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BY C. N. ROSS, M.Sc., B.M.E., M.I.E., Aust.

4. The General Formulae of Moment-Distribution and their Application to Space Frames

BY DINO ANTHONY MORELLI, B.E.

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REDUNDANT FRAMES.

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Redundant Frames.

By C. N. Ross, M.Sc., B.M.E., M.I.E.Aust.*

Summary.—The author describes a modification of existing methods for finding the forces in the members of a redundant frame. By the use of trigonometrical resolution or otherwise, the forces in all members are written on the frame. Then each redundant bay in the structure is considered by *itself*, and from the strains in the six members of the bay an equation is formed which contains some of the unknown forces. Two numerical examples are used to illustrate its application.

NOTATION.

- l =length of a member.
- A = cross-sectional area of a member.
- E = Young's Modulus for the material.
- P = total force in a member caused by the load system.
- μ = force in a member caused by a unit load at the point where a deflection is required and in the same direction as the required deflection.

Method.

Let Fig. 1 be any redundant bay forming portion of a frame.



Fig. 1.

The difference of the deflections in the direction of L_1U_1 , of U_1 and L_1 relative to L_2 is equal to the strain of member U_1L_1 . Thus an equation is obtained connecting the six forces in the six members of the bay.

It is interesting to note that the deflections of any pair of adjacent nodes, in the direction of the line joining them, relative to either of the other nodes, can be used, thus giving eight different ways of forming the same equation.

Two more ways for obtaining this equation are to equate the relative deflections of two opposite nodes to the strain in the member joining them, e.g., the deflection of U_2 , in the direction U_2L_1 , relative to L_1 , is equal to the strain in U_2L_1 . The latter procedure will be used here.

Any suitable members are selected as the redundant ones and symbols are used for the unknown forces in these members. The forces in all the other members in the frame caused by the external load system, together with these unknown forces, are then found by trigonometrical resolution, or by some other standard method.

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Each redundant bay in the frame is then treated by itself, thus giving as many equations as there are unknown forces.



Since E has the same value for all the members in most frames, it is assumed to be equal to unity in the following numerical examples.

Example 1. The frame shown in Fig. 2 (a) is taken from *Deflections and Statically Indeterminate Stresses* by Hudson (p. 162).

The bays are 30 ft. x 30 ft. and the loads are in units of 1,000 lb.

Choose U_2L_1 and U_3L_2 as the redundant members and let the forces in them be $-x\sqrt{2}$ and $-y\sqrt{2}$ respectively as they are obviously both compressions. It is convenient to choose these forces so that their vertical components are x and y respectively.

From the given external forces and these assumed forces in the redundant members, compute the force in each member by trigonometrical resolution and write it on the frame as shown in Fig. 2 (a).

The forces in the members of a bay caused by a unit force in the direction of a diagonal member are shown in Fig. 2 (b).

The deflection of U_2 towards L_1 is equal to $\Sigma \frac{P \mu l}{A}$ of the bay $U_1 U_2 L_1 L_2$ for all members excepting $U_2 L_1$ of Fig. 2 (a), and the values of μ are given on Fig. 2 (b). Member l/A P μ $\frac{P \mu l}{\Delta}$

$U_1 U_2$ $L_1 L_2$ $U_2 L_1$	0.769 - 180 + x 1.364 90 + x 2.143 + x		97.9 — 86.75	-0.544x -0.965x -1.515x	
$U_2L_2 U_1L_2$	$3.333 x + y - 90 3.03 90 \sqrt{2} - x \sqrt{2}$	- 0.707 I.0	212.00 385.50	-2.355x - $-4.285x$	– 2.355 <i>y</i>

 $\therefore \text{ deflection of } U_2 \text{ towards } L_1 = 608.75 - 9.664x - 2.355y$ and this must be equal to the strain of U_2L_1 , that is to $x\sqrt{2} \times \frac{l}{A} = x\sqrt{2} \times 3.03 = 4.285x$

 $\therefore \ 608.75 - 9.664x - 2.355y = 4.285x$ or 13.949x + 2.355y = 608.75

The deflection of U_3 towards L_2 is equal to $\Sigma \frac{P\mu}{A}$ for all members of the bay $U_2U_3L_2L_3$ excepting U_3L_2 of Fig. 2 (a) and the values of μ are given on Fig. 2 (b).



