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FREE-SURFACE FLUCTUATIONS IN HYDRAULIC JUMPS: EXPERIMENTAL OBSERVATIONS

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Abstract: A hydraulic jump is the rapid and sudden transition from a high-velocity supercritical open channel flow to a subcritical flow. It is characterised by the dynamic interactions of the large-scale eddies with the free-surface. New series of experimental measurements were conducted in hydraulic jumps with Froude numbers between 3.1 and 8.5 to investigate these interactions. The dynamic free surface measurements were performed with a non-intrusive technique while the two-phase flow properties were recorded with a phase-detection probe. The shape of the mean free surface profile was well defined and the turbulent fluctuation profiles highlighted a distinct peak of turbulent intensity in the first part of the jump roller, with free-surface fluctuation levels increasing with increasing Froude number. The dominant free-surface fluctuation frequencies were typically between 1 and 4 Hz. A comparison between the acoustic sensor signals and conductivity probe data suggested that the air-water "free-surface" detected by the acoustic sensor corresponded to about the boundary between the turbulent shear layer and the upper free-surface layer. Simultaneous measurements of free surface and bubbly flow fluctuations for $Fr = 5.1$ indicated that the frequency ranges of both sensors were similar ($F < 5$ Hz) whatever the position downstream of the toe. The present results highlighted that the dynamic free-surface measurements can be conducted successfully using acoustic displacement meters, and the time-averaged depth measurements was a physical measure of the free-surface location in hydraulic jumps.

Keywords: Hydraulic jumps, Free-surface fluctuations, Frequency, Turbulent shear flow, Air bubble entrainment, Roller length, Experimental techniques.

1. INTRODUCTION

Turbulent free-surface flows including hydraulic jumps are characterised by some interfacial aeration (Fig. 1). In the highly aerated flow where the void fraction C ranges from 0.3 to 0.7, the microscopic two-phase flow structure is complex, and it consists of a wide range of entities including air–water projections, foam, and complicated air–water imbrications (RAO and KOBUS 1971, CHANSON 1997). In high-velocity open

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channel flows, KILLEN (1968) suggested the possibility of surface waves riding over a bubbly flow region, while TOOMBES and CHANSON (2007) studied the influence of surface waves and fluctuations of the flow depth on the two-phase air-water flow properties. MOUAZE et al. (2005) investigated specifically the free-surface fluctuations in a hydraulic jump roller. Figure 2 documents some air-water flow structures above a hydraulic jump roller.

A hydraulic jump is the rapid and sudden transition from a high-velocity supercritical open channel flow to a subcritical flow (Fig. 1). Hydraulic jumps are commonly experienced in rivers and canals, in industrial applications and in manufacturing processes. The application of the momentum principle to the hydraulic jump gives a relationship between the flow properties upstream and downstream of a hydraulic jump. In a horizontal rectangular prismatic channel, it yields:

$$\frac{d_2}{d_1} = \frac{1}{2} \times \left(\sqrt{1 + 8 \times Fr^2} - 1 \right) \quad (1)$$

where d and V are the flow depth and velocity respectively, the subscripts 1 and 2 refer to the upstream and downstream flow conditions respectively, Fr is the inflow Froude number: $Fr = V_1 / \sqrt{g \times d_1}$, and g is the gravity acceleration. Turbulence measurements in hydraulic jumps were conducted by several researchers, including ROUSE et al (1959), RESCH and LEUTHEUSSER (1972a,1972b), CHANSON and BRATTBERG (2000), LIU et al (2004), LENNON and HILL (2006), CHANSON (2007) and KUCUKALI and CHANSON (2008). These studies focused either on the turbulent water flow properties with relatively low Froude number conditions or on the air-water flow properties in the jump roller. Related experiments investigated the fluctuations of hydraulic jump properties including the toe position and water level (LONG et al. 1990, MOSSA 1999) (Table 1).

In a hydraulic jump, the characterisation of free-surface fluctuations may be conducted with different techniques: e.g., the wire gage and the acoustic displacement sensor. The first kind is particularly accurate for periodic waves in absence of wave breaking as well as details of the free surface turbulence length and time scales for low Froude numbers (MOUAZE et al 2005). When the flow becomes strongly turbulent with large variations of the air-water interface associated with bubbles, splashes and droplets, the output signal must be considered with some caution because the sensitive part of the probe is not continuously immersed. Further most sensors have a dynamic response that is generally lower than 12 Hz (MURZYN et al. 2007). The second technique (acoustic displacement sensor) is non intrusive (Fig. 1 B) and more accurate in terms of the dynamic response. Herein, the sensor manufacturer specified with a response time less than 50 ms. While the acoustic displacement sensor seems a well-defined measurement technique, two key questions remain: what do these sensors exactly measure? what is the accuracy?

In the present study, the free-surface fluctuations and air-water flow properties in hydraulic jumps were investigated. Both acoustic displacement sensors and phase-detection probes were used. It is the aim of this work to assess the suitability of the acoustic displacement sensor, and to examine the free surface motion as well as the air-water flow properties in hydraulic jumps for a relatively broad range of Froude numbers (3.1

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< Fr < 8.5).

2. EXPERIMENTAL SET-UP AND INSTRUMENTATION

New experiments were performed in a 3.2 m long 0.5 m wide horizontal rectangular flume (Fig. 1, Table 1). The glass sidewalls were 0.45 m high and the channel bed was PVC. The water discharge was measured with a Venturi meter located in the supply line which was calibrated on-site with a large V-notch weir. The clear-water flow depths were measured using rail mounted point gages. The inflow conditions were controlled by a vertical gate with a semi-circular rounded shape ($\varnothing = 0.3$ m). The upstream gate aperture was fixed during all experiments ($d_1 = 0.018$ m).

The unsteady free surface measurements were conducted using six ultrasonic displacement meters Microsonic™ Mic+25/IU/TC. The acoustic displacement sensors were mounted above the flow at fixed locations for all series of experiments (Fig. 1). Each probe signal output was scanned at 50 Hz per sensor for 10 minutes (unless stated).

The air-water flow properties were measured with a phase-detection conductivity probe ($\varnothing = 0.25$ mm). The phase-detection intrusive probe was excited by an electronic system (model ref. UQ82.518) designed with a response time of less than 10 μ s. During the experiments, the probe sensor was sampled at 20 kHz for 45 s per sampling point, and each vertical profile contained at least 30 points. The displacement and the position of the probe in the vertical direction were controlled by a fine adjustment system connected to a Mitutoyo™ digimatic scale unit with a vertical accuracy Δy of less than 0.1 mm.

Further details on the experimental facility and data sets were reported in MURZYN and CHANSON (2007).

2.1 Signal outputs and processing

The principle of the acoustic displacement meters is based upon an acoustic beam emitted in air by the sensor. Once the beam hits the air-water interface, it is reflected back to the sensor. A simple measure of the travel time provides the distance between the sensor and the free surface. Although the measurement principle is basic, the present experience highlighted a number of situations when the outputs were meaningless. For example, when the free surface was not horizontal, the acoustic beam did not reflect back to the receiver; when the free surface was a bubbly foam, the sensor response might correspond to an unknown location in the intermediate flow region (discussion below); when measurements were made above the roller with a large Froude number, data errors were caused by bubbles, water splashes and droplets coming into contact with the emitter. Herein, the acoustic displacement sensor signal was filtered to remove and replace erroneous points. In most cases, less than 7% of the data samples were also removed. For two data sets, up to 20% were removed due to the large number of droplets impacting the ultrasonic displacement meter sensor.

The analysis of the phase-detection probe output was based upon a single threshold technique, with a

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threshold set between 45% and 55% of the air–water voltage range (TOOMBES 2002, CHANSON and CAROSI 2007). A number of air-water flow properties were derived from the signal analysis, including the instantaneous void fraction and the time-averaged void fraction C defined as the volume of air per unit volume of air and water. The sampling rate (20 kHz) and duration (45 s) of the conductivity probe was selected based upon the detailed sensitivity analysis (CHANSON 2007b). When the conductivity probe was sampled simultaneously with the displacement sensor, the sampling rate was reduced because the dynamic response of the acoustic displacement meters was much lower, while the sampling duration was not a relevant parameter since the focus of that study was the instantaneous free-surface and void fraction data.

2.2 Measurement accuracy and error

The water discharge was measured with an accuracy of about 2%. The clear-water flow depths were measured using a point gage with a 0.2 mm accuracy. The ultrasonic displacement meters had a 0.18 mm accuracy and 50 ms response time. The translation of the conductivity probes in the direction normal to the channel invert was controlled with an error of less than 0.1 mm. The accuracy on the longitudinal probe position was estimated as $\Delta x < \pm 0.5$ cm. The error on the transverse position of the probe was less than 1 mm. With the double-tip conductivity probe, the error on the void fraction measurements was estimated as: $\Delta C/C = 4\%$ for $0.05 < C < 0.95$, $\Delta C \approx 0.002/(1 - C)$ for $C > 0.95$, and $\Delta C/C \approx 0.005/C$ for $C < 0.05$. The minimum detectable bubble chord length was about 50 μm in a 1 m/s flow based upon a data acquisition frequency of 20 kHz per channel.

