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An actively constrained two degree-of-freedom manipulator for passive deployment applications

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Abstract: A two degree-of-freedom manipulator using actively constrained revolute joints is presented in this paper. Each revolute joint drive mechanism consists of two motor-driven worms coupled to a single worm wheel. A mathematical model of the manipulator system is used in order to develop a computed-torque control algorithm. Experimental results show that the position of the end-effector can be successfully controlled to track a path generated from a user-input force command signal while cancelling backlash at the gear interface. This system has been designed for the purpose of following a predefined path under the direct physical control of the user. The joint mechanism and control strategy used in this paper allow for backlash to be continuously cancelled. The safety of the user is ensured by enabling joint motions only if a user force is applied, and this force is in a direction that allows the controller to restrict joint motion along a predefined path.

Keywords: robotics, computed-torque control, passive deployment

NOTATION

a_n	position of the centre of mass of link n
$\mathbf{B}$	torque conversion matrix
B_{ij}	element ij of $\mathbf{B}$
$\mathbf{C}$	manipulator viscous damping matrix
C_{ij}	element ij of $\mathbf{C}$
$\mathbf{C_r}$	referred manipulator viscous damping matrix from worm mechanisms
$\mathbf{C}_{r11}$	element ij of $\mathbf{C_r}$
F_u	user-input force
$F_\mathrm{u,max}$	user-input force at which the maximum end-effector speed is reached
$F_\mathrm{u,min}$	minimum user-input force allowed to generate motion
$\mathbf{J}$	manipulator arm Jacobian matrix
$\bar{J}_n$	mass moment of inertia about the centre of mass of link n
$J_{\theta n}$	residual inertia of link n
$K_{\mathrm{d}\alpha n}$, $K_{\mathrm{i}\alpha n}$, $K_{\mathrm{p}\alpha n}$	derivative, integral and proportional controller gains for α unwinding of joint n
$K_{\mathrm{d}\beta n}$, $K_{\mathrm{i}\beta n}$, $K_{\mathrm{p}\beta n}$	derivative, integral and proportional controller gains for β unwinding of joint n
l_n	length of link n
m_n	mass of link n
$\mathbf{M}$	manipulator inertia matrix
$\mathbf{M_r}$	referred inertia matrix from worm mechanisms
$\mathbf{M}_\theta$	residual inertia matrix
$\boldsymbol{q}$	vector of joint positions
$\boldsymbol{q}_\mathrm{d}$	vector of desired joint positions
$r_{\alpha n}$	radius of contact of α worm with worm wheel for link n
$r_{\beta n}$	radius of contact of β worm with worm wheel for link n
$r_{\theta n}$	radius of contact of worm wheel with α and β worms for link n
$V_\mathrm{d,max}$	maximum allowable speed of the end-effector
$V_\mathrm{d,min}$	minimum allowable speed of the end-effector
$\mathbf{V_m}$	matrix of Coriolis and centrifugal terms
γ_n	worm lead angle of joint n
ε_n	friction constant for joint n
θ_n	position of joint n
$\theta_{n\mathrm{d}}$	desired position of joint n

$\mu_{\alpha n}, \mu_{d\beta n}$	coefficient of dynamic friction of α and β worms of joint n
$\mu_{s\alpha n}, \mu_{s\beta n}$	coefficient of static friction of α and β worms of joint n
$\boldsymbol{\tau}$	joint actuator torque vector
$\boldsymbol{\tau}_{act}$	vector of joint torques obtained from strain gauges measurements
$\tau_{act,n}$	measured torques of joint n
$\boldsymbol{\tau}_m$	vector of α and β motor torques
$\tau_{m\alpha n}, \tau_{m\beta n}$	α and β motor torques of joint n

1 INTRODUCTION

Development of manipulators that interact closely with humans is increasingly a focus of research in the field of robotics. In particular, robotic devices have been introduced into the operating theatre to help the surgeon perform intricate invasive procedures with a higher degree of accuracy and reliability than would otherwise be possible with more traditional methods.

A wide variety of operations have benefited from the introduction of a robotic assistant. For example, orthopaedic, neurological, cardiac and cosmetic surgical procedures have all been enhanced by robotic assistance. However, many of the robotic devices currently in use are based on modified industrial manipulators which have typically been designed to meet the requirements of high-speed and high-torque applications. The accuracy of such devices is unsurpassed but in such safety-critical environments the use of high-powered mechanisms to perform these tasks has the potential to compromise the safety of patient, surgeon and operating room staff.

Without very expensive and necessary software and hardware safety features, these robots cannot be allowed to operate in safety-critical environments such as an operating theatre. As a result more recent research has focused on designing semi-active and passive devices that are designed to mechanically remove the potential dangers posed by high-powered manipulators. In particular, passive arms with dynamic constraints have been identified as a safe alternative to modified industrial manipulators. Robots such as PADyC (passive arm with dynamic constraints) [1], PTER (passive trajectory enhancing robot) [2] and COBOT (collaborative robot) [3] have been used to demonstrate that computer controlled clutches and brakes can limit the range of motion of the end-effector to remain within a predefined safe working region. Although these devices have proved relatively successful in accomplishing this task, they have had only limited success in constraining the tool (i.e. the end-effector) to remain on a predefined path, as may be required by more complex machining processes such as bone milling and drilling involved in modern surgical procedures.

In this paper, a two degree-of-freedom (2 DOF) revolute joint planar manipulator is presented that is suitable for use in applications where direct control of the end-effector by a human is required. The robot is developed based on the mechanical and control principles given in a previous paper [4]. A conceptual overview of the system is given and a mathematical model of the 2 DOF system is used to develop a computed-torque control algorithm for trajectory tracking. A novel pointer mechanism used in conjunction with the control strategies to help the user to complete the desired task is presented. A description of the experimental set-up and control system architecture is also included. Experimental results using the computed-torque control algorithm to guide the end-effector around a circular path under direct user control are presented, followed by concluding remarks.

2 CONCEPTUAL OVERVIEW

2.1 Dual-worm driven joint

Each joint, shown schematically in Fig. 1, consists of two worms, each driven by a low-power d.c. servomotor. The two worms follow a worm wheel that is fixed to the robot link. The inability of the worm mechanism to be back-driven means that no matter how much force the operator applies, motion is not allowed until both motors are controlled to move in a positive or negative direction. The dual-worm mechanism also has the ability to eliminate backlash using a control strategy that is discussed below.

The parallel nature of this overactuated joint structure makes this active system safer than modified industrial robots. Should there be a failure, both motors would have to drive in opposite directions to achieve motion.
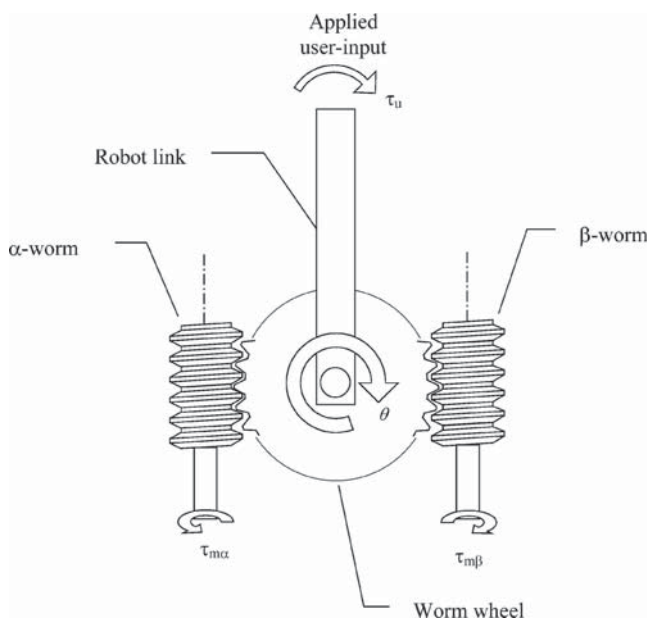

Fig. 1 Single DOF dual-worm drive joint

However, if this failure occurs, speed of the joints would be very small, giving the operator plenty of time to react, i.e. to release the dead-man's-handle. The motors were selected to give a peak joint speed of 2 r/min.

