

New Reynolds equation for line contact based on the Carreau model modification by Bair

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A B S T R A C T

This paper presents a new form of the one-dimensional Reynolds equation for lubricants whose rheological behaviour follows a modified Carreau rheological model proposed by Bair. The results of the shear stress and flow rate obtained through a new Reynolds–Carreau equation are shown and compared with the results obtained by other researchers.

1. Introduction

The classic Reynolds formula, Eq. (1), is obtained under the Newtonian fluid hypothesis. It therefore does not include the variation in viscosity (η) through the film thickness (h).

$$\frac{\partial}{\partial x} \left(\frac{\rho h^3}{\eta} \frac{\partial p}{\partial x} \right) = 12u_m \frac{\partial(\rho h)}{\partial x} \quad (1)$$

For contacts that are highly loaded with small film thickness, which is typical of elastohydrodynamic lubrication (EHL), the fluid ceases to be Newtonian. The use of Reynolds equations that include non-Newtonian effects has led to improvements in the calculation of film thickness [1,2] and in the prediction of the friction coefficient.

The development of generalised Reynolds equations has been frequently based on two-dimensional models [3], such as the one presented in this article. This type of models is generally used in some applications of gears, cams and bearings [4,5].

However, the development of generalised exact Reynolds equations for non-Newtonian fluids has mainly focussed on fluids that fulfil the Eyring rheological model [6–11] or Ellis model [1,6], both of which have their limitations for predicting the variation in viscosity over a wide range of shear rate [12]. For other non-Newtonian liquid models, like the Ostwald-de Waele, Spriggs or Rabinowitsch models, exact Reynolds equations can be found in Ref. [13].

2. Rheological Carreau model

The Carreau model [14] is a non-Newtonian model where viscosity depends on shear rate, as follows:

$$\eta = \mu \left(1 + \left(\frac{\mu \dot{\gamma}}{G} \right)^2 \right)^{(n-1)/2} \quad (2)$$

When trying to find the generalised Reynolds equation for a non-Newtonian fluid, it is simpler to use a modified Carreau Eq. (3), proposed by Bair [1,15], with τ as the independent variable. This expression is valid for $0.2 < n < 1$, which means it can be applied to most non-Newtonian fluids that fit the Carreau model. Otherwise, a numerical calculation would be required to find the velocities field.

$$\eta = \mu \left(1 + \left(\frac{\tau}{G} \right)^2 \right)^{(1-1/n)/2} \quad (3)$$

3. Reynolds–Carreau equation

Bair and Khonsari [13,16] have put forward an approximate Reynolds equation adapted to the Carreau model. It is based on a numerical adjustment of the equations for flow rate (Q) and mid-plane shear stress (τ_m) taken from 103 numerical simulations for different operating conditions. The dimensionless expressions they obtained are as follows for the flow rate and mid-plane shear stress, and will serve for making a comparison

Nomenclature

α	variable of integration (m)
β	variable of integration (m)
$\dot{\gamma}$	shear rate (s^{-1})
η	viscosity (Pa s)
$\bar{\eta}$	dimensionless viscosity
μ	low shear viscosity (Pa s)
ρ	density (kg/m^3)
Σ	$(u_2 - u_1)/u_m$, slide-to-roll ratio (SRR)
τ	shear stress (Pa)
τ_m	mid-plane shear stress (Pa)
$\bar{\tau}_m$	mid-plane dimensionless shear stress
$\bar{\tau}_{m;Bair}$	mid-plane dimensionless shear stress calculated using analytical formula by Bair
$\bar{\tau}_{m;new}$	mid-plane dimensionless shear stress calculated using new analytical formula proposed in this article
$\bar{\tau}_{m;sim}$	mid-plane dimensionless shear stress calculated using numerical simulation by Bair
G	shear modulus (Pa)