2.3 Experimental flow conditions

The experimental measurements were performed with inflow Froude numbers ranging 3.1 to 8.5 (Table 1). The corresponding Reynolds numbers ranged from 24 to $64 \cdot 10^5$, and were large enough to minimise scale effects (CHANSON and GUALTIERI 2008, MURZYN and CHANSON 2008).

The jump toe was located at $x_1 = 0.75$ m downstream of the rounded gate, and clear-water velocity measurements showed that the inflow conditions were partially-developed. The downstream flow properties were controlled by an overshoot gate located 2.45 m downstream of the jump toe.

3. EXPERIMENTAL RESULTS: FREE-SURFACE FLUCTUATIONS

Two basic characteristic of the hydraulic jump are its mean longitudinal free surface profile and its free-surface fluctuations. Although the free-surface was well-defined upstream of the toe (nearly flat), it became strongly turbulent downstream of the impingement point with large vertical fluctuations and a bubbly/foamy structure (Fig. 1 & 2). In the roller, high amplitude motions and strong fluctuations in time and space occurred with increasing amplitude with increasing Froude number. Herein the mean and turbulent profiles, as well as the free-surface fluctuation frequencies, are presented first. The measurements were performed

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using the acoustic displacement meters and were compared with some pertinent studies (Table 1).

Visual observations showed that the air-water interface exhibited some small amplitude motions for the lowest Froude number, whereas it became strongly turbulent for the larger Froude numbers with large air packets and splashes projected above the air-water interface (Fig. 2). Figure 3 presents some results obtained in terms of the mean free surface profile for six experiments with Froude numbers ranging from 3.1 to 8.5 (Table 1). In Figure 3, the position of the jump toe corresponds to $(x-x_1)/d_1 = 0$, where x is the longitudinal distance from the upstream gate, and x_1 is the toe position, and η is the flow depth. Upstream of the jump toe ($x < x_1$), the free surface profile was flat with small fluctuations. The upstream flow depths were between 0.017 m and 0.020 m which were close to the upstream gate aperture ($d_1 = 0.018$ m). The difference of about 10% was satisfactory considering the uncertainties of the sensors and the expected contraction coefficient of unity for the upstream rounded gate. Downstream of the jump toe ($x > x_1$), a monotonic increase of the mean water level was noted. This pattern was in accordance with visual observations during the experiments and photographic evidences (Fig. 1A). For inflow Froude numbers less than 6.5, the jump roller surface was followed by a flat region where the flow was less turbulent: i.e., with large scale motion and low fluctuation frequencies (Fig. 3). For these jumps ($Fr \leq 6.4$), the roller length (L_r) was estimated where L_r was defined as the distance over which the mean free surface level increased monotonically. The results are presented on Figure 4 in terms of the dimensionless roller length (L_r/d_1). The present roller length data were in agreement with the experiments of MURZYN et al (2007) ($1.9 < Fr < 4.8$) and the correlation of HAGER et al (1990) developed for $2.5 < Fr < 8$ in horizontal rectangular channels.

3.1 Free-surface turbulent fluctuations

The free surface fluctuations were investigated and the results are given in Figure 5. Figure 5 presents the dimensionless standard deviation of the water depth η'/d_1 as a function of the dimensionless distance from the jump toe $(x-x_1)/d_1$. Upstream of the toe, the turbulent fluctuations η' were small. Immediately downstream of the jump toe (i.e. $(x-x_1)/d_1 > 0$), a marked increase in free-surface fluctuation was recorded for all Froude numbers, reaching a maximum value η'_{max} which increased with increasing Froude number (Fig. 5). Further downstream, the free-surface fluctuations decayed with increasing distance from the jump foot. Far downstream, the turbulence levels were small, with magnitudes comparable to those observed upstream of the impingement point. This pattern was consistent with the earlier studies of MOUAZE et al. (2005) and KUCUKALI and CHANSON (2008), while the downstream decay in free-surface fluctuations would correspond to a dissipative region.

The peak of turbulent fluctuations was observed in the first half of the roller (Fig. 5). This was in agreement with the findings of MOUAZE et al. (2005) who observed an intense turbulent area with a length of about 30% of the roller length. In the roller region, the flow was characterized by strong turbulence production, large recirculation vortices and coherent structures reaching the free surface. For $Fr = 8.5$, the maximum fluctuation was close to $\eta'_{max}/d_1 = 1.5$ whereas it was only about 0.3 for $Fr = 3.1$ (Fig. 6). The finding

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highlighted that the free surface motion became more turbulent with increasing inflow Froude number. Since the instantaneous variation in water depth is equal to the vertical velocity component at the free-surface:

$$\frac{\partial \eta}{\partial t} = V_y (y = \eta) \quad \text{at the free-surface (2)}$$

some algebraical considerations show that the fluctuations of the water depth η are linked to the turbulent kinetic energy per unit volume ε : $\eta'^2 \propto v_y'^2 \propto \varepsilon$. In a rectangular, horizontal, prismatic channel, the dimensionless rate of energy dissipation equals:

$$\frac{\Delta H}{d_1} = \frac{\left(\frac{d_2}{d_1} - 1\right)^3}{4 \times \frac{d_2}{d_1}} = \frac{\left(\sqrt{1 + 8 \times Fr^2} - 3\right)^3}{16 \times \left(\sqrt{1 + 8 \times Fr^2} - 1\right)} \quad (3)$$

For large Froude numbers, it gives: $\Delta H \propto Fr^2$. Combining the continuity equation at the free-surface (Eq. (2)) with the integral form of the energy equation (Eq. (3)), this simple reasoning suggests that, assuming $\Delta H \propto \varepsilon$, the fluctuation of the water level is proportional to the inflow Froude number: $\eta' \propto Fr$. In Figure 6, the present results are compared with the data of MADSEN (1981), MOUAZE et al. (2005) and KUCUKALI and CHANSON (2008) (Table 1). The experimental data show that $\eta'_{\max} \propto (Fr - 1)$ for $Fr > 5$ and the data trend is identical to the theoretical development.

Overall all the experimental data collapsed into a monotonic curve best fitted by:

$$\frac{\eta'_{\max}}{d_1} = 0.116 \times (Fr - 1)^{1.235} \quad (4)$$

with a normalised correlation coefficient of 0.985. Equation (4) is compared with the experimental data in Figure 6.

3.2 Frequency range(s) in hydraulic jumps

Some spectral analyses of the free-surface fluctuations were performed. The data provided new information on the time scales of the flow next to the free-surface. A typical Fast Fourier Transform (FFT) of a sensor output signal is presented in Figure 7, where the smoothed FFT data were obtained using a smoothing window of 20 points. The data highlighted some dominant frequency at the given position downstream of the impingement point. For example, in Figure 7, a peak is clearly marked at $F_{fs} = 3$ Hz depicting the main frequency of the free surface fluctuations at $(x-x_1)/d_1 = 13.3$ for $Fr = 4.2$. This approach was applied to all flow conditions and the results yielded the dominant frequencies of the free-surface fluctuations in the hydraulic jumps. The results are summarised in Table 2 and Figure 8, where the main frequency data are plotted as a function of the dimensionless distance from the jump toe $(x-x_1)/L_r$ where L_r is the roller length. The graph includes the experimental results obtained in the roller ($(x-x_1)/L_r < 1$) and downstream of the roller

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$((x-x_1)/L_r > 1)$. Note that the data for $Fr > 6.8$ are not shown because the acoustic displacement meter response was adversely affected by the spray and splashing on the emitter. Figure 8 indicates that the dominant frequency range was between 1 and 4 Hz. Additional experiments showed little influence of the sampling rate: i.e., 50 Hz during 1,200 s, or 5,000 Hz for 12 s. For a given inflow Froude number, the free-surface fluctuation frequency appeared to be constant in the hydraulic jump roller $(x-x_1)/L_r < 1$ while F_{fs} decreased downstream for $(x-x_1)/L_r > 1$. Downstream of the jump, the results (Fig. 8) were in good agreement with visual observations.

The horizontal oscillations of the jump toe were also recorded and the data are reported in Table 2. The dimensionless results are plotted in Figure 9 in terms of the Strouhal number defined as

$$St_{toe} = \frac{F_{toe} \times d_1}{V_1} \quad (5)$$

where F_{toe} is the toe oscillation frequency. The Strouhal number is a dimensionless term characterising the oscillations of a physical system. The results in terms of jump toe oscillation frequencies were compared with the dimensionless free-surface fluctuation frequencies St_{fs} in the jump roller and with two earlier studies of jump toe oscillations (Table 2, Fig. 9). The present data were close to the findings of MOSSA and TOLVE (1998) and CHANSON and GUALTIERI (2008) with steady jumps. It is noteworthy that the range of fluctuation frequency was similar to the free-surface fluctuation frequency observation of MOSSA (1990) in an oscillating hydraulic jump and turbulent velocity fluctuations of LIU et al. (2004) in weak jumps.

