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**DYNAMIC MODEL
OF A CONTINUOUS
COLD ROLLING MILL**

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Abstract

The dynamic model of a continuous cold reduction mill is developed. The model makes use of the force-torque equation expansion technique developed in an earlier report and presents the generalized $(7N+8)$ th order state equation for an N -stand rolling mill. The equations of interstand tensions are found to be highly nonlinear with the arguments delayed due to the transition of strip between the stands. The model developed can be used in the design of a controller for the entire rolling process.

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I. INTRODUCTION

In this report a nonlinear model of an N-stand rolling mill is developed. The model represents an improvement on previously proposed models [1,2].

Figure 1 shows an N-stand mill presented with coiler, decoiler, roll screw-down and drive motors.

As seen, the decoiler feeds the strip into the first stand where its thickness is reduced by a combination of interstand tension and roll separating force. The rolls of each stand are driven by a dc-motor which in turn is connected to its generator. The generators, the circuit breakers and emergency stopping circuits are not shown for simplicity of the diagram [3]. The drive motors can be controlled by both armature and field, while the screwdown motors are only armature controlled.

The model developed in this report will be used in the design of a controller for the rolling process. Each stand, coiler and decoiler are considered as dynamic subsystems coupled by an elastic strip. In general the coupled model is nonlinear and its order is high. Consequently, straight forward applications of optimal control theory would be virtually impossible and some approximation methods are needed. The development of this model takes into consideration properties of a class of near optimum design methods. In particular, it is adapted for an application of singular perturbation [4-6], decoupling methods [7], three time scale design method [8] and design of systems with time lags [9-12]. This model, although assumes the electrical features of

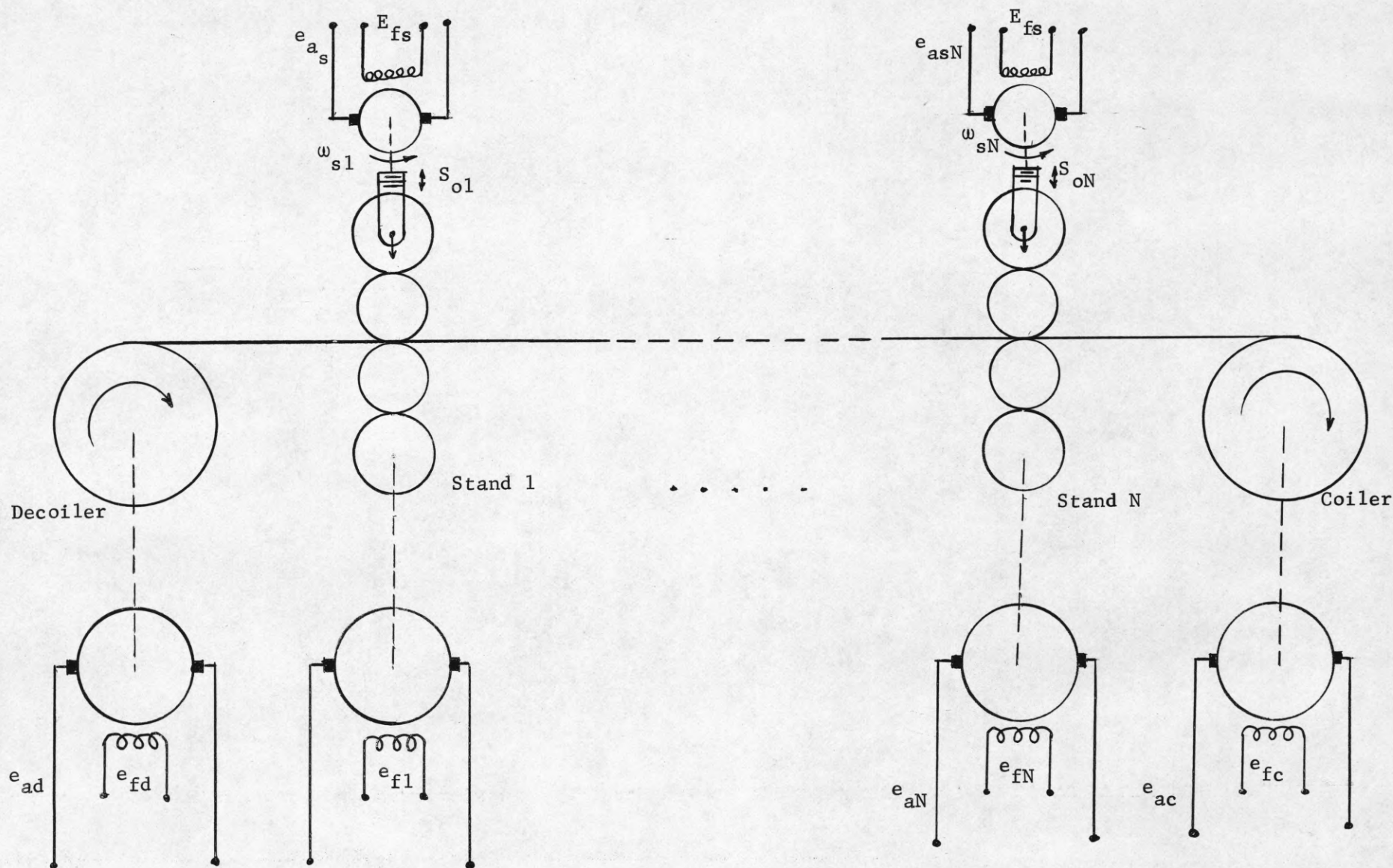


Figure 1. An N-stand continuous cold rolling mill.

Figure 1, remains general in its mechanical subsystems and formulation of state equations for the mill in interstand variables. The N-stand mill system of Figure 1 is divided into two major subsystems: mechanical and electrical.

2. MECHANICAL SUBSYSTEM

A brief review of rolling theory will explain the deformation process, interstand relations and other rolling equations relating mill process variables such as thickness, tension, torque, force, and coefficients of friction. Appendix 1 presents the notation and nomenclature for the entire report.

2.1 Deformation Process

In general, there is no "exact" rolling theory presented in the literature. The "exactness" of a rolling theory depends on assumptions involved. Thus far, the theory of Orowan [13] is most "exact". It permits variation of both yield stress and coefficient of friction. It is furthermore useful for both cold and hot strip rolling. Orowan's theory has been extended by Finne, et al [14]. However, due to its complexity, this theory is approximated by Bland and Ford [15].