h	film thickness (m)
n	carreau exponent
p	pressure (Pa)
\bar{p}	$(h/G)(\partial p/\partial x)$, dimensionless pressure gradient
Q	flow rate per unit length (m^2/s)
\bar{Q}	dimensionless flow rate
\bar{Q}_{Bair}	dimensionless flow rate calculated using the analytical formula by Bair
\bar{Q}_{new}	dimensionless flow rate calculated using the new analytical formula proposed in this article
\bar{Q}_{sim}	dimensionless flow rate calculated using numerical simulation by Bair
u	velocity (m/s)
u_m	average velocity (m/s)
u_i	velocity of solid i ($i = 1, 2$) (m/s)
\bar{u}	dimensionless velocity
\bar{u}_m	dimensionless mean velocity
\bar{u}_i	dimensionless velocity of solid i
x	coordinate in flow direction (m)
z	coordinate across flow direction (m)

with the new equations obtained in this article:

$$\bar{\tau}_m = \Sigma \bar{u}_m (1 + 0.53 \bar{p}^4)^{1 - (1/n)/4} \left[1 + (\Sigma \bar{u}_m)^2 \right]^{(n-1)/2}$$

$$\bar{Q} = \begin{cases} \bar{u}_m - \frac{\bar{p}}{12} + 0.57 \frac{(n-1)}{(1+(1/n)\bar{p})} \left| \frac{\bar{p}}{2} \right|^{(1/n)+(4/3)} \\ \bar{u}_m \left[(1 + 0.0047)(|\bar{p}|^{2.72} |\bar{\tau}_m|) \right]^{1 - (1/n)} - \frac{\bar{p}}{12} + 0.57 \frac{(n-1)}{(1+(1/n)\bar{p})} \left| \frac{\bar{p}}{2} \right|^{(1/n)+(4/3)} \end{cases} \quad (4)$$

These equations are fitted for the following dimensionless operating ranges:

$$\begin{aligned} n &= 0.3, 0.5 \text{ and } 0.75 \\ 0.01 &\leq \bar{u}_m \leq 1 \text{ and } \bar{u}_m = -1 \\ -27 &\leq \bar{p} \leq 27 \\ -0.5 &\leq \Sigma \leq 1 \end{aligned} \quad (5)$$

4. New Reynolds–Carreau equation

The procedure followed is identical to that for finding the Newtonian Reynolds equation. The first step is to balance the forces of an element of the contacting fluid, as Fig. 1 shows.

This gives

$$\frac{\partial p}{\partial x} = \frac{\partial \tau}{\partial z} \quad (6)$$

To obtain the Reynolds equation the velocities field needs to be calculated. The field can be defined by taking the integral expression as follows, where the origin of z is the mean point of the film thickness ($-h/2 < z < h/2$):

$$u(z) = u_1 + \int_{-h/2}^z \frac{\partial u}{\partial z} dz \quad (7)$$

Inserting the relationship between viscosity (η) and shear stress (τ) into the velocity gradient gives Eq. (8)

$$u = u_1 + \int_{-h/2}^z \frac{\tau}{\eta} dz = u_1 + \frac{1}{\mu} \int_{-h/2}^z \tau \left(1 + \left(\frac{\tau}{G} \right)^2 \right)^{[(1/n)-1]/2} dz \quad (8)$$

By taking the balance of forces obtained in Eq. (6) for $\frac{\partial p}{\partial x} \neq 0$ and making a substitution in the integration variable

$$\frac{\partial p}{\partial x} = \frac{\partial \tau}{\partial z} \Rightarrow \tau = \tau_m + z \frac{\partial p}{\partial x} \Rightarrow d\tau = dz \frac{\partial p}{\partial x} \quad (9)$$

where τ_m is defined as the value of the shear stress at point $z=0$. By making the corresponding change in the integration limits, expression (8) is transformed to

$$u = u_1 + \frac{1}{\mu(\partial p/\partial x)} \int_{\tau_m}^{\tau} \tau \left(1 + \left(\frac{\tau}{G} \right)^2 \right)^{[(1/n)-1]/2} d\tau \quad (10)$$