2.2 Two DOF manipulator

The proposed system is intended for medical/surgical applications. Many surgical assistant systems use computer graphics to convey information to the surgeon. However, this can be distracting because of the need, for the surgeon, to be able to focus not only on the task in hand but also on the monitor that is giving directions. The use of a graphical interface in this manner requires the surgeon to transform the directions from the information provided on the screen to the real-world robot. This may not always be easy or possible and would increase the time required to complete a task, contradicting the purpose of developing the manipulator in the first instance. Extensive training of the user in the operation of the system may also be required. The use of revolute joint architectures for passive deployment applications has been shown to be a significant problem. In particular, the PADyC system requires the user to 'feel' his or her way along a narrow corridor that describes the actual desired path. Sudden changes in the system dynamics caused by the clutches have to be compensated for by the user. This may require training of the user in the use of the system. Revolute joint devices such as PADyC, which rely on the user 'feeling' his or her way by applying a force in an appropriate direction, have clearly identified that it is not sufficient to simply show the user what the task is by, for example, displaying the path on a computer screen. It should be noted that

motion of the end-effector along a particular path is possible only if the torque applied on each joint, as a result of the user force, produces motion (i.e. rotation) of the corresponding joint in the desired direction. This is explained below using a random kinematic configuration of a 2 DOF system.

Consider a 2 DOF revolute joint manipulator as shown in Fig. 2a that is required to follow a predefined path, $P(x, y)$. For every combination of joint positions the workspace of the manipulator can be divided into four regions based on the sign of the motion of each joint, as shown in Table 1. Similarly, for every combination of joint positions the workspace at the end-effector can also be divided into four regions based on the signs of the joints torques caused by the application of the user force, as shown in Fig. 2b and Table 2.

Table 1 Joint motion of a 2 DOF revolute joint manipulator

Region	Motion of joint 1	Motion of joint 2
①	+	+
②	+	−
③	−	−
④	−	+

Table 2 Joint torques caused by applied user input force

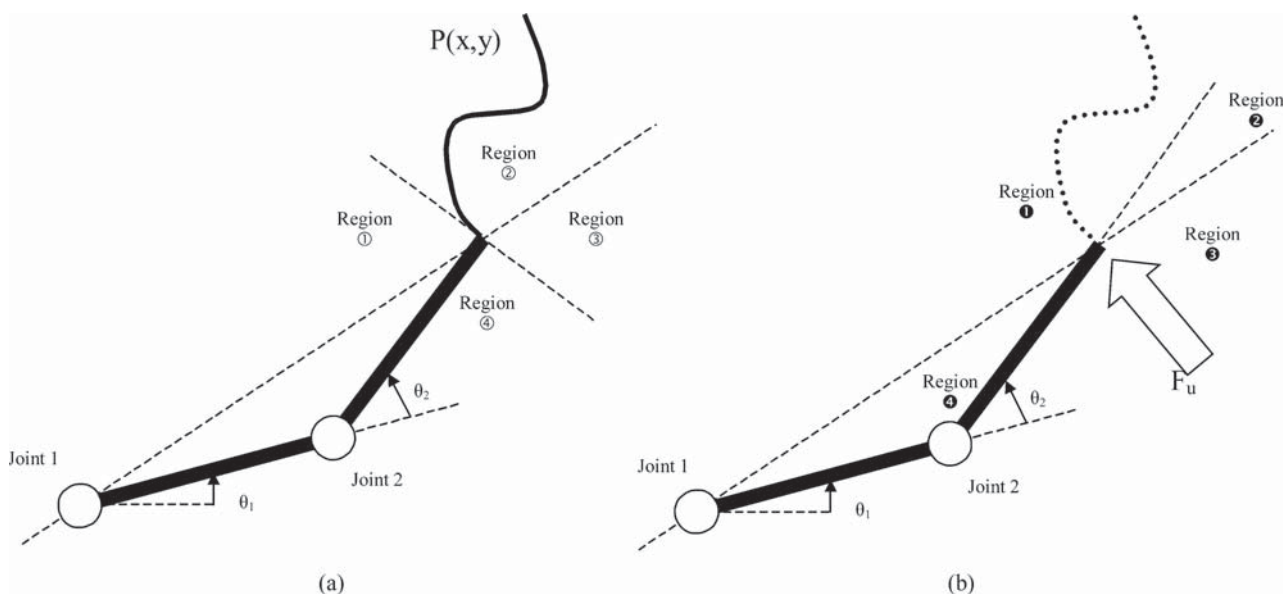| Region	Torque at joint 1	Torque at joint 2
❶	+	+
❷	+	−
❸	−	−
❹	−	+

Fig. 2 Joint motion of a 2 DOF revolute joint manipulator

To ensure that the passive robot can move along a desired path it is necessary that the sign of the motion of the joint and the sign of the torque caused by the applied user force are in the same direction. It can be seen from Tables 1 and 2 that motion is possible in region ①, which requires positive directions of θ_1 and θ_2, only if the user force is directed into region ❶; i.e. only if the user force produces positive torques at joints 1 and 2. Similarly, motion is possible into region ② only when the user force is in region ❷, etc.

It can clearly be seen that if the user tries to apply a force along the desired path, which is intuitive, then no motion of the end-effector may result. For example, for the kinematic configuration shown in Fig. 2b, if the path of the end-effector has a direction within region ② and the direction of the user force is in region ❶ along the path, the torques produced at the joints would result in a different direction of the end-effector. Therefore, no motion is allowed by the control system until the direction of the user force is changed to be into region ❷, which is much smaller than region ❶ in this particular configuration. Hence, if the user is shown the desired path on a computer screen, which is the case with PADyC, then the initial action of the user is to push in the shown direction of the path and then 'feel' his or her way, by changing the direction of the force, until motion is allowed by the control system. For this, the user must have significant experience to operate the robot effectively.

Subsequently, in this work a control handle that incorporates a pointer mechanism has been implemented. It is used in combination with the control strategy described below to inform the user about the direction

that the user-input force needs to be applied in order for the controller to allow motion. Figure 3 shows a schematic diagram of the control handle. The handle/pointer mechanism consists of four main components: a base unit, a motor encoder, a direction indicator and a rotating grip. The direction indicator is mounted inside the control handle.

Mounting the indicator at a similar position to the robot end-effector has two distinct advantages. Firstly, the surgeon does not have to perform a coordinate transformation from the direction indicator to the real world. Secondly, the indicator is mounted in such a position that it can always be seen without large head movements and large changes of visual focus, thus reducing the need for training in the use of the manipulator and the speed at which tasks can be performed.

3 CONTROL STRATEGY

The aim of the control algorithm is to make the manipulator end-effector track a desired position command when the user-input force is in a specified direction (as described above) while cancelling backlash at the worm interface. In order to ensure operator safety the robot and control method *must not* exhibit:

(a) any motion against the user;
(b) any motion without direct control from the user;
(c) any backlash at either of the worm/wheel interfaces.