The theory of Bland and Ford is used throughout our analysis. Using this theory several authors have developed graphical techniques and iterative methods for computations of force and torque to be applied to the rolls during the process [1,2,16-20]. In an earlier report we developed a computational procedure for determining the coefficients of a three-term Taylor series expansion of rolling

force and torque [21].

The deformation of strip takes place at each stand's roll gap.

A roll gap configuration is shown in Figure 2.

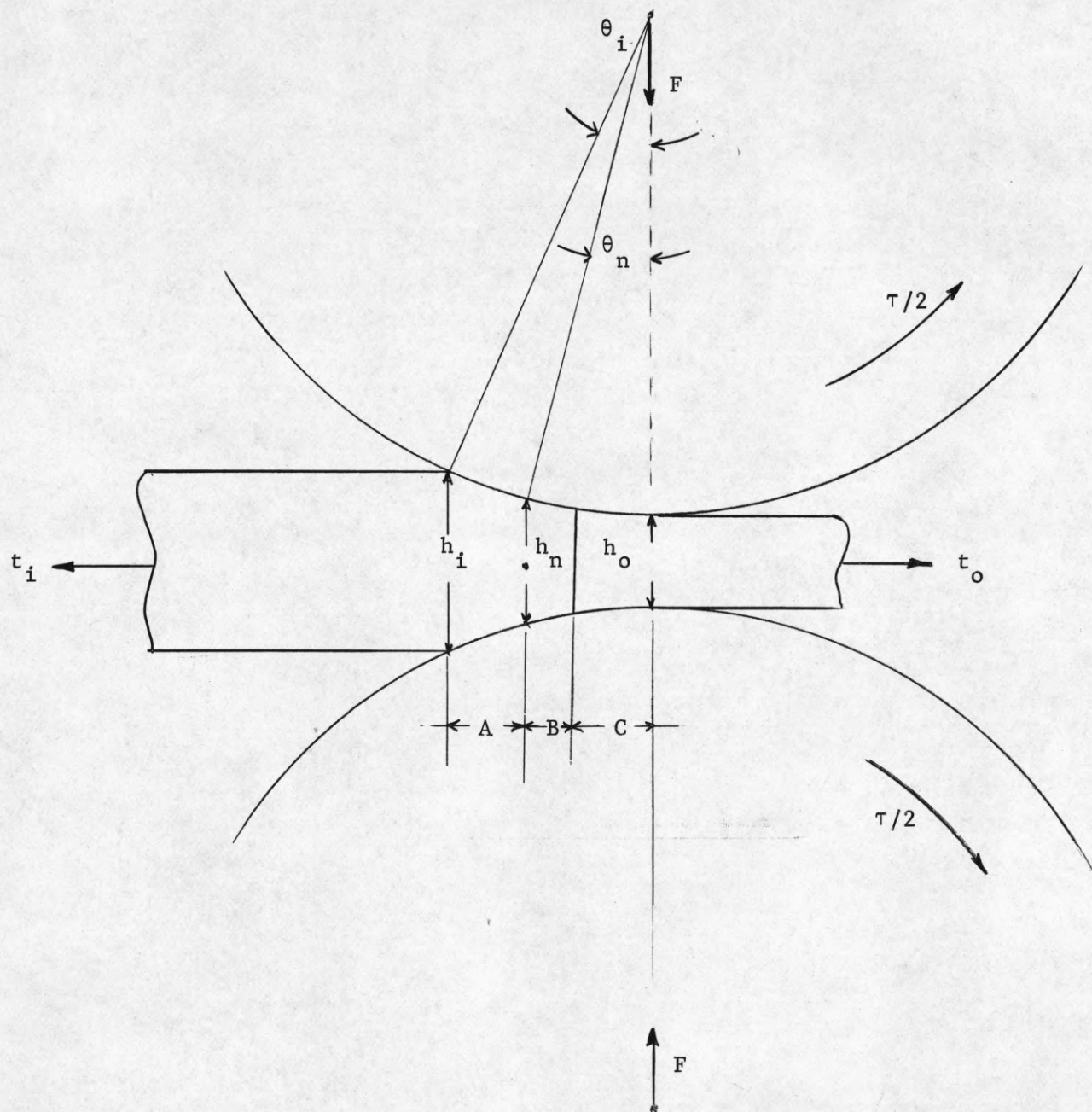


Figure 2. A roll gap configuration.

As seen the arc of contact is divided into three segments:

A is the elastic arc at the entry.

B is the plastic arc.

C is the elastic recovery arc at the exit.

In the two elastic regions the frictional force exerted by the rolls to the strip is commonly assumed to [13-15] be proportional to roll pressure with proportionality being the coefficient of friction. It is in these two regions where the rolling metal is said to be "slipping" [13-20]. In region B, however, the frictional force increases up to the magnitude of the yield stress in shear and a plastic shear occurs in the metal. The surface of the rolling metal is then said to be "sticking" to the rolls [13]. One of the basic assumptions made in [13] is that the deformation in the plastic region is a "homogeneous compression", i.e. the strip is divided into a number of vertical segments which remain plane and perpendicular to the direction of rolling. Three of these planes are shown in Figure 2. As seen from this figure, in the neighborhood of the entry plane these vertical segments (Planes) are "squeezed backward" while close to the exit plane they are "squeezed forward". In the middle of the gap there is a segment where the strip is being pulled out to its right and pushed back to its left. Such a segment is called the "neutral plane" [13,20].

The rolling theory of Bland and Ford calculates roll force and torque items of the "mill variables", h_i , h_o , t_i , t_o , and μ ,

$$F = F(h_i, h_o, t_i, t_o, \mu) \quad (2.1)$$

$$t = t(h_i, h_o, t_i, t_o, \mu) . \quad (2.2)$$

As it will become clear later a third equation for the neutral point thickness, h_n (See Figure 2) is also required.

$$h_n = h_n(h_i, h_o, t_i, t_o, \mu) \quad (2.3)$$

Analytic expressions for (2.1, 2 and 3) developed by Bland and Ford are given in Appendix 2.

