



ASIA TURBOMACHINERY & PUMP SYMPOSIUM  
MARCH 2018 | SUNTEC SINGAPORE

# Failure, Repair and Overhaul of 5 MW Right Angle Reduction Gear Box

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# Presenter/Author

**Haseeb Bukhari:** The author has more than 7 years experience in maintenance and Reliability of Turbomachinery. The author has also written Technical Paper in other Journals as well. Author has carried out RCAs for equipment failures and have also been part of many Projects

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# Abstract

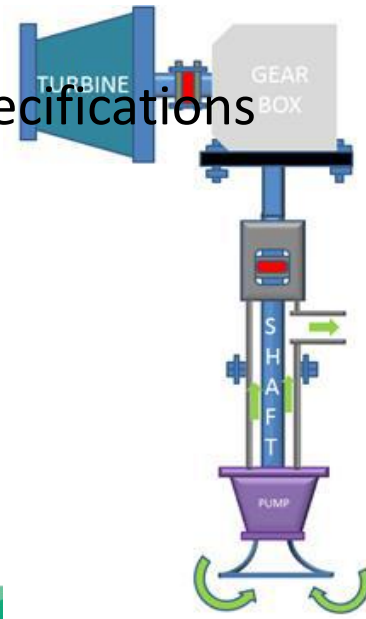
L-1403A & B are 5 MW Right Angle –Double reductions gearboxes installed between Turbine (TP-1403) and Cooling water Pump (P-1403). Both gearboxes reported high vibration after 15 months of continuous operation one after the other. Vibration had a continuous increasing trend. CSI® was used to check the gearboxes that both showed looseness spectrum.

During inspection of L-1403 B Gear Box, broken tooth of LS Bevel Gear was observed. While L-1403 A inspection revealed pitting on HS gears. Gearboxes were repaired in house by increasing interference of bevel pinion with shaft.

