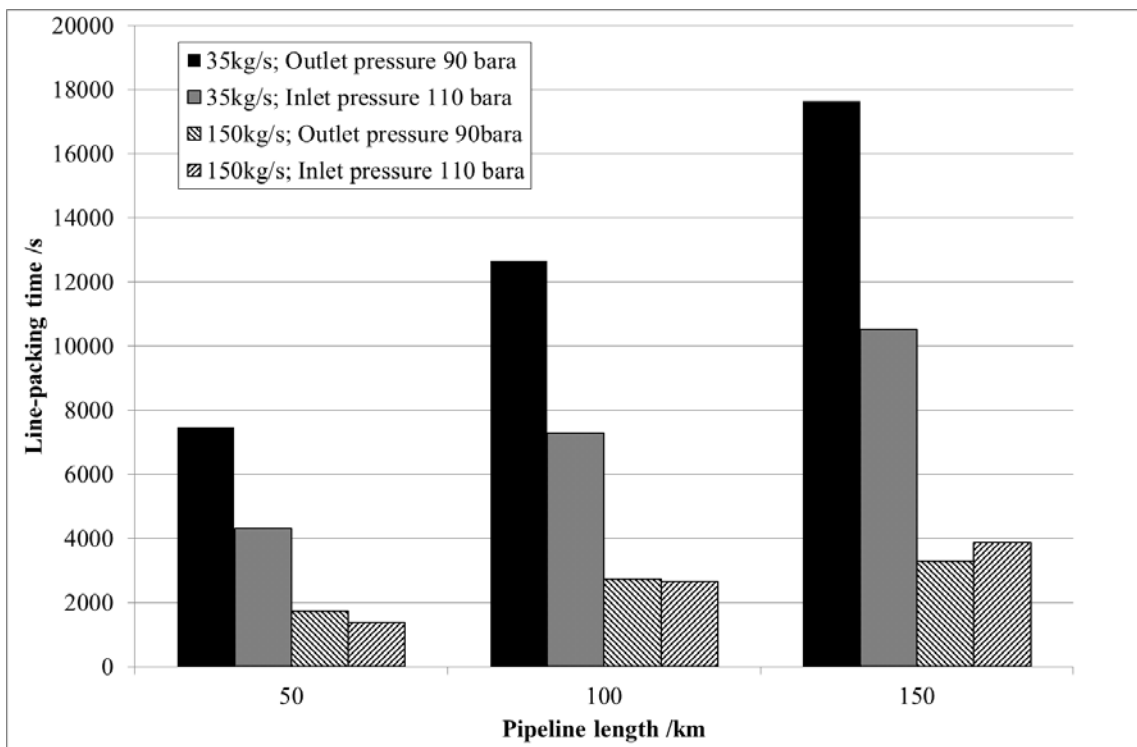


Fig. 4: Effect of changes in flow rate and inlet and outlet pressure management on the line-packing time for a 457mm OD, 11mm wall thickness pipeline.

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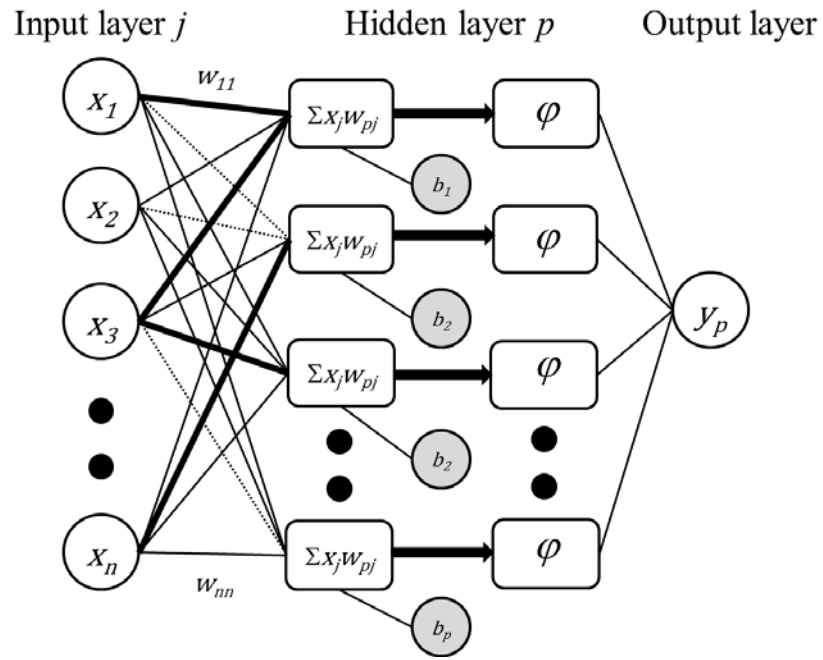


Fig. 5: Schematic of a typical ANN architecture

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