| 1 | Performance and emissions study of diesel and waste biodiesel blends |
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| 2 | with nanosized CZA2 of high oxygen storage capacity |
| 3 | P. Pimenidou ^{1*} , N. Shanmugapriya ² , N. Shah ³ |
| 4 | ¹ School of Engineering, University of Bradford, West Yorkshire BD7 1DP, UK |
| 5 | ² Department of Mechanical Engineering, Siddaganga Institute of Technology, |
| 6 | Tumkur-572103. Karnataka, India |
| 7 | ³ Built Environment Research Institute, University of Ulster, Newtonabbey, Northern |
| 8 | Ireland, BT37 0QB, UK |
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| 11 | Abstract |
| 12 | In this work, the effect of the nanosized CZA2 (cerium-zirconium-aluminium) on the |
| 13 | performance and emissions in a two- cylinder indirect injection (IDI) diesel engine, |
| 14 | was studied. CZA2 was dispersed in diesel (D100) and waste cooking oil and tallow |
| 15 | origin biodiesel-diesel blends (B10, B20, B30) and tested at different engine loads |
| 16 | and constant speed. The nanocatalyst (CZA2) increased the brake specific fuel |
| 17 | consumption (BSFC) and decreased the brake thermal efficiency (BTE, %) of all |
| 18 | tested fuels, at all loads, except B20 at the lowest load. CZA2 reduced nitrogen oxides |
| 19 | (NOx) from D100 at low and high engine loads, as well as carbon monoxide (CO) |
| 20 | and unburned hydrocarbons (HC) at medium and high tested loads. The dispersion of |
| 21 | CZA2 promoted the combustion of the biodiesel blends by almost eliminating HC |
| 22 | while reducing NOx and CO emissions at various loads. Thermogravimetric analysis |
| 23 | (TGA) coupled with Attenuated Total Reflectance- Fourier Transform Infrared (ATR- |
| 24 | FTIR) spectroscopy revealed that the addition of CZA2 in diesel and biodiesel under |
| 25 | pyrolysis and oxidation conditions resulted in the presence of saturated species like |
| 26 | ketones and final oxidation products such as CO2, supporting their improved |
| 27 | combustion and emissions' reduction in the engine tests. |
| 28 | Keywords: Redox, waste biodiesel, nanocatalyst, NOx, HC |
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1 **1. Introduction**

2 Biodiesel continues to gain attention in combustion and energy conversion 3 technologies due to its sustainability. Especially waste origin biodiesel such as waste 4 cooking oil-sourced biodiesel is deemed the most challenging regarding NOx 5 minimisation. Employment of oxygenated compounds (fatty acid methyl esters) found in biodiesel compared to straight and branched-chain paraffins and aromatic 6 7 hydrocarbons found in diesel make their combustion more challenging. Therefore, the 8 chemical structure of the biodiesel compounds limits the penetration of biodiesel in 9 the transportation, agriculture and construction sectors. Under lean conditions, 10 biodiesel produces less unburned hydrocarbons (HC) and carbon monoxide (CO), 11 whose presence indicates incomplete combustion, hence insufficient utilisation of the 12 fuel's stored energy expressed by its calorific value. Key challenges in biodiesel 13 combustion are significant emissions production as nitrogen oxides (NOx), a harmful 14 pollutant to humans and the environment. Biodiesel of various origins and similar 15 oxygen content gives different physicochemical properties and produces dissimilar 16 NOx emissions [1] attributed to the structure and unsaturation of the fuel's constituent 17 compounds. Therefore, it is the structure and degree of unsaturation of fatty acids 18 from virgin, waste, plant oils and animal fats either resulting in increased [2-12] or 19 decreased NOx emissions [13-15]. NOx formation is dependent on oxygen 20 concentration relevant to the fuel mixed, combustion temperature and effective 21 volume of the combustion zone according to the Zeldovich mechanism. Higher NOx 22 is noted at high pressure and temperature as in high and full load conditions, as well 23 with a higher rate of combustion leading to higher combustion temperature [9]. 24 Additionally, NOx can be produced during the combustion process at the flame front 25 by CH₂, CH radicals reacting with N₂ via the Fenimore mechanism [16].

Different types of biodiesel-diesel blends compared to diesel show increased brake specific fuel consumption (BSFC) [17-20] in direct injection (DI), compressed ignition engines (CIE) and the less studied indirect injection (IDI) engines [5, 21].

Different approaches have been examined to minimising emissions due to the incomplete combustion of biodiesel-diesel blends. Some of these approaches include the use of additives such as organic peroxides [22] and antioxidants, to aid the limited physicochemical properties of biofuels affected by their chemical composition [16, 23]. Alkaline earth, post-transition metals, metalloids, lanthanides particles and their oxides of a few microns have been tested to enhancing the performance and
 combustion of biodiesel in diesel engines.

3 Catalytic combustion has already been applied to reduce NOx emissions below 4 1500°C in furnace burners and gas turbine engine combustors. However, catalytic 5 combustion is minorly examined in the transportation sector due to the additional mass, bulk catalysts can add to the system [24-26]. Today, this disadvantage is 6 7 overcome by the progress in nanoscience and nanotechnology to formulate nanosized 8 catalysts whose enhanced catalytic activity requires small amounts. Emissions' post-9 treatment by three-way catalysts (TWCs) utilises platinum-rhodium (Pt/ Rh) blends in 10 the reduction of NOx and combined platinum-palladium (Pt/ Pd) to oxidise CO, 11 unburned HC at the exhaust. Cerium oxide mixed with zirconium oxide in TWCs 12 store excess oxygen under fuel-lean conditions and release oxygen under fuel rich 13 conditions [27-28]. Bulk cerium oxide (CeO_2) as a coating in ICEs for enhancing 14 combustion, is reported as early as 1926 [24]. Throughout the 60s, 70s and 80s, 15 modified cerium oxide is reported to boost fuel burning rates while minimising 16 unburned HC emissions from gasoline and diesel engines [24]. Oxygen storage 17 capacity (OSC) of cerium, zirconium (Zr) and aluminium (Al) mixed oxides are 18 reported for hydrocarbons oxidation of combustion [29, 30]. Ce-Zr oxides promote 19 thermal and reduction- oxidation (redox) behaviour of the material and increase their 20 OSC [27, 31].