The jump toe oscillation frequencies were consistently smaller than the free-surface fluctuation frequencies for a given Froude number (Fig. 9). The results indicated that the Strouhal number $F_{toe} \times d_1 / V_1$ of the toe oscillations was nearly constant independently of the Reynolds number $\rho \times V_1 \times d_1 / \mu$, whereas the Strouhal number $F_{fs} \times d_1 / V_1$ of the roller surface fluctuations decreased with increasing Reynolds number. These results were in agreement with the earlier observations of MOSSA and TOLVE (1998).

4. DISCUSSION: DEFINITION OF THE FREE-SURFACE

What did the acoustic displacement meter detect? The hydraulic jump roller was highly aerated. Figure 10A presents typical vertical distributions of void fraction along the roller for a given Froude number. The air bubble entrainment occurred at the impingement of the supercritical flow into the roller. Bubbles and air packets were entrained at the flow singularity and were advected downstream in the turbulent shear flow. In the developing shear layer, the data compared favourably with an analytical solution of the advective diffusion equation for air bubbles (CHANSON 1995,1997) :

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$$C = C_{\max} \times \exp \left(- \frac{\left(\frac{y - y_{C_{\max}}}{d_1} \right)^2}{4 \times D^* \times \left(\frac{x - x_1}{d_1} \right)} \right) \quad (6)$$

where C_{\max} is the maximum void fraction in the shear layer, $y_{C_{\max}}$ is the vertical elevation of the maximum void fraction C_{\max} , D^* is a dimensionless turbulent diffusivity. Equation (6) is compared with some data in Figure 10. Such a close agreement was previously observed in hydraulic jumps with partially-developed inflow conditions, including a re-analysis of the data of RESCH and LEUTHEUSSER (1972b), CHANSON and BRATTBERG (2000), MURZYN et al. (2005) and CHANSON (2007).

Above the turbulent shear layer, the upper free-surface region was characterized by large air contents, splashes and recirculation areas, with large eddies, a wavy free surface pattern and air-water projections (Fig. 2). The void fraction profiles illustrated the two distinct regions (Fig. 10B), and y^* is defined herein as the transition between the upper flow region and the air-water shear flow.

4.1 Detection of the air-water interface

The present data were analysed to bring new information on the free surface fluctuation characteristics in hydraulic jumps. The relationship between turbulent fluctuation η'_{\max}/d_1 and the maximum amplitude of the free surface $\Delta\eta/d_1$ was tested, where $\Delta\eta$ was the free-surface fluctuation range recorded during the sampling duration ($\Delta\eta = \eta_{\max} - \eta_{\min}$). The results indicated that the largest turbulent fluctuations were closely linked to the most important free surface amplitudes according to a well-defined linear trend, and this trend was consistent with visual observations.

Two earlier studies tested acoustic displacement sensors above bubbly flows. In a bubbly column with up to 10% void fraction, CHANSON et al. (2002) observed that the ultrasonic probe readings corresponded to about y_{50} to y_{60} where y_{xx} is the elevation where the void fraction is xx%. KUCUKALI and CHANSON (2008) reported that the response of acoustics displacement meters corresponded to the range y_{60} to y_{80} in hydraulic jumps. Herein a comparative analysis was conducted systematically between the acoustic displacement meter and void fraction data. The time-averaged depth recorded with the acoustic displacement meter was compared with the void fraction profile measured with the leading tip of the dual-tip conductivity probe. Some results are presented on Figure 11. Figure 11 shows that the time-averaged "free-surface" elevation η measured by the acoustic displacement sensor was slightly above the characteristic location y^* for all investigated Froude numbers. The finding suggested that the interface measurement by the acoustic sensor was within the upper flow region ($\eta > y^*$, Fig. 10B). This region was typically a thin layer where the void fraction was basically larger than 20%, rapidly reaching 90% and more (Fig. 10). The present description is thought to be a more physical measure of the free-surface location in hydraulic jumps because it refers to a specific region of the flow. Altogether, the present findings were in agreement with the

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experimental data of KUCUKALI and CHANSON (2008).

4.2 Simultaneous measurements of free-surface and bubbly flow properties

For one Froude number ($Fr = 5.1$), simultaneous measurements of free surface fluctuations and air-water flow properties were conducted using an acoustic displacement meter mounted directly above the phase-detection probe control leading tip as shown in Figure 1B. The sampling rate was 5,000 Hz, the acquisition duration was 12 s for both sensors and the experimental flow conditions are summarised in Table 3. Although the measurements were performed at several different vertical elevations, the comparative results are focused herein on the air-water flow properties at $y = y_{30}$ (i.e. $C = 0.3$).

The spectral analyses were performed on the processed signals of both sensors. The original output signal was filtered using a band pass (0-25 Hz) and the low-pass filtered signal was averaged over 100 points. Figure 12 presents a comparison between the raw probe signal (phase-detection probe only) and the processed signals. Figure 12B illustrates the time-series of simultaneous processed signals. Typical results of spectral analyses are presented in Figure 13. In Figure 13, the FFT analysis of the phase detection probe signal is on the left and that of the free-surface signal on the right. The results showed that the dominant frequencies were less than 5 Hz for both conductivity probe and acoustic displacement sensors (Fig. 13), implying that the bubble generation and free surface vertical motion might be dependant processes.

A cross-correlation analysis was performed on the processed signals of both phase detection probe and acoustic sensor. Typical cross correlation functions, for the two conditions shown in Figure 13, are presented on Figure 14. In Figure 14, the vertical axis is the normalised coefficient of correlation R_{xz} where x is the phase-detection probe signal and z is the free-surface fluctuation signal, and the horizontal axis is the time lag. Altogether the results exhibited some large negative and positive values with some periodic shape (Fig. 14). A spectral analysis of these cross-correlation functions revealed that the dominant frequencies were 1.12 Hz, 1.56 Hz and 2.34 Hz for $(x-x_1)/d_1 = 5.0, 9.2$ and 13.3 respectively. These values were consistent with results presented earlier herein.

5. CONCLUSION

In a hydraulic jump, the large vortices develop beneath the free-surface and are advected in the developing shear layer. The interactions of the large-scale eddies with the free-surface yield a complex two-phase flow structure, consisting of a wide range of entities including air–water projections, foam, and complicated air–water imbrications. Some new series of experimental measurements were conducted in hydraulic jumps with Froude numbers between 3.1 and 8.5 to characterise the turbulent free-surface fluctuations. Dynamic free surface measurements were performed with non-intrusive acoustic displacement meters while the two-phase flow properties were recorded with a phase-detection probe. For one Froude number, instantaneous free surface and void fraction data were simultaneously recorded to study some correlation between their

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respective frequency ranges.

For Froude numbers between 3.1 and 8.5, the shape of the mean free surface profile was well defined and the results were in agreement with visual observations and earlier studies. The turbulent fluctuation profiles highlighted a distinct peak of turbulent intensity in the first part of the jump roller, and the peak of free-surface fluctuation levels increased with increasing Froude number. The free-surface fluctuation frequencies were typically between 1 and 4 Hz. The dominant frequency was typically higher in the roller than in the downstream flow. It was consistently larger than the jump toe oscillation frequency for $Fr < 5.3$, but tended to comparable values for larger Froude numbers ($Fr > 5.3$) (Fig. 9). A comparison between the acoustic sensor and phase-detection probe signals suggested that the air-water "free-surface" detected by the acoustic sensor corresponded to about the boundary between the turbulent shear layer and the upper free-surface layer ($\eta \sim y^*$).

Simultaneous measurements of free surface and bubbly flow fluctuations for $Fr = 5.1$ were performed. The spectral analyses indicated that the frequency ranges of both sensors were similar ($F < 5$ Hz) whatever the position downstream of the toe. The signal cross correlations showed some large positive and negative values implying with some periodic shapes. The findings suggested the air bubble entrainment and free surface vertical motion might be dependant processes.

Ultimately, the present results highlighted that the dynamic free-surface measurements can be conducted successfully using acoustic displacement meters, and the time-averaged depth measurements was a physical measure of the free-surface location in hydraulic jumps. The data hinted furthermore some interactions between the free-surface fluctuations and the two-phase flow properties of the developing shear flow, although further investigations are needed.

