



タイトル Title	Improvement of separation performance by fluid motion in the membrane module with a helical baffle
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掲載誌・巻号・ページ Citation	Separation and Purification Technology,198:52-59
刊行日 Issue date	2018-06-08
資源タイプ Resource Type	Journal Article / 学術雑誌論文
版区分 Resource Version	author
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DOI	10.1016/j.seppur.2017.07.012
Jalcdoi	
URL	http://www.lib.kobe-u.ac.jp/handle_kernel/90004910

PDF issue: 2020-11-20

1	Title: Improvement of separation performance by fluid motion in the membrane module with
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#### 26 Abstract

27Pressure-driven membrane filtration processes such as microfiltration and ultrafiltration are still 28hindered by concentration polarization and membrane fouling. Generally in these filtration processes, 29concentration polarization causes decline of permeate flux and rejection, and fouling leads to permeate 30 flux decline with the increase of rejection. The use of high shear stress for cross flow filtrations has 31long been considered one of the most efficient methods for overcoming these problems. However, 32circumferential fluid motion of the hollow fiber membrane surface is also important to avoid formation 33 of a high concentration layer on the surface. In this study, ultrafiltration of humic acid aqueous solution 34using a polyethersulfone hollow fiber membrane was selected as a model case, and a membrane 35 module with a helical baffle installed around the membrane was used. With the insertion of the baffle, 36 normalized permeate flux and rejection became higher than those without the baffle at the wide range 37 of the feed flow rate. In order to identify the cause of the improvement, CFD simulation was conducted 38for different baffle geometries. Swirling flow motion generated by the helical baffle around the 39 membrane became more dominant with the lower aperture ratio of the cross sectional area, and there 40 existed the optimum value for the swirling flow generation in terms of the variation of the helical 41 baffle pitch length. The intensity of this fluid motion was characterized by Swirl number and it was 42found out that high separation performance was obtained at the high Swirl number.

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#### 51 **1. Introduction**

52Membrane filtration is applied to a wide variety of industrial fields, such as wastewater, food, 53pharmaceutical and petrochemical processing and so on. Among these, pressure-driven processes such 54as microfiltration and ultrafiltration have been still seriously hindered by concentration polarization 55and membrane fouling. Some of the components in the solution are rejected by the membrane, and the 56rejected components are concentrated at the upstream membrane surface. This is called concentration 57polarization which is often the reason for the serious limitation due to its negative influence on the 58transmembrane flux in microfiltration and ultrafiltration. Furthermore, the emergence of concentration 59gradient at the membrane and solution interface promotes the transfer of the rejected components 60 through the membrane and results in the decrease in the separation performance. Fouling is often the 61result of concentration polarization and could be described as adsorption on pore walls or 62 accumulation of foulant to form a second layer on the membrane surface which causes the decline of 63 permeate flux [1]. However, with the increase of the resistance to the flux, the fouling also leads to the 64 increase of the ability to reject the foulant due to the pore capacity decline or cake layer formation.

65 Hollow fiber membrane filtration is mostly carried out in a cross-flow manner because the axial 66 fluid flow along the membrane surface increases mass transfer from the membrane surface to the bulk 67 solution. This mass transfer improvement reduces the concentration of the concentration polarization 68 layer at the membrane surface and thus inhibits the fouling formation as a result of the concentration 69 polarization [1]. In addition, the axial flow generates shear stress at the vicinity of the membrane 70surface and is capable of removing the membrane fouling. Generally the flow rate of the processing 71fluid is quite high and the flow regime is turbulent to enhance the mass transfer and shear stress. This 72simple method is widely employed in industry but contains a problem that the high flow rate causes 73 the pressure drop increment leading to the high power consumption. However, there are also reports 74about inhibition of declines in performance by utilizing fluid motion such as vortex and periodic 75unsteady flow; applications of Taylor vortices [2], Dean vortices [3] and oscillating motions [4, 5].