In this work, we study the effect of a ternary nano-catalyst, Ce_{0.6}Zr_{0.2}Al_{0.26}O₂ (CZA2) 21 22 [32] on IDI engine performance indicators and emissions from diesel only (D100) and 23 waste cooking oil and tallow origin biodiesel- diesel blends (B10, B20, B30). Loss of oxygen from CZA2 and reduction of Ce^{4+} to Ce^{3+} increases its thermal expansion 24 coefficient by 11.9% in the temperature region of 600-1000°C compared to that of 25 26 200-600°C [33]. Therefore, thermal stability under combustion conditions is 27 increased, which potentially further supports enhanced catalytic combustion of the 28 tested fuels by CZA2. All tested fuels are analysed by thermogravimetric analysis 29 (TGA) coupled with Attenuated Total Reflectance-Fourier transform infrared (ATR-30 FTIR) spectroscopy, under inert (pyrolysis) and oxidative (ignition) conditions 31 relative to their performance and emissions in the IDI engine tests.

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2 2. Experimental

3 2.1 Nano-catalysts and fuel blends

4 Nanosized CZA2 ($Ce_{0.6}Zr_{0.2}Al_{0.26}O_2$, 4.7 nm ± 0.7 nm) catalysts were produced by the 5 sol-gel method. The method of synthesis, characterisation, initial optimisation, 6 stability and reduction-oxidation capacity were described in previous work [33]. 7 Doping with ZrO₂ and Al₂O₃ overcame inherent limitation of CeO₂ like, low thermal 8 stability and high redox temperature. Additionally, anti-sintering characteristics of 9 Al₂O₃ helped to avoid the particles' agglomeration. CZA2 exhibited high oxygen 10 storage capacity (OSC) and coefficient of thermal expansion (CTE) [33].

11 Commercial diesel (D100), biodiesel (B100) and 10, 20 and 30% by volume biodiesel 12 in diesel blends, B10, B20 and B30 respectively were tested. Biodiesel originated 13 from waste cooking oil and tallow at a ratio of 60:40 by weight (Green Biofuels 14 Ireland). Homogeneity of blends and uniform dispersion of 0.06 wt% CZA2 in all 15 fuels were ensured with the aid of a magnetic stirrer (IKA C-Mag, model: HS7) for 16 two hours before testing. The relative stability of the nanofuel was observed to be 100 17 % for the period of 2 hrs. All blends with CZA2 were continuously stirred to maintain 18 suspension throughout the engine feeding. More than 92% relative stability gave 19 sufficient dispersion for the fuel use till 18 hours from its preparation.

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22 2.2 TGA- FTIR

23 Thermal degradation (pyrolysis) and oxidation (ignition conditions) of the above-24 described fuels were carried out using a Mettler Toledo TGA/SDTA 851e, with 25 temperature accuracy $\pm 0.5^{\circ}$ C, equipped with an alumina pan. All samples were heated under a constant dynamic linear rate of 10° C/ min, in a 50 cm³/ min nitrogen flow for 26 27 thermal degradation. The same volumetric flow rate under air from 25 to 600°C was 28 employed. Attenuated Total Reflectance- Fourier Transform Infrared (ATR- FTIR) 29 spectroscopy was carried out on a NEXUS ATR, Thermo Nicolet Spectrometer, in the range of 4000-400 cm^{-1} with a resolution of 4 cm^{-1} . The interval of time between 30 31 spectra was 60 s.

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2 2.3 Characterisation of fuels

Kinematic viscosity was determined in a thermal bath at 40°C by using an Ubbelohde
viscometer (PAC (Herzog) Multi Range Viskosimeter HVM 472) according to EN
ISO 3104.

6 Analysis of the inorganic elements was conducted according to EN 14538. Samples of 7 all fuel blends were analysed by PE2400 CHN Elemental Analyser (Perkin Elmer) for 8 carbon, hydrogen, sulphur and nitrogen content. K and Na were determined by 9 inductively coupled plasma mass spectrometry (ICP MS; Varian 820) according to 10 EN 14108/ 14109 and EN 14538 respectively. Ca and Si were determined by 11 inductively coupled plasma optical emission spectrometry (ICP OES; Varian "Vista 12 pro") (EN 14538). Biodiesel samples were diluted in kerosene solvent only, without 13 any other further pre-treatment. Water content was determined using coulometric Karl 14 Fisher titration (Metrohm, KF Coulometer 756) (DIN EN ISO 12937). Gross calorific 15 values were determined using a combustion calorimeter (C 2000- IKA) (DIN 51900).

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18 2.4 Indirect injection engine (IDI) set-up and experimental procedure

All fuels were tested in a four stroke LDW 702 M (FOCS series, Lombardini Group) indirect injection (IDI) engine. The main parameters of the engine are given in Table 1. A schematic diagram of the engine test rig is shown in Figure 1. A net weighted glass cylinder and a digital scale were used to measure the mass of fuel used and a stopwatch to measure the duration of fuel consumption per experiment.