$$\text{where } \alpha = \tau_m - \frac{\partial p h}{\partial x} \quad (11)$$

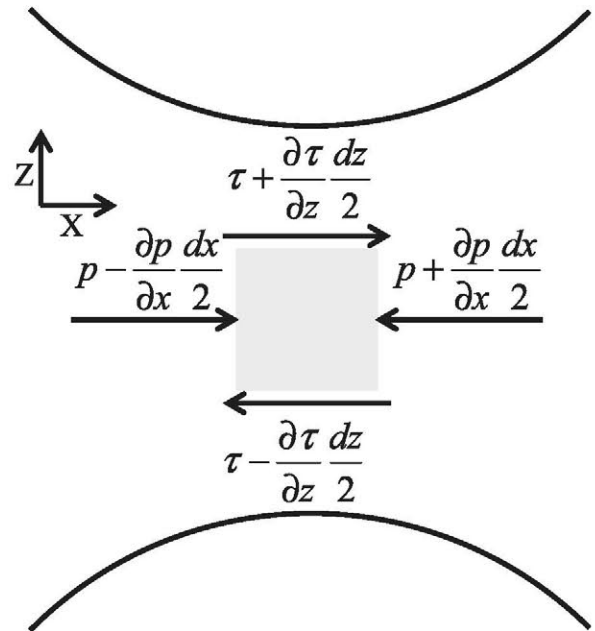


Fig. 1. Line contact diagram.

Integrating Eq. (10) with $\tau > 0$ gives the fluid velocity field across the film thickness:

$$u = u_1 + \frac{1}{\mu(\partial p/\partial x)} \left[\frac{G^{1-(1/n)n}((G^2 + \tau^2)^{(1+n)/2n} - (G^2 + \alpha^2)^{(1+n)/2n})}{1+n} \right] \quad (12)$$

Transforming back to terms of z and developing leads to:

$$u = u_1 + \frac{1}{\mu(\partial p/\partial x)} \frac{G^{1-(1/n)n}}{1+n} \left[\left(G^2 + \tau_m^2 + 2\tau_m z \frac{\partial p}{\partial x} + \left(z \frac{\partial p}{\partial x} \right)^2 \right)^{(1+n)/2n} - \left(G^2 + \tau_m^2 - \tau_m h \frac{\partial p}{\partial x} + \frac{1}{4} \left(h \frac{\partial p}{\partial x} \right)^2 \right)^{(1+n)/2n} \right] \quad (13)$$

By particularising u for $z=h/2$, Eq. (14) is obtained. This expression provides the value of τ_m , which a priori was unknown.

$$\frac{(u_2 - u_1)\mu(\partial p/\partial x)(1+n)}{G^{1-(1/n)n}} = \left(G^2 + \tau_m^2 + \tau_m h \frac{\partial p}{\partial x} + \frac{1}{4} \left(h \frac{\partial p}{\partial x} \right)^2 \right)^{(1+n)/2n} - \left(G^2 + \tau_m^2 - \tau_m h \frac{\partial p}{\partial x} + \frac{1}{4} \left(h \frac{\partial p}{\partial x} \right)^2 \right)^{(1+n)/2n} \quad (14)$$

This expression (14) has no analytical solution in general. Therefore, in order to find the value of τ_m numerical calculation is required. In this case, the Newton-Raphson method is used.

The next step is to calculate the flow rate per unit length by integrating the velocities field:

$$Q = \int_{-h/2}^{h/2} \left(u_1 + \frac{1}{\mu(\partial p/\partial x)} \frac{G^{1-(1/n)n}}{1+n} \left[\left(G^2 + \tau_m^2 + 2\tau_m z \frac{\partial p}{\partial x} + \left(z \frac{\partial p}{\partial x} \right)^2 \right)^{(1+n)/2n} - \left(G^2 + \tau_m^2 - \tau_m h \frac{\partial p}{\partial x} + \frac{1}{4} \left(h \frac{\partial p}{\partial x} \right)^2 \right)^{(1+n)/2n} \right] \right) dz \quad (15)$$