These expressions are complicated nonlinear functions of mill variables and a simplification of their calculation is needed. In [18] a graphical method was proposed. Some computer iterative methods have been suggested [1,2], where small (1%) incremental changes of different mill variables are made to perform linearization of the force-torque equations. A computationally simpler and in the same time more accurate approximation is given in [21] where 2.1, 2 and 3 are expanded using explicit differentiation. The truncated series are

$$\hat{F} = \langle k, q \rangle + \langle q, Kq \rangle \quad (2.4)$$

$$\hat{t} = \langle l, q \rangle + \langle q, Lq \rangle \quad (2.5)$$

$$\hat{h}_n = \langle m, q \rangle + \langle q, Mq \rangle \quad (2.6)$$

where q is a vector with the mill variables as its components.

$$q^T = [\hat{h}_i \quad \hat{h}_o \quad \hat{t}_i \quad \hat{t}_o \quad \hat{\mu}], \quad (2.7)$$

k is a 5×1 normalized vector, $k_i = \frac{q_i^*}{F^*} \frac{\partial F}{\partial q_i}$ $i = 1, \dots, 5$ and K is a 5×5 symmetric matrix whose elements are, $k_{ij} = \frac{1}{2} \frac{q_i^* q_j^*}{F^*} \frac{\partial^2 F}{\partial q_i \partial q_j}$. Similar definitions apply for ℓ , L , m , and M for (2.5) and (2.6). The computational scheme developed in [21] consists of two main subroutines which allows all the mill coefficients to be obtained in less than 20 seconds on a CDC-1604 computer.

A common assumption in all rolling theories is that the volume of the material per unit time passing through each vertical segment is constant, i.e.

$$v_i h_i = \dots = v_n h_n = \dots = v_o h_o \quad (2.8)$$

2.2. Stand Relations

The output tension of the j th stand is assumed to be equal to the input tension of the $(j+1)$ th stand, i.e.

$$t_{i,j+1}(t) = t_{oj}(t) \quad (2.9)$$

The output tension is related to the strip velocity by the elasticity principle (Hook's law) [22-24],

$$\dot{t}_{oj} = \frac{E}{T} (\hat{v}_{i,j+1} - \hat{v}_{o,j}) \quad (2.10)$$

where it is assumed that the delay time T (transit time between stands j and $j+1$) is constant [14,18]. The input thickens at the $(j+1)$ th stand is the same as the output thickens at the j th stand delayed by T , i.e.

$$h_{i,j+1}(t) = h_{o,j}(t-T) . \quad (2.11)$$

As noted before when force is applied to the rolls an elastic deformation occurs to the "mill housing" and the rolls, i.e. the entire structure (stand and its four highs) is considered to be a stiff spring with a modulus of elasticity (spring constant) M' [1,2,25]. Thus,

$$F_j = M'(S_j - S_{o,j}) \quad (2.12)$$

where S_j is the screw-down setting (separation of rolls during rolling, positive upward) of the j th stand and $S_{o,j}$ is the screw-down setting when $F_j = 0$. If the elastic recovery of the metal is neglected as in the case in Bland and Ford's theory [1,2,15,25] then the output thickness $h_{o,j} = S_j$ and (2.12) in incremental form becomes [2],

$$\hat{h}_{o,j} = \left(\frac{1}{h_{o,j}} \right) S_{o,j} + \left(\frac{F}{Mh_{o,j}} \right) \hat{F}_j . \quad (2.13)$$

3. ELECTRICAL SUBSYSTEM

The electrical subsystem consists of the control rectifier motor groups for coiler, decoiler, stands, drives, and screw-down servomechanisms. In order to preserve generality, our model is derived with the assumption that the armature voltages of the motors are control variables. In a solution of an optimum control problem the dynamics of the generators or rectifiers will have to be included, and the motor armature voltages will become state variables.

In this section the dynamic equations of motors are given. A block diagram of both mechanical and electrical subsystems shows the interconnection between process variables.

3.1 Coiler and Decoiler

The coiler and decoiler are driven by a dc motor which is controlled by both armature and field voltages.

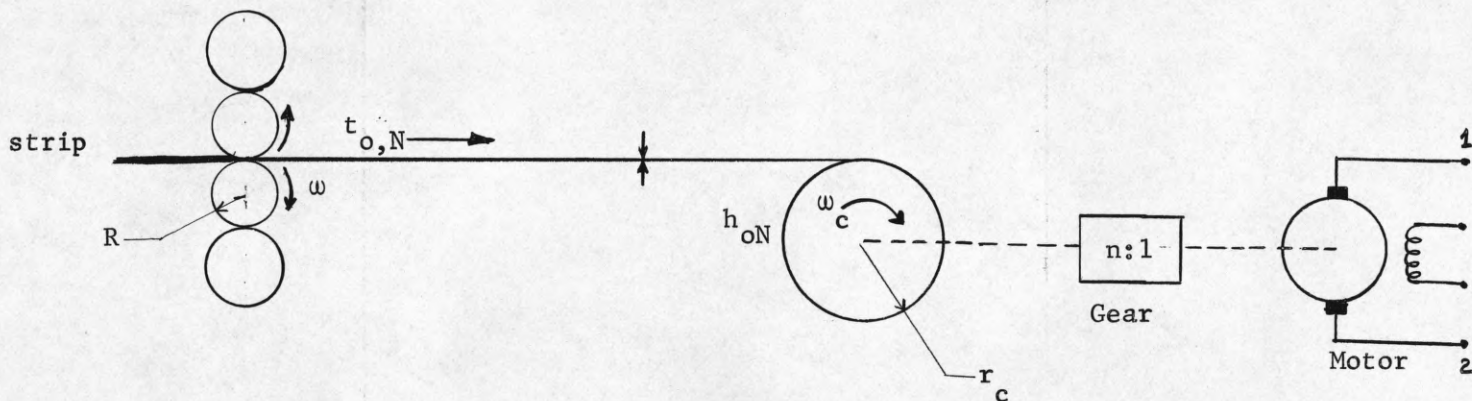


Figure 3. Coiler schematic.