24 A set of transmitters sent the values of the operation parameters (rotation speed, 25 torque, fuel weight, cooling water flow rate and engine inlet and outlet temperature, 26 fuel- airflow rate, exhaust temperature) to an automatic data acquisition system and 27 were logged by a runtime type software package (LabVIEW). A hydraulic 28 dynamometer (TD-315v4, STEM ISI Impianti) was used to adjust and measure 29 variable engine load (torque) conditions and speed (max operating speed 7000 rpm 30 for dyno rpm f) respectively. Engine load was measured by an electronic load cell 31 with a panel-type instrument for torque measurement range 0.263 N m. Air was 32 monitored by a fixed choke nozzle (-40÷0 mbar for combustion air flow rate 33 measurement).

The engine was warmed up for 30 min with diesel for the biodiesel blends
 experiments. Each target load was reached with adjustments of speed and torque. The
 torque was measured with the help of moment arm.

4 The experiments were conducted at variable loads (based on torque) and constant 5 speed of the shaft of 3600 rpm at steady state conditions resulting in small mechanical losses. These conditions would lead to conclusions on the effect all tested fuels and 6 7 CZA2 had on the engine's performance characteristics and emissions. A four gases 8 emissions analyser (AUTOplus automotive) measured nitrogen oxides (NOx, ppm), 9 carbon monoxide (CO, %), carbon dioxide (CO₂, %) and total unburned hydrocarbons 10 (HC, ppm) at the exhaust. The analyser had an accuracy of \pm 5% of reading. At each 11 load, the engine was stabilised for 4 min, and then the measurement parameters per 12 tested blend were recorded. All fuels were tested three times under steady state 13 conditions, and the results were averaged to decrease the uncertainty.

14 Performance indicators such as brake power (BP, kW), brake specific fuel 15 consumption (BSFC, kg/ (kW·s)) and % of brake thermal efficiency (BTE,%) of all 16 fuels were calculated as follows:

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$$BP(kW) = \frac{2 \cdot \pi \cdot RPM \cdot \tau}{60 \cdot 1,000}$$
 Eq. (1)

18 where τ is the brake toque (Nm),

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$$BSFC = \frac{\dot{m}_f \left(\frac{kg}{s}\right)}{BP(kW)}$$
 Eq. (2)

20 where \dot{m}_f , fuel consumption rate (kg/s),

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$$BTE, \% = \frac{3,600}{BSFC \cdot LHV} \cdot 100$$
 Eq. (3).

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24 2.5 Engine performance and emissions statistical analysis

25 Engine performance and emissions statistical analysis were conducted on the 26 calculated engine performance parameters, aka of the average BSFC, BTE and 27 emissions of all fuels. The two-way analysis of the variance (ANOVA) with Fischer's 28 least significance difference (LSD) post-hoc analysis [34] were applied to show the 29 level of significance of the load, the addition of CZA2 in each fuel and the synergy of 30 the addition of CZA2 and the load, by using SPSS (v24) software (IBM). The F-ratio 31 is the ratio of the two mean square values, comparing each variation source, and the 32 *p*-value determined the statistical level of significance. When a *p*-value was less than

or equal to 0.05 the mean values of BSFC, BTE (%) and emissions of all fuels would
 indicate the different performance and emissions of each fuel. A *p*-value greater than
 0.05 would show statistical non-significance.

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3. Results and discussion

7 3.1 TGA-FTIR analysis

8 Figures 2. (a) and (b) showed single step curves of pyrolysis and oxidation 9 respectively for all tested fuels. One-step mass remaining curves of all tested fuels 10 suggested that operational engine problems, such as slow and inhibited multi-step 11 burning under pyrolysis and ignition conditions, would not be met [35]. Two 12 distinctive phases of decomposition were identified: (i) low- (< 230°C) and (ii) high-13 temperature (> 230° C). Pyrolysis of D100 and B100 started at 60° C and 50° C 14 respectively while the volatilisation of pure biodiesel (B100) started at 120°C, a 15 higher temperature than the rest of the tested fuels as seen in Fig. 2 (a1). Pyrolysis of 16 D100 finished at 290°C, an earlier temperature compared to the remainder of the 17 tested fuels (Fig. 2. (a1)). B100 was fully decomposed by 310°C, which was attributed 18 to more unsaturated bonds present compared to diesel. Lower volatility linked to 19 higher intramolecular forces of B100 giving to the fuel higher viscosity. The 20 volatilisation of B100-CZA2 started at 140°C, a temperature 20°C higher compared to 21 B100, and the pyrolysis of D100-CZA2 was initiated at 65°C. B100-CZA2 and D100-22 CZA2 pyrolysis ended at around 270°C and 300°C respectively. Overall, dispersion of 23 CZA2 reduced the temperature range of pyrolysis of D100 by postponing its initial 24 and advancing the final pyrolysis to higher temperatures and lower temperatures 25 respectively compared to the corresponding tests in the absence of CZA2. The initial 26 and final pyrolysis temperatures of biodiesel and biodiesel blends with CZA2 were 27 raised. Such an outcome was further reflected in the increased D100-CZA2 and 28 decreased B100-CZA2 mass losses in Fig. 2. (a2) compared to D100 and B100 29 pyrolysis in the low-temperature region in Fig. 2. (a1). The presence of CZA2 in the 30 low-temperature region increased mass losses of D100 from 66.81% (33.19% residual 31 mass) to 77.82% (22.18% residual mass) and marginally increased the mass losses of 32 B100 from 32.16% (67.84% residual mass) to 32.68% (67.32% residual mass) 33 respectively. All biodiesel blends followed the pattern of D100 pyrolysis in the low-34 temperature region; the biodiesel content dominated the pyrolysis of B30 in the high

temperature region (Fig. 2. (a1)). Pyrolysis of all biodiesel blends with CZA2
demonstrated a unique catalytic thermal cracking identity, incomparable to D100CZA2 and B100-CZA2 at all temperatures except B30-CZA2 which pyrolysed
similarly to B100-CZA2 in the high-temperature region.