 $P \mu l$

Member 1/A Р μ A U_2U_3 0.769 - 270 + y - 0.707 146.9 -0.544v L_2L_3 0.937 180 + y -0.707 - 119.2-0.662v $U_{2}L_{2}^{2}$ 3.333 x + y - 90 $U_{3}L_{3}^{2}$ 3.333 y- 0.707 212.0 - 2.355x - 2.355y- 0.707 $U_{3}L_{3}$ 3.333 y -2.355y U_2L_3 3.535 90 $\sqrt{2} - y\sqrt{2}$ I.0 449.5 -5.000y: deflection of U_3 towards $L_2 = 689.2$ and this must be equal to the strain of U_3L_2 , 689.2 - 2.355x - 10.916ythat is to $y\sqrt{2} \times \frac{l}{A} = y\sqrt{2} \times 3.535 = 5.00y$ $\therefore 689.2 - 2.355x - 10.916y = 5.00y$ or 2.355x + 15.916y = 689.2(2) or x = 292.7 - 6.76ySubstituting in (I)

4080 -94.2y + 2.355y = 608.75 $\therefore y = 37.8$ $\therefore x = 292.7 - 6.76 \times 37.8 = 37.3.$ Substituting these values for x and y in the forces shown on Fig. 2 (a), gives the actual forces in the members as

shown on Fig. 2 (c). *Example II.*—The frame shown in Fig. 3 (a), is taken from *Statically Indeterminate Stresses* by Parcel and Maney,

p. 115. Since the frame and loading are both symmetrical,

only one half of the frame is shown. The ratio of the length of a diagonal to a vertical to a horizontal member is as 1.28: 1.0: 0.8.

The loads are in units of 1,000 lb.

Choose U_1L_0 , U_2L_1 , U_3L_2 , and U_4L_3 as the redundant members and let the forces in them be -1.28w, -1.28x, -1.28y and -1.28z respectively, as they are obviously all compressions.

From the given external forces and these assumed forces in the redundant members compute the force in each member by trigonometrical resolution and write it on the frame as shown in Fig. 3 (a).

The forces in the members of a bay caused by a unit force in the direction of a diagonal member are shown in Fig. 3 (b).

The deflection of U_1 towards L_0 is equal to $\Sigma \frac{P\mu l}{A}$, for all members of the bay $U_0U_1L_0L_1$ excepting U_1L_0 of Fig. 3 (a), and the values of μ are given on Fig. 3 (b).

Member	l/A	Р	μ	$\frac{P \mu l}{A}$
$\begin{matrix} U_0U_1\\ L_0L_1\\ U_0L_0\\ U_1L_1\\ U_0L_1 \end{matrix}$	I0.4 I0.4 I2 I6.7 I9.2	$\begin{array}{r} -280 + 0.8w \\ 0.8w \\ -400 + w \\ -350 + w + \\ 448 - 1.28w \end{array}$	$ \begin{array}{r} -0.625 \\ -0.625 \\ -0.781 \\ x -0.781 \\ 1.0 \\ \end{array} $	$\begin{array}{r} 1,820 - 5.20w \\ - 5.20v \\ 3,750 - 9.37w \\ 4,565 - 13.04w - 13.04x \\ 8,600 - 24.58w \end{array}$
∴ deflec and this 1.28 <i>w</i> × ∴ 18,72 or 81.97	ction of must $\langle \frac{l}{A} =$ 35 - 7w +	of U_1 towards I be equal to th $1.28w \times 19$ 57.39w - 13 13.04x = 18	$L_0 = 1$ e strain of l_0 2 = 24.58u .04x = 24. $.735 \dots$	18,735 - 57.39w - 13.04x J_1L_0 , that is to 58w (1)