The torque equation at the coiler's shaft, Figure 3, is [23,26].

$$\Sigma\tau = \tau_c = J_c \dot{\omega}_c + \dot{J}_c (\omega_c - \omega_N) + B_c \omega_c + \frac{1}{n} r_c t_{oN} \quad (3.1)$$

where the term $\dot{J}_c (\omega_c - \omega_N)$ accounts for the fact that the inertia of the coiler, J_c , is a time-varying function [27-29]. The term $\omega_c - \omega_N$ is the magnitude of the angular velocity of incoming strip relative to that of the coiler. The remaining relationships for the system in Fig. 3 are

$$\tau_c = K_{lc} \phi_c i_{ac} \quad (3.2)$$

$$e_{ac} = R_{ac} i_{ac} + L_a \frac{di_a}{dt} + K_{bc} \omega_c \quad (3.3)$$

$$e_{fc} = R_{fc} i_{fc} + N_{fc} \frac{d\phi_c}{dt} \quad (3.4)$$

$$\dot{r}_c = \frac{h_{oN}}{2\pi} \omega_c \quad (3.5)$$

$$i_{fc} = c_1 \phi_c + c_3 \phi_c^3 \quad (3.6)$$

$$J_c(r_c) = J_m + \frac{1}{n^2} J_L = J_m + \frac{1}{2n^2} \pi \rho W r_c^4 \quad (3.7)$$

$$\text{and } B_c = B_m + \frac{1}{n^2} B_L.$$

All the quantities are defined in the nomenclature (Appendix 1).

A similar set of equations can be written for decoiler by interchanging subscripts c (coiler) to d (decoiler) and N to 1. For example, as an equivalent to 3.1 one obtains,

$$\Sigma\tau = \tau_d = J_d \dot{\omega}_d + \dot{J}_d (\omega_1 - \omega_d) + B_d \omega_d - \frac{1}{n} r_d t_{i1} \quad (3.8)$$

where $\tau_d = K_{ld} \phi_d i_{ad}$, etc.

Note that a nonlinear magnetization relation, (3.6) is assumed between the field current and the flux of the motors.

3.2 Main Drive

The torque, armature and field circuit equations for the j th stand are presented. The stand's drive motor must provide torque sufficient to balance frictions rolling torque, torque caused by inlet tension and the acceleration torque. The torque caused by outlet tension is helping the rolls to rotate in the prescribed directions. Thus,

$$\Sigma \tau = \tau_m + \tau_o - J\dot{\omega} - \tau_f - \tau_i - \tau_j = 0 \quad (3.9)$$

where τ_m , o , f , i , j indicate the torque of motor, outlet tension, friction, inlet tension, and j th rolls, respectively. Equation (3.9) can be written for the j th stand as

$$\tau_{mj} = J\dot{\omega}_j + B\omega_j + \frac{1}{n} R t_{ij} = \frac{1}{n} R t_{oj} + \tau_j \quad (3.10)$$

where,

$$\tau_{mj} = K_1 \phi_j i_{aj} \quad (3.11)$$

The armature and field currents are governed by,

$$e_{aj} = L_a \frac{d(i_{aj})}{dt} + R_a i_{aj} + K_b \omega_j \quad (3.12)$$

$$e_{fj} = R_f i_{fj} + N_f \dot{\phi}_j \quad (3.13)$$

The field current-flux are related by the familiar magnetization characteristic approximated by third order polynomial,

$$i_{fj} = c_1 \phi_j + c_3 \phi_j^3 \quad (3.14)$$

Thus three variables, ω_j , i_{aj} , and ϕ_j ; are sufficient for a state space model of the main drive.

3.3 Screw-Down

The screw-down motors are assumed to be of the armature-controlled direct current type. The task of these motors is the adjustment of the vertical position of rolls (settings or screw pitch) [30]. The equations for the j th stand screw-down motor are,

$$\tau_{sj} = J_s \dot{\omega}_{sj} + B_s \omega_{sj} \quad (3.15)$$

$$\tau_{sj} = K_l \phi_{sj} i_{asj} \quad (3.16)$$

$$e_{asj} = R_a i_{asj} + L_a \frac{d(i_{asj})}{dt} + K_b \omega_{sj} \quad (3.17)$$

and the screw speed is assumed to be directly proportional to the angular velocity of the motor,

$$\dot{S}_{oj} = K_s \omega_{sj} \quad (3.18)$$

where S_{oj} is the screw pitch of the j th stand rolls.

3.4 Mill Block Diagram

The mechanical and electrical subsystems for the j th stand can be shown in a general block diagram as in Fig. 4. Note that for $j = 1$ and N , the diagram involves the decoiler or coiler motors, respectively and are considered as special cases of this diagram.