In order to ensure that the robot does not exhibit motion against the user, the robot should only be allowed to move if the torque acting on each joint is in the same direction as the desired motion of that joint. These conditions can be expressed mathematically by equation (1), and for safety reasons they must both be satisfied in order for the robot to be able to move. To implement this strategy successfully, strain gauges have been mounted on each link in order to measure the link torques, τ_{act1} and τ_{act2}:

$$\text{sgn}(\dot{\theta}_{1d}) = \text{sgn}(\tau_{\text{act1}}) \quad \text{and} \quad \text{sgn}(\dot{\theta}_{2d}) = \text{sgn}(\tau_{\text{act2}})$$

(1)

As with similar devices, PADyC and PTER, the user controls motion of the end-effector along the path by grasping the control handle and applying a force. Only if the conditions of equation (1) are satisfied is the robot end-effector allowed to move along the path. The desired velocity of the end-effector, V_d, is in a direction determined by the path at a magnitude derived from the magnitude of the user input force and the relationship given by

$$|V_d| = \begin{cases} 0 & \forall |\mathbf{F}_u| < F_{u,\text{min}} \\ V_{d,\text{max}}\left[-\dfrac{2(|\mathbf{F}_u| - F_{u,\text{min}})^3}{(F_{u,\text{max}} - F_{u,\text{min}})^3} + \dfrac{3(|\mathbf{F}_u| - F_{u,\text{min}})^2}{(F_{u,\text{max}} - F_{u,\text{min}})^2} \right] & \forall F_{u,\text{min}} \leqslant |\mathbf{F}_u| \leqslant F_{u,\text{max}} \\ V_{d,\text{max}} & \forall |\mathbf{F}_u| > F_{u,\text{max}} \end{cases}$$

(2)

where $V_{d,\text{max}}$, $F_{u,\text{min}}$ and $F_{u,\text{max}}$ are all positive constants. This velocity-limiting algorithm is also shown graphically in Fig. 4 and the relationship ensures that velocity and acceleration demands for the end-effector are continuous.

In order to cancel backlash in the worm mechanism the following control strategy is used [4]. For motion of a joint in the clockwise direction (i.e. $\dot{\theta} > 0$) the motor command voltage for the α worm is set to a constant value and the β motor torque, $\tau_{\text{m}\beta n}$, is controlled to unwind the β worm to track the trajectory. In this condition the β worm *leads* and the α worm is used to *follow* the motion of the worm wheel without applying unnecessary frictional forces to the system. Control of motion in this manner shall be termed β-unwinding control. However, in the anticlockwise direction (i.e. $\dot{\theta} < 0$) the β worm motor command voltage is set to a constant value and $\tau_{\text{m}\alpha n}$ is used to control the unwinding of the α worm to track
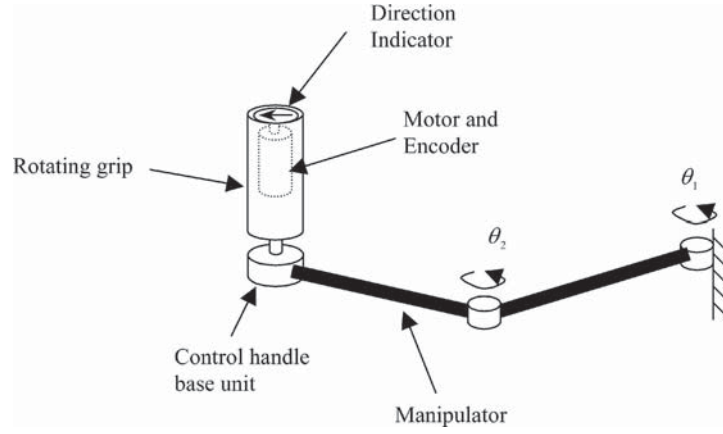
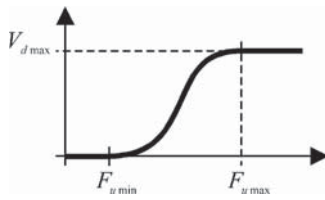**Fig. 3** Manipulator control handle schematic

Fig. 4 User-defined velocity demand

the trajectory. This is termed α-unwinding control. In this instance, the α worm *leads* and the β worm *follows*.

4 MATHEMATICAL MODEL

Consider the 2 DOF revolute joint, planar manipulator shown in Fig. 5. In general the dynamic equation for the system is given by the following equation, where τ is a vector of joint torque provided by the joint actuators, F_u is the user-input force and $\mathbf{J}$ is the arm Jacobian matrix for the manipulator:

$$\mathbf{M}\ddot{q} + \mathbf{C}\dot{q} + \mathbf{V}_m\dot{q} = \tau + \mathbf{J}^T F_u \quad (3)$$

where

$$\mathbf{M} = \begin{bmatrix} a_1^2 m_1 + [l_1^2 + a_2^2 + 2l_1 a_2 \cos(\theta_2)]m_2 + \bar{J}_1 + \bar{J}_2 & [a_2^2 + l_1 a_2 \cos(\theta_2)]m_2 + \bar{J}_2 \\ (a_2^2 + l_1 a_2 \cos(\theta_2))m_2 + \bar{J}_2 & a_2^2 m_2 + \bar{J}_2 \end{bmatrix}$$

$$q = \begin{bmatrix} \theta_1 \\ \theta_2 \end{bmatrix}, \qquad \mathbf{C} = \begin{bmatrix} C_{11} & 0 \\ 0 & C_{22} \end{bmatrix}$$

$$\mathbf{V}_m\dot{q} = \begin{bmatrix} V_{m1} \\ V_{m2} \end{bmatrix} = \begin{bmatrix} -m_2 l_1 a_2 \sin(\theta_2)(2\dot{\theta}_1\dot{\theta}_2 + \dot{\theta}_2^2) \\ m_2 l_1 a_2 \sin(\theta_2)\dot{\theta}_1^2 \end{bmatrix}$$

$$\mathbf{J} = \begin{bmatrix} -l_1 \sin(\theta_1) - l_2 \sin(\theta_1 + \theta_2) & -l_2 \sin(\theta_1 + \theta_2) \\ l_1 \cos(\theta_1) + l_2 \cos(\theta_1 + \theta_2) & l_2 \cos(\theta_1 + \theta_2) \end{bmatrix}$$

$$\tau = \begin{bmatrix} \tau_1 \\ \tau_2 \end{bmatrix} \quad \text{and} \quad F_n = \begin{bmatrix} F_{nx} \\ F_{uy} \end{bmatrix}$$

From the model of the worm-driven joint [4], the torque provided by the mechanisms for joints 1 and 2 can be written respectively

$$\tau_1 = \frac{\tau_{m\alpha1} r_{\theta1}}{r_{\alpha1} \Delta_{\alpha1}} + \frac{\tau_{m\beta1} r_{\theta1}}{r_{\beta1} \Delta_{\beta1}} - J_{\alpha1} \frac{\ddot{\theta}_1 r_{\theta1}^2}{\Delta_{\alpha1} r_{\alpha1}^2 \tan \gamma_1}$$

$$- J_{\beta1} \frac{\ddot{\theta}_1 r_{\theta1}^2}{\Delta_{\beta1} r_{\beta1}^2 \tan \gamma_1} - C_{f\alpha1} \frac{\dot{\theta}_1 r_{\theta1}^2}{\Delta_{\alpha1} r_{\alpha1}^2 \tan \gamma_1}$$

$$- C_{f\beta1} \frac{\dot{\theta}_1 r_{\theta1}^2}{\Delta_{\beta1} r_{\beta1}^2 \tan \gamma_1} \quad (4)$$

$$\tau_2 = \frac{\tau_{m\alpha2} r_{\theta2}}{r_{\alpha2} \Delta_{\alpha2}} + \frac{\tau_{m\beta2} r_{\theta2}}{r_{\beta2} \Delta_{\beta2}} - J_{\alpha2} \frac{\ddot{\theta}_2 r_{\theta2}^2}{\Delta_{\alpha2} r_{\alpha2}^2 \tan \gamma_2}$$

$$- J_{\beta2} \frac{\ddot{\theta}_2 r_{\theta2}^2}{\Delta_{\beta2} r_{\beta2}^2 \tan \gamma_2} - C_{f\alpha2} \frac{\dot{\theta}_2 r_{\theta2}^2}{\Delta_{\alpha2} r_{\alpha2}^2 \tan \gamma_2}$$

$$- C_{f\beta2} \frac{\dot{\theta}_2 r_{\theta2}^2}{\Delta_{\beta2} r_{\beta2}^2 \tan \gamma_2} \quad (5)$$