By again making the same substitution in the integration variable, Eq. (9), the following equation is obtained:

$$Q = \int_{\alpha}^{\beta} \frac{1}{(\partial p/\partial x)} \times \left(u_1 + \frac{1}{\mu(\partial p/\partial x)} \left[\frac{G^{1-(1/n)n}((G^2 + \tau^2)^{(1+n)/2n} - (G^2 + \alpha^2)^{(1+n)/2n})}{1+n} \right] \right) d\tau \quad (16)$$

where β is given by the expression:

$$\beta = \tau_m + \frac{\partial p h}{\partial x 2} \quad (17)$$

Under the following integration rule, obtained through Maple's symbolic integration modulus (where the integrand is positive) the solution can be calculated:

$$\int (G^{1-(1/n)n}(G^2 + \tau^2)^{(1+n)/2n}) d\tau = nG^2 \tau \cdot {}_2F_1 \left(\left[\frac{1}{2}, -\frac{1+n}{2n} \right], \left[\frac{3}{2} \right], -\frac{\tau^2}{G^2} \right) = nG^2 \tau \text{ hypergeom} \left(\left[\frac{1}{2}, -\frac{1+n}{2n} \right], \left[\frac{3}{2} \right], -\frac{\tau^2}{G^2} \right) \quad (18)$$

where ${}_2F_1$ is the Gauss hypergeometric function [14], that corresponds to a particular case of the generalised form of the hypergeometric function or Barnes extended hypergeometric function. It is denoted by ${}_jF_k(a, b, z)$, where j is the length of vector a and k is the length of vector b . For further clarity the following notation is used in this paper: hypergeom($[a_1, a_2, \dots], [b_1, b_2, \dots], z$) [17,18].

By developing expression (16) and simplifying, we obtain the expression for the flow rate:

$$Q = u_1 h - \frac{G^{(n-1)/n} n(G^2 + (\tau_m - (\partial p/\partial x)(h/2))^2)^{(1+n)/2n} h}{(1+n)\mu(\partial p/\partial x)} + \frac{nG^2(\tau_m + (\partial p/\partial x)(h/2))}{(1+n)\mu(\partial p/\partial x)^2} \times \text{hypergeom} \left(\left[\frac{1}{2}, -\frac{1+n}{2n} \right], \left[\frac{3}{2} \right], -\frac{(\tau_m + (\partial p/\partial x)(h/2))^2}{G^2} \right) - \frac{nG^2(\tau_m - (\partial p/\partial x)(h/2))}{(1+n)\mu(\partial p/\partial x)^2} \times \text{hypergeom} \left(\left[\frac{1}{2}, -\frac{1+n}{2n} \right], \left[\frac{3}{2} \right], -\frac{(\tau_m - (\partial p/\partial x)(h/2))^2}{G^2} \right) \quad (19)$$

Applying the law of conservation of mass will give the Reynolds-Carreau analytical equation:

$$\frac{d(\rho Q)}{dx} = 0 \quad u_1 \frac{d(\rho h)}{dx} = \frac{d}{dx} \left\{ \begin{array}{l} \rho \frac{G^{(n-1)/n} n(G^2 + (\tau_m - (\partial p/\partial x)(h/2))^2)^{(1+n)/2n} h}{(1+n)\mu(\partial p/\partial x)} \\ - \frac{nG^2(\tau_m + (\partial p/\partial x)(h/2))}{(1+n)\mu(\partial p/\partial x)^2} \text{hypergeom} \left(\left[\frac{1}{2}, -\frac{1+n}{2n} \right], \left[\frac{3}{2} \right], -\frac{(\tau_m + (\partial p/\partial x)(h/2))^2}{G^2} \right) \\ + \frac{nG^2(\tau_m - (\partial p/\partial x)(h/2))}{(1+n)\mu(\partial p/\partial x)^2} \text{hypergeom} \left(\left[\frac{1}{2}, -\frac{1+n}{2n} \right], \left[\frac{3}{2} \right], -\frac{(\tau_m - (\partial p/\partial x)(h/2))^2}{G^2} \right) \end{array} \right\} \quad (20)$$

The resulting expression (20) is a function of τ_m , which means that to find the flow rate, all that is required is to insert the values of τ_m calculated from Eq. (14).