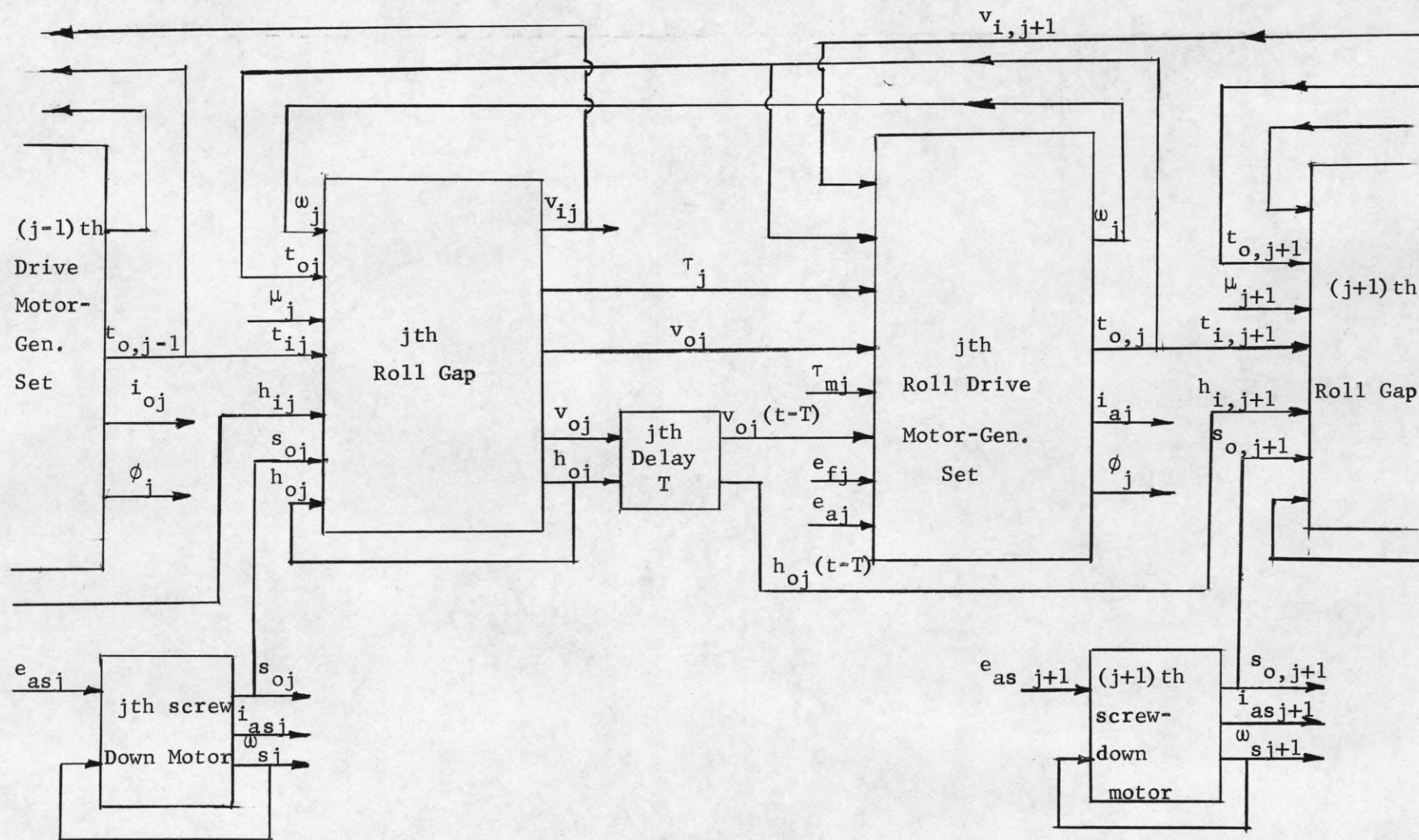


Figure 4. Mechanical-electrical subsystem block diagram.

Table 1 shows a possible selection of state, control and output variables of the system of Figures 1,2. For a single stand rolling mill, i.e., $j = N = 1$ there may be as many as $7 + 8 = 15$ state variables, $3 + 4 = 7$ control variables and 2 output variables.

Let

$$\begin{aligned}
 x_{1j} &= \omega_j & u_{1j} &= e_{aj} \\
 x_{2j} &= \tilde{t}_{0j} & u_{2j} &= e_{fj} \\
 x_{3j} &= i_{aj} & u_{3j} &= e_{asj} \\
 x_{4j} &= \phi_j \\
 x_{5j} &= \omega_{sj} \\
 x_{6j} &= S_{0j} \\
 x_{7j} &= i_{asj}
 \end{aligned} \tag{4.1}$$

Considering (2.4) and rewrite it as:

$$\begin{aligned}
 \hat{F}_j &= \langle \underline{k}, q_j \rangle + \langle q_j, \underline{K} q_j \rangle \\
 &= k_{22} \hat{h}_{0j}^2 + (k_2 + k_{12} \hat{h}_{ig} + k_{23} \tilde{t}_{ij} + k_{25} \mu_j) \hat{h}_{aj} \\
 &\quad + \langle \underline{k}, q_j \rangle + \langle q_j, \underline{K} q_j \rangle
 \end{aligned} \tag{4.2}$$

where

$$\underline{k} = \underline{I}k, \underline{K} = \underline{I}K, \underline{I} = \text{diag} [1 \ 0 \ 1 \ 1 \ 1],$$

note that vector k and matrix K are previously computed for stand j and

Table 1
System Variables in an N-Stand Rolling Mill

Type of Variable	Stand j			Decoiler	Coiler	Total no. of Variables
	Roll Gap	Drive Motor	Screw-Down			
State	$\tilde{\tau}_{0j}$	ω_j, i_{aj}, ϕ_j	$\omega_{sj}, S_{0j}, i_{asj}$	$\omega_d, i_d, \phi_d, r_d$	$\omega_c, i_c, \phi_c, r_c$	$7N + 8$
Control	-	e_{aj}, e_{fj}	e_{asj}	e_{ad}, e_{fd}	e_{ac}, e_{fc}	$3N + 4$
Output	\hat{h}_{oj} $\tilde{\tau}_{oj}$	-	-	-	-	$2N$

the subscript has been omitted on them for simplicity.

Eliminating \hat{F}_j from (4.2) and (2.13),

$$\beta_j k_{22} \hat{h}_{0j}^2 + [\beta_j (k_2 + k_{12} \hat{h}_{ih} + k_{23} \tilde{\tau}_{ij} + k_{24} \tilde{\tau}_{0j} + k_{25} \mu_j) - 1] \hat{h}_{0j} + (\alpha_j S_{0j} + \beta_j \langle \underline{k}, q_j \rangle + \beta_j \langle q_j, \underline{K} q_j \rangle) = 0$$

where $\alpha_j = (\frac{1}{h_{0j}^*})$, $\beta_j = (F_j^* / M h_{0j}^*)$ as defined in (2.13).

Solving for \hat{h}_{0j} from the above and considering (2.9) and notations of (4.1),

$$\hat{h}_{0j} = a_{3j} x_{2,j-1} + a_{4j} x_{2j} + \eta_j(t-T) + a_{5j} \quad (4.3)$$

where, a_{ij} , $i = 3, 4, 5$ and $\eta_j(t-T)$ are defined in Appendix 3.

Similarly torque and thickness equations of (2.5) and (2.6) after the substitution of \hat{h}_{0j} from (4.3) can be rewritten as,

$$\hat{t}_j = b_{3j} x_{2,j-1} + b_{4j} x_{2j} + \phi_j(t-T) + b_{5j} \quad (4.4)$$

b_{ij} , $i = 3, 4, 5$ and $\phi_j(t-T)$ are expressed in Appendix 3.

Similarly 2.6 can be reduced to,

$$\hat{h}_{nj} = C_{3j} x_{2,j-1} + C_{4j} x_{2j} + \psi_j(t-T) + C_{5j} \quad (4.5)$$

where C_{nj} , $n = 3, 4, 5$ and ψ_j have expressions identical with b_{nj} , $n = 3, 4, 5$ and ϕ_j when replacing \underline{l} and \underline{L} by m and \underline{M} , respectively.