	The	deflection	of	U_2	towards	L_1	is	equal	to .	$\Sigma \frac{P \mu l}{A}$
for	all m	embers of	the	bay	$U_1 U_2 L$	${}_{1}L_{2}$ iven	exc	epting	U_2	L ₁ of
Fig	. 3 (a)	, and the v	alue	s of	f μ are g		on	Fig. 3	(b)	

Member l,	A P	μ	$\frac{P\mu!}{A}$			
U_1U_2 6	5 - 480 + 0.8	x - 0.625	1,800	-3.00x		
$L_1 L_2 = 6$	5 280 + 0.8	3x - 0.625	- 1,050	-3.00x		
U_1L_1 16	5.7 - 350 + w	+x - 0.781	4,565 - 13.05	5w - 13.05x		
U_2L_2 20	-250 + x -	+y - 0.781	3,905	-15.62x		
				— 15.62 <i>y</i>		
U_1L_2 24	320 — I.2	.8x I.0	7,680	-30.72x		
∴ deflectio	on of U_2 towards	$s L_1 =$	16,900 — 13.0	5w - 65.39x - 15.62y		
and this m	ust be equal to	o the strain of	U_2L_1 ,	2 0		
that is to a	$1.28x \frac{1}{A} = 1.28$	$x \times 24 = 30$	0.72x			
. 16,900 -	-13.05w - 65	5.39x - 15.62	y = 30.72x			
or 13.05w	+ 96.11x + 1	5.62y = 16,9		(2)		
for all members of the bay $U_2U_3L_2L_3$ excepting U_3L_2 of Fig. 3 (a), and the values of μ are given on Fig. 3 (b).						
Fig. 3 (a),	and the valu	es of μ are g	L_2L_3 excepting given on Fig.	$\begin{array}{c} g & U_3 L_2 & 0 \\ g & (b). \end{array}$		
Fig. 3 (a),	and the valu	es of μ are g	L_2L_3 excepting given on Fig. $P \mu l$	$\begin{array}{c} g \ U_3 L_2 \ 0 \\ g \ (b). \end{array}$		
Fig. 3 (a), Member l	and the valu	es of μ are μ	given on Fig. $\frac{P\mu l}{A}$	$g 0_3 L_2 $ of 3 (b).		
Fig. 3 (a), $\frac{Member}{U_2U_3} = \frac{l}{4}$	and the value A A P A P A	$\frac{\mu}{y - 0.625}$	$\frac{\frac{P \mu l}{A}}{1,838}$	$\frac{g}{3} = \frac{U_3 L_2}{(b)}$ or $\frac{U_3 L_2}{(b)}$		
Fig. 3 (a), $ \frac{Member l}{U_2U_3 4} $ $ \frac{U_2U_3 4}{4} $	and the valu A P 9 - 600 + 0.8 9 - 480 + 0.8	$\frac{\mu}{y - 0.625}$	$\frac{P \mu l}{A}$ 1,838 - 1,471	$\frac{g \ U_3 L_2 \ 0}{3 \ (b).}$		
Fig. 3 (a), Member $l/$ $U_2U_3 4$ $U_2L_3 4$ $U_2L_2 20$	and the valu A P -9 - 600 + 0.8 -9 - 480 + 0.8 -250 + x + 100	$\mu = \frac{\mu}{y - 0.625} - \frac{1}{y} - 0.781$	$2_{2}L_{3}$ excepting given on Fig. $\frac{P \mu l}{A}$ 1,838 - 1,471 3,910 - 15.62	$\frac{g}{3} = \frac{0.3L_2}{(b)}$ $\frac{1}{2.45y}$ $\frac{1}{2.45y}$ $x = 15.62y$		