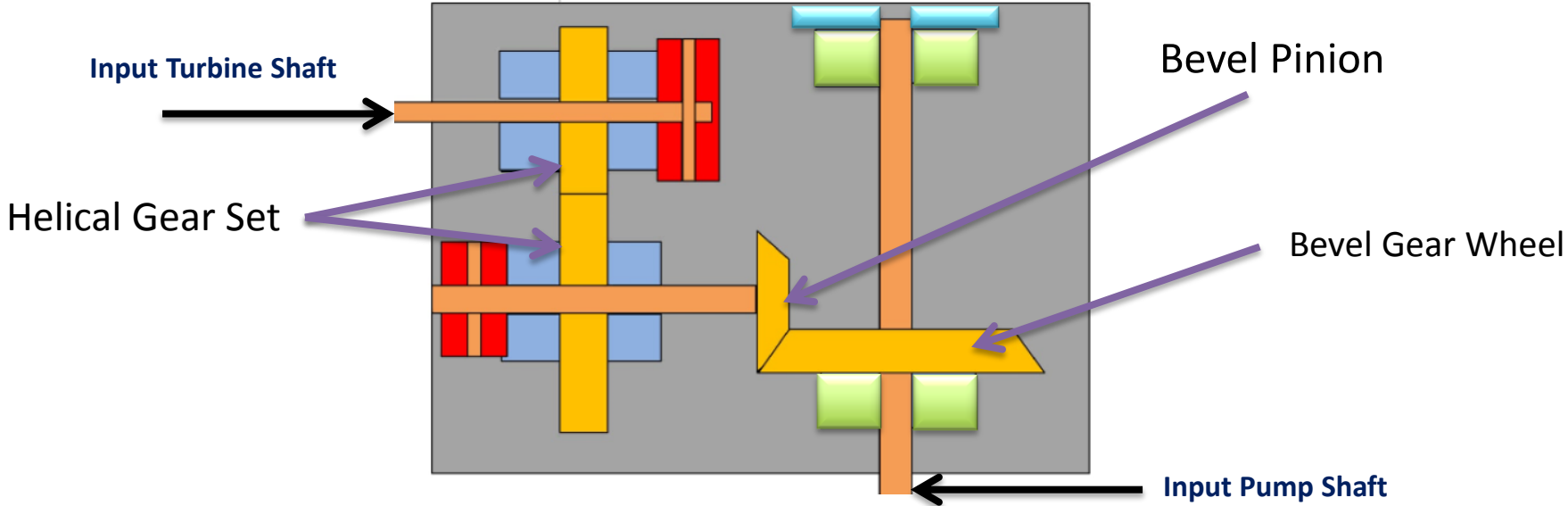


# Cooling Water Pump Schematic

- There are **four** Cooling water (CW) pumps
  - ✓ **P-1403 A/B** are Steam driven machine through Gearbox Reduction
  - ✓ **P-1403 C/D** are Motor driven. No issue/ abnormality with motor driven pumps
- **Operating philosophy:** Two steam driven and one motor driven pumps in service. Remaining one motor driven as standby
- L-1403A/B are cooling water gearboxes, bearing following specifications
  - **Power** – 5 MW
  - **Gear Types** - Bevel & Helical Gears (Reduction)
  - Gearbox Ratio – 1:8.9
  - **Bearings** – Tilting Pad & Ball Bearings
  - **Accelerometer on Casing**



# Gearbox Construction



Sleeve Bearing



Roller Bearings



Anti-Rotation Device



# Observations and Anomalies



- On **Jan,14** Abnormal noise from **L-1403A** gearbox and increased gear backlash was observed during window inspection
- Similar abnormal noise with lower extent observed from **L-1403B** (***damaged one***) Gearbox. Hence its inspection was planned for latter
- Surveillance Frequency was increased on Both pumps post this incident
- Accelerometer probes Vibrations reading on both increased but remained under Alarm limits



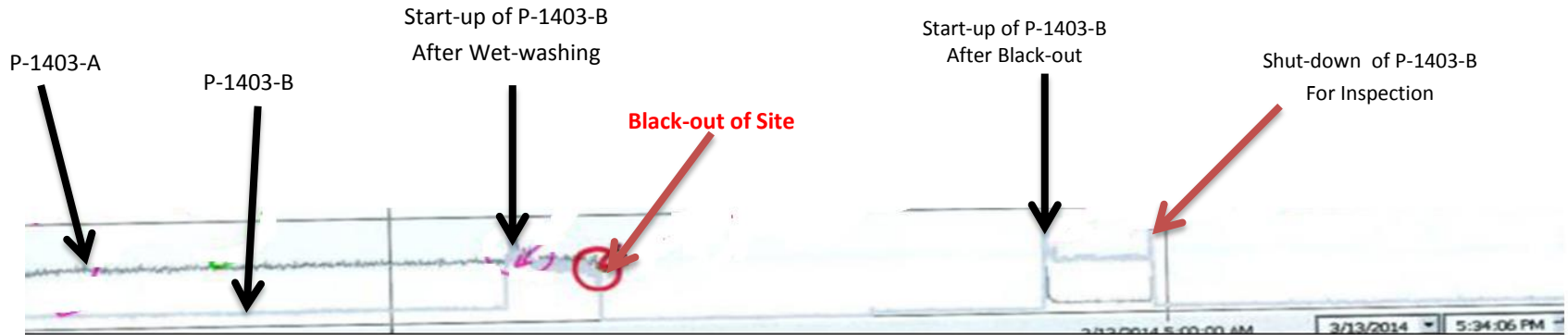
# Observations and Anomalies



- On March 12<sup>th</sup>, during startup , L-1403 B was reported to have very high vibration Data was collected using CSI<sup>®</sup>
- Data showed high vibration and failure of Gears. It was then immediately shutdown and inspected.



# Chronology of incident



On Mar 12<sup>th</sup>, 2014 after Power failure, both turbine driven pumps remained in operation & were tripped manually due to low steam pressure