To derive the first state equation let us write the torque equation at the shaft of the j^{th} stand drive motor,

$$\begin{aligned} J\dot{\omega}_j &= -B\omega_j + K_t i_{aj}\phi_j - \frac{R}{n} A_{ij} \tilde{\tau}_{ij} + \frac{RA}{n} \tilde{\tau}_{oj} - \\ &\frac{1}{n} \tau_j^* \hat{\tau}_j - \frac{1}{n} (R(t_{ij}^* - t_{oj}^*) + \tau_j^*) \end{aligned} \quad (4.6)$$

Substituting for $\hat{\tau}_j$ from (4.7)

$$\begin{aligned} \dot{x}_{1j} &= -\frac{B}{J} x_{1j} - \frac{1}{Jn} (b_{3j} \tau_j^* + RA_{ij}) x_{2,j-1} - \\ &\frac{1}{Jn} (b_{4j} \tau_j^* - RA_{0j}) x_{2j} - \frac{\tau_j^*}{Jn} \phi_j - d_{1j} \end{aligned} \quad (4.7)$$

where

$$d_{1j} = \frac{1}{Jn} (R(t_{ij}^* - t_{oj}^*) + (b_{5j} + 1) \tau_j^*) \quad (4.8)$$

To derive the state quantum for $\tilde{\tau}_{oj}$, consider equations (2.8), (2.10), and (2.11),

$$\begin{aligned} \hat{v}_{i,j+1} &= -1 + \frac{v_{i,j-1}}{v_{i,j+1}^*} = -1 + \frac{1}{v_{i,j+1}^*} \frac{v_{n,j+1} h_{n,j+1}}{h_{i,j+1}} \\ &= -1 + \frac{Rh_{n,j+1}^*}{v_{i,j+1}^* h_{i,j+1}} \frac{\omega_{j+1} (h_{n,j+1} + 1)}{(h_{0j} (t-T) + 1)} \end{aligned} \quad (4.9)$$

Similarly,

$$\hat{v}_{o,j} = \frac{v_{o,j}}{v_{o,j}^*} - 1 = -1 + \frac{Rh_{n,j}^*}{v_{o,j}^* h_{oj}^*} \cdot \frac{\omega_j (\hat{h}_{nj} + 1)}{(\hat{h}_{oj} + 1)} \quad (4.10)$$

then by (2.10)

$$\dot{x}_{2j} = g_{2j}(\underline{x}, \underline{x}(t-T), \underline{x}(t-2T)) \quad (4.11)$$

where g_{2j} is defined in Appendix 3. Note that by \underline{x} in (4.11) it is meant a vector containing all state variables as defined by (4.1).

The remaining state equations are simple and are written according to (3.12-18).

$$\dot{x}_{3j} = -\frac{R_a}{L_a} x_{3j} - \frac{K_b}{L_a} x_{1j} + \frac{1}{L_a} u_{1j} \quad (4.12)$$

$$\dot{x}_{4j} = -\frac{R_f e_1}{N_f} x_{4j} - \frac{R_f e_3}{N_f} x_{4j}^3 + \frac{1}{N_f} u_{2j} \quad (4.13)$$

$$\dot{x}_{5j} = -\frac{B_s}{J_s} x_{5j} + \frac{K_{is}}{J_s} x_{7j} \quad (4.14)$$

$$\dot{x}_{6j} = -K_s x_{5j} \quad (4.15)$$

$$\dot{x}_{7j} = -\frac{R_{as}}{L_{as}} x_{7j} - \frac{K_b}{L_{as}} x_{5j} + \frac{1}{L_{as}} u_{3j} \quad (4.16)$$

Thus 4.7, 4.11, 4.12 to 4.16 are seven state equations describing the dynamics of the j^{th} stand. In addition to the above, eight other equations are needed to describe coiler and decoiler;

let

$M = 7N$, N is the number of stands in the rolling mill installation, and

Decoiler:

$$x_{M+1} = \omega_d$$

$$x_{M+2} = L_{ad}$$

$$x_{M+3} = \phi_d$$

$$x_{M+4} = r_d$$

$$u_{2N+1} = e_{ad}$$

$$u_{3N+2} = e_{fd}$$

Coiler:

$$x_{M+5} = \omega_c$$

$$x_{M+6} = L_{ac}$$

$$x_{M+7} = \phi_c$$

$$x_{M+8} = r_c$$

$$u_{3N+3} = e_{ac}$$

$$u_{3N+4} = e_{fc}$$

(4.17)

The corresponding state equations are,

$$\begin{aligned} \dot{x}_{M+1} = & -B_d \frac{x_{M+1}}{J_d(x)} - 2Wph_{ij}^* \frac{x_{M+4} x_{11}}{J_d(x)} \\ & + 2Wph_{i1}^* \frac{x_{M+1}^2 x_{M+4}^3}{J_d(x)} + \frac{A_{i1}}{n} \frac{x_{M+4} x_{M+5}}{J_d(x)} \\ & + K_{1d} \frac{x_{M+2} x_{M+3}}{J_d(x)} \end{aligned} \quad (4.18)$$

where

$$\begin{aligned} J(x) = & J_m + \frac{1}{2n^2} \pi \rho W x_{M+4}^4 \\ \dot{x}_{M+2} = & \frac{R_{ad}}{L_{ad}} x_{M+2} - \frac{K_{bd}}{L_{ad}} x_{M+1} + \frac{1}{L_{ad}} u_{3N+1} \end{aligned} \quad (4.19)$$

$$\dot{x}_{M+3} = -\frac{R_{fd1}}{N_{fd}} x_{M+3} - \frac{R_{fd3}}{N_{fd}} x_{M+3}^3 + \frac{1}{N_{fd}} u_{3N+2} \quad (4.20)$$

$$\dot{x}_{M+4} = \frac{h_{i1}}{2\pi} x_{M+1} \quad (4.21)$$

$$\begin{aligned} \dot{x}_{M+5} = & -B_c \frac{x_{M+5}}{J_c(x)} + 2Wph_{oN}^* \frac{x_{M+8}^3 x_N}{J_c(x)} + \\ & 2Wph_{oN}^* \frac{x_{M+5}^2 x_{M+8}^3}{J_c(x)} - \frac{A_{oN}}{n} \frac{x_{M+8} x_{2N}}{J_c(x)} + K_{1c} \frac{x_{M+8} x_{M+7}}{J_c(x)} \end{aligned} \quad (4.22)$$

when

$$J_c(x) = J_m + \frac{1}{2n} \pi \rho W x_{M+8}^4$$

$$\dot{x}_{M+6} = -\frac{R_{ac}}{L_{ac}} x_{M+6} - \frac{K_{bc}}{L_{ac}} x_{M+5} + \frac{1}{L_{ac}} u_{3N+3} \quad (4.23)$$

$$\dot{x}_{M+7} = -\frac{R_{fc}^c}{N_{fc}} x_{M+7} - \frac{R_{fc}^c}{N_{fc}} u_{3N+3} \quad (4.23)$$

$$\dot{x}_{M+7} = -\frac{R_{fc}^c}{N_{fc}} x_{M+7} - \frac{R_{fc}^c}{N_{fc}} x_{M+7}^3 + \frac{1}{N_{fc}} u_{3N+4} \quad (4.24)$$

$$\dot{x}_{M+8} = \frac{h_{oN}^*}{2\pi} x_{M+5} \quad (4.25)$$

The above equations as well as those for the j^{th} stand can now be written in a vector form for an N -stand rolling mill as follows,

Let

$$\dot{x} = Ax + Bu + f(x) + g(x, x(t-T), x(t-2T)) + d(t)$$

where

A $7N + 8 \times 3N + 8$ constant matrix

B $7N + 8 \times 2N + 4$ constant matrix (4.27)

f, g, d, and x are $7N + 8$ vector

u is $7N + 4$ vector

f and g are nonlinear vector functions.