5 Under ignition conditions, decomposition of D100 and B100 concluded at 280 and 310°C respectively (Fig. 2. (b1)). All tested biodiesel blends in the absence and 6 7 presence of CZA2 were fully oxidised by approximately 300°C (Fig. 2. (b2)). Mass 8 losses of D100 and B100 at 230°C were 70.02% (29.98% residual mass) and 23.32% 9 (76.68% residual mass) under ignition conditions (Fig. 2. (b1)), compared to 66.81% (33.19% residual mass) and 32.16% (67.84% residual mass) respectively under 10 11 pyrolysis conditions (Fig. 2. (a1)). Mass losses of D100-CZA2 and B100-CZA2 at 12 230°C were 70.02% (29.98% residual mass) and 23.32% (76.68% residual mass) 13 under ignition conditions (Fig. 2. (b2)), compared to 77.82% (22.18% residual mass) 14 and 32.68% (67.32% residual mass) under pyrolysis conditions (Fig. 2. (a2)). The 15 oxidation curves of all biodiesel blends showed a similar rate of consumption within 16 the same temperatures to those of D100 rather than B100 (Fig. 2. (b1)). Dispersion of 17 CZA2 in all biodiesel blends under ignition conditions led to their almost identical 18 behaviour by shifting the oxidation of these blends to higher temperatures than in the 19 absence of CZA2. Dispersion of CZA2 under ignition conditions gave to all biodiesel 20 blends a different behaviour to D100-CZA2 and B100-CZA2.

FTIR spectra were obtained simultaneously to the TGA data to gain insight on the thermal decomposition and oxidation of the tested fuels in the absence and presence of CZA2 (Fig. 3. (a) and (b)). The mid- IR spectral region (4000- 400 cm⁻¹) showed in the past some incomparable peaks between biodiesel and diesel. Here, FTIR spectra of all TGA experiments at 150°C were chosen since CZA2 oxygen transfer was initiated at this temperature [33].

27 CZA2 dispersion in D100 under pyrolysis conditions resulted in the additional peaks of C=O assigned to CO₂ (2343 and 2283 cm⁻¹), C=O attributed to CO (2100 cm⁻¹), 28 C=O attributed to ketones (1740 cm⁻¹) (Fig. 3. (a2)). D100-CZA2 under ignition 29 conditions produced less pronounced peaks of CH₂ (1450 cm⁻¹), C-H bending (1380 30 cm⁻¹), CO₂, CO and ketones (Fig. 3. (b2)) compared to D100-CZA2 under pyrolysis 31 32 conditions and D100 under ignition conditions (Fig. 3. (b1)). Pyrolysis of B100 at 33 150°C produced the same but much weaker absorption peaks than D100. Dispersion of CZA2 into B100 produced significant bands at 2954 and 2854 cm⁻¹ by sp³ 34

hybridised C-H stretch (Fig. 3. (a4)). The absence of the band attributed to CO was
noted for the same fuel, and stronger absorption peaks were recorded due to ketones
and CH₂, CH bending compared to D100-CZA2 under pyrolysis (Fig. 3. (a2)).
Oxidation of B100 resulted in weaker bands absorption (Fig. 3. (b3)) than B100CZA2 (Fig. 3.(b4)). Ketones absorption band was more pronounced for B100-CZA2
(Fig. 3. (b4)) than D100-CZA2 under ignition conditions (Fig. 3. (b2)).
Wexler [36] found that esters in the presence of metals such as cerium and zirconium

8 could lead to the formation of hydroperoxides, which by oxidation-reduction 9 produced aldehydes and ultimately ketones. Phenol could inhibit further oxidation 10 reactions by reacting with the free peroxy radical. In the same work [36] the 11 formation of lower aldehydes was considered a product of secondary oxidation (chain 12 propagation) of fatty acid esters from unsaturated ester peroxides (ROOH) which 13 were produced from the primary oxidation of esters. In particular, metals acting as 14 oxygen carriers could oxidise esters through different steps in forming 15 hydroperoxides by:

16 (1) attack of the oxygen atom to an activated methylene group and the produced alkyl17 radical

$$M - O - O \xrightarrow{-CH_2CH=CH-} M - O - O - H + -\dot{C}HCH = CH - CH - CH + -\dot{C}HCH = CH - CH + -CH +$$

18 (2) direct attack of a metal ion to an olefinic double bond

$$RCH = CH_2 + M^{3+} \rightarrow RCH = CH_2^{+} + M^{2+}$$

19 (3) direct attack of a metal ion to a saturated bond

$$RH + M^{3+} \rightarrow R \cdot + M^{2+} + H^+$$

The radical produced by all possible mechanisms could react with oxygen to produce a hydroperoxide by chain propagation. The hydroperoxides could then either decompose to produce an alkoxy and OH radical or break down to produce lower aldehydes:

$$R_1CH_2OOH \rightarrow \underline{R_1CH_2O \cdot + OH}$$

$$R_1CHO + H \cdot \underline{\text{or}} R_1 + HCHO$$

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Further oxidation of aldehydes led to the formation of acids, including some
unsaturated acids, and CO [36]. Oxidative attack of acid radicals resulted in methylketones and CO₂:

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$$R_1CH_2CH_2COOR_2 \xrightarrow{O_2} R_1CHCH_2COOR_2 \rightarrow R_1COCH_2COOR_2 \rightarrow R_1COCH_3 + CO_2$$

0 - 0 - H

Methyl- ketones could also be produced by the decomposition of vinyl ketones
 formed by the oxidation of the double bond:

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$$5 \qquad +R_2 \stackrel{O_2}{\cdot} \xrightarrow{RCCH} = CHCO_2H \rightarrow RCCH = CH_2 + CO_2$$

Formation of these ketones and carboxylic acids could also occur during thermal
cracking between 245- 260°C.