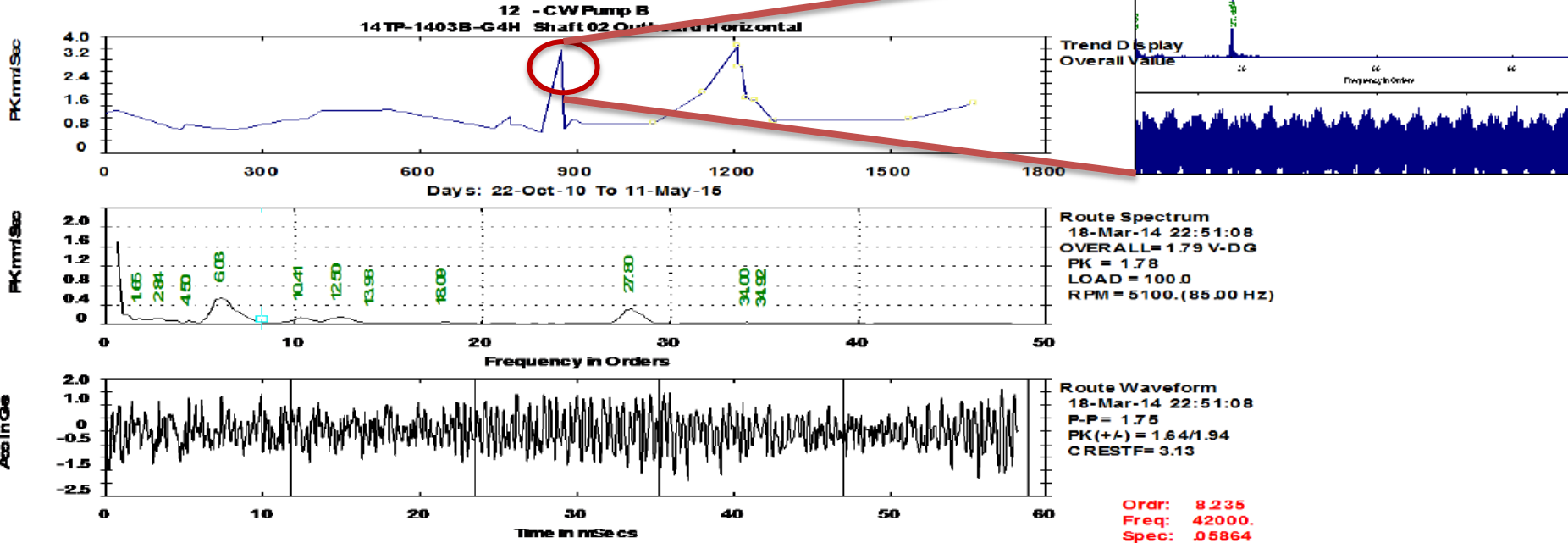


# Chronology of incident

- **Post blackout**, on restart of CW pumps
  - ✓ L-1403A gearbox noise & vibrations remained same as they were prior to Blackout
  - ✓ L-1403B (***broken tooth***) gearbox noise & vibrations increased & machine was decided to be inspect on 14<sup>th</sup> March
- L-1403-A was kept in service – for Business continuity – and inspected every 03 months