So that the results from the equations obtained can be compared with those found by Bair-Khonsari, Eq. (20) is made dimensionless with the same dimensionless parameters used by Bair-Khonsari. These parameters are

$$\bar{\tau} = \frac{\tau}{G}, \quad \bar{u}_i = \frac{u_i \mu}{hG}, \quad \bar{z} = \frac{z}{h}, \quad \bar{\eta} = \frac{\eta}{\mu}, \quad \bar{p}' = p' \frac{h}{G}, \quad \bar{Q} = \frac{Q \mu}{h^2 G}, \quad \bar{\Sigma} = \frac{(u_2 - u_1)}{u_m} \quad (21)$$

By inserting these parameters in Eqs. (3), (13), (14) and (19), dimensionless equation is obtained for the Carreau model, the mid-plane shear stress, the flow rate and the Reynolds-Carreau equation:

$$\bar{\eta} = (1 + \bar{\tau}^2)^{[1-(1/n)]/2} \quad \bar{u} = \bar{u}_1 + \frac{n}{(1+n)} \frac{1}{\bar{p}'} \left[(1 + \bar{\tau}_m^2 + 2\bar{\tau}_m \bar{z} \bar{p}' + (\bar{z} \bar{p}')^2)^{(1+n)/2n} - \left(1 + \bar{\tau}_m^2 - \bar{\tau}_m \bar{p}' + \frac{1}{4} \bar{p}'^2 \right)^{(1+n)/2n} \right] \quad (\bar{u}_2 - \bar{u}_1) \frac{(1+n)}{n} \bar{p}' = \left[\left(1 + \bar{\tau}_m^2 + \bar{\tau}_m \bar{p}' + \frac{1}{4} \bar{p}'^2 \right)^{(1+n)/2n} - \left(1 + \bar{\tau}_m^2 - \bar{\tau}_m \bar{p}' + \frac{1}{4} \bar{p}'^2 \right)^{(1+n)/2n} \right] \quad \bar{Q} = \left\{ \begin{array}{l} \bar{u}_1 - \frac{n}{\bar{p}'(1+n)} \left(1 + (\bar{\tau}_m - \bar{p}'/2)^2 \right)^{(1+n)/2n} \\ + \frac{n}{\bar{p}'^2(1+n)} (\bar{\tau}_m + \bar{p}'/2) \text{hypergeom} \left(\left[\frac{1}{2}, -\frac{1+n}{2n} \right], \left[\frac{3}{2} \right], -(\bar{\tau}_m + \bar{p}'/2)^2 \right) \\ - \frac{n}{\bar{p}'^2(1+n)} (\bar{\tau}_m - \bar{p}'/2) \text{hypergeom} \left(\left[\frac{1}{2}, -\frac{1+n}{2n} \right], \left[\frac{3}{2} \right], -(\bar{\tau}_m - \bar{p}'/2)^2 \right) \end{array} \right\} \quad (22)$$