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10 3.2 Engine performance

The addition of biodiesel into diesel decreased BP to a small degree at 21 and 25 Nm whereas it marginally increased BP at lower loads. The dispersion of CZA2 decreased BP on average by 0.2- 5% for all biodiesel blends (Fig. 4) and by -4- 1.3% for D100, from low to high load conditions.

15 All biodiesel blends exhibited lower BTE than diesel at all loads. Lower BTE of the 16 biodiesel blends compared to diesel fuel is mainly due to high viscosity and low 17 heating value of the blend (Table 2). The BTE of D100-CZA2 declined at all load 18 conditions compared to D100 credited to the oxygenated compounds produced under 19 pyrolysis and ignition conditions relevant to the FTIR analysis in paragraph 3.1. Table 20 4 shows the results of ANOVA and LSD along with the F ratio of the BSFC and BTE 21 for each fuel and the associated p-value. The statistical analysis affirmed the 22 significant effect of load and addition of CZA2 on the reduction of BTE of diesel 23 (Table 4). The BTE was reduced by dispersion of CZA2 in all biodiesel blends and at 24 all loads. These results indicated that the combustion of all fuels with the CZA2 at all 25 loads was almost complete, despite that at the high load conditions the duration of 26 combustion was expected to decrease. All biodiesel blends' BTE reduction was more 27 pronounced at high than at low loads. The load had a substantial impact on the 28 decrease in BTE as seen in Table 4. Additionally, the dispersion of CZA2 became 29 more significant by increasing the biodiesel content in the fuel.

All blends with CZA2 under the majority of applied loads exhibited the mirroring
effect of an increased BSFC against a decreased BTE as expected. The brake specific

1 fuel consumption (BSFC) increased with the addition of biodiesel into diesel at all 2 loads (Fig. 5), reflecting on the decrease of the BTE at the same loads (Fig. 6). The 3 BSFC of the same blends decreased with increasing load due to the increasing BP at 4 higher loads [17]. Here, the decreasing trend of BSFC (Fig. 5) with adding biodiesel 5 in diesel at higher loads reflected on the increasing BTE (Fig. 6), similarly to earlier 6 performance studies at a constant speed and variable loads [20, 21, 37]. Such an effect 7 could be assigned to less fuel needed to operate the engine at high loads due to 8 minimisation of heat losses compensating for the increase in brake power (BP).

9 Furthermore, BSFC was calculated on a mass basis while the heating value of D100 10 was 2-3.8% higher than those of B10, B20 and B30 (Table 3). The increase in BSFC 11 when using D100-CZA2 on an average by approximately 16% compared to D100 at 12 all loads, suggested faster combustion due to catalysis that led to additional fuel 13 consumption per power produced, compared to the D100 engine tests. The 14 pronounced effect of the addition of CZA2 and the various loads on the BSFC when 15 D100 was tested was confirmed by the statistical analysis (Table 4). The BSFC was 16 amplified by B10-CZA2 versus B10 mainly at the high load conditions, between B20-17 CZA2 and B20 from medium to high loads (15, 21 and 25 Nm) and by dispersion of 18 CZA2 in B30 at all loads on average by approximately 12% (Fig. 5; Table 4). These 19 results underlined the different chemistry of the fuels during catalytic combustion 20 based on the groups of compounds identified in FTIR representing the fuel 21 compounds in the fuel rich and lean zones. As it was illustrated by the TGA-FTIR 22 study of diesel and biodiesel, the routes towards catalytic combustion of diesel and 23 biodiesel blends were expected to be different. The various mechanisms of pyrolysis 24 and oxidation could have led to intermediate products which would have given rise to 25 different catalytic combustion steps and whose complexity was apparent through the 26 various correlations of BSFC and BTE when CZA2 was dispersed in the tested fuels.

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29 **3.3 Engine emissions characteristics**

30 3.3.1 HC emissions

Diesel produced more unburned HC than all biodiesel blends in the engine tests at the high and full load conditions (Fig. 7.(a)), a well- established phenomenon as per previous studies [1, 8, 20, 38] due to the higher oxygen content of biodiesel (Table 3). The dispersion of CZA2 reduced the HC emissions from D100 at the high and full

1 loads appreciably (Fig. 7. (a)), underlining the benefit catalytic combustion offers as 2 well as the advantage of dispersion of oxygen released from CZA2. More specifically, 3 in the FTIR spectrum of the D100-CZA2 profile the peaks of CO, CO₂ and ketones 4 identified, which were not found in the FTIR profile of D100 under pyrolysis 5 conditions, affirmed an easier combustible mixture here evidenced by the reduced HC emissions during the catalytic combustion of D100. The profound effect of load, the 6 7 addition of CZA2 and the synergy of load and addition of CZA2 on HC emissions 8 from D100 was affirmed by the statistical analysis (Table 4). All biodiesel blends 9 with CZA2 blends had an overall significant effect on diminishing the presence of HC 10 found at the exhaust from the full load engine tests (Fig. 7. (a)). The lowest HC 11 emissions resulted in 0.06 ppm with B30 at 10 Nm, 0.03 ppm with B10-CZA-2 at 5 12 Nm, 0.05 ppm with B30-CZA2 at 21 Nm, 0.03 ppm with B30-CZA2 at 15 and 25 13 Nm. Unburned HC emissions were linked in the past to misfire in the locally lean 14 region or rich region. Here the redox property of ceria enabled the release of oxygen 15 in the fuel rich zone and absorbed oxygen from the lean fuel zone, which could assist 16 combustion to reach completion. An overall decreasing trend of HC emissions 17 decrease was identified with less biodiesel blended in diesel with CZA2 in the low 18 load (B10-CZA2< B20-CZA2< B30-CZA2 (Fig. 7.(a)). This outcome could be traced 19 in the shift of the decomposition of B10-, B20- and B30- CZA2 towards the one of 20 B100-CZA2 at high temperatures under ignition conditions (Fig. 2. (b2)), which gave 21 a prominent release of oxygenated compounds, such as ketones, in the FTIR (Fig. 3. 22 (b4) further justified the minimisation of the HC emissions.