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FIGURE CAPTIONS

Fig. 1 - Photographs of a hydraulic jump in the experimental channel

(A) General view: $Fr = 7.9$, $d_1 = 0.018$ m, flow from left to right (shutter: 1/80 s) - Note the five acoustic displacement sensors mounted above the channel

(B) General view: $Fr = 7$, $d_1 = 0.024$ m, flow from bottom to top (shutter: 1/80 s) - Note the phase detection probe with its sensor located at $x-x_1 = 0.2$ m and the acoustic displacement sensor mounted above

Fig. 2 - High-speed photographs of the free-surface deformation and water ejections above the hydraulic jump roller, looking upstream towards the toe: $Fr = 8.45$, $d_1 = 0.018$ m, flow from background to foreground (shutter: 1/150 s) - Note the phase-detection probe on the left with its tip located at $x-x_1 = 0.225$ m - Each photograph was taken a few second apart

Fig. 3 - Dimensionless mean free surface profile (η/d_1) measurements in hydraulic jumps ($Fr = 3.1$ to 8.5)

Fig. 4 - Measurements of dimensionless hydraulic jump roller length L_r/d_1 for $Fr < 6.5$ - Comparison with the correlations of HAGER et al. (1990) and MURZYN et al. (2007)

Fig. 5 - Longitudinal distributions of dimensionless free surface fluctuations η'/d_1 in hydraulic jumps for several experimental conditions ($Fr = 3.1$ to 8.5)

Fig. 6 - Maximum of turbulent fluctuations η'_{max}/d_1 in hydraulic jumps as a function of Froude number Fr - Comparison with the data of MADSEN (1981), MOUAZE et al. (2005) and KUCUKALI and CHANSON (2008), and Equation (4)

Fig. 7 - Spectral analysis of the free-surface fluctuations: $Fr = 4.2$, $d_1 = 0.018$ m, $x_1 = 0.75$ m, $x-x_1 = 0.24$ m

Fig. 8 - Dominant free surface frequencies F_{fs} of the free-surface fluctuations in hydraulic jumps

Fig. 9 - Dimensionless hydraulic jump toe oscillation frequency $F_{toe} \times d_1/V_1$ and dimensionless free-surface fluctuation frequency $F_{fs} \times d_1/V_1$ in hydraulic jumps - Comparison with the jump toe oscillation data of MOSSA and TOLVE (1998) and CHANSON and GUALTIERI (2008)

Fig. 10 - Vertical distributions of void fraction in a hydraulic jump with partially-developed inflow

(A) Experimental data: $Fr = 7.62$, $d_1 = 0.018$ m, $x_1 = 0.75$ m, $x-x_1 = 0.225, 0.30, 0.45$ m

(B) Definition sketch of the vertical profile

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Fig. 11 - Comparison of free surface and void fraction measurements using acoustic displacement sensor and phase detection probe - Experimental results

Figure 12 - Simultaneous measurements of free-surface and bubbly flow properties - Flow conditions: $Fr = 5.1$, $(x-x_1)/d_1 = 5.0$, $y/d_1 = 3.5$

(A) Phase detection probe raw voltage output

(B) Averaged signals (average over 100 points) with equal interval (0.02 s) of the low-pass filtered signal component (Band pass 0-25 Hz)

Figure 13 - Spectral analyses (FFT) of both phase-detection probe and acoustic displacement sensor signals - Flow conditions: $Fr = 5.1$ - (Left) conductivity probe signal; (Right) acoustic displacement sensor signal

(A) $(x-x_1)/d_1 = 5.0$, $y/d_1 = 3.5$

(B) $(x-x_1)/d_1 = 13.33$, $y/d_1 = 5.67$

Fig 14 - Normalised cross-correlation functions between the conductivity probe signal and acoustic displacement meter sensor - Flow conditions: $Fr = 5.1$

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TABLES

Table 1 - Experiments measurements of hydraulic jump fluctuations

Reference	x_1 (m)	d_1 (m)	Fr	Re	W (m)	Instrumentation
MADSEN (1981)	0.1	0.024	2	23,000	0.15	Resistance gage.
LONG et al. (1990)	0.042 0.082 0.081 0.077	0.025	4.0 6.0 8.0 9.0	49,200 73,700 98,300 110,600	0.47	High-speed video.
MOSSA & TOLVE (1998)	0.9	0.020	6.4	57,000	0.45	PIV and photography.
MOSSA (1999)	--	0.009 to 0.0165	1.3 to 8.9	17,000 to 54,800	0.3	Electrical probe and video.
	--	0.011 to 0.0385	3.0 to 9.9	30,600 to 122,500	0.4	
LIU et al. (2004)	-- -- --	0.071 0.071 0.041	2.0 2.5 3.32	118,600 147,700 86,100	0.46	High-speed video.
MOUAZE et al. (2005)	0.35	0.059 0.032	1.98 3.65	88,230 64,965	0.30	Wire gages.
MURZYN et al. (2007)	0.43 0.44 0.34 0.36 0.18 0.29 0.43	0.059 0.046 0.032 0.021 0.029 0.045 0.059	2.0 2.4 3.7 4.8 2.1 1.9 2.0	87,901 74,930 65,156 45,679 32,836 56,316 87,901	0.30	Wire gage.
CHANSON & GUALTIERI (2008)	0.5 1.0 1.0	0.013 0.028 0.029 0.029 0.025 0.027 0.028 0.027 0.024	8.5 4.6 5.0 6.7 7.5 5.1 6.9 7.3 8.6	38,000 69,000 77,000 100,000 94,000 68,000 100,000 98,000 98,000	0.25 0.50	Visual observations and conductivity probes.
KUCUKALI & CHANSON (2008)	1.0	0.024	4.7 5.0 5.8 6.9 8.5	54,335 57,800 67,050 79,770 98,265	0.50	Acoustic displacement meters and conductivity probes.
Present study	0.75	0.018	3.1 4.2 5.3 6.4 7.6 8.5	23,750 31,850 39,800 48,600 57,050 64,100	0.50	Acoustic displacement meters and conductivity probes.

Notes: Hydraulic jumps with partially-developed inflow conditions; d_1 : inflow depth; Fr: inflow Froude number; Re: Reynolds number defined as $Re = \rho \times V_1 \times d_1 / \mu$; W: channel width; x_1 : distance between sluice gate and jump toe; (--): information not available.

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Table 2 - Experimental observations of jump toe longitudinal oscillations and free-surface oscillations

Reference	x_1 (m)	d_1 (m)	Fr	F_{toe} (Hz)	St_{toe}	F_{fs} (Hz)	St_{fs}
LONG et al. (1990)	0.082	0.025	6	--	0.062 (*)		
	0.081	0.025	8	--	0.023 (*)		
	0.077	0.025	9	--	0.033 (*)		
MOSSA & TOLVE (1998)	0.90	0.020	6.4	0.67	0.0047	--	--
MOSSA (1999)	--	0.0287	4.52	--	--	1 to 3	0.015 to 0.045
CHANSON & GUALTIERI (2008)	0.5	0.013	8.5	1.27	0.0055	--	--
			4.6	0.59	0.0069	--	--
			5.0	0.75	0.0081	--	--
			6.7	1.18	0.0096	--	--
			7.5	1.27	0.0086	--	--
	1.0	0.027	5.1	1.25	0.013	--	--
			6.9	1.47	0.011	--	--
			7.3	1.59	0.011	--	--
			8.6	2.0	0.011	--	--
Present study	0.75	0.018	3.1	--	--	3.83	0.053
			4.2	--	--	3.14	0.032
			5.1	0.47	0.0040	--	--
			5.3	--	--	2.35	0.019
			6.4	--	--	1.13	0.0076
			7.6	0.68	0.0038	--	--
			8.3	0.77	0.0039	--	--

Notes: Hydraulic jumps with partially-developed inflow conditions; F_{fs} : free-surface fluctuation dominant frequency; F_{toe} : jump toe oscillation frequency; St : Strouhal number defined as $St = F \times d_1 / V_1$; (*): large vortical structure angular frequency; (--) : data not available.

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Table 3 - Experimental conditions for simultaneous measurements of free surface fluctuations and bubbly flow properties

Fr	d_1 (m)	x_1 (m)	$x-x_1$ (m)	$(x-x_1)/d_1$	Nb points	y_{30}/d_1
5.1	0.018	0.75	0.090	5.0	10	3.50
			0.165	9.17	9	4.83
			0.240	13.33	9	5.67

Note: Nb points = number of experiments with phase-detection probe at different vertical elevations.