The terms $\Delta_{\alpha n}$ and $\Delta_{\beta n}$ in equations (4) and (5) represent the torque transfer ratio between the worms and the worm

wheel of joint n. These terms include an exponential stick-slip model of friction in the dual worm mechanism of joint n, as shown by

$$\Delta_{\alpha n} = \frac{\sin \gamma_n + [\mu_{d\alpha n} + (\mu_{s\alpha n} - \mu_{d\alpha n}) e^{|\dot{\theta}_n|/(\varepsilon_n \sin \gamma_n)}] \operatorname{sgn}(\dot{\theta}_n) \cos \gamma_n}{\cos \gamma_n - [\mu_{d\alpha n} + (\mu_{s\alpha n} - \mu_{d\alpha n}) e^{|\dot{\theta}_n|/(\varepsilon_n \sin \gamma_n)}] \operatorname{sgn}(\dot{\theta}_n) \sin \gamma_n}$$

$$(6)$$

$$\Delta_{\beta n} = \frac{\sin \gamma_n - [\mu_{d\beta n} + (\mu_{s\beta n} - \mu_{d\beta n}) e^{|\dot{\theta}_n|/(\varepsilon_n \sin \gamma_n)}] \operatorname{sgn}(\dot{\theta}_n) \cos \gamma_n}{\cos \gamma_n + [\mu_{d\beta n} + (\mu_{s\beta n} - \mu_{d\beta n}) e^{|\dot{\theta}_n|/(\varepsilon_n \sin \gamma_n)}] \operatorname{sgn}(\dot{\theta}_n) \sin \gamma_n}$$
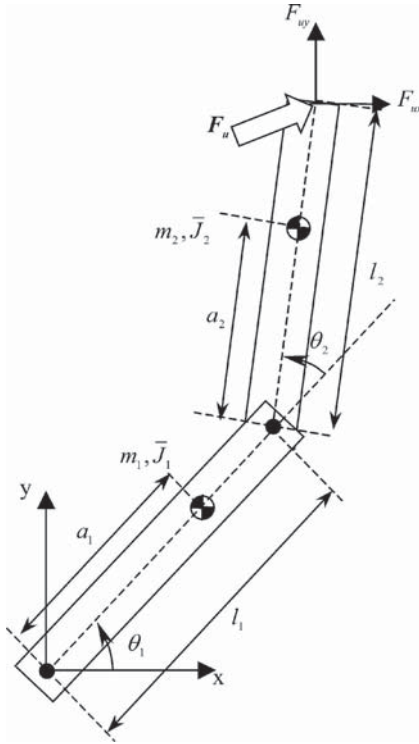
$$(7)$$

Fig. 5 A two DOF planar manipulator

Substitution of the joint torque equations (4) and (5) into the system equation of motion (3) yields

$$\mathbf{M}\ddot{\boldsymbol{q}} + \mathbf{M}_r\ddot{\boldsymbol{q}} + \mathbf{C}\dot{\boldsymbol{q}} + \mathbf{C}_r\dot{\boldsymbol{q}} + \mathbf{V}_m\dot{\boldsymbol{q}} = \mathbf{B}\boldsymbol{\tau}_m + \mathbf{J}^T\boldsymbol{F}_u \qquad (8)$$

where

$$\mathbf{M}_r = \begin{bmatrix} \dfrac{J_{\alpha1}r_{\theta1}^2}{\Delta_{\alpha1}r_{\alpha1}^2\tan\gamma_1} + \dfrac{J_{\beta1}r_{\theta1}^2}{\Delta_{\beta1}r_{\beta1}^2\tan\gamma_1} & 0 \\[3mm] 0 & \dfrac{J_{\alpha2}r_{\theta2}^2}{\Delta_{\alpha2}r_{\alpha2}^2\tan\gamma_2} + \dfrac{J_{\beta2}r_{\theta2}^2}{\Delta_{\beta2}r_{\beta2}^2\tan\gamma_2} \end{bmatrix}$$

$$\mathbf{C}_r = \begin{bmatrix} \dfrac{C_{f\alpha1}r_{\theta1}^2}{\Delta_{\alpha1}r_{\alpha1}^2\tan\gamma_1} + \dfrac{C_{f\beta1}r_{\theta1}^2}{\Delta_{\beta1}r_{\beta1}^2\tan\gamma_1} & 0 \\[3mm] 0 & \dfrac{C_{f\alpha2}r_{\theta2}^2}{\Delta_{\alpha2}r_{\alpha2}^2\tan\gamma_2} + \dfrac{C_{f\beta2}r_{\theta2}^2}{\Delta_{\beta2}r_{\beta2}^2\tan\gamma_2} \end{bmatrix} = \begin{bmatrix} \mathbf{C}_{r11} & 0 \\ 0 & \mathbf{C}_{r22} \end{bmatrix}$$

$$\mathbf{B} = \begin{bmatrix} \dfrac{r_{\theta1}}{\Delta_{\alpha11}r_{\alpha1}} & \dfrac{r_{\theta1}}{\Delta_{\beta11}r_{\beta1}} & 0 & 0 \\[3mm] 0 & 0 & \dfrac{r_{\theta2}}{\Delta_{\alpha12}r_{\alpha2}} & \dfrac{r_{\theta2}}{\Delta_{\beta12}r_{\beta2}} \end{bmatrix}$$

$$= \begin{bmatrix} B_{11} & B_{12} & 0 & 0 \\ 0 & 0 & B_{23} & B_{24} \end{bmatrix}$$

$$\boldsymbol{\tau}_m = [\tau_{m\alpha1}\,\tau_{m\beta1}\,\tau_{m\alpha2}\,\tau_{m\beta2}]^T$$

However, due to the placement of the strain gauges on each link, it is possible to simplify equation (8) by introducing the terms τ_{act1} and τ_{act2} which represent measurements from the two sensors. The use of the sensors

in this manner mathematically decouples the dynamic equations because the forces acting on each joint are explicitly measured. In particular, the forces caused by acceleration and the Coriolis effect are measured. However, the position of the sensors does not allow for the measurement of the total inertial forces. This *residual* inertia includes the inertia of the worm wheel, the main shaft and a small portion of the link between the joint and the strain gauge module. Therefore, the torque vector measured by the two link strain gauge sensors is given as

$$\boldsymbol{\tau}_{act} = -(\mathbf{M} - \mathbf{M}_\theta)\ddot{\boldsymbol{q}} - \mathbf{V}_m\dot{\boldsymbol{q}} + \mathbf{J}^T\boldsymbol{F}_u \qquad (9)$$

Equation (9) can also be written as

$$\mathbf{J}^T\boldsymbol{F}_u = \boldsymbol{\tau}_{act} + (\mathbf{M} - \mathbf{M}_\theta)\ddot{\boldsymbol{q}} + \mathbf{V}_m\ddot{\boldsymbol{q}} \qquad (10)$$

where

$$\mathbf{M}_\theta = \begin{bmatrix} J_{\theta1} & 0 \\ 0 & J_{\theta2} \end{bmatrix}, \qquad \boldsymbol{\tau}_{act} = [\tau_{act1}\,\tau_{act2}]^T$$

and $J_{\theta n}$ represents the residual inertia of link n. Combining equations (8) and (10) yields the decoupled system dynamics given by

$$(\mathbf{M}_\theta + \mathbf{M}_r)\ddot{\boldsymbol{q}} + (\mathbf{C} + \mathbf{C}_r)\dot{\boldsymbol{q}} = \mathbf{B}\boldsymbol{\tau}_m + \boldsymbol{\tau}_{act} \qquad (11)$$

5 COMPUTED-TORQUE CONTROLLER

The difference between the desired trajectory and the actual trajectory is defined as

$$\mathbf{E} = \begin{bmatrix} e_1 \\ e_2 \end{bmatrix} = \boldsymbol{q}_d - \boldsymbol{q} \qquad (12)$$

Using equations (11) and (12) it is possible to write the error dynamics as

$$\ddot{\mathbf{E}} = \ddot{\boldsymbol{q}}_d - \ddot{\boldsymbol{q}} = (\mathbf{M}_\theta + \mathbf{M}_r)^{-1}((\mathbf{M}_\theta + \mathbf{M}_r)\boldsymbol{q}_d$$
$$+ (\mathbf{C} + \mathbf{C}_r)\dot{\boldsymbol{q}} - \mathbf{B}\boldsymbol{\tau}_m - \boldsymbol{\tau}_{act}) \qquad (13)$$