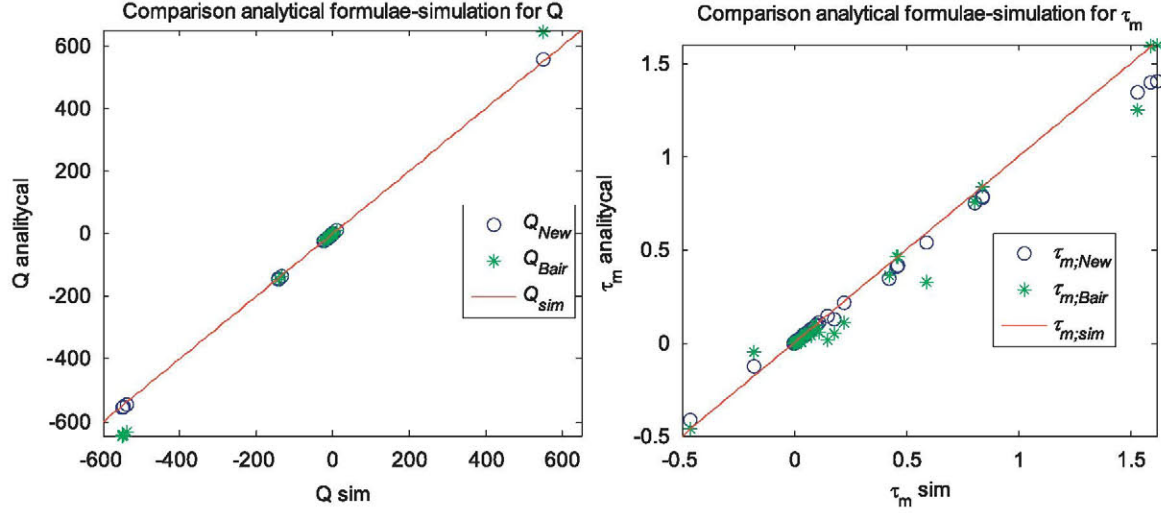


Fig. 2. Dimensionless results for flow rate and mid-plane shear stress compared with those obtained by Bair–Khonsari [16] through simulation and analytical formulation.

5. Results obtained with the Carreau model modification by Bair

Having presented the new equations, we will now compare the results for the flow rate and mid-plane shear stress obtained for different operating conditions with those obtained by Bair–Khonsari through their equations and simulations.

The appendix shows different operating conditions (\bar{u}_m , Σ , \bar{p} and n) together with the results obtained by Bair–Khonsari [16], (\bar{Q}_{sim} and \bar{Q}_{Bair} , $\bar{\tau}_{m;sim}$ and $\bar{\tau}_{m;Bair}$), and the new results (\bar{Q}_{new} and $\bar{\tau}_{m;new}$) through Eq. (22).

Fig. 2 shows the results obtained in graphic form. The values calculated through analytical expressions (5) and (22) are shown compared with the values obtained through simulations, which are represented by the line ($y=x$).

The errors obtained are less than 6% for the values of $\bar{\tau}_m$ and less than 8% for the values of \bar{Q}_{new} in all conditions. If a comparison is made with the data obtained by Bair–Khonsari [16], the errors for flow rate are of the same order, but the results for mid-plane shear stress over the whole simulated range are generally better.

6. Particularisation for the Ellis and Rabinowitsch models

After verifying the new equations, they are particularised for $n=1/3$ where the Carreau model is equivalent to Rabinowitsch and Ellis equations [1]. For these two models analytical results of the Reynolds equation are published in Refs. [1,13,19], and therefore the results derived from the new equations can be compared with the existing equations for this particular case.

By substituting $n=1/3$ in (3) and (14) the particular rheological model (23) is found, whereas Eq. (24) gives the shear stress in the mid-plane, τ_m .

$$\eta = \mu \left(1 + \left(\frac{\tau}{G} \right)^2 \right)^{-1} \quad (23)$$

$$4(u_2 - u_1)\mu \frac{\partial p}{\partial x} G^2 = 4\tau_m G^2 h \frac{\partial p}{\partial x} + \left(\left(\tau_m + \frac{h}{2} \frac{\partial p}{\partial x} \right)^4 - \left(\tau_m - \frac{h}{2} \frac{\partial p}{\partial x} \right)^4 \right) \quad (24)$$

In order to verify the flow rate equation, expression (18) particularised to $n=1/3$ leads to the polynomial

$$\begin{aligned} nG^2\tau \text{ hypergeom} \left(\left[\frac{1}{2}, -\frac{1+n}{2n} \right], \left[\frac{3}{2} \right], -\frac{\tau^2}{G^2} \right) \Big|_{n=1/3} \\ = \frac{1}{3} G^2 \tau \left(1 - \frac{2}{3} \left(-\frac{\tau^2}{G^2} \right) + \frac{1}{5} \left(-\frac{\tau^2}{G^2} \right)^2 \right) \end{aligned} \quad (25)$$

By inserting Eq. (25) into (19) and developing, flow rate expression (26) is obtained.