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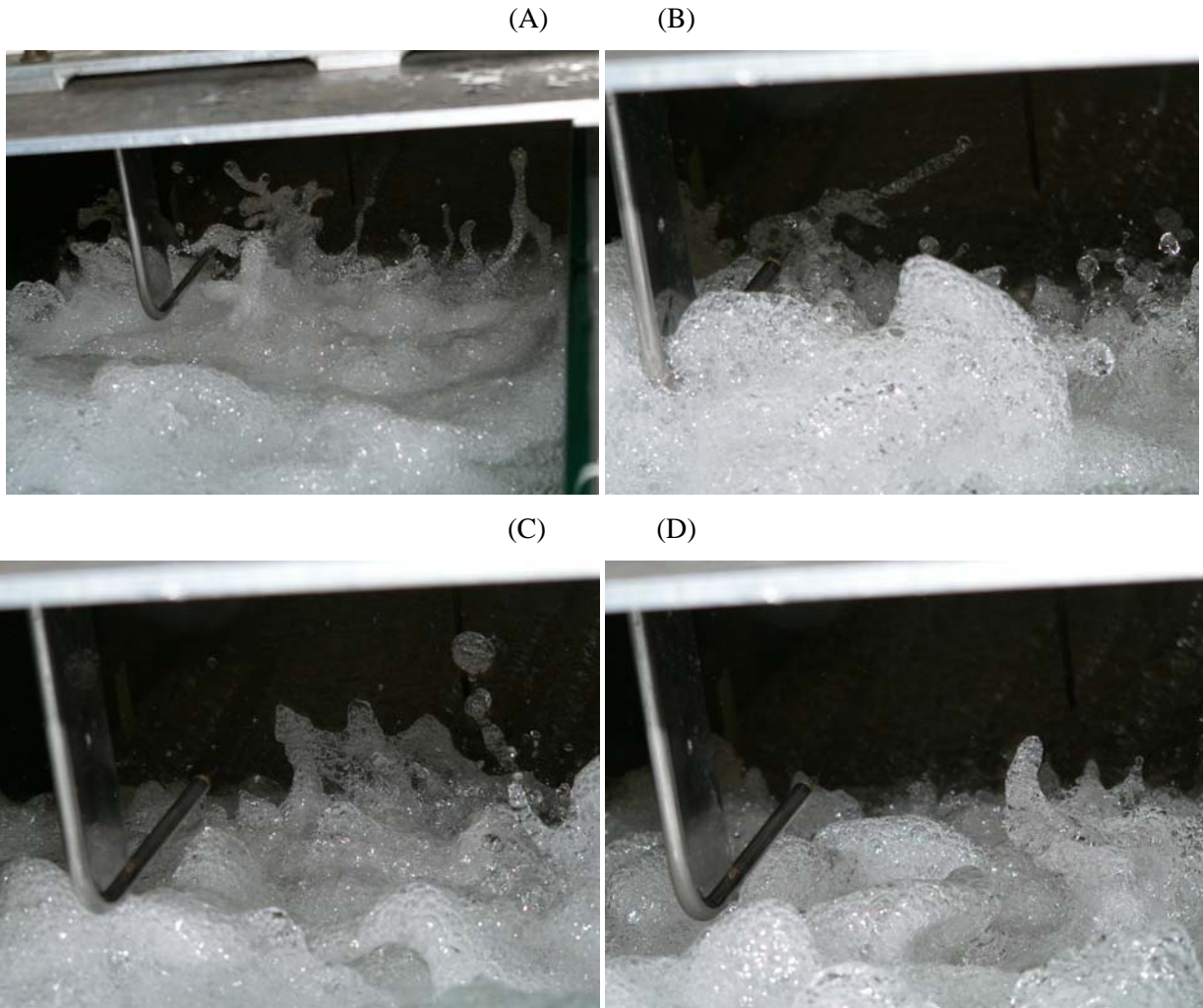


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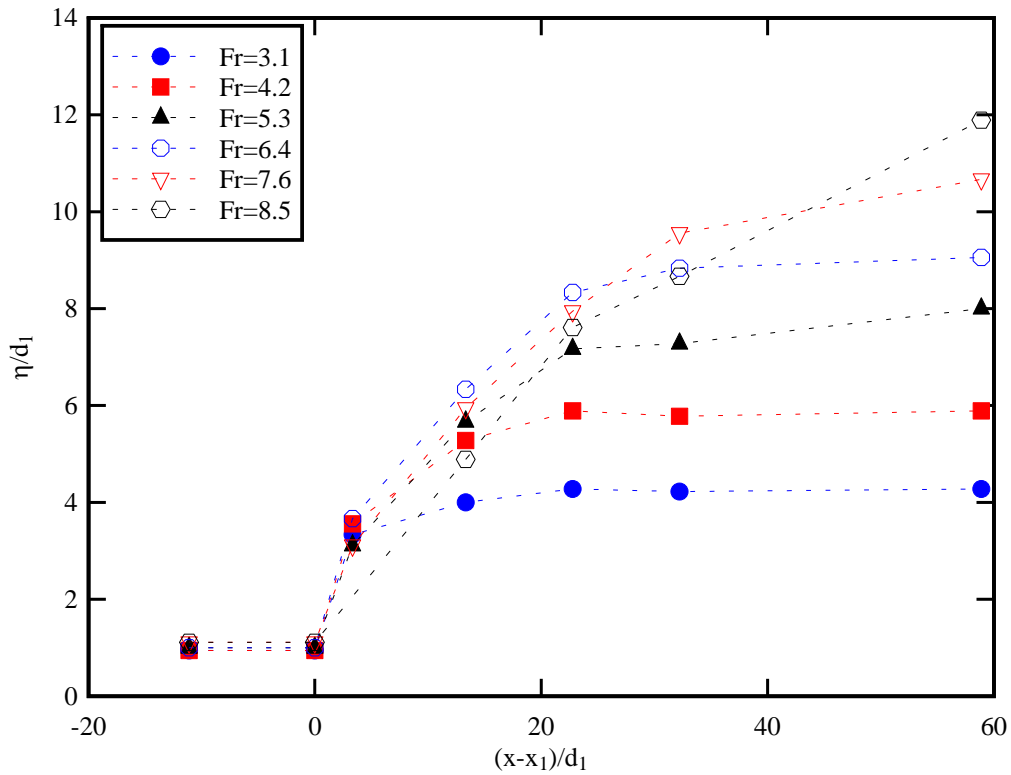


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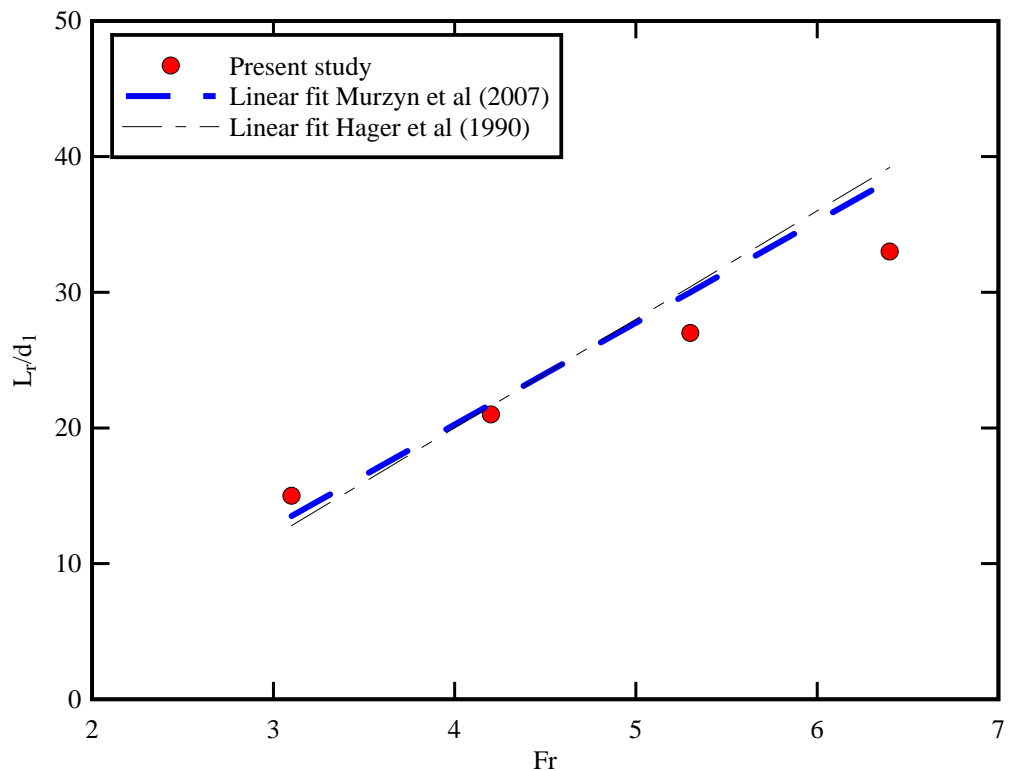
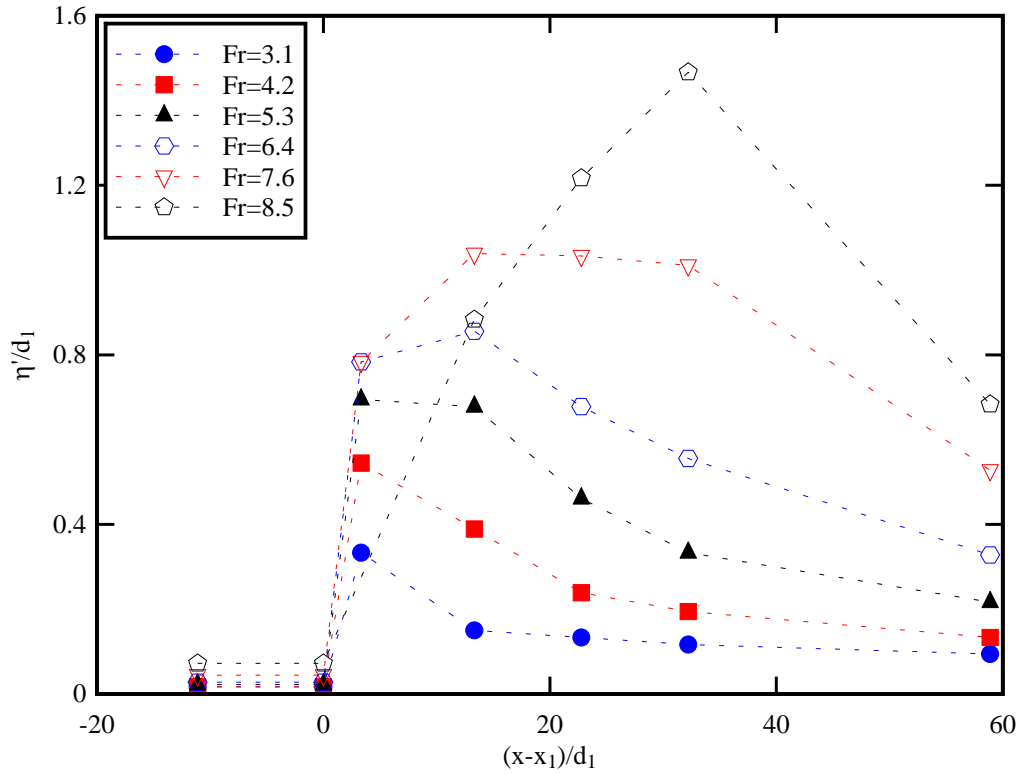
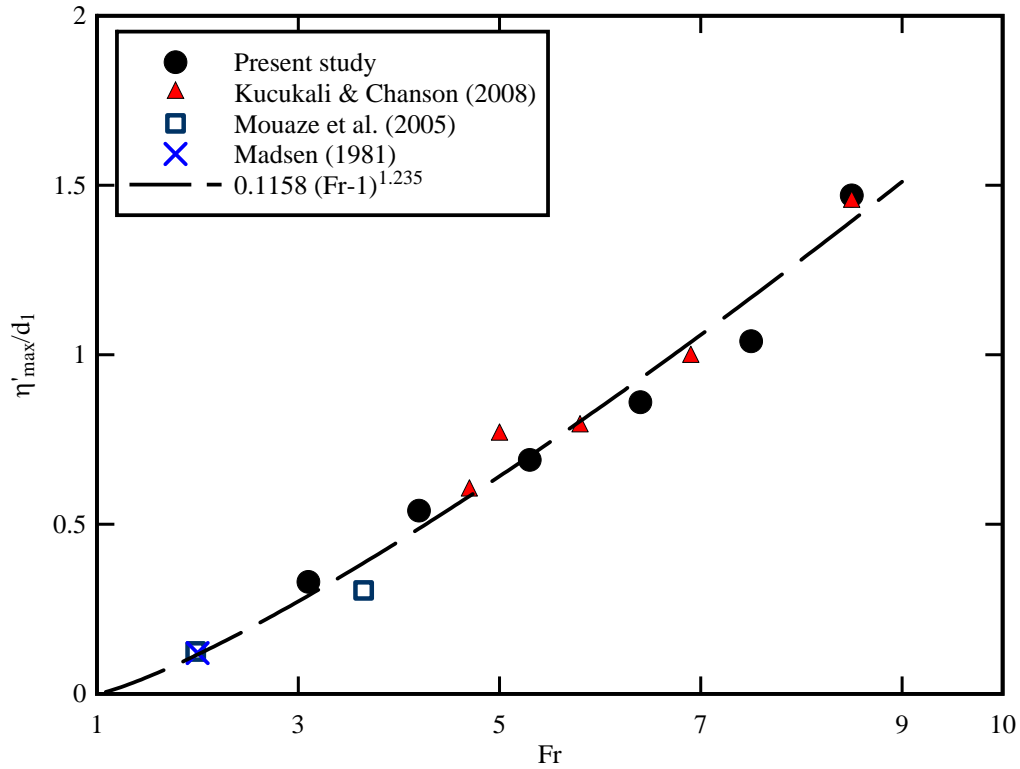


