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ase anary internal combustion engine with butanol

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> > **HYPOTHESIS**

### OBJECTIVES

- To study the behaviour of pure n-butanol as a fuel in a spark ignition engine
- To compare combustion analysis results in a variable compression ratio CFR research engine fuelled with butanol and other fuels as ethanol and commercial gasoline.
- To analyse the knocking behaviour of n-butanol in the CFR engine

### **Butanol properties**

Property	Gasoline	Ethanol	Butan
Carbon number	C4-C12	C2	C4
(RON*MON)/2	97	100	89
Density (kg/dm <sup>3</sup> )	0.736	0.790	0.810
L. heating value (kJ/kg)	43.0	26.8	33.0
Stoiquiom. A/F ratio	14.7	9.0	11.2
Boiling Temp. (°C)	24-215	78	118
Latent heat vap. (kJ/kg)	380-500	904	716
Laminar flame speed (cm/s)	51	63.6	58.5
Solubility in water (ml/100ml)	<0.1	Fully	7.7



### Single cylinder research engine:

- Variable compression ratio
- Electronic injection

**RESULTS**:

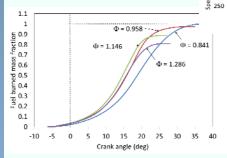
"Hook" curves.

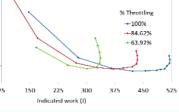
and fuel consumption

- Cylinder pressure sensor
- Labview data acquisition
- Mathlab data processing

# 850 ÷ 750 % Throttling 650

Indicated thermodynamic work calculated from cylinder pressure





## Combustion analysis

Fuel burned mass fraction computed from cylinder pressure analysis

Fuel	Ign. timing	W <sub>i</sub> (J)	Energy (J)	pmi (kPa)	g/kWh	$\eta_i$
Gasoline	10 deg	502	1492	821	253.5	0.337
Ethanol	15 deg	496	1433	810	388.5	0.3495
Butanol	10 deg	505	1474	825	313.9	0.343

550

150

ne] 350

### CONCLUSIONS

- n.-butanol is a suitable fuel for spark ignition engines
- Less heating value than gasoline increase absolute fuel consumption but increase
- But reduce indicated thermal efficiency ⇒ less specific fuel consumption.

# **METHODOLOGY**

wastes by cost - effective techniques.

ignition internal combustion engines.

"Hook" curves to asses n-butanol behaviour as spark ignited fuel

n-butanol is expected to be a suitable substitute of the ethanol as a biofuel for spark

n-butanol has demonstrated different physical and chemical properties than iso-butanol,

being both future biofuels candidates if it could be obtained from waste agro - food

- Combustion and Heat Release analysis based on cylinder pressure data recording was commanded by an external time base with angle encoder of 1800 pulses / revolution.
- Cold start only possible re-fuelling with gasoline
- Self-ignition analysis based on ignition timing for heavy knocking, based on cylinder pressure oscillations analysis along with knocking noise.

### Heat Release Analysis

Based on 1º law of thermodynamics for the gas inside the cylinder.

$$\frac{\Delta Q_{L}}{\Delta \alpha} - p \frac{\Delta V}{\Delta \alpha} + \dot{m}_{f} \cdot h_{f} = \frac{\Delta U}{\Delta \alpha}$$

 $\frac{\Delta Q_n}{\Delta \alpha} = \frac{\Delta Q_f}{\Delta \alpha} - \frac{\Delta Q_p}{\Delta \alpha} = p \frac{\Delta V}{\Delta \alpha} + \frac{\Delta U}{\Delta \alpha}$ The net heat release rate  $(\Delta Q_n)$  is:

Assuming ideal gas (Heywood, 1988), the heat released by the fuel is:

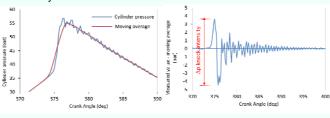
$$\frac{\Delta Q_{f}}{\Delta \alpha} = \frac{\Delta Q_{n}}{\Delta \alpha} + \frac{\Delta Q_{p}}{\Delta \alpha} = (1 + \frac{C_{v}}{R})p\frac{\Delta V}{\Delta \alpha} + \frac{C_{v}}{R} \cdot V \cdot \frac{\Delta p}{\Delta \alpha} + \frac{\Delta Q_{p}}{\Delta \alpha}$$

And the mass burning rate can be calculated

### **RESULTS:**

### Knocking combustion behaviour

Fuel burned mass fraction computed from cylinder pressure analysis



Pressure oscillation when knocking. Knock intensity definition.

Ignition angle to obtain 2 bar Δp knocking level

Fuel	O. N.	r = 7	r = 8	r = 9	r = 10
Gasoline	87 - 90	9	4	0	0
Ethanol	100	21	13	7	4
Butanol	86 -89	7	3	0	0

- Knocking behaviour is similar to gasoline
- But is worse than ethanol
- Lower volatility than gasoline and ethanol  $\Rightarrow$  cold start assistance

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