The above matrices and vectors are defined on the following pages.

$$\begin{array}{ccccccc|c}
 a_{11} & a_{12} & 0 & 0 & 0 & 0 & 0 & \\
 0 & 0 & 0 & 0 & 0 & 0 & 0 & \\
 a_{31} & 0 & a_{33} & 0 & 0 & 0 & 0 & \\
 0 & 0 & 0 & a_{44} & 0 & 0 & 0 & \\
 0 & 0 & 0 & 0 & a_{55} & 0 & a_{57} & \\
 0 & 0 & 0 & 0 & a_{65} & 0 & 0 & \\
 0 & 0 & 0 & 0 & a_{75} & 0 & a_{77} &
 \end{array}$$

$$\begin{array}{ccccccc|c}
 & & \dots & & & & & \\
 & & 0 & & & & & \\
 & & a_{7N-6,7N-12} & 0 & 0 & a_{7N-6,7N-6} & a_{7N-6,7N-5} & 0 \\
 & & & & & 0 & 0 & 0 \\
 & & & & & a_{7N-7,7N-6} & 0 & a_{7N-4,7N-4} \\
 & & & & & 0 & 0 & 0 \\
 & & & & & 0 & 0 & 0 \\
 & & & & & 0 & 0 & 0 \\
 & & & & & 0 & 0 & 0
 \end{array}$$

A=

										Stand 1
										...
0	0	0	0							Stand N
0	0	0	0							
0	0	0	0							
$a_{7N-3, 7N-3}$	0	0	0							
0	$a_{7N-2, 7N-2}$	0	$a_{7N-2, 7N}$							
0	$a_{7N-1, 7N-2}$	0	0							
0	$a_{7N, 7N-2}$	0	$a_{7N, 7N}$							
				0	0	0	0	0	0	Coiler and Decoiler
				$a_{7N+2, 7N+1}$	$a_{7N+2, 7N+2}$	0	0	0	0	
				0	0	$a_{7N+3, 7N+3}$	0	0	0	
				$a_{7N+4, 7N+1}$	0	0	0	0	0	
				0	0	0	0	0	0	
				0	0	0	0	$a_{7N+6, 7N+5}$	$a_{7N+6, 7N+6}$	
				0	0	0	0	0	$a_{7N+7, 7N+7}$	
				0	0	0	0	$a_{7N+8, 7N+5}$	0	
									0	

	0	0	0						Stand 1
	0	0	0						
	b_{31}	0	0						
	0	b_{42}	0						
	0	0	0						
	0	0	0						
	0	0	b_{73}						
		...							
		...							
	
		...							
		...							
B=	0	0	0						
	0	0	0						
	0	$b_{7N-4, 3N-2}$	0						
	0	0	$b_{7N-3, 3N-1}$						
	0	0	0						
	0	0	0						
	0	0	0						
	0	0	$b_{7N, 3N}$						Stand N
	0	0	0						
	$b_{7N+2, 3N+1}$	0	0						
	0	$b_{7N+3, 3N+2}$	0						
	0	0	0						
	0	0	0						
	0	0	0						
	0	0	$b_{7N+6, 3N+3}$						
	0	0	0				$b_{7N+7, 3N+4}$		
	0	0	0				0		Coiler and Decoiler

5. CONCLUSIONS

A dynamic model of an N-stand continuous cold reduction mill has been developed. A generalized state equation of $(7N + 8)$ th order was derived taking into account the dynamics of coiler, stands, drive, screw-down motors and decoiler. The equations turned out to be highly nonlinear and in general involve time lags due to the transition of strip from one stand to the next.

This model is expected to be a more accurate description of a rolling mill than the models proposed thus far since it uses less approximations such as linearizations of force-torque as well as the conservation of the strip volume (eq. 2.8). The main reasons for avoiding these approximations is that with the existence of digital computers and efficient numerical methods more realistic models can be handled.

The state equations can be modified for rolling mills with different electrical equipment and introduction of new variables cause no difficulty in formulating the new set of equations.

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Appendix 1.

Notations and NomenclatureNotations:

i	= "entry" or inlet plane
o	= "exit", or outlet plane
n	= neutral plane
j	= number of stand $j = 1, \dots, N$
N	= maximum number of stands in the mill
\hat{F}	= $\hat{F} = \frac{F - F^*}{F^*}$, denotes the operating value
t_{ij}	= the inlet tension, t_i belonging to the j th stand.
s	= screw down motors
c	= coiler
d	= decoiler
m	= motor (coiler, decoiler)
L	= load

Nomenclature:

e_f	= field voltage
e_a	= armature voltage
ω	= angular velocity
S_{oj}	= screw-down setting of j th stand rolls
θ	= arc of contact (rolls and strip)
R	= radius of undeformed roll
R'	= radius of deformed roll
h	= thickness of the strip

ξ	= tension per unit area
A	= area of the strip
τ	= roll torque per unit width
F	= roll force per unit width
μ	= coefficient of friction
v	= strip velocity
E	= modulus of elasticity (Young's modulus)
T	= time lag (strip transit period)
M'	= modulus of elasticity (spring constant) of "mill housing"
J	= moment of inertia
B	= friction loss
n	= gear ratio
r	= coiler or decoiler radius
K_1	= torque constant
\emptyset	= magnetic flux
i_a	= armature current
R_a	= armature resistance
L_a	= armature inductance
K_b	= back emf constant
L_f	= field current
K_b	= back emf constant
L_f	= field inductance
R_f	= field resistance
N_f	= field winding number of turns
C_1, C_3	= coefficients of field current polynomial

ρ	= density of rolled material
w	= width of the strip
K_s	= screw-down position gears constant
d	= disturbance vector
x	= state vector
u	= control vector

Appendix 2.