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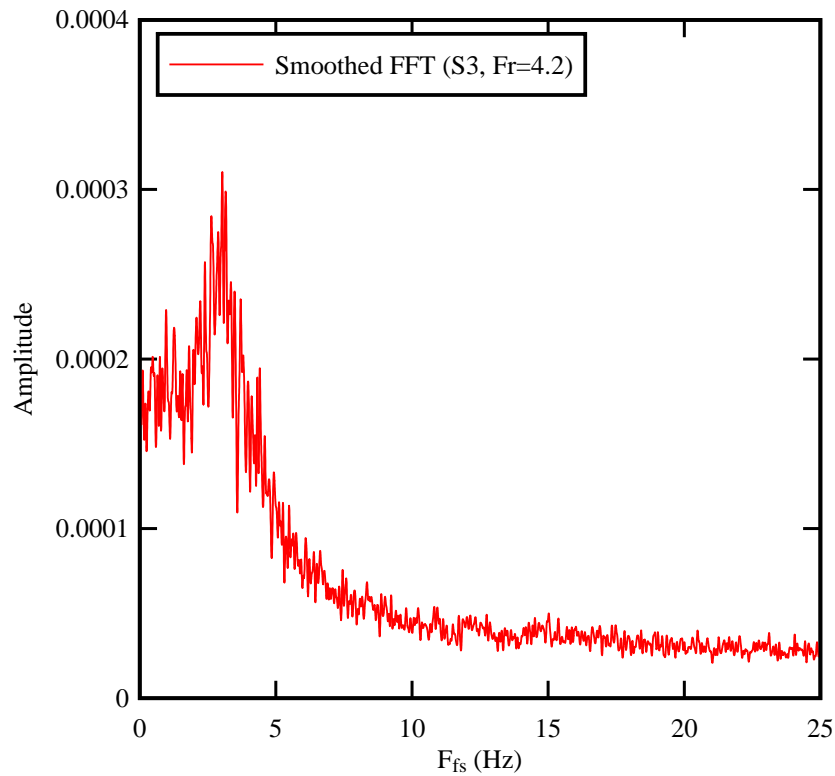
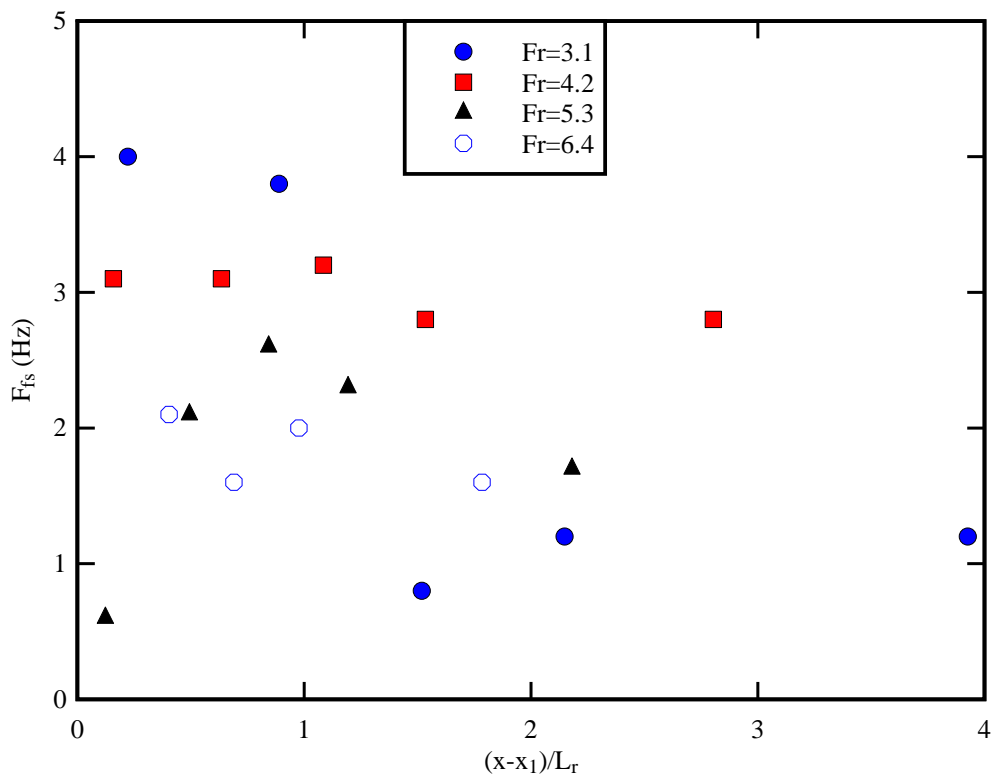