# CSI Trends



Typical trend of Broken tooth-Fish Tail wave forms

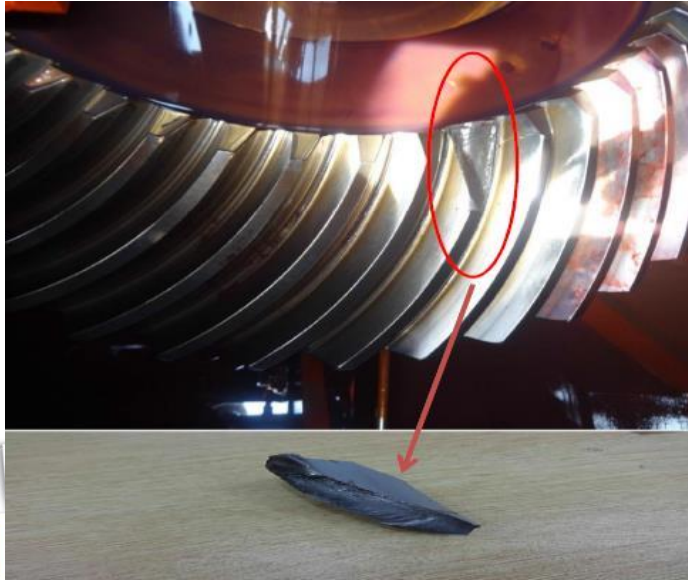


# Observation and Wear Pattern



# Bevel Pinion & Gear

L-1403-B, Broken LS  
Bevel Gear tooth



L-1403-B worn HS  
Bevel Pinion Teeth



# Key and Keyways



Bevel Pinion sitting Collar



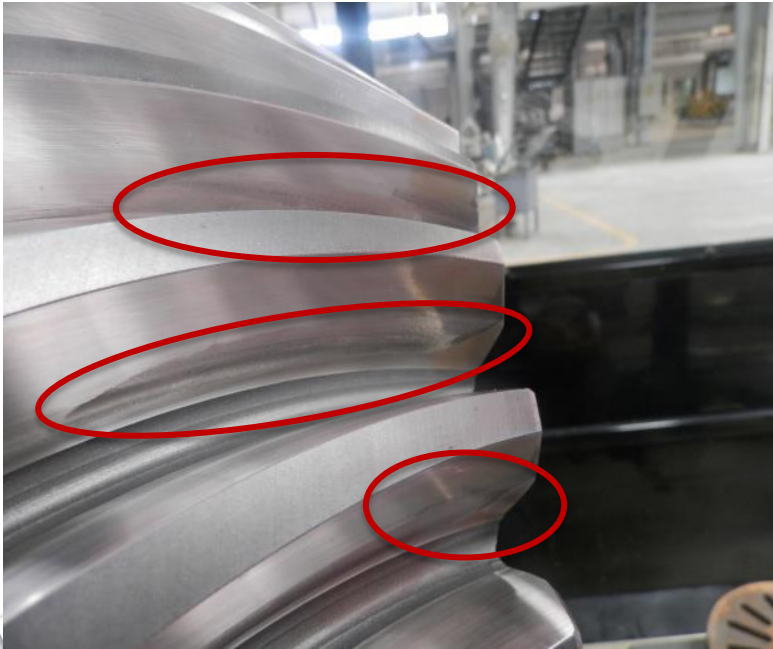
Extruded Keys – High Torque



Loose Bevel Pinion

Bevel Pinion (which meshes with LSS gear) was found **0.018"** loose on shaft (which ideally should be interface fit with the shaft)

# Bevel Pinions Wear



L-1403-A



L-1403-B

# Shaft Condition underneath Bevel Pinion

Fretting  
Corrosion



L-1403-A



L-1403-B

# Other Observations

- All Bearings – Normal
- Clearances in Bearings – Within Range
- Anti-Rotation Device – Normal
- Roller Bearings - Normal
- Casing Internals - Normal





# Root Cause Analysis



# RCA-Fretting Corrosion

- One of the critical observations taken during the removal of HS Bevel Pinion was the evidence of fretting in the Hub of Bevel gear and Shaft.
- "Fretting is a special wear process that occurs at the contact area between two materials under load and subject to minute relative motion by vibration or some other force." ASM Handbook



# RCA-Fretting Corrosion

- Fretting Corrosion aggravates in
  - Presence of lubrication
  - Cyclic loading
  - High Torque
- All of the above was present in this case.
- Case was then taken up with OEM for his review and in parallel in house repairs were done.
- Immediate solution to address this was to increase the Fit.