Roll Force and Torque Equations

[Bland and Ford]

$$F = F(h_i, h_o, t_i, t_o, r) = R' \left[\int_0^{\theta_n} \frac{kh}{h_o} \left(1 - \frac{t_o}{k_o}\right) e^{\mu H(h_o, \theta)} d\theta \right. \\ \left. + \int_{\theta_n}^{\theta_i} \frac{kh}{h_i} \left(1 - \frac{t_i}{k_i}\right) e^{\mu (H_i(h_o, \theta_i) - H(h_o, \theta))} d\theta \right] \quad (2.1)$$

$$\tau = \tau(h_i, h_o, t_i, t_o, \mu) = RR' \left[\int_0^{\theta_n} \frac{kh}{h_o} \left(1 - \frac{t_o}{k_o}\right) \theta e^{\mu + 1} d\theta + \right. \\ \left. \int_{\theta_n}^{\theta_i} \frac{kh}{h_i} \left(1 - \frac{t_i}{k_i}\right) + \frac{h_i t_i - h_o t_o}{2R'} d\theta \right] \quad (2.2)$$

where

$$H(h, \theta) = 2 \sqrt{\frac{R'}{h}} \tan^{-1} \sqrt{\frac{R'}{h}} \theta$$

and

$$h_n = h_n(h_i, h_o, t_i, t_o, \tau) = h_o \left[1 + \frac{1}{2} \tan^2 \left[\sqrt{\frac{R'}{h_o}} \tan^{-1} \right. \right. \\ \left. \left. \sqrt{\frac{R'}{h_o}} \theta_i - \frac{1}{2\mu} \ln \frac{h_i (1 - t_o/k_o)}{h_o (1 - t_i/k_i)} \right]^2 \right] \quad (2.3)$$

Appendix 3: Expressions Involving Equations (4.3), (4.4), and (4.11)

In Equation (4.3):

$$a_{3j} = -\frac{k_{23}}{2k_{22}}, \quad a_{4j} = -\frac{k_{24}}{2k_{22}},$$

$$a_{5j} = -\frac{k_{25}^{\mu_j}}{2k_{22}} + \frac{\beta_j k_{2-1}}{2\beta_j k_{22}}, \quad \text{all } k\text{'s are for } j\text{th stand,}$$

$$\begin{aligned} \pm \eta_j(t-T) = & -\frac{k_{12}}{2k_{22}} \hat{h}_{o,j-1}(t-T) \pm \{[\beta(k_{2+k_{12}} h_{o,j-1}(t-T) + k_{23} x_{2,j-1} \\ & + k_{24} x_{2,j} + k_{25}^{\mu_j}) - 1]^2 - 4\beta k_{22} (\alpha_j x_{6j} + \beta_j \langle k, q_j \rangle \\ & + \beta_j \langle q_j, \underline{k} q_j \rangle)\}^{\frac{1}{2}}. \end{aligned}$$

Note that here

$$q_j = [h_{o,j-1}(t-T) \quad h_{oj}(t) \quad x_{2,j-1} \quad x_{2,j} \quad \mu_j]^T.$$

For Equation (4.4):

$$b_{3j} = (l_2 + l_{25}^{\mu_j}) a_{3j} + 2l_{22} a_{3j} a_{5j} + l_{23} a_{5j} + l_3$$

$$b_{4j} = (l_2 + l_{25}^{\mu_j}) a_{4j} + 2l_{22} a_{4j} a_{5j} + l_{24} a_{5j} + l_4$$

$$b_{5j} = l_2 a_{5j} + l_{22} a_{5j}^2 + l_{25} a_{5j}^{\mu_j} + l_5^{\mu_j}$$

and

$$\begin{aligned}
\phi_j(t-T) = & (l_{22}a_{3j}^2 + l_{23}a_{3j})x_{2,j-1}^2 + (l_{22}a_{4j}^2 + l_{24}a_{4j})x_{2,j}^2 \\
& + (l_{23}a_{4j} + l_{24}a_{3j} + 2l_{22}a_{3j}a_{4j})x_{2,j-1}x_{2,j} \\
& + [(l_{23} + 2l_{22}a_{3j})x_{2,j-1} + (l_{24} + 2l_{22}a_{4j})x_{2,j} + l_{12}h_{o,j-1}(t-T) \\
& \pm (l_2 + 2l_{22}a_{5j} + l_{25}\mu_j)]\eta_j(t-T) + [l_{12}(a_{3j}x_{2,j-1} + a_{4j}x_{2,j} \\
& + a_{5j}) + l_2]h_{o,j-1}(t-T) + \langle q_j, \underline{L}, q_j \rangle + l_{22}\eta_j^2(t-T) .
\end{aligned}$$

Note that $\underline{L} = \underline{I}L$ and q_j is as in (4.2). Equation (4.11):

$$\begin{aligned}
g_{2j} \triangleq \frac{ER}{T} & \left[\frac{h_{nj}^*}{v_{oj}^* h_{oj}^*} \frac{x_{1j}(c_{3j}x_{2,j-1} + c_{4j}x_{2j} + \Psi_j(t-T) + c_{5j} + 1)}{(a_{3j}x_{2,j-1} + a_{4j}x_{2j} + \eta_j(t-T) + a_{5j} + 1)} \right. \\
& - \frac{h_{n,j+1}^*}{v_{i,j+1}^* h_{i,j+1}^*} \cdot \frac{x_{1,j+1}(c_{3,j+1}x_{2,j} + c_{4,j+1}x_{2,j+1} + \\
& \left. \frac{\Psi_{j+1}(t-T) + c_{5,j+1} + 1}{\eta_j(t-2T) + a_{5j} + 1})}{(a_{3j}x_{2,j-1}(t-T) + a_{4j}x_{2j}(t-T) +} \right] .
\end{aligned}$$

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13. ABSTRACT The dynamic model of a continuous cold reduction mill is developed. The model makes use of the force-torque equation expansion technique developed in an earlier report and presents the generalized (7N+8)th order state equation for an N-stand rolling mill. The equations of interstand tensions are found to be highly nonlinear with the arguments delayed due to the transition of strip between the stands. The model developed can used in the design of a controller for the entire rolling process.			

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