76 The use of baffles in a tube is a simple way to induce specific flow fields [6-12]. Wide variety of 77 baffle configurations has been tested such as central baffles [10], rod baffles [11] and helical baffles 78[6-9]. With regard to the helical baffle, the flow visualization [13] and the effect of heat transfer by 79 changing baffle geometry were carried out experimentally [14]. It was also shown that the performance 80 for membrane filtration was improved and greatly affected by the baffle geometry inserted into the 81 membrane tube [6, 7]. The effects of feed concentration, net flow rate and transmembrane pressure 82 were investigated by flow analysis using CFD in the membrane module with a helical baffle [8, 9], 83 but the investigations in terms of the systematic changes of the baffle geometries were not carried out, 84 or in other words they focused on the comparison of the cases with and without baffles. In addition, 85 these studies were conducted under turbulent flow regime based on the standard operational condition [7-12]. Therefore, the flow patterns are so complex that a specific flow motion affecting on the 86 87 improvement the filtration performance haven't been sufficiently described.

In order to realize more efficient filtration processes, operation at a low flow rate is required from the aspects of low pressure drop or power consumption. By utilizing a specific fluid motion induced by a helical baffle high permeate flux and rejection can be achieved under low flow rate condition. In this study a cross-flow-type membrane module with a helical baffle around a hollow fiber membrane was employed and the relationship between fluid motion and filtration performance was investigated experimentally and numerically to identify the cause of the performance improvement.

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#### 95 **2. Materials and methods**

## 96 **2.1. Experimental setup and filtration medium**

Fig. 1 shows the experimental setup for ultrafiltration. A polyethersulfone (PES) hollow fiber membrane (molecular weight cut off (MWCO) = 150 kDa; inner diameter = 0.80 mm; outer diameter = 0.95 mm; length = 110 mm; effective surface area  $3.3 \times 10^{-4}$  m<sup>2</sup>) purchased from Daicen Membrane-Systems LTD. (Product No. FUS1582) was installed concentrically in a clear acrylic resin pipe with a

101 4 mm inner diameter. The membrane is asymmetric but has skin layers for filtration on the inner and 102 outer sides [15]. A helical baffle made of a bronze wire was inserted into the gap between the 103 membrane and the resin pipe. As for the water purification experiment, humic acid, a major 104 contaminant of drinking water, was selected as a model filtrated component in this study. The 105concentration of humic acid in the aqueous solution was fixed at 50 mg  $\cdot$ L<sup>-1</sup>, and pH was adjusted to 106 8.4 by adding sodium hydrogen carbonate to be 1 mM. Each filtration experiment was carried out by 107 forcing a feed solution to permeate from the outside to the inside of the hollow fiber membrane by 0.5 108 bar transmembrane pressure.

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Fig. 1 (a) Schematic of the external-pressure-type cross flow filtration module with a hollow fiber
membrane and a helical baffle. (b) Picture of the membrane module with a helical baffle

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### 114 **2.2. Evaluated experimental conditions**

An aperture ratio of the open cross-sectional area of the module,  $\Phi$  [%], and a baffle pitch length, b [m], were varied at 27, 34 and 53 %, and 2 – 12 mm, respectively. The aperture ratio can be varied by changing the diameter of the baffle wires and calculated by subtracting the projected areas of the membrane and helical baffle from the cross sectional area of the module. The baffle pitch length is a distance between the coils. The inner diameter of the module,  $d_m$  [m], was fixed at 4 mm, and the baffle pitch was normalized by being divided by  $d_{\rm m}$ . The effect of Reynolds number, Re, was also examined by changing the flow rate of the feed solution. All of the examined experimental conditions are listed in Table 1.