Furthermore, the statistical analysis confirmed the profound effect CZA2 addition and load had on HC reduction when B10 was used (Table 4). The use of CZA2, load and synergy between the catalyst and load had a significant impact on the HC emissions from B20. From Table 4, the reduction of HC emissions with the use of B30-CZA2 compared to B30 was attributed mainly to the effect of the load and synergy of the catalyst with load.

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30 3.3.2 Carbon oxides

Carbon monoxide (CO) appeared reduced at all high loads by the addition of biodiesel
compared to D100 except at the lowest load (Fig. 7(b)) contrary to earlier reports
where CO emissions of same biodiesel blends appeared larger compared to those of
diesel [39]. The addition of CZA2 in D100 reduced CO emissions at high loads (Fig.

7(b)) affected by the addition of CZA2 (Table 4). CZA2 addition to B10 and B30
reduced CO emissions only at the middle loads and to B20 in the low loads (Fig.
7(b)). The effect of the load was pronounced for the reduction of CO emissions for all
biodiesel blends with CZA2, while the impact of CZA2 addition with the load on CO
production was significant for B20 and B30 (Table 4).

6 Carbon dioxide (CO_2) emissions from the catalytic combustion of all biodiesel blends 7 increased compared to D100 at the high and full load conditions (Fig. 7(c)). The 8 statistical analysis also indicated the significant impact CZA2, load and both CZA2 9 and load had on the increase of CO_2 from D100 (Table 4). The decrease of CO by the 10 addition of CZA2 into biodiesel blends was not accompanied by the same extent of 11 CO₂ emissions decrease. Such an outcome could signify two phenomena, which 12 occurred in parallel and consecutively: (i) the enhancement of combustion of 13 biodiesel blends directly to CO and CO₂, (ii) oxidation of produced CO to CO₂ by an 14 increase of the overall CO₂ emitted. The different possible mechanisms of aldehydes 15 and ketones formation via peroxides production were discussed in paragraph 3.1. 16 Organic peroxides indicated to catalytically reduce emissions possibly through the 17 formation of ketones [22]. Earlier research also showed that the catalytic oxidation of 18 aldehydes increased the rate of the termination step, through an increased peroxy 19 formation resulting in enhanced CO₂ production [40]. Therefore, CZA2 promoted 20 here the termination mechanism of combustion in the direction of CO₂ formation by 21 contributing towards oxygen delivery and producing more CO and CO₂ through the 22 mechanisms described in paragraph 3.1, therefore, reducing HC emissions. Increased 23 oxygen contact time with the tested fuels during combustion also reflected on the 24 marginal increase in the BP of all biodiesel blends with CZA2, consequently utilising 25 the energy of the fuels. The reduction of CO₂ emissions by addition of biodiesel 26 content in diesel with CZA2 was significant regarding load for B10, load and the 27 synergy of CZA2 and load for B20 and CZA2, load and combination of CZA2 and 28 load for B30.

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30 3.3.3 Oxides of nitrogen

Our results showed higher NOx emissions from D100 compared to B10, B20 and B30 at all loads except at full load (Fig. 7(d)) similarly to previous work [5, 9]. Blends with higher content in biodiesel produced less NOx at the low loads and more NOx than D100 at the full load. NOx production followed an increasing trend from low to high and full load conditions due to increasing adiabatic flames temperatures allowing
higher combustion temperature and time at full load while cooling was not sufficient
[5, 41].

4 D100-CZA2 minimised NOx emissions compared to D100 on an average by 8.04% at 5 all conditions. D100-CZA2 had the lowest NOx emissions compared to all biodiesel 6 blends-CZA2 at the low loads. The significant effect of load on the NOx emissions 7 between D100 and D100/CZA2 is shown in Table 4. The addition of organo-mono-8 metallic oxides to diesel showed enhancement of NOx emissions and reduction of the 9 specific consumption of fuel [42], whereas here we achieved reduction of NOx and 10 increase of BSFC by the use of CZA2.

11 B10-CZA2 generated slightly lower NOx emissions at 15 and 21 Nm, compared to 12 B10. B20-CZA2 produced less NOx than B20 at all loads, except at the highest load. 13 B30-CZA2 achieved an average reduction of NOx emissions at all loads by 13.34%. 14 FTIR spectra showed earlier that oxidation of biodiesel (unsaturated, oxygenated 15 compounds) with CZA2 produced more noticeable saturated (alkane) and ketone 16 peaks. Earlier research found that fuel with higher content of unsaturated compounds 17 produced more NOx compared to saturated compounds [12]. Therefore, the presence 18 of CZA2 catalysed the fuel blends resulting in their products combustion, which led 19 to reduced NOx emissions (Fig. 7(d)). The minimisation of NOx emissions could be 20 advanced due to a delay in the ignition [22]. It was described earlier that all biodiesel 21 blends with CZA2 under ignition conditions oxidised at a higher temperature than in 22 the absence of CZA2 (Fig. 2. (b2), paragraph 3.1). NOx emissions of both B10-CZA 23 and B20-CZA2 were mainly affected by the load while those from the use of B30-24 CZA2 by the addition of CZA2, load and the synergy of CZA2 and load (Table 4).