Fig. 3 (a), Member $l/$ $U_2U_3 4$ $L_2L_3 4$ $U_2L_2 20$ $U_3L_3 30$	and the valu A P -9 - 600 + 0.8 -9 - 480 + 0.8 -250 + x + -250 + x + -150 + y + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -150 + -1	$\mu = \frac{\mu}{y - 0.625} - \frac{\mu}{y - 0.781} - \frac{\mu}{z - 0.781}$	$2_{2}L_{3}$ excepting given on Fig. $\frac{P \mu l}{A}$ 1,838 -1,471 3,910 - 15.62 3,516	$\frac{2.45y}{-2.45y}$ - 2.45y x - 15.62y - 23.43y		
Fig. 3 (a), Member l_{j} $U_{2}U_{3} + 4$ $U_{2}L_{3} + 4$ $U_{2}L_{2} + 20$ $U_{3}L_{3} + 30$	and the valu A $P-9 - 600 + 0.8-9 - 480 + 0.8-250 + x + 4-150 + y + 4$	$ \begin{array}{r} \nu & \nu & \nu & \nu & \nu \\ \nu & \nu & \nu & \nu \\ \nu & \nu &$	$\frac{P \mu l}{A}$ $\frac{P \mu l}{A}$ $\frac{1,838}{1,471}$ 3,910 - 15.62 3,516	$\begin{array}{r} g U_3 L_2 \text{ of } \\ 3 \text{ (b).} \\ \hline \\ \hline \\ \hline \\ - 2.45y \\ x - 15.62y \\ - 23.43y \\ - 23.43y \\ - 23.43z \\ \end{array}$		
Fig. 3 (a), Member l_{j} $U_{2}U_{3} + 4$ $U_{2}L_{2} + 20$ $U_{3}L_{3} - 30$ $U_{2}L_{3} - 32$	and the valu A $P-9 - 600 + 0.8-9 - 480 + 0.8-250 + x + -150 + y + -150 + y + -150 + y + -150 + y + -150$	$ \begin{array}{c} \text{bay} & 0 & 20 & 31\\ \text{es of } \mu & \text{are } g\\ \mu\\ y & -0.625\\ -y & -0.781\\ -z & -0.781\\ 8y & 1.0\\ \end{array} $	$2_{2}L_{3}$ excepting given on Fig. $\frac{P \mu l}{A}$ 1,838 - 1,471 3,910 - 15.62 3,516 6,141	$\begin{array}{r} & -2.45y \\ \hline & -2.45y \\ -2.45y \\ x - 15.62y \\ -23.43y \\ -23.43z \\ -40.95y \end{array}$		
Fig. 3 (a), Member $l/$ $U_2U_3 4$ $L_2L_3 4$ $U_2L_2 20$ $U_3L_3 30$ $U_2L_3 32$ \therefore deflection	and the value A P -9 - 600 + 0.8 -9 - 480 + 0.8 -250 + x + 1 -150 + y + 1 192 - 1.22 an of U_3 towards	$\mu = \frac{\mu}{y - 0.625}$ $\mu = 0.625$ $\mu = 0.781$	$2_{2}L_{3}$ excepting given on Fig. $\frac{P \mu l}{A}$ 1,838 - 1,471 3,910 - 15.62 6,141 13,934 - 15.62	$\begin{array}{r} g U_3 L_2 \text{ of } \\ 3 \text{ (b).} \\ \hline \hline \\ - 2.45y \\ - 2.45y \\ x - 15.62y \\ - 23.43y \\ - 23.43z \\ - 40.95y \\ \hline \\ x - 84.90y \\ - 23.43z \end{array}$		
Fig. 3 (a), Member l_{j} $U_{2}U_{3}$ 4 $U_{2}L_{3}$ 4 $U_{2}L_{2}$ 20 $U_{3}L_{3}$ 30 $U_{2}L_{3}$ 32 \therefore deflection and this mutanic	and the valu A P -9 - 600 + 0.8 -9 - 480 + 0.8 -250 + x + -150 + y + -150 + y + -150 + y + -192 - 1.21 -1	$\mu = \frac{\mu}{y_{1}^{2} - 0.625}$ $\mu = 0.625$ $\mu = -0.781$ $\mu = -2 - 0.781$	$2_{2}L_{3}$ excepting given on Fig. $\frac{P \mu l}{A}$ 1,838 -1,471 3,510 $-15.626,14113,934$ $-15.62U_{3}L_{2},$	$\begin{array}{r} g U_3 L_2 \text{ of } \\ 3 \text{ (b).} \\ \hline \\ \hline \\ - 2.45y \\ x - 15.62y \\ - 23.43y \\ - 23.43z \\ - 40.95y \\ \hline \\ x - 84.90y \\ - 23.43z \end{array}$		