5.1 Joint 1 α-unwinding controller

During α-unwinding control of joint 1 the torque, $\tau_{m\alpha1}$, is used to regulate position. In order to track the desired

trajectory it is possible to select the computed-torque control law as

$$\tau_{\mathrm{m}\alpha1} = \frac{1}{B_{11}}\left[\left(J_{\theta1} + \frac{J_{\alpha1}r_{\theta1}^2}{\Delta_{\alpha1}r_{\alpha1}^2\tan\gamma_1} + \frac{J_{\beta1}r_{\theta1}^2}{\Delta_{\beta1}r_{\beta1}^2\tan\gamma_1}\right)(\ddot\theta_{\mathrm{d}1} + u_{\alpha1})\right.$$
$$\left. + (C_{11} + C_{\mathrm{r}11})\dot\theta_1 - B_{12}\tau_{\mathrm{m}\beta1} - \tau_{\mathrm{act}1}\right] \qquad (14)$$

where $u_{\alpha1}$ is the proportional + integral + derivative (PID) control law given by

$$u_{\alpha1} = K_{\mathrm{p}\alpha1}e_1 + K_{\mathrm{v}\alpha1}\dot e_1 + K_{\mathrm{i}\alpha1}\int e_1\,\mathrm{d}t \qquad (15)$$

5.2 Joint 1 β-unwinding controller

Similarly, $\tau_{\mathrm{m}\beta1}$ is used to regulate position for β-unwinding control of joint 1. Hence the computed-torque PID controller is given by

$$\tau_{\mathrm{m}\beta1} = \frac{1}{B_{12}}\left[\left(J_{\theta1} + \frac{J_{\alpha1}r_{\theta1}^2}{\Delta_{\alpha1}r_{\alpha1}^2\tan\gamma_1} + \frac{J_{\beta1}r_{\theta1}^2}{\Delta_{\beta1}r_{\beta1}^2\tan\gamma_1}\right)(\ddot\theta_{\mathrm{d}1} + u_{\beta1})\right.$$
$$\left. + (C_{11} + C_{\mathrm{r}11})\dot\theta_1 - B_{11}\tau_{\mathrm{m}\alpha1} - \tau_{\mathrm{act}1}\right] \qquad (16)$$

where $u_{\beta1}$ is the PID control law given by

$$u_{\beta1} = K_{\mathrm{p}\beta1}e_1 + K_{\mathrm{v}\beta1}\dot e_1 + K_{\mathrm{i}\beta1}\int e_1\,\mathrm{d}t \qquad (17)$$

5.3 Joint 2 α-unwinding controller

For α-unwinding control of joint 2, the computed-torque PID controller is chosen in a similar manner and is given by

$$\tau_{\mathrm{m}\alpha2} = \frac{1}{B_{23}}\left[\left(J_{\theta2} + \frac{J_{\alpha2}r_{\theta2}^2}{\Delta_{\alpha2}r_{\alpha2}^2\tan\gamma_2} + \frac{J_{\beta2}r_{\theta2}^2}{\Delta_{\beta2}r_{\beta2}^2\tan\gamma_2}\right)(\ddot\theta_{\mathrm{d}2} + u_{\alpha2})\right.$$
$$\left. + (C_{22} + C_{\mathrm{r}22})\dot\theta_2 - B_{24}\tau_{\mathrm{m}\beta1} - \tau_{\mathrm{act}2}\right] \qquad (18)$$

where $u_{\alpha2}$ is the PID control law given by

$$u_{\alpha2} = K_{\mathrm{p}\alpha2}e_2 + K_{\mathrm{v}\alpha2}\dot e_2 + K_{\mathrm{i}\alpha2}\int e_2\,\mathrm{d}t \qquad (19)$$

5.4 Joint 2 β-unwinding controller

The β-unwinding control of joint 2 is accomplished using the computed-torque PID controller of the following equations:

$$\tau_{\mathrm{m}\beta2} = \frac{1}{B_{24}}\left[\left(J_{\theta2} + \frac{J_{\alpha2}r_{\theta2}^2}{\Delta_{\alpha2}r_{\alpha2}^2\tan\gamma_2} + \frac{J_{\beta2}r_{\theta2}^2}{\Delta_{\beta2}r_{\beta2}^2\tan\gamma_2}\right)(\ddot\theta_{\mathrm{d}2} + u_{\beta2})\right.$$
$$\left. + (C_{22} + C_{\mathrm{r}22})\dot\theta_2 - B_{23}\tau_{\mathrm{m}\alpha1} - \tau_{\mathrm{act}2}\right] \qquad (20)$$
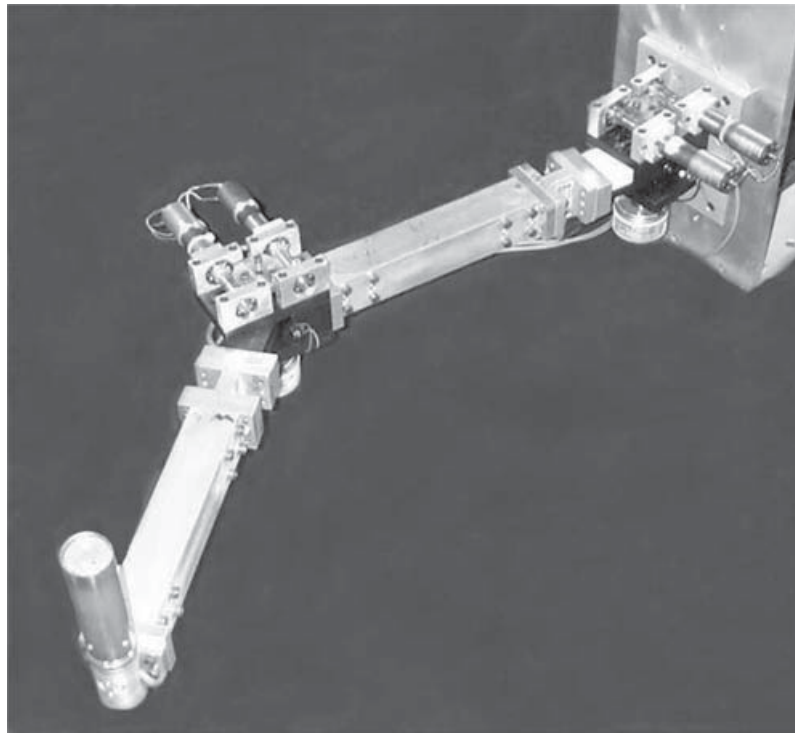

Fig. 6 A two DOF manipulator

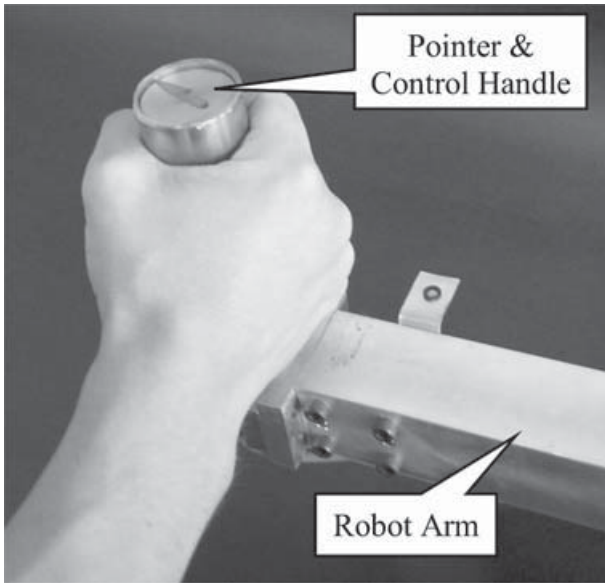**Fig. 7** Control handle and pointer mechanism

where $u_{\beta 2}$ is the PID control law given by

$$u_{\beta 2} = K_{\mathrm{p}\beta 2} e_2 + K_{\mathrm{v}\beta 2} \dot{e}_2 + K_{\mathrm{i}\beta 2} \int e_2 \, \mathrm{d}t \qquad (21)$$

6 EXPERIMENTAL SET-UP

The 2 DOF dual-worm driven joint planar manipulator is shown in Fig. 6. The control handle and pointer mechanism are shown in more detail in Fig. 7. Control of the robot is performed using a Pentium 233 MHz personal computer running the QNX 4.25 real-time operating system. A schematic diagram of the control system is shown in Fig. 8.