$$\begin{aligned} Q = u_1 h + \frac{\tau_m h^2}{2\mu} - \frac{\partial p}{\partial x} \frac{h^3}{12\mu} + \frac{1}{4G^2 \mu (\partial p / \partial x)^2} \\ \times \left(\frac{(\tau_m + (h/2)(\partial p / \partial x))^5}{5} - \frac{(\tau_m - (h/2)(\partial p / \partial x))^5}{5} - \frac{h}{2} \frac{\partial p}{\partial x} \left(\tau_m - \frac{h}{2} \frac{\partial p}{\partial x} \right)^4 \right) \end{aligned} \quad (26)$$

If Eqs. (24) and (26) are developed to deduce the mid-plane shear stress and flow rate, a complete equivalency is found with the results presented by Bair [1,13] when regarding the Rabinowitsch model and the particular case of the Ellis model when $n=1/3$.

7. Conclusions

This article proposes a new Reynolds–Carreau equation for line contact and expressions for the mid-plane shear stress and flow rate. The equations have been obtained in a general way, unlike the approximate expressions published previously for the Carreau model, which are obtained by fitting the simulations to specific conditions. Therefore, the new equations are applicable to a wider range of conditions.

The new formulae are analytical, simple and easy to implement since the hypergeometric function can be found in most programming libraries (Matlab, Mathematica, Maple among others) and moreover the results obtained with new equations show a good correlation with simulations presented by Bair–Khonsari [16].

In addition, the presented equations have been particularised for $n=1/3$ in order to obtain comparable expressions to the Ellis and Rabinowitsch models. As analytical equations are available for the Ellis and Rabinowitsch models, it has been verified that the particular solution attained by using the new equations is identical to that presented previously.

The Carreau model describes the behaviour of a large number of lubricants and we hope that the equations deduced will be of help in calculating friction and film thickness in line contacts under elastohydrodynamic lubrication. Moreover, the use of analytical equations facilitates the solving process because it avoids numerical integration of the velocity field of the fluid, unlike numerous generalised Reynolds equations where flow factors are used [6,20,21].

Appendix

See Table A.1

Table A.1
Summary of results.