Fretting  
Corrosion

# In-house Repair Methodology

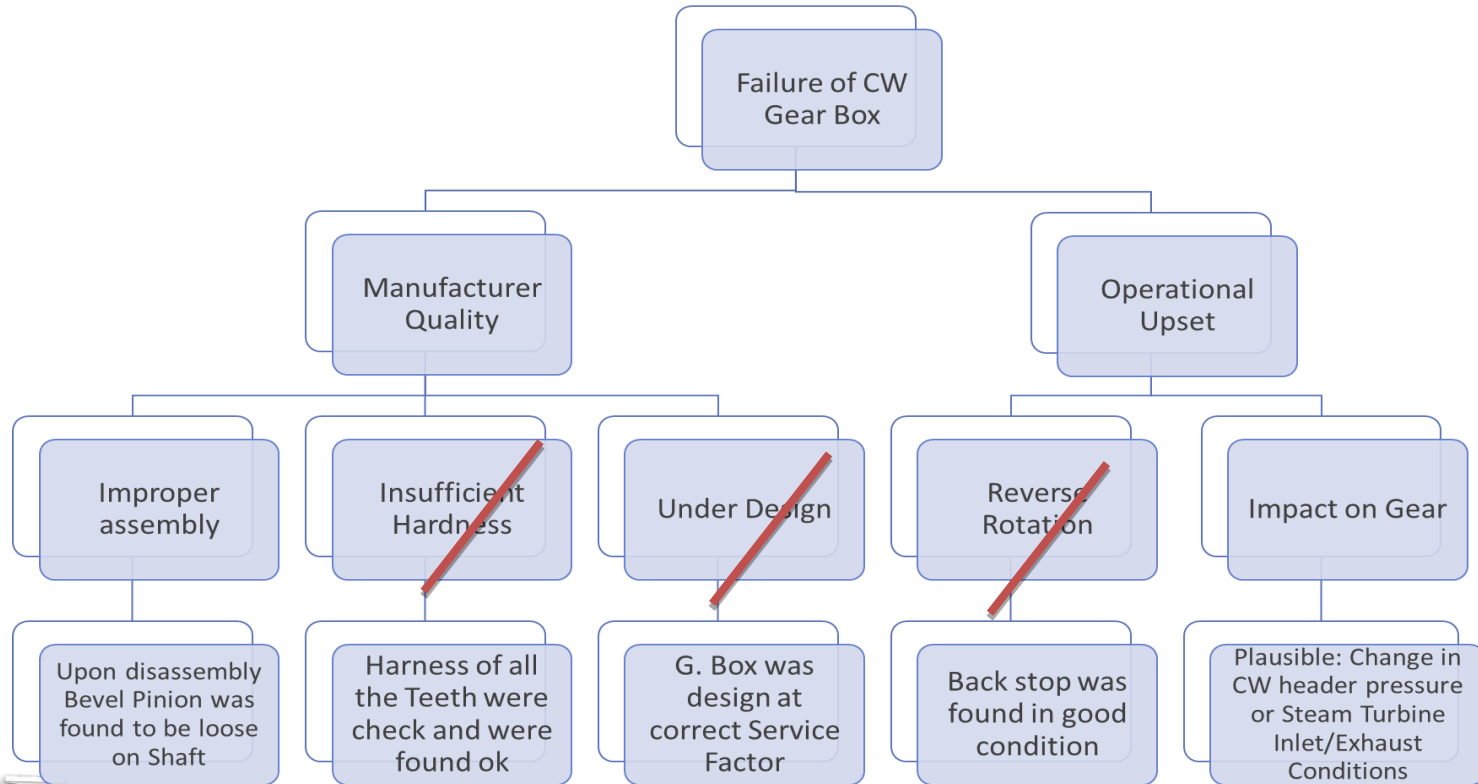
- Machining of Internal Diameter of Bevel Pinion to ensure uniform Diameter
- Build-up of Shaft using Electrode E-6010 followed by PWHT
- New Keys and Keyway Machining. Keys to be Snug fit
- DPT & MPI of all parts
- Increased interference to 0.003" between shaft and pinion
- Smoothing of broken tooth area on Bevel Gear.

Removing material thru pencil grinder and reuse Bevel

Gear.