Each control algorithm requires measurements of motor torque, joint torque and joint position. The torque acting on each joint is measured using four strain gauges, in a Wheatstone bridge configuration, mounted on a specially designed section of the link. Each joint uses a pair of
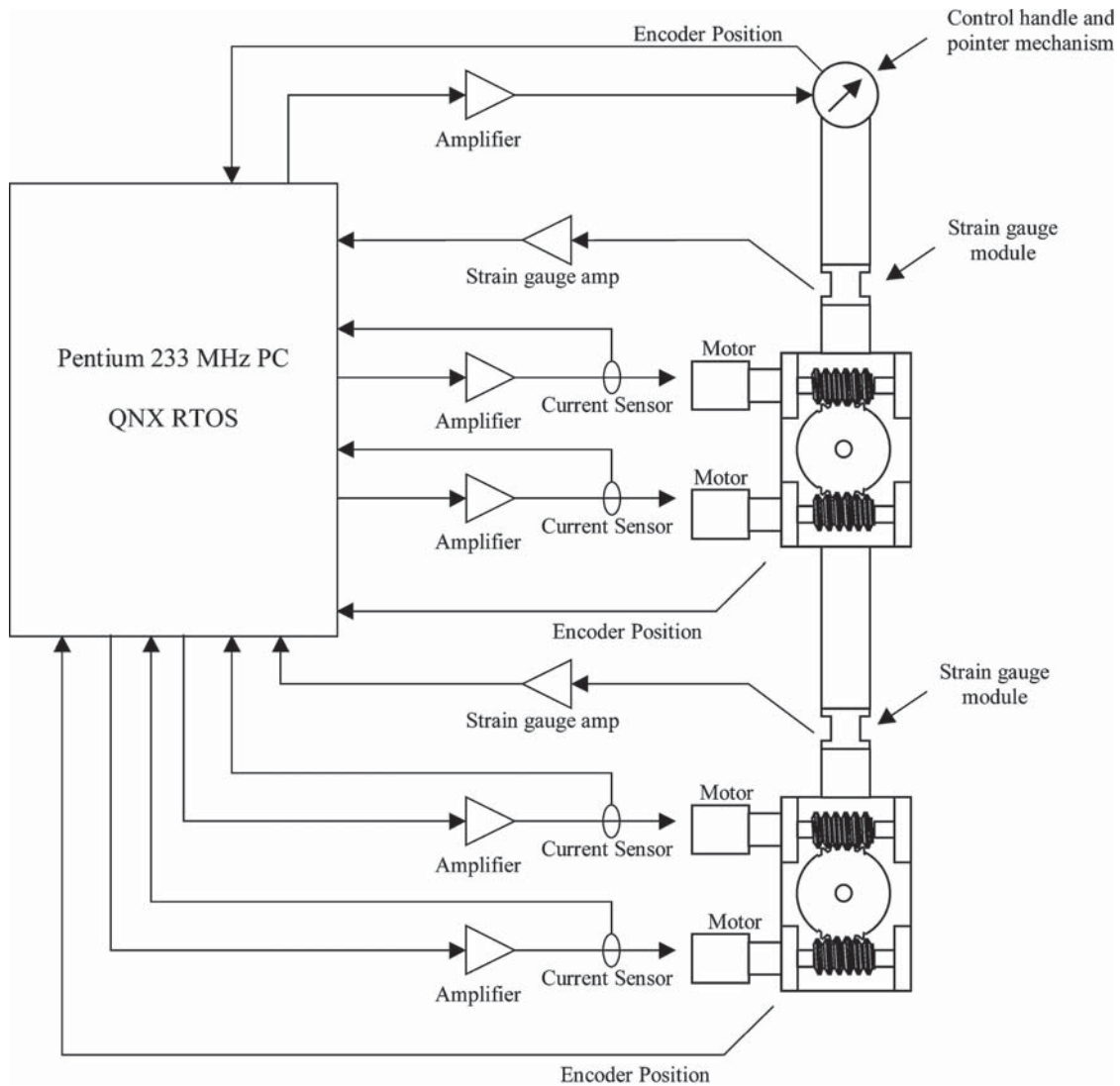

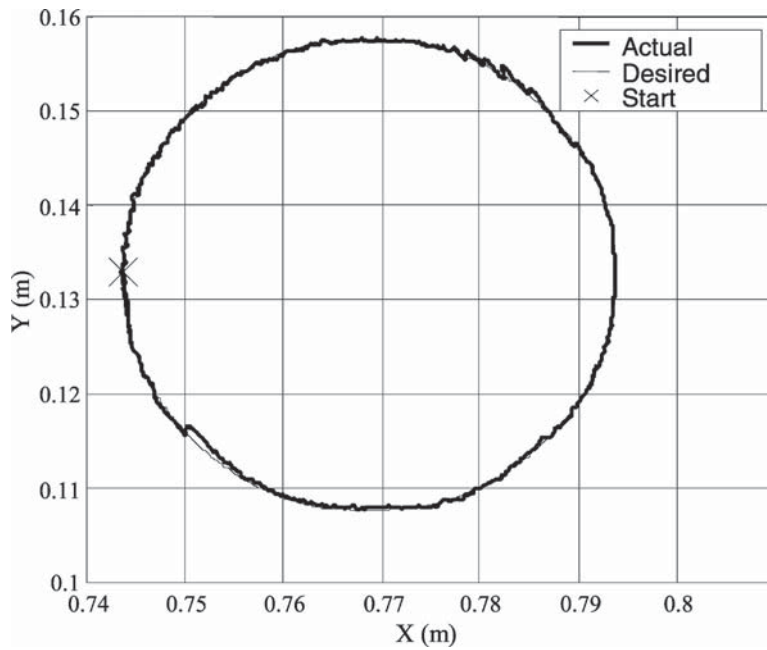**Fig. 8** Shematic diagram of the control system
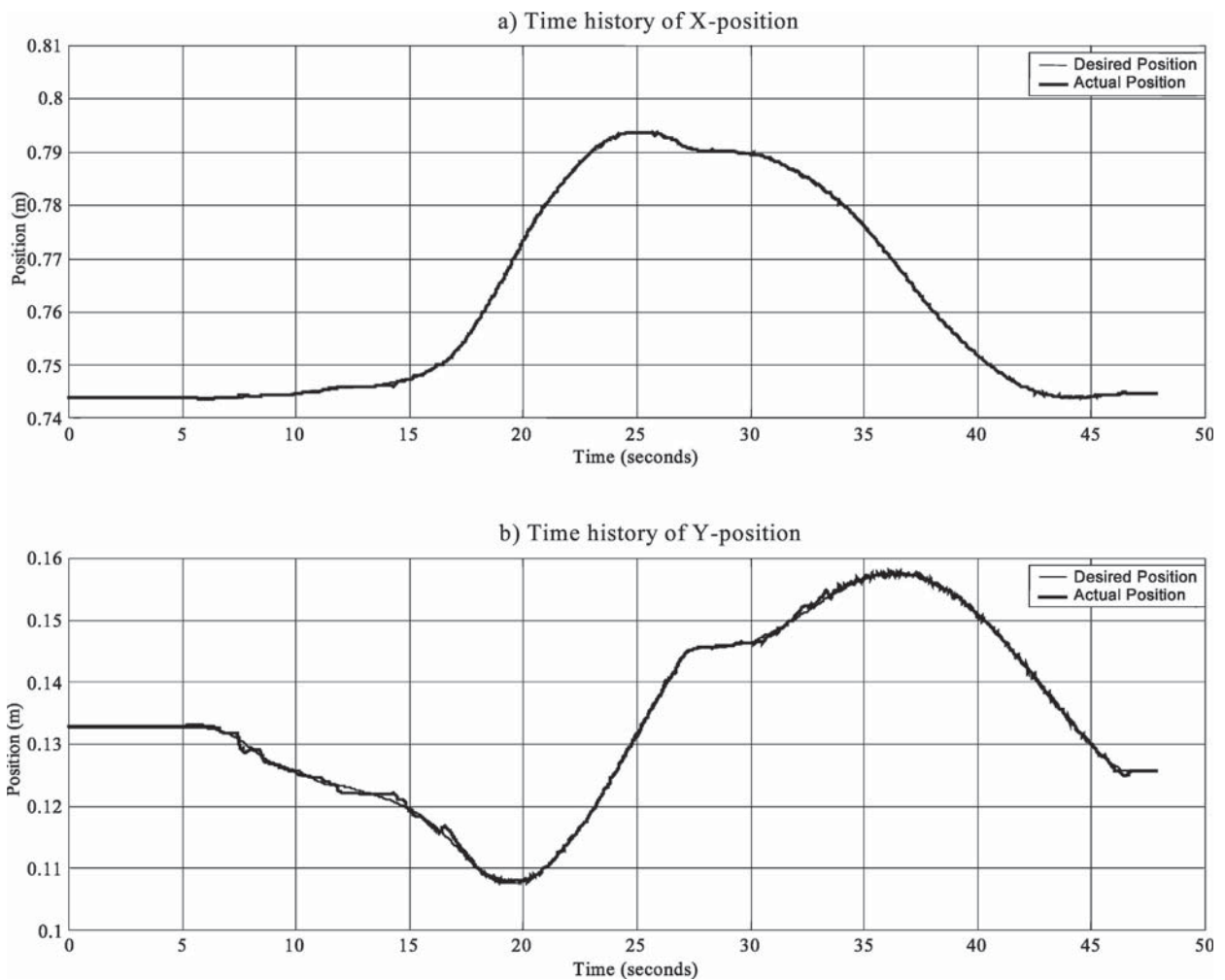
Fig. 9 Tracking of a circular path