Fig. 3 (a), Member $l/$ $U_2U_3 \qquad 4$ $U_2L_2 \qquad 20$ $U_3L_3 \qquad 30$ $U_2L_3 \qquad 32$ \therefore deflection and this muthat is to 1	and the value A P A P -9 - 600 + 0.8 -9 - 480 + 0.8 -250 + x + 1 -150 + y + 192 - 1.22 an of U_3 towards ust be equal to $-28y \times \frac{l}{A} = 1$	$\mu = \frac{\mu}{y - 0.625}$ $\mu = 0.781$	$2_{2}L_{3}$ excepting given on Fig. $\frac{P \mu l}{A}$ 1,838 - 1,471 3,910 - 15.62 6,141 13,934 - 15.62 $U_{3}L_{2},$ 40.95y	$ \begin{array}{r} g \ U_3 L_2 \ 01 \\ 3 \ (b). \\ \hline - 2.45y \\ - 2.45y \\ x - 15.62y \\ - 23.43y \\ - 40.95y \\ \hline x - 84.90y \\ - 23.43z \end{array} $		
Fig. 3 (a), Member $l/$ $U_2U_3 \qquad 4$ $L_2L_3 \qquad 4$ $U_2L_2 \qquad 20$ $U_3L_3 \qquad 30$ $U_2L_3 \qquad 32$ \therefore deflection and this muthat is to I \therefore 13,934 -	and the value A P A P A P A A P A A A A A A A A A A	$\mu = \frac{\mu}{y - 0.625}$ $\mu = 0.781$ $\mu = 0.$	$ \begin{array}{r} 2_{2}L_{3} & \text{excepting}\\ given on Fig. \\ \frac{P \mu l}{A} \\ 1,838 \\ -1,471 \\ 3,910 - 15.62 \\ 3,516 \\ 6,141 \\ 13,934 - 15.62 \\ U_{3}L_{2}, \\ 40.95y \\ x = 40.95y \\ x = 40.95y $	$ \begin{array}{r} g \ U_3 L_2 \ 01 \\ 3 \ (b). \\ \hline - 2.45y \\ - 2.45y \\ x - 15.62y \\ - 23.43y \\ - 23.43z \\ \hline x - 84.90y \\ - 23.43z \end{array} $		
Fig. 3 (a), Member $l/$ $U_2U_3 4$ $U_2L_3 4$ $U_2L_2 20$ $U_3L_3 30$ $U_2L_3 32$ \therefore deflection and this muthat is to I \therefore 13,934 - or 15.62x -	and the value A P A P	$\mu = \frac{\mu}{\sqrt{9} - 0.625}$ $\mu = 0.781$ $\mu =$	$ \begin{array}{r} 1_{2}L_{3} \text{ excepting}\\ \underline{P \ \mu l}\\ \underline{A} \\ 1,838 \\ - 1,471 \\ 3,910 - 15.62 \\ 3,516 \\ 6,141 \\ 13,934 - 15.62 \\ U_{3}L_{2}, \\ 40.95y \\ 4 = 40.95y \\ 34 - \dots \\ 35 - \dots \\ 34 - \dots \\ 35 - 15 - 15 - 15 - 15 - 15 $	$ \begin{array}{r} g U_3 L_2 01 \\ \hline 3 (b). \\ \hline - 2.45y \\ - 2.45y \\ - 23.43y \\ - 23.43z \\ - 40.95y \\ \hline x - 84.90y \\ - 23.43z \\ \hline \dots \dots$		