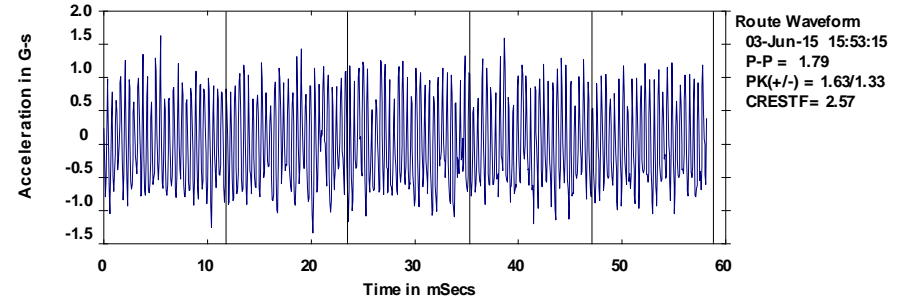
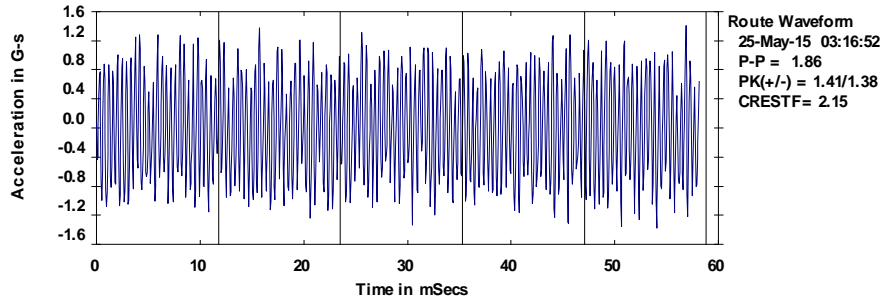
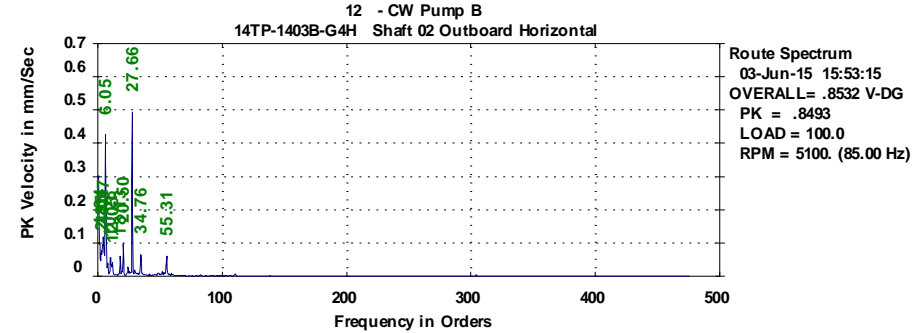
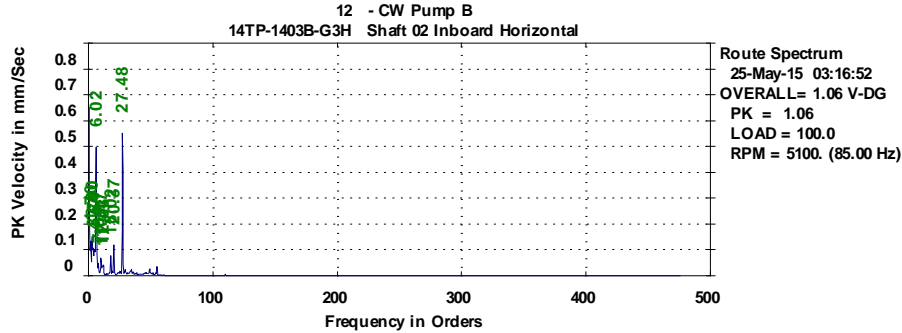
# RCA



K.F



# Pot Repair startup



## Analysis:

- The maximum vibration recorded post startup was less than 1 mm/sec RMS value. With PK-PK value of about 1.8 mm/sec.
- No impacting in waveforms



# Conclusion and Lesson Learnt



# Conclusion

- Loose pinion on shaft had resulted in repeated cyclic loading on the LS gear which eventually generated the high stress points on gear teeth
- During Startups, the initial torque on gears are normally high, therefore post black out, upon TP-1403B startup the high stressed portion on a LS gear teeth had given away, resulting in high noise & vibration
- Bevel gear pinion interference with the shaft was not adequate (which is a design Work-man ship related flaw). That further aggravates during the course of time & eventually resulted in failure.





# Conclusion

- OEM latter accepted the fault and increased tolerance from **270 H7/n5(transition fit)** to 270 H7 270 r6 shaft leads to a min interference of 0.0018” and a max of 0.005”. It was hence a design error.
- The gear set has been running from 03 years with no increase in vibration level and within acceptable noise levels



# Lessons Learnt

- Fit Type to be given due consideration during design phase
- FAT to be carried out to assess design considerations
- No reliance on Accelerometer
- Proper Vibration Monitoring Regime Development
- Use of Sound meter for Noise Monitoring
- Oil analysis for metallic particle through a Gear Box Return Oil drain to be made part of routine monitoring