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Table 1 Experimental conditions and geometric dimensions of helical baffles

Reynolds number <i>Re</i> [-]	Pitch length <i>b</i> [mm]	Tube diameter d <sub>m</sub> [mm]	Baffle diameter d <sub>b</sub> [mm]	Aperture ratio $\Phi$ [%]	Trans membrane pressure [bar]
68 - 615	2 - 12	4	0.5, 0.85, 1.0	53, 34, 27	0.5

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# 126 **2.3** Methods to determine permeate flux and humic acid concentration

127A permeability was evaluated by a normalized permeate flux,  $J/J_0$  to eliminate the individual 128membrane variability. J and  $J_0$  were permeate fluxes when the humic acid solution and distilled water 129were processed, respectively. Samples permeated through the membrane and dripping out from the 130 one side of the membrane were collected and weighed to determine a permeate flux. A separation 131performance was represented by rejection,  $R = 1 - (C/C_0)$  where C and  $C_0$  are the concentrations of 132humic acid in the permeated samples and the feed solution, respectively. This is not an intrinsic 133rejection but an observed rejection. The concentration was determined by measuring absorbance at 134254 nm light [16] using a UV-vis spectrophotometer, SHIMADZU MPS-2400.

## 135 **2.4.** Numerical simulation of fluid motion in the membrane module

The numerical simulation was conducted using commercial CFD software, (R-flow, RFLOW Co. Ltd.) in order to observe the flow field in the module. The permeation through the membrane and transmembrane pressure were not considered in the calculation because the permeate flux was much lower than that of the feed flow. The geometry of the module used for the simulation was the same as the experiment shown in Fig. 1 and Table 1, and the properties of the fluid were assumed to be the same as water. The walls of the membrane surface, the inner surface of the cylinder and the outer surface of the helical baffle were considered to be non-slip condition. The pressure-velocity-coupling scheme was resolved with SIMPLE algorithm. Mesh was 24 in the radial direction, 1700 in the axial direction and 100 in the circumference direction. The governing equations used for the simulation are the conservation equations of momentum and mass are given as

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$$\frac{\partial \boldsymbol{u}}{\partial t} + (\boldsymbol{u} \cdot \nabla)\boldsymbol{u} = -\frac{\nabla p}{\rho} + \frac{1}{\rho} \nabla \cdot (\eta \nabla \boldsymbol{u}) + \boldsymbol{g}$$
(1)

$$147 \qquad \nabla \cdot \boldsymbol{u} = 0 \tag{2}$$

148 where  $\boldsymbol{u}$  is the fluid velocity, t is the time, p is the pressure,  $\rho$  is the density,  $\eta$  is the viscosity and  $\boldsymbol{g}$  is 149 the gravitational acceleration. In this simulation, the steady state was assumed. The validation of the 150 simulation and mesh sizes were conducted by comparing the pressure drops of the simulation results 151 and experimental results.

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### 153 **3. Results and discussion**

## 154 **3.1 Determination of fouling mechanism**

In order to examine the effect of fluid motion on fouling removal, it is necessary to comprehend the mechanism of the fouling deposition on the membrane surface in advance. Table 2 shows the list of the filtration rate equations expressing the 4 different blocking filtration models under constant pressure condition [17]. In the equations,  $(d\nu/d\theta)_0$  is the initial filtration rate and  $K_{cb}$ ,  $K_{sb}$ ,  $K_{ib}$  and  $K_c$ are constants for each model. These equations are originally from the equation (3)

160 
$$\frac{\mathrm{d}^2\theta}{\mathrm{d}v^2} = k_{\mathrm{p}} \left(\frac{\mathrm{d}\theta}{\mathrm{d}v}\right)^n \tag{3}$$

161 where  $\theta$  is a filtration time, *v* is a cumulative filtrate volume per unit effective membrane area,  $k_p$  and 162 *n* are constants. *n* can be determined corresponding to each model as shown in Table 2. The schematic 163 diagrams of these models were presented in the papers [17-19]. (a) is a complete-blocking model: 164 Particles larger than the membrane pores uniformly block pores on the membrane surface. (b) is a

165standard-blocking model: Particles smaller than the membrane pores deposit on the wall of the pores 166 and the pore capacity gradually decreases. (c) is an intermediate-blocking model: This is similar in 167that particles larger than the membrane pores uniformly block. Most of the pores are blocked at the 168initial stage while a lot of open pores exist, but as the blocking proceeds it becomes more difficult for 169this to occur due to the reduced number of pores. In order to describe the process, the model equation 170was constructed by Hermia (1982) based on a stochastic model [20]. (d) is a cake filtration. It is not 171pore blocking but cake formation on the membrane surface, and the filtration resistance increases 172gradually by the cake formation and its growth on the membrane surface. Although these 4 processes 173may occur simultaneously, the most dominant blocking mechanism can be determined by conducting 174the linear plotting of experimental data in accordance with the model equations.