Fig. 10 Time history of the end-effector X and Y positions for the circular path shown in Fig. 9

geared d.c. motors selected to give a peak joint speed of only 2 r/min. Closed-loop regulation of the motor output torque is accomplished by measuring the armature current and implementing a digital PI (proportional + integral) controller. All analogue signals are measured and generated using 12-bit analogue-to-digital converters and 12-bit digital-to-analogue converters. The link rotation is recorded using an encoder and appropriate electronics to generate 20 000 counts per revolution, giving a resolution of 3.142×10^{-4} rads/count (or 0.018°/count) for each joint. The link rotational velocity measurement is obtained in software by using a backwarddifference algorithm.

7 RESULTS AND DISCUSSION

To test the ability of the manipulator end-effector to track a trajectory, a circular path of diameter 50 mm was chosen. The parameters used to compensate for system dynamics in the control algorithm are given in Table 3. It should be noted that the static and dynamic friction coefficients at the two joints are different because they are dependent on the physical condition of the joint mechanism and gear interface, including the surface condition and lubrication. They were obtained experimentally.

The desired and actual paths of the end-effector, expressed in Cartesian coordinates x and y, are shown in Fig. 9. The start position of the end-effector on the path is also illustrated. Figure 10 shows the circular path expressed as the time history of both the x and y positions of the end-effector. It can be seen that the end-effector of the robot tracks the path with little error. While the plot of the x position illustrates good tracking performance it can be seen that the graph of the y position shows some deviation from the path. Figure 11 illustrates the error in the position of the end-effector shown in Cartesian coordinates. The x-axis error, shown in Fig. 11a, does not exceed 0.6 mm at any time while the y-axis error (Fig. 11b) remains under 1.5 mm. Figure 12 shows the components of the force calculated from the joint torque recorded by the strain gauges on each link.

The difference in the magnitudes of the error in the x and y positions depends on the particular configuration of the manipulator with respect to the chosen directions of
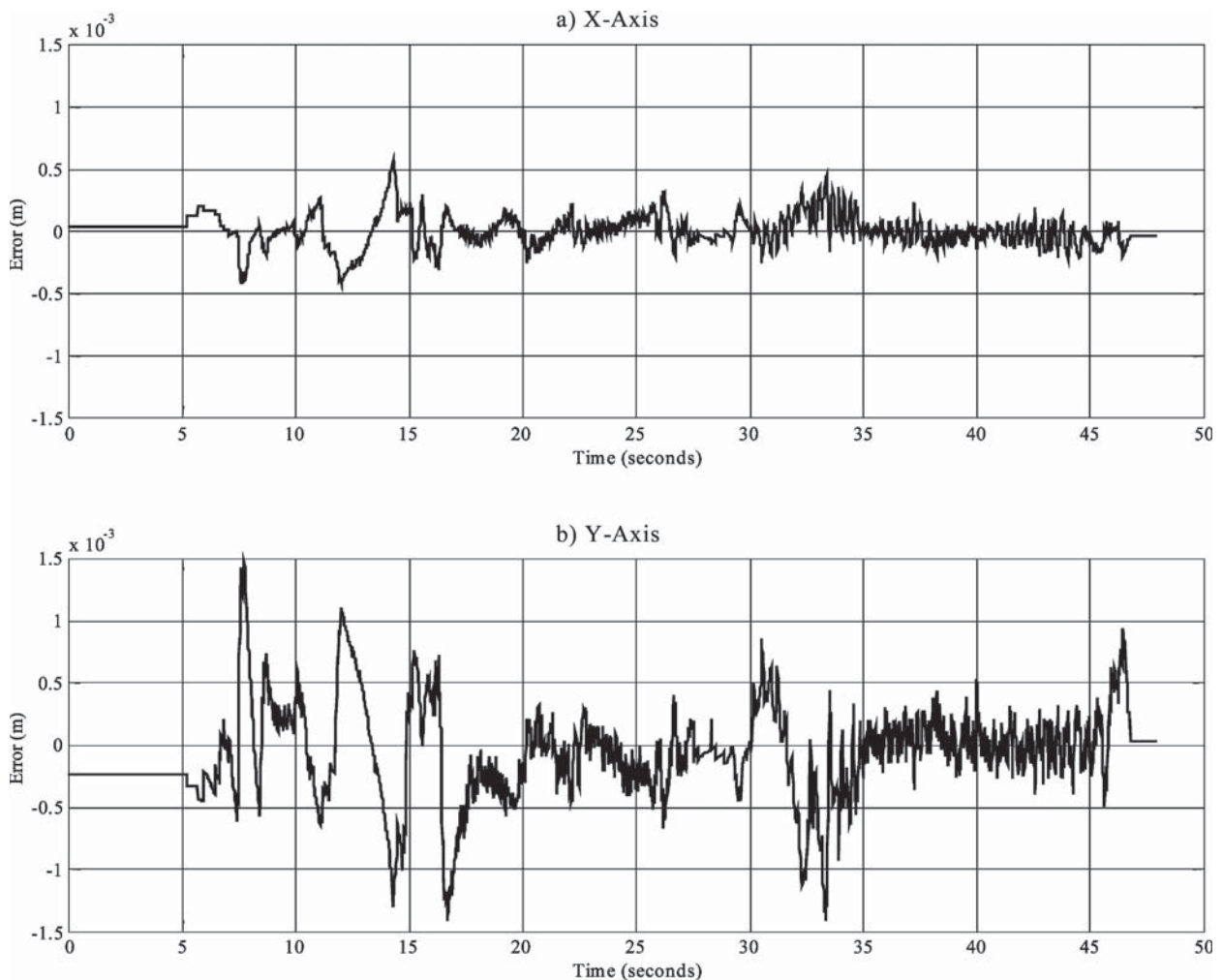