The deflection of U_4 towards L_3 is equal to $\Sigma \frac{P \mu l}{A}$ for all members of the bay $U_3 U_4 L_3 L_4$ excepting $U_4 L_3$ of Fig. 3 (a), and the values of μ are given on Fig. 3 (b).



Fig. 3 (b).



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Member	l/A	P		μ			
$\begin{matrix} U_{3}U_{4} \\ L_{3}L_{4} \\ U_{3}L_{3} \\ U_{4}L_{4} \\ U_{3}L_{4} \end{matrix}$	4.5 - 4.5 30 - 30 - 38.4	640 + 0.600	.8z	- 0.625 - 0.625 - 0.781 - 0.781 I.0	1,800 1,688 3,516 2,343 2,458	– 23.43 <i>y</i>	$\begin{array}{rrrr} - & 2.25z \\ - & 2.25z \\ - & 23.43z \\ - & 46.86z \\ - & 49.15z \end{array}$
: deflect and this that is to	tion of must b 0 1.28 <i>z</i>	U_4 towar be equal $\times \frac{l}{A} =$	ds $L_3 =$ to the s 1.28z	train of × 38.4 =	8,429 - U ₄ L ₃ = 49.152	- 23.43y -	– 123.94 <i>z</i>
∴ 8,429	- 23	43 <i>y</i> – 1	23.94z	= 49.1	5 <i>z</i>		
or 23.43	$y + \mathbf{r}$	73.09z =	8,429		•••••	•••••	(4)
15.62	x + 12	25.85y +	23.432	= 13	، 934		(3)
13.05	w + q	96.11x +	15.62y	, = 16	,900		(2)
	8	81.97w +	13.04 <i>x</i>	= 18	، 735		(1)
From substitut or 94.04 or $x =$	m (1) x ing in x x + 15 147.9	w = 228 (2) 2,998 (.62 $y = -0.166y$.6 — 0 — 2.07 13,902	x + 96	.11x +	15.62 <i>y</i> =	16,900
Substitut	ting in	(3) 2.310	- 2.50	$\sigma v + \tau 2$	25.85V +	- 23.432	= 13.034
or 123.2	6v + 2	$(3)^{-3} ($	11,624			~~	-3773-
or $y =$	94.3 -	0.19 <i>z</i> .					
Substitut	ting in	(4) 2,210	- 4.4	5z + 17	73.092 =	= 8,429	
or 168.6	4z = 6	5,219 or 2	ε = <u>3</u> 6	.9			
∴ y =	94.3	- 0.19	× 36.	9 = 8	7.3		
$\therefore x =$	147.9	- 0.166	× 87.	3 = 13	3.4		
∴ w =	228.6	- 0.159	× 133.	4 = 20	7.4.		

Substituting these values for w, x, y, and z in the forces shown on Fig. 3 (a) gives the actual forces in the members as shown on Fig. 3 (c).

CONCLUSION.

This method considers the forces caused by the load system and by the redundant members together, and is therefore shorter than the usual method.

All forces are shown on the members of the frame so that from the first step the procedure can be visualised. The sets of quantities from which the equations are formed have similar signs, thus eliminating the possibility of an error in sign. Obviously these signs change beyond the node where the shearing force changes sign.