175

176 Table 2 Filtration rate equations under constant pressure condition for various blocking filtration

177

models [17]

Blocking filtration law	n	Filtration rate equation
(a) Complete blocking	2	$\frac{\mathrm{d}v}{\mathrm{d}\theta} = \left(\frac{\mathrm{d}v}{\mathrm{d}\theta}\right)_0 - K_{\mathrm{cb}}v$
(b) Standard blocking	1.5	$\frac{\mathrm{d}\nu}{\mathrm{d}\theta} = \left(\frac{\mathrm{d}\nu}{\mathrm{d}\theta}\right)_0 \left(1 - \frac{K_{\rm sb}\nu}{2} \left(\frac{\mathrm{d}\nu}{\mathrm{d}\theta}\right)_0^{-1/2}\right)^2$
(c) Intermediate blocking	1	$\frac{\mathrm{d}v}{\mathrm{d}\theta} = \left(\frac{\mathrm{d}v}{\mathrm{d}\theta}\right)_0 \exp\left(-K_{\mathrm{ib}}v\right)$
(d) Cake filtration	0	$\frac{\mathrm{d}\theta}{\mathrm{d}\nu} = \left(\frac{\mathrm{d}\theta}{\mathrm{d}\nu}\right)_0 + K_{\mathrm{c}}\nu$

<sup>178</sup> 

Fig. 2 shows the experimental results modified with each of the model equations when the humic acid solution and polyethersulfone membrane were used without baffles. Each coefficient of determination was compared in the wide range of the cumulative filtrate volume per unit effective membrane area,  $v \, [m^3 \cdot m^{-2}]$ . The value for the cake filtration was the closest to 1 compared with the other 3 cases, and thus the cake filtration turned out to be a dominant model. This indicates that most of the foulant particles accumulated on the membrane surface. There are two possibilities to inhibit concentration polarization and fouling thus maintaining high filtration performance. The first is the removal of deposited foulant particles by a high shear stress due to a high flow rate. The other is to generate a flow which keeps foulant particles away from the membrane surface by replacing the fluid at the surface.

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Fig. 2 Linear fitting results with the 4 different filtration blocking models listed in Table 2 when the
humic acid and polyethersulfone membrane were used.

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## 194 **3.2 Time variation of permeate flux and rejection**

Fig. 3 shows that time variation of permeate flux and rejection for the cases without and with a baffle (Re = 68,  $b/d_m = 1.5$ ,  $\Phi = 34\%$ ). Only by inserting the helical baffle, the permeate flux and rejection were improved from the initial filtration time. As for the cake filtration mechanism, foulant

198is considered to be concentrated quickly on the membrane surface because the permeate flux is higher 199 at the initial stage. The concentrated foulant passes through the membrane with the driving force of 200the concentration gradient at the membrane and solution interface (i.e. concentration polarization) and 201causes the rejection decrease as a result. The rejection for the case with the helical baffle was much 202higher at the initial stage and kept higher value than that without the baffle. It indicates that the specific 203fluid motion induced by the helical baffle promoted mixing at the vicinity of the membrane surface 204and kept the foulant away from this region. In addition, the permeate flux also kept at higher value 205than that without the baffle. Since it is assumed that the cake layer leading to the permeate flux 206reduction might be formed after the concentrated foulant was adsorbed and accumulated on the 207 membrane surface, the mixing of the high concentrated region inhibited the formation of the cake layer 208as well and led to the preservation of higher permeate flux.

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Fig. 3 Time variation of the normalized permeate flux  $J/J_0$  and rejection *R* for the membrane module with the helical baffle (Baffle) and without the helical baffle (Control) under the condition *Re* = 68,  $b/d_m$  = 1.5 and  $\Phi$  = 34 %.