Fig. 11 End-effector position error

Table 3 System parameters

	Worm wheel	α worm	β worm
Inertia (kg m^2)	$J_{\theta 1} = J_{\theta 2} = 0.251\,22$	$J_{\alpha 1} = J_{\alpha 2} = 0.003\,698$	$J_{\beta 1} = J_{\beta 2} = 0.003\,698$
Viscous damping (N m s/rad)	$C_{f\theta 1} = C_{f\theta 2} = 0.005$	$C_{f\alpha 1} = C_{f\alpha 2} = 0.001$	$C_{f\beta 1} = C_{f\beta 2} = 0.001$
Gear pitch circle diameter (PCD) (m)	$r_{\theta 1} = r_{\theta 2} = 0.025$	$r_{\alpha 1} = r_{\alpha 2} = 0.0065$	$r_{\beta 1} = r_{\beta 2} = 0.0065$
Static friction		$\mu_{s\alpha 1} = \mu_{s\alpha 2} = 0.12$	$\mu_{s\beta 1} = \mu_{s\beta 2} = 0.16$
Dynamic friction		$\mu_{d\alpha 1} = \mu_{d\alpha 2} = 0.10$	$\mu_{d\beta 1} = \mu_{d\beta 2} = 0.14$
Friction constant		$\varepsilon_1 = \varepsilon_2 = 0.001$	$\varepsilon_1 = \varepsilon_2 = 0.001$
Worm lead angle		$\gamma_1 = \gamma_2 = 0.052\,36$	$\gamma_1 = \gamma_2 = 0.052\,36$
Joint link lengths (m)	$l_1 = 0.450$		
	$l_2 = 0.415$		
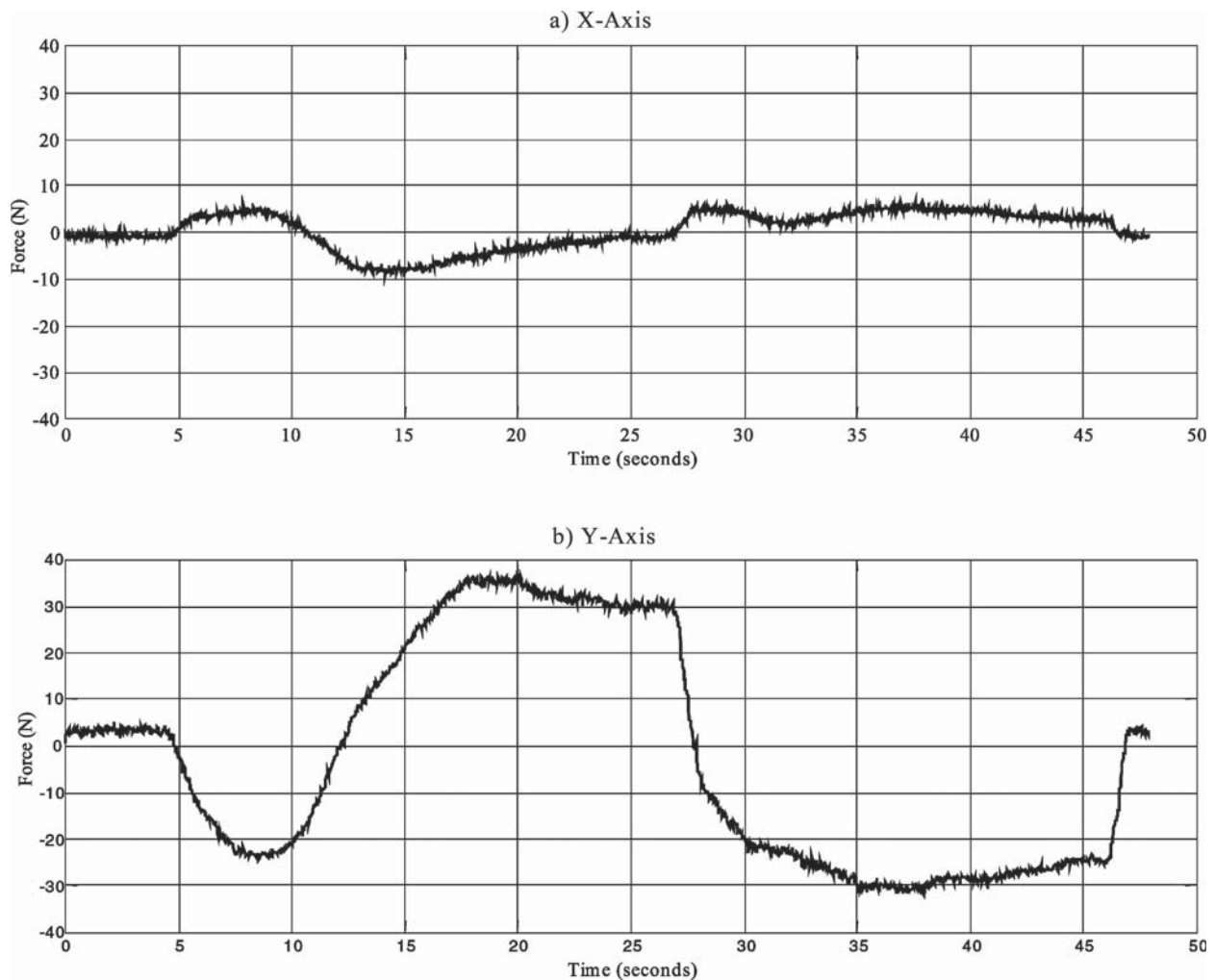
Fig. 12 End-effector force computed from the joint torques generated when the path shown in Fig. 9 is followed

the x and y axes, and is attributed to the flexibility in the mechanism of joint 1. The end-effector error, caused by system compliance, is dependent upon the manipulator configuration but can be highlighted in a static test. For example, the test for a particular configuration has shown that for every 10 N of user-input force in the y direction the end-effector is deflected by approximately 1.15 mm in the y axis in the given manipulator configuration. For the same configuration and force in the x direction the end-effector is deflected by only 0.1 mm in the x axis. By increasing the stiffness of the main shaft

of joint 1 the error in the y axis due to flexibility can be significantly reduced.

Using these control strategies the tracking error over the 50 mm diameter circular path was shown to be less than 1.6 mm in magnitude and was caused by the interaction of the unmodelled joint flexibility, user applied force and the controller. In view of the large errors that could be obtained due to the application of a force by the user (approximately 4 mm for 35 N), this is considered to be reasonable. However, since the user can easily apply forces to the control handle in the order of

50 N, the error shown in these experiments with this 2 DOF manipulator cannot be tolerated in most medical applications without reducing the effect of the stiffness and/or introducing instrumentation to raise an alarm whenever the user exceeds a maximum allowable force.

8 CONCLUSIONS

It has been shown that by implementing a pointing device to direct the user to apply a force in the correct way the end-effector of a 2 DOF manipulator can be guided along a predefined path under the direct physical control of a human operator. PADyC uses a computer monitor to display the desired path and requires the user to have some knowledge/experience of the system and its behaviour in order to successfully negotiate the corridors that describe the path [1]. A distinct advantage of using the pointing device presented in this work is that the user is shown how to apply a force to the control handle in order to accomplish the desired task more efficiently. Therefore it is not necessary for the user to make the decision about how to apply the force based on learned knowledge of the system, significantly reducing the need for training. However, control of the 2 DOF manipulator with the strategies described in this paper requires measurement of the user-input force and implementation of a pointing device. The increase in tracking performance over other solutions far outweighs the increased cost incurred by the addition of force measurement and pointing devices.

Safe deployment of the end-effector has been assured by allowing motion only if the direction of the applied user-input force would allow the controller to restrict motion along the predefined path. The effects of system compliance have been shown to have an adverse affect on controller performance. However, a mechanical redesign of the system and reconfiguration of the manipulator would result in increased robustness to large user-input forces. Using the control strategy described in this paper it would also be possible to replace the motorized joints with brake or clutch mechanisms without loss of performance. Future work will address the implementation of these control strategies on a passive manipulator.

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