25 The reduced form of CZA2, through absorption of available oxygen during 26 combustion, expected the minimisation of NOx; when this was not possible at 27 extremely high combustion temperatures, a lessened catalytic activity during 28 combustion or fast kinetics of combustion surpassing the rate of the reduction 29 capability of the catalyst was expected. Fast combustion as depicted in the increased 30 BSFC at all loads of B30-CZA2 was expected to cause additional air pumped which 31 would promote oxidation of CO towards CO2 and N2 in the air towards NOx; the 32 latter did not occur. On the contrary, here CZA2, when dispersed in the fuel blends, 33 acted oxidatively and released oxygen by potentially setting the endothermic catalytic 34 combustion of biodiesel-diesel blends dominant at high load conditions. Therefore,

the decrease of both NOx and combustion carbon emissions revealed the simultaneous occurrence of the otherwise antagonistic mechanisms of reducing flames temperature and fast combustion. This effect could be possible through reducing the flame temperature due to the highly endothermic catalytic combustion reactions resulting in the minimised HC emissions (Fig. 7(a)) and hindering NOx formation (Fig. 7(d)). Adiabatic flames temperature was found to be lower when produced by combustion of saturated than of unsaturated molecules [12].

8 Overall, catalytic combustion of diesel and biodiesel-diesel blends by using CZA2 9 revealed its valuable oxidative character by promoting the consumption of heat 10 released by combustion of the highly endothermic catalytic reactions and production 11 of intermediate products leading to almost complete combustion with fewer HC 12 emissions and less favourable conditions for NOx formation.

13

14 **4.** Conclusions

This study explored the effect of nanocatalytic combustion with CZA2 on diesel and waste- origin biodiesel-diesel blends performance and emissions from an indirect injection diesel engine under various loads. Our experimental results showed the reduction of the emissions from diesel and B10, B20, B30 when using CZA2.

19 The presence of CZA2 enhanced the pyrolysis of all investigated fuels in the TGA 20 tests. All biodiesel blends (B10, B20, B30) behaved similarly to D100 at all 21 temperatures under ignition conditions. On the other hand, biodiesel blends behaved 22 similarly to D100 at all temperatures under the pyrolysis conditions with the 23 exception of B30, which thermally decomposed similarly to B100 over 230°C. FTIR 24 under pyrolysis and ignition conditions showed a different distribution of compounds 25 for diesel and biodiesel when dispersing CZA2. In particular, ketone bands were 26 identified via the formation of hydroperoxides, which could have enhanced the 27 combustion in the engine tests. However, definitive results concerning the specific 28 compounds formed by all tested fuels (D100, B10, B20, B30) with CZA2 during 29 combustion in the engine tests could not be established.

Fuel consumption increased for all tested fuels with the dispersion of CZA2, which was attributed to the improved combustion as evidenced by the reduction of the unburned hydrocarbons (HC) and carbon monoxide (CO) emissions from the exhaust. At the same time, the reduced nitrogen oxides (NOx) emissions suggested that CZA2

- 1 induced a trade-off between the mechanisms of NOx formation and close to complete
- 2 combustion.
- 3
- 4
- +
- 5

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| Technical parameters | Technical data |
|----------------------|---------------------------------|
| Engine type | 4-strokes diesel engine |
| Number of cylinders | 2 |
| Bore | 75 mm |
| Stroke | 77.6 mm |
| Dry weight | 66 kg |
| Compression ratio | 22.8 : 1 |
| Displacement | 686 cm^3 |
| Injection system | Indirect injection |
| Aspiration | Normal |
| Cooling system | Water cooled via heat exchanger |
| Rated speed | 3,600 rpm |
| Rated power | 10.7 kW @ 3600 rpm |
| Maximum torque | 40.5 N m @ 2000 rpm |

Table 1. Specification of test engine.

| Table 2. P | Percentage unc | ertainties and | errors of | instruments, | measurements | and pe | erformance | indicators | |
|------------|----------------|----------------|-----------|--------------|--------------|--------|------------|------------|--|
| | | | | | | | | | |

| • | Instrument | Emission | Measuring range | Error |
|---|------------|----------|-----------------|----------------|
| | AUTOplus | NOx | 0- 15,000 ppm | <u>+</u> 1 ppm |
| | automotive | | | |
| | | CO | 0-10% vol | $\pm 0.01\%$ |
| | | CO_2 | 0-16% vol | <u>±</u> 0.1 % |
| | | HC | 0- 5,000 ppm | <u>+</u> 1 ppm |
| | | | | |

| Measurements | Measuring range | Accuracy |
|--------------|-----------------|----------------------|
| Load | 0- 236 Nm | $\pm 0.2 \text{ Nm}$ |
| monitoring | | |
| Speed | 0- 7,000 rpm | \pm 15 rpm |
| measuring | | |
| Fuel | 0- 20,000 g | ± 2 g |
| consumption | | |
| (mass) | | |