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# 215 **3.3 Effect of Reynolds number on filtration performance**

Fig. 4 shows the permeate flux and rejection against Reynolds number for the cases without and with the helical baffle ( $b/d_m = 1.5$ ,  $\Phi = 34\%$ ). Each of the plots was obtained at 180 min when the time 218variations of the permeate flux and rejection were considered to be stable, but it should be note that it 219didn't reach the inherent steady state. As for the case without the baffle, both of the permeate flux and 220rejection increased with Reynolds number increase. When the flow rate becomes higher, the mass 221transfer from the membrane surface of the feeding side can be promoted and leads to the reduction of 222the concentration polarization layer. This reduction of the concentration gradient between the feeding 223and permeation sides of the membrane resulted in the rejection increment with the mechanism 224explained in the previous section. Besides, in the case of pressure driven processes, concentration 225gradient between the feeding and permeation sides of the membrane causes an increase of the osmotic 226pressure gradient in the membrane, which reduces the net driving pressure gradient [21]. The reduction 227 of the layer thickness also increased the permeate flux by reducing this osmotic pressure. Another cause of the permeate flux increment was the high shear stress removing the accumulated foulant on 228229the membrane surface and maintaining the cake layer thinner. However, the flux and rejection 230improvement became lower at the higher flow because the streamlines were uniform along the axial 231direction of the membrane fiber, and even when the flow rate became higher, concentration 232polarization layer still existed to some extent which caused fouling development.

233On the other hand, for the case with the helical baffle, higher filtration performance was obtained 234at the low Reynolds number. Even when the case with the baffle at Re = 68 and the case without the 235baffle at Re = 615 were compared, the former showed higher filtration performance. This is because 236the specific fluid motion induced by the helical baffle gave a disturbance on the concentration 237polarization layer, and led to the reduced concentration. Additionally, since the highly concentrated 238region was difficult to be formed on the surface, the less amount of foulant could be adsorbed and 239accumulated. Thus the thick cake layer was not formed and the high permeate flux could be achieved. 240It is inferred that the specific fluid motion generated by the guide of the helical baffle was more 241effective on maintaining the high filtration performance than the high flow rate.

However, in the case with the helical baffle, the degree of the improvement obtained by the flow

rate (*Re*) increase was less than the case without the baffle. Although the major cause of the high performance at the low flow rate was the fluid motion induced by the helical baffle, the proportion of the by-pass flow passing through the gap between the baffle and membrane would be increased when the flow rate was higher. In other words, the fluid motion induced by the baffle became less dominant at the higher flow rate.





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Fig. 4 Dependency of Reynolds number on the normalized permeate flux  $J/J_0$  and rejection *R* for the membrane module with the helical baffle (B) and without the helical baffle (C) under the condition  $b/d_m = 1.5$  and  $\Phi = 34$  % after the 180 min filtration operation.

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### 254 **3.4** Numerical analysis of fluid motions with different baffle geometries

255In order to identify the specific fluid motion induced by the helical baffle and enhancing the 256filtration performance, the numerical investigation was carried out. Fig. 5 shows (a) the streamlines 257starting from the randomly selected initial positions and (b) the contour of the circumferential flow 258velocity to the counterclockwise direction in the cross section at the aperture ratios,  $\Phi = 27\%$  and 53% 259with the fixed pitch length of the helical baffle  $(b/d_m = 1.5)$ . It should be noted that (a) the dots on the 260baffle have no meaning and (b) the violet areas at the center and at the lower right are the membrane 261without considering the hollow space and the cross-section of helical baffle, respectively. At the lower 262aperture ratio, it was clearly seen that the streamlines had the similar trajectory as the geometry of the helical baffle. Additionally, high velocity region above the baffle in the circumferential velocity field could be observed. This specific fluid motion induced by the helical baffle is called as "swirling flow" in the following. When the higher aperture ratio of 53%, the swirling flow became less dominant compared with the axial component because the flow passed though the gap more easily.

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Fig. 5 (a) Streamlines starting from the randomly selected initial positions and (b) circumferential velocity fields in the cross section at the aperture ratios,  $\Phi = 27\%$  and 53% under the condition Re = 68 and  $b/d_m = 1.5$ .

