

Blade Failure on an Axial Compressor Caused by Unexpected Operating at Choke

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1	Background
2	Description of the machine
3	Findings
4	Root Cause Failure Analysis (RCFA)
5	Actions
6	Conclusion

Background

- Two parallel trains consisting of axial air compressors driven by steam turbines
- Application: blast furnace
- In operation since more than forty years (trouble free)
- Recently : shutdown (planed) of one train with inspection of the compressor.
- After three months of operation break-down of this compressor.
- Blades damaged
- RCFA performed:
 - Reasons of the damage
 - Countermeasures



Description of the machine

- Axial compressor, 19 stages
 - gas handled : wet air
 - p_s= 1.0 bara
 - p_d= 5.5 bara
 - V = 450,000 Nm³/h

- P = 40.7 MW
- n_{operation} = 3,200 rpm
- n_{min} = 2,856 rpm (89%)
- n_{max} = 3,360 rpm (105%)



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Rotor: Major damages on stages 16 to 19



Findings

 Stator: Major damages on stages 16 to 19 minor damages on blade carrier & lever system



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Root Cause Failure Analysis

 In order to clear up the compressor break down cause, following investigations were performed:

on-site inspection

- ➢ RCFA
 - Iaboratory analysis
 - timeline and operation data evaluations
 - > aerodynamic investigations

Root Cause Failure Analysis - Overview



RCFA – Laboratory analysis

- Blades failed/fractured in stage 18 & 19 (blades of stage 18 failed by material fatigue)
- Fracture initiation and propagation always on/from the backside of the blades (transition area airfoil-blade root)



RCFA – Laboratory analysis

- Deposits with corrosive potential
- In all deposits, remarkable sulphur and chloride - enrichments in various concentrations.
- Deposits contain dilutable elements
 - → corrosion of blade material (with moisture)
- Fracture surface Corrosion attack "Pitting" ck
- Local corrosion reduces material fatigue resistance
 - → corrosion fatigue caused by cracks with simultaneous influence of mechanical alternating load and of attacking medium.

RCFA – Timeline Evaluation, Operation Data Analysis

- February: completion of compressor installation (new blading installed)
- May : termination of commissioning
- July 11th : silencer failure
- July 31st : compressor breakdown

Operation data during compressor breakdown

13 >



May – June :

operation at chokepoint 1 for several hundred hours

standstill of compressor





Silencer failure leads to a sudden pressure drop and a resulting negative pressure wave hits the compressor blading.



Result of silencer failure:

operation at chokepoint 2 for approx. 8 hours (cumulative) from June 12th until July 31st

- ① 100% Speed (1.5h)
- ② Opening of IGV (4 min)
- ③ Increasing of backpressure (1s)
 - Silencer incident
- ④ Dropping of backpressure to 0.2 barg



choke operation: explanation



choke operation: explanation



→ specific flow conditions (stall), which can stimulate the natural frequencies of the blades



choke operation: first bending mode (B1)





RCFA – Breakdown causes



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In order to avoid any further compressor breakdown due to above mentioned reasons, 3 countermeasures were implemented:

- 1. realization of a choke control system
 - (see next slide)

RCFA – Countermeasures



- 2. improved resistance against overload and fatigue fractures with modified rotor blade design
- avoidance of corrosion attacks by application of an inorganic aluminum coating with chromate/phosphate inert sealant (rotor blades only)

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Conclusion

- The encountered case was the consequence of the concatenation of many factors:
 - Choke operation
 - High load on blades
 - Corrosion
- Not only the surge but also the operation in (deep) choke is an issue for the reliability of an axial compressor
- To prevent any future damage appropriate countermeasures are necessary:
 - Choke control system
 - Increased blade thickness
 - Application of coating

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Thank you very much for your attention

Questions?