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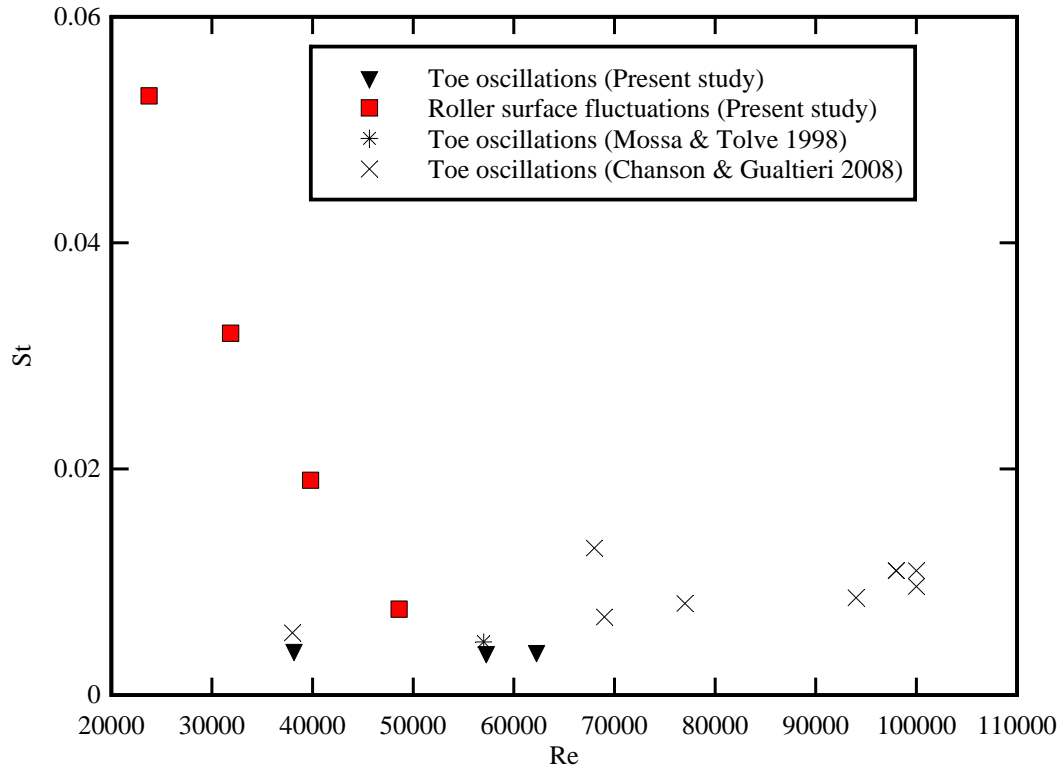
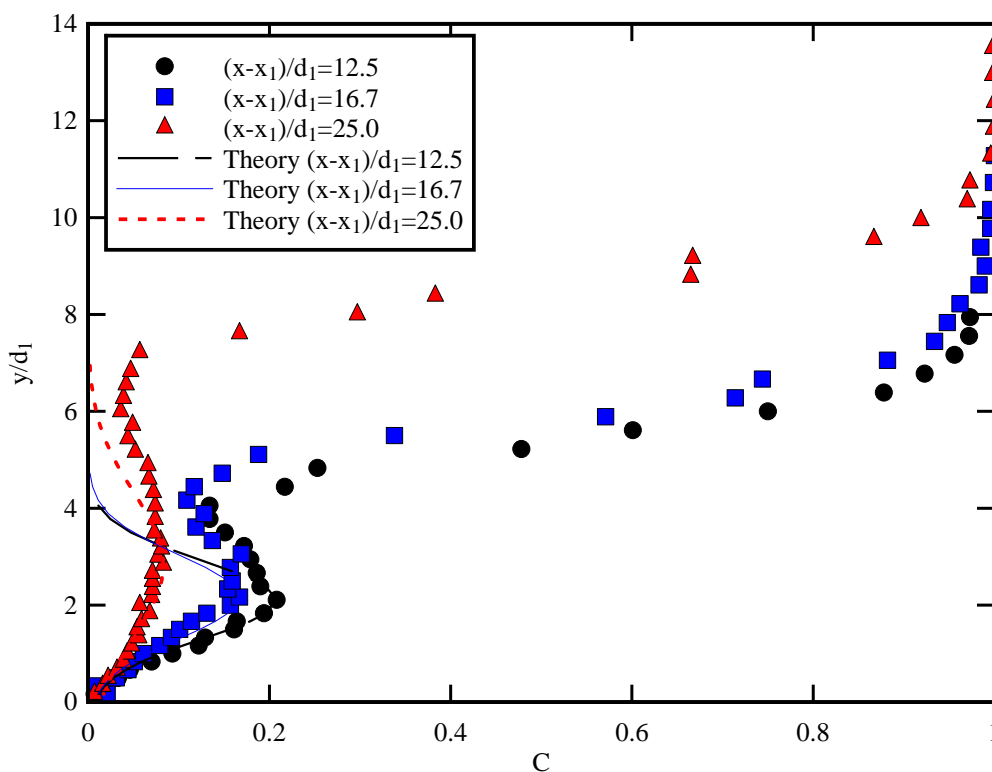
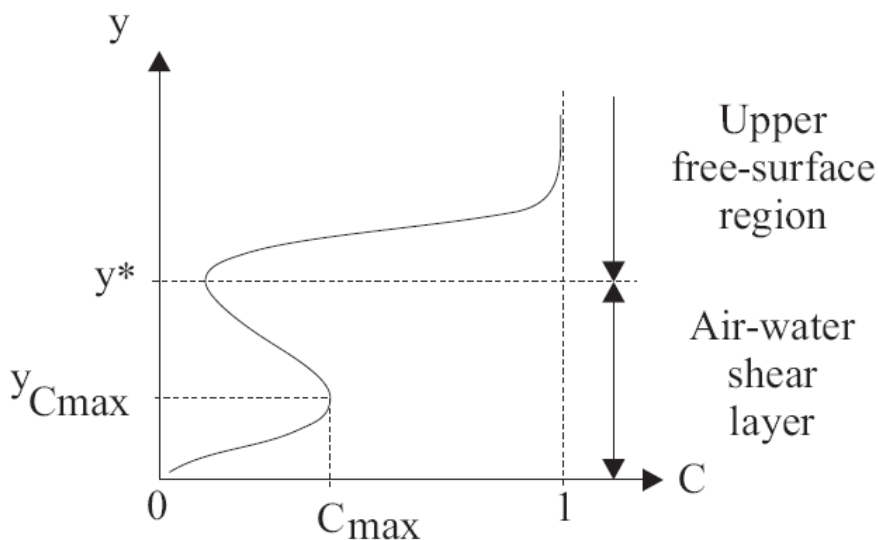


Fig. 10 - Vertical distributions of void fraction in a hydraulic jump with partially-developed inflow

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(B) Definition sketch of the vertical profile