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Fig. 6 also shows the streamline and the circumferential flow velocity in the cross section when the normalized pitch,  $b/d_m$ , were 1.0, 1.5 and 3.0 and the aperture ratio was constant at 38%. The number of wire coils around the membrane was adjusted by changing the pitch length. At the lower  $b/d_m$ , (i.e. more coils around the membrane), the streamline was predominantly axial, and swirling or rotational movement was weak. Conversely, at a higher  $b/d_m$ , the number of the rotations of the streamline was nearly equal to the coils of the helical baffle. In comparison of 1.5  $b/d_m$ , and 3  $b/d_m$ , coil rates, the streamline completed more rotations at 1.5  $b/d_m$ . Also, when examining the circumferential velocity, a higher velocity region above the baffle was observed at 1.5  $b/d_m$ , although there was not a significant difference. This shows that in order to maximize the number of rotations and rotation velocity of the fluid, there exists an optimum  $b/d_m$ .

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Fig. 6 (a) Streamlines starting from the randomly selected initial positions and (b) circumferential velocity fields when the normalized pitch  $b/d_m$  were 1.0, 1.5 and 3.0 under the condition Re =88 68 and  $\Phi = 38\%$ .

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#### 290 **3.5 Evaluation of performance using Swirl number**

Figs. 7 and 8 show the permeate flux and rejection when the aperture ratio was changed at the constant  $b/d_m$  and when the  $b/d_m$  was changed at the constant aperture ratio,  $\Phi$ , respectively. In Fig. 7, the lower the aperture ratio became, the higher permeate flux and rejection were achieved. The lower aperture ratio corresponds to the narrower gap between the baffle and the other surfaces and made the swirling flow stronger due to the reduction of the by-pass flow as shown in the previous section. The swirling flow generated the flow which was not parallel to the membrane surface, and this non-parallel flow might cause the mixing effect on the concentration polarization layer to decrease the 298concentration at the membrane surface even under the laminar flow condition. Additionally, since the 299fouling was formed as a result of the concentration polarization layer, the fouling was also suppressed 300 and the permeate flux showed a better performance at the low aperture ratio. In Fig. 8, the lower  $b/d_{\rm m}$ 301 caused the by-pass flow because it made more difficult to flow along the helical baffle, and thus led 302to the less number of the rotations of the streamline. Additionally, at the higher  $b/d_m$ , the number of 303 the helical baffle coils per unit length of the membrane was less, and the number of the rotations of 304 the streamline necessarily became less even though the by-pass flow was suppressed. Therefore, the 305 permeate flux showed the optimum value around  $b/d_m = 1.5$ . On the other hand, the decreasing 306 tendencies in the rejection at the lower  $b/d_m$  and at the higher  $b/d_m$  were less obvious than the case in 307 Fig. 7. It might be said that the circumferential velocity was lower than the case in Fig. 7 and could 308 not suppress the cake layer formation on the membrane surface, and this thicker cake layer also 309 rejected the foulant to increase the rejection. However, the mechanisms of the disturbance for the 310 concentration layer and cake layer were not elucidated sufficiently and further investigation would be 311required.

The swirling flow induced by the helical baffle could reduce the formation of the concentration polarization and cake layer rather than the high shear stress. Swirl number, *m* [-], was employed in order to evaluate both of the effects of flow rate and the baffle geometry. This dimensionless number is defined as the ratio of the axial flux of angular momentum to the axial flux of axial momentum [22]. It was originally proposed by Chigier and Beer [23] and simplified by Sheen et al. [24]. This represents the intensity of the swirling flow.