| Analysis fuel | D100 | B10 | B20 | B30 | B100 |
|---|------------------|------------------|------------------|------------------|------------------------------|
| Standards | EN 590 | EN 16734 | EN 16709 | EN 16709 | EN 14214 |
| Kinematic viscosity (mm ² / s) | 2.824 | 2.957 | 3.109 | 3.231 | 4.535 |
| (@40°C) | | | | | |
| Test method and limits | EN ISO 3104 |
| | 2.000-4.500 | 2.000-4.500 | 2.000-4.620 | 2.000-4.650 | 3.500- 5.000 |
| Proximate | | | | | |
| Calorific value, upper | 45.49 | 44.68 | 44.22 | 43.68 | 40.33 |
| (MJ/kg) | DIN 51900-1 mod. |
| Test method and limits | No limit |
| Calorific value, lower | 42.62 | 41.75 | 41.61 | 40.96 | 37.67 |
| (MJ/kg) | DIN 51900-2 mod. |
| Test method and limits | No limit |
| Water content (mg/kg) | 44 | 71 | 75 | 100 | 177 |
| Test method and limits | EN ISO 12937 |
| | < 200 | < 200 | < 260 | < 290 | < 500 |
| Ultimate | | | | | |
| C (wt%-mol. fr.) | 84.34-0.329 | 84.11-0.333 | 83.28- 0.332 | 83.82-0.340 | 76.93-0.328 |
| H (wt%- mol. fr.) | 14.37-0.668 | 14.01-0.662 | 13.92-0.664 | 13.52-0.653 | 12.56-0.639 |
| N (wt%- mol. fr.) | < 0.5- 0.02 | < 0.5- 0.02 | < 0.5- 0.02 | < 0.5- 0.02 | < 0.5- 0.02 |
| S (wt%- mol. fr.) | 0.65-0.001 | 0.55-0.0008 | 0.57-0.0009 | 0.54-0.0008 | < 0.5- 0.0008 |
| K (µg/kg) | < 50 | N/A | N/A | N/A | < 50 |
| Test method and limits | EN 14108/ 14109 |
| Na (µg/kg) | < 50 | N/A | N/A | N/A | < 50 |
| $Ca (\mu g/kg)$ | < 0.5 | N/A | N/A | N/A | < 0.5 |
| | | | | | |
| Si (µg/kg) | < 0.5 | N/A | N/A | N/A | < 0.5 |
| Test method and limits | EN 14538 |
| | No limit | N/A | N/A | N/A | sum of K and Na < 5.0 mg/kg |
| | | | | | sum of Ca and Si < 5.0 mg/kg |

Table 3. Physical and chemical properties of diesel, diesel-biodiesel blends and biodiesel.

| Table 4. | Results | of ANOVA | analysis | |
|----------|---------|----------|----------|--|
| | | | | |

| | D100/D10 | 0-CZA2 | B10/B10-CZA2 | | B20/B2 | 20-CZA2 | B30/B30-CZA2 | |
|-----------------|----------|--------|--------------|-------|--------|---------|--------------|-------|
| BSFC | F | р | F | р | F | р | F | р |
| Fuel | 22.898 | 0.000 | 0738 | 0.399 | 0.087 | 0.771 | 39.885 | 0.000 |
| Load | 16.593 | 0.000 | 24.038 | 0.000 | 81.862 | 0.000 | 391.918 | 0.000 |
| Fuel×Load | 0.378 | 0.822 | 0.291 | 0.881 | 4.400 | 0.008 | 2.579 | 0.080 |
| BTE (%) | F | р | F | р | F | р | F | р |
| Fuel | 17.920 | 0.000 | 0.625 | 0.437 | 1.168 | 0.290 | 152.244 | 0.000 |
| Load | 4.975 | 0.004 | 8.202 | 0.000 | 53.954 | 0.000 | 647.937 | 0.000 |
| Fuel×Load | 0.643 | 0.636 | 0.323 | 0.860 | 3.643 | 0.018 | 10.837 | 0.000 |
| НС | F | р | F | р | F | р | F | р |
| Fuel | 21.124 | 0.000 | 5.619 | 0.026 | 12.311 | 0.002 | 0.132 | 0.721 |
| Load | 6.513 | 0.001 | 0.457 | 0.767 | 9.908 | 0.000 | 5.118 | 0.008 |
| Fuel×Load | 8.291 | 0.000 | 3.355 | 0.026 | 3.627 | 0.018 | 6.012 | 0.004 |
| СО | F | р | F | р | F | р | F | р |
| Fuel | 3.445 | 0.073 | 1.250 | 0.274 | 0.789 | 0.383 | 0.186 | 0.672 |
| Load | 0.526 | 0.717 | 3.954 | 0.013 | 7.176 | 0.001 | 19.138 | 0.000 |
| Fuel×Load | 1.536 | 0.217 | 1.888 | 0.144 | 7.942 | 0.000 | 8.462 | 0.001 |
| CO ₂ | F | р | F | р | F | р | F | р |
| Fuel | 4.615 | 0.040 | 0.111 | 0.742 | 1.461 | 0.238 | 25.007 | 0.000 |
| Load | 9.101 | 0.000 | 5.526 | 0.002 | 10.562 | 0.000 | 32.985 | 0.000 |
| Fuel×Load | 6.732 | 0.001 | 2.259 | 0.091 | 7.369 | 0.000 | 10.346 | 0.000 |
| NOx | F | р | F | р | F | р | F | р |
| Fuel | 1.478 | 0.234 | 0.477 | 0.496 | 0.060 | 0.808 | 20.461 | 0.000 |
| Load | 8.820 | 0.000 | 16.856 | 0.000 | 6.958 | 0.001 | 29.876 | 0.000 |
| Fuel×Load | 0.450 | 0.772 | 0.726 | 0.583 | 2.312 | 0.086 | 7.633 | 0.001 |



Fig. 1. Schematic diagram of engine test rig.





Fig. 2. TGA (a) pyrolysis and (b) oxidation curves of D100, B10, B20, B30 and B100 at heating rate, $\beta = 10^{\circ}$ C/min, without (1) and with the CZA2 dispersion (2).





Fig. 3. FTIR spectra at 150°C under (a) pyrolysis and (b) oxidation conditions of (1) D100, (2) D100-CZA2, (3) B100, and (4) B100-CZA2.



Fig. 4. Brake power (BP) of all tested fuels without and with the CZA2, at variable load conditions.



Fig. 5. Brake specific fuel consumption (BSFC) of all tested fuels without and with the CZA2, at variable load conditions.



Fig. 6. Brake thermal efficiency (BTE) of all tested fuels without and with CZA2 at variable load conditions.









Fig. 7. (a) Unburned hydrocarbon (HC), (b) carbon monoxide (CO), (c) carbon dioxide (CO₂) and (d) nitrogen oxide (NOx) emissions of all tested fuels without and with CZA2 at variable load conditions.