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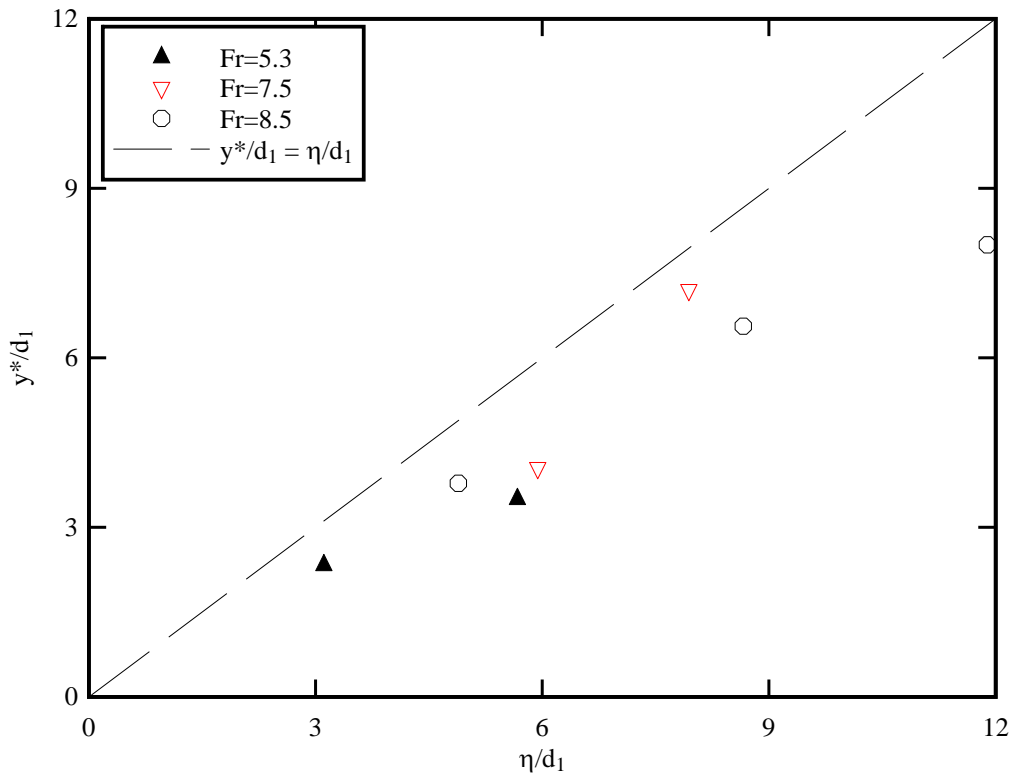
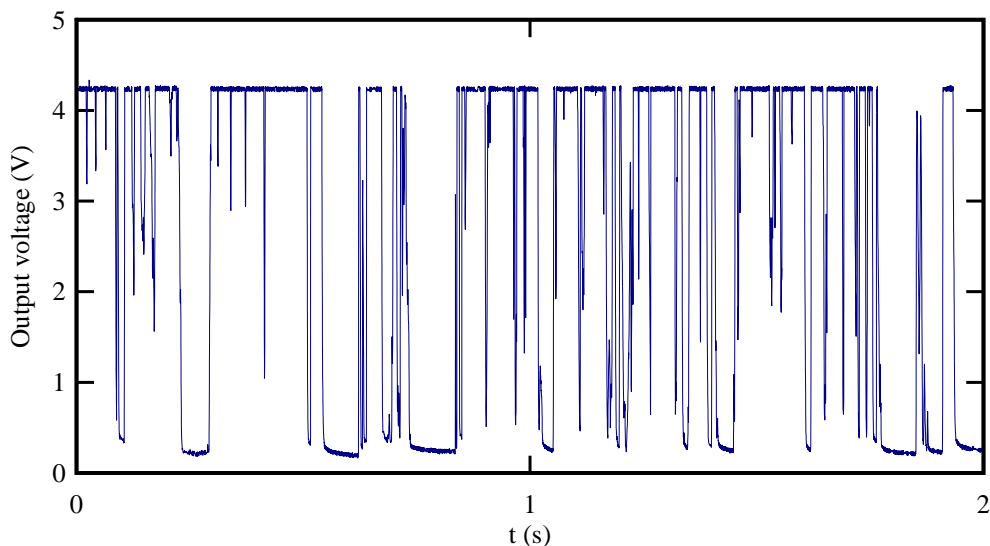
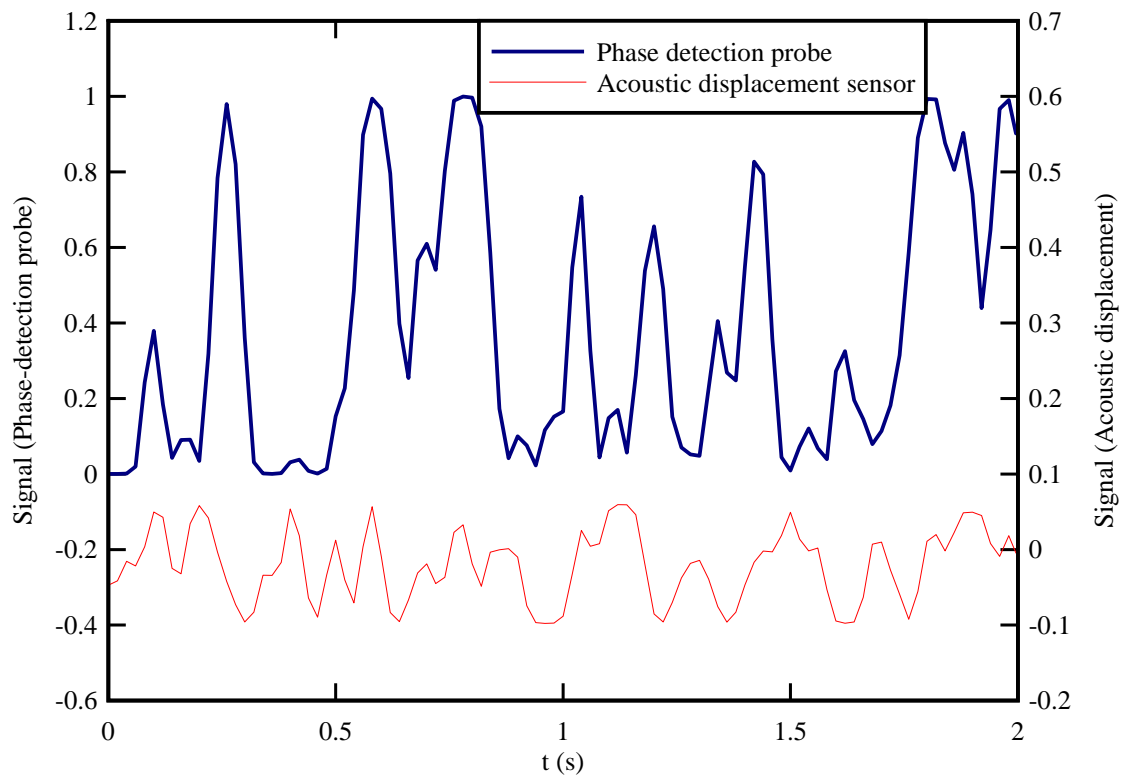


Figure 12 - Simultaneous measurements of free-surface and bubbly flow properties - Flow conditions: $Fr = 5.1$, $(x-x_1)/d_1 = 5.0$, $y/d_1 = 3.5$

(A) Phase detection probe raw voltage output



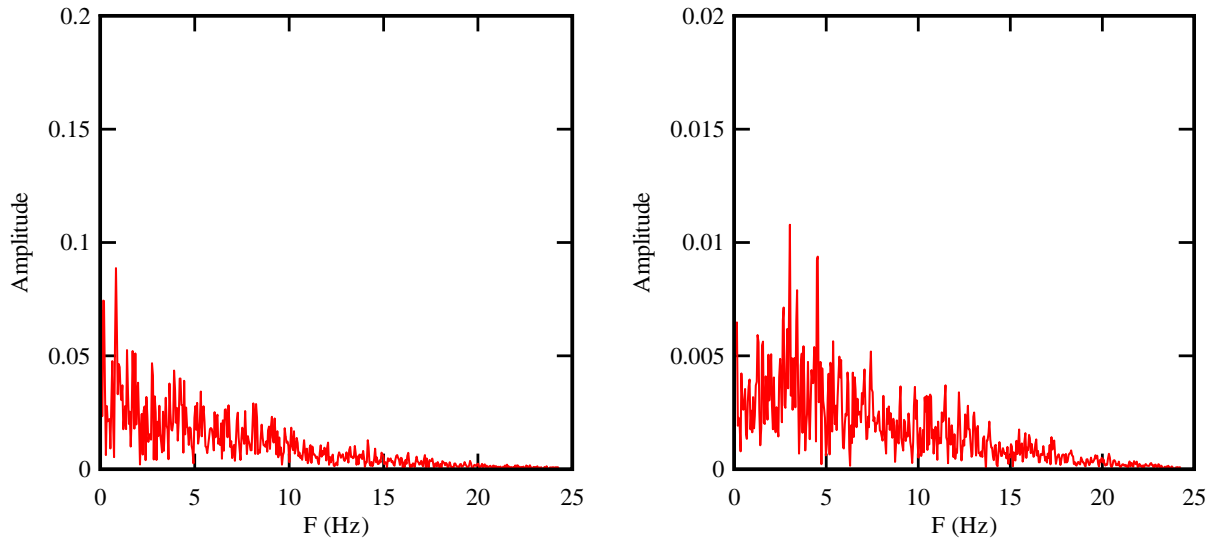
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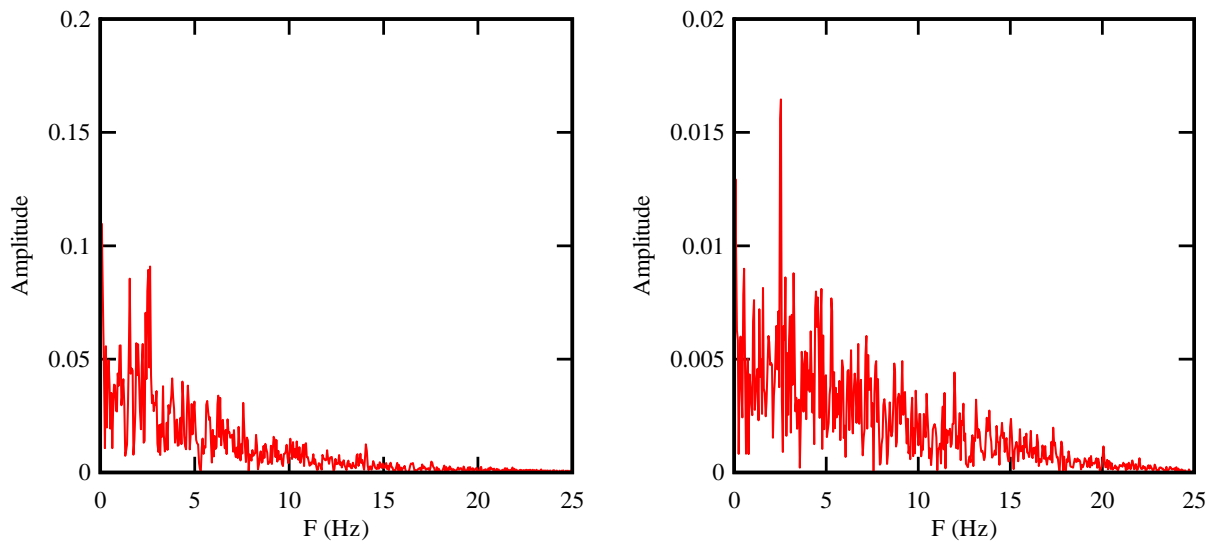


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