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$$m = \frac{\int_{0}^{R_{i}} \rho U V r 2\pi r dr}{R_{i} \int_{0}^{R_{i}} \rho U^{2} 2\pi r dr} = \frac{\int_{0}^{R_{i}} U V r^{2} dr}{R_{i} \int_{0}^{R_{i}} U^{2} r d}$$
(4)

where *r* [m] is a radial position, *R* [m] is a tube radius, U [m·s<sup>-1</sup>] is a axial fluid velocity, V [m·s<sup>-1</sup>] is a circumferential fluid velocity and  $\rho$  [kg·m<sup>-3</sup>] is a fluid density. The velocity components were obtained from the results of the CFD simulation in the section 3.4. Fig. 9 shows the relationship between Swirl number and the filtration performance. The permeate flux and rejection was raised with Swirl number increase. It is assumed that the high mixing intensity could be obtained due to the swirling motion of fluid at high Swirl number and inhibited the formation of high concentrated layer at the membrane surface. It is concluded that the filtration performance was improved by enhancing the intensity of swirling motion in the membrane module.

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Fig. 7 Effect of the aperture ratio on the normalized permeate flux  $J/J_0$  and rejection *R* after the 180 min operation under the condition Re = 68 and  $b/d_m = 1.5$  and the hollow marks indicates the results without the baffle.

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Fig. 8 Effect of the aperture ratio on the normalized permeate flux  $J/J_0$  and rejection *R* after the 180 min operation under the condition Re = 68 and  $\Phi = 34\%$  and the hollow marks indicates the results without the baffle.



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339Fig. 9 Dependency of Swirl number on the filtration performance after 180 min operation: Filled and340hollow marks represent the normalized permeate flux  $J/J_0$  and rejection R respectively.

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# 342 **4. Conclusion**

343 The effect of fluid motion on the filtration performance was investigated for the membrane module 344with the helical baffle. Humic acid solution and polyethersulfone membrane were used for the model 345filtration process, and the dominant fouling mechanism model was determined to be cake filtration. 346 With the helical baffle, the filtration performance became higher even at the low flow rate of the 347processing fluid than that without the baffle. Swirling flow induced by the helical baffle was more 348effective for suppressing concentration polarization and fouling than high flow rate such as turbulent 349 flow. The swirling flow has a characteristic flow pattern to increase the mixing effect around the 350membrane surface and reduce the deposition of the foulant. As the intensity of the swirling flow was 351characterized by the dimensionless number, Swirl number, the concentration and deposition of the 352foulant were reduced and the filtration performance was improved with the Swirl number increase.

353

## 354 Acknowledgments



356 The authors wish to thank Mr. Norihisa Kumagai for his experimental support.

358	Nome	enclature
359	b	pitch of helical baffles (m)
360	С	concentrations of humic acid in the permeate (kg m <sup>-3</sup> )
361	$C_0$	concentrations of humic acid in the feed solution (kg m <sup>-3</sup> )
362	$d_{ m m}$	inner diameter of the module cylinder (m)
363	g	gravitational acceleration (m s <sup>-2</sup> )
364	J	permeate flux (m <sup>3</sup> m <sup>-2</sup> s <sup>-1</sup> )
365	$J_0$	pure water permeate flux $(m^3 m^{-2} s^{-1})$
366	$k_{ m p}$	constant in Eq.(3) $(m^{n-2} s^{1-n})$
367	Kcb	constant in Table 2 (s <sup>-1</sup> )
368	$K_{\rm sb}$	constant in Table 2 ( $m^{-1/2} s^{-1/2}$ )
369	$K_{\rm ib}$	constant in Table 2 (m <sup>-1</sup> )
370	Kc	constant in Table 2 (s m <sup>-2</sup> )
371	т	Swirl number (-)
372	n	constant in Eq.(3) (-)
373	р	pressure (Pa)
374	r	radial position (m)
375	R	rejection (%)
376	$R_{ m i}$	inner radius of the membrane module cylinder (m)
377	t	time (s)
378	и	fluid velocity (m s <sup>-1</sup> )
379	U	axial fluid velocity (m s <sup>-1</sup> )
380	v	cumulative filtrate volume per unit effective membrane area (m <sup>3</sup> m <sup>-2</sup> )

381	V circumferential fluid velocity (m s <sup>-1</sup> )
382	
383	Greek letters
384	$\Phi$ aperture ratio of the open cross-sectional area (%)
385	$\rho$ fluid density (kg m <sup>-3</sup> )
386	$\mu$ viscosity (Pa s)
387	$\theta$ filtration time (s)
388	
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