\bar{u}_m	Σ	\bar{p}'	n	\bar{Q}_{sim}	\bar{Q}_{Bair}	\bar{Q}_{new}	$\bar{\tau}_{m,sim}$	$\bar{\tau}_{m,Bair}$	$\bar{\tau}_{m,new}$
0.1	0	0.111	0.3	9.07E-02	9.075E-02	9.073E-02	0	0	0
0.1	0	0.333	0.3	7.20E-02	7.219E-02	7.171E-02	0	0	0
0.1	0	1	0.3	1.19E-02	1.304E-02	1.877E-03	0	0	0
0.1	0	3	0.3	-3.69E-01	-3.536E-01	-5.817E-01	0	0	0
0.1	0	9	0.3	-1.40E+01	-1.208E+01	-1.532E+01	0	0	0
0.1	0	27	0.3	-5.49E+02	-6.444E+02	-5.548E+02	0	0	0
0.1	0	18	0.3	-1.42E+02	-1.466E+02	-1.453E+02	0	0	0
0.1	0.5	0.111	0.3	9.10E-02	9.075E-02	9.065E-02	5.00E-02	4.995E-02	4.968E-02
0.1	0.5	0.333	0.3	7.23E-02	7.219E-02	7.148E-02	4.96E-02	4.977E-02	4.830E-02
0.1	0.5	1	0.3	1.20E-02	1.304E-02	1.429E-03	4.55E-02	3.898E-02	3.849E-02
0.1	0.5	3	0.3	-3.69E-01	-3.536E-01	-5.819E-01	1.82E-02	5.499E-03	1.264E-02
0.1	0.5	9	0.3	-1.40E+01	-1.208E+01	-1.532E+01	9.90E-04	4.293E-04	1.414E-03
0.1	0.5	27	0.3	-5.49E+02	-6.444E+02	-5.548E+02	2.90E-04	3.308E-05	1.145E-04
0.1	0.5	18	0.3	-1.42E+02	-1.466E+02	-1.453E+02	2.60E-04	8.520E-05	2.925E-04
1	0	0.111	0.3	9.90E-01	9.907E-01	9.907E-01	0	0	0
1	0	0.333	0.3	9.72E-01	9.722E-01	9.717E-01	0	0	0
1	0	1	0.3	9.12E-01	9.130E-01	9.019E-01	0	0	0
1	0	3	0.3	5.31E-01	5.464E-01	3.183E-01	0	0	0
1	0	9	0.3	-1.31E+01	-1.118E+01	-1.442E+01	0	0	0
1	0	27	0.3	-5.48E+02	-6.435E+02	-5.539E+02	0	0	0
1	0	18	0.3	-1.41E+02	-1.457E+02	-1.444E+02	0	0	0
1	0.5	0.111	0.3	9.91E-01	9.907E-01	9.850E-01	4.62E-01	4.624E-01	4.143E-01
1	0.5	0.333	0.3	9.67E-01	9.722E-01	9.552E-01	4.58E-01	4.607E-01	4.067E-01
1	0.5	1	0.3	8.93E-01	9.130E-01	8.653E-01	4.21E-01	3.608E-01	3.457E-01
1	0.5	3	0.3	5.05E-01	5.464E-01	3.014E-01	1.78E-01	5.091E-02	1.256E-01
1	0.5	9	0.3	-1.31E+01	-1.118E+01	-1.442E+01	1.50E-02	3.974E-03	1.414E-02
1	0.5	27	0.3	-5.48E+02	-6.435E+02	-5.539E+02	1.18E-03	3.062E-04	1.145E-03
1	0.5	18	0.3	-1.41E+02	-1.457E+02	-1.444E+02	2.20E-03	7.887E-04	2.925E-03
10	0	0.111	0.3	9.99E+00	9.991E+00	9.991E+00	0	0	0
10	0	0.333	0.3	9.98E+00	9.972E+00	9.972E+00	0	0	0
10	0	1	0.3	9.92E+00	9.913E+00	9.902E+00	0	0	0
10	0	3	0.3	9.53E+00	9.546E+00	9.318E+00	0	0	0
10	0	9	0.3	-4.12E+00	-2.185E+00	-5.416E+00	0	0	0
10	0	27	0.3	-5.39E+02	-6.345E+02	-5.449E+02	0	0	0
10	0	18	0.3	-1.32E+02	-1.367E+02	-1.354E+02	0	0	0
10	0.5	0.111	0.3	9.94E+00	9.991E+00	9.916E+00	1.62E+00	1.598E+00	1.403E+00
10	0.5	0.333	0.3	9.67E+00	9.972E+00	9.750E+00	1.59E+00	1.593E+00	1.397E+00
10	0.5	1	0.3	9.29E+00	9.913E+00	9.279E+00	1.53E+00	1.247E+00	1.346E+00
10	0.5	9	0.3	-4.20E+00	-2.185E+00	-5.503E+00	1.49E-01	1.374E-02	1.412E-01
10	0.5	27	0.3	-5.39E+02	-6.345E+02	-5.449E+02	9.90E-03	1.059E-03	1.145E-02
10	0.5	18	0.3	-1.32E+02	-1.367E+02	-1.355E+02	2.90E-02	2.726E-03	2.925E-02
-1	0	0.111	0.3	-1.01E+00	-1.009E+00	-1.009E+00	0	0	0
-1	0	3	0.3	-1.47E+00	-1.454E+00	-1.682E+00	0	0	0
-1	0	27	0.3	-5.50E+02	-6.455E+02	-5.559E+02	0	0	0
1	0	-0.111	0.3	1.009E+00	1.009E+00	1.009E+00	0	0	0
1	0	-3	0.3	1.47E+00	1.454E+00	1.682E+00	0	0	0
1	0	-27	0.3	5.50E+02	6.455E+02	5.559E+02	0	0	0
1	-0.5	0.111	0.3	9.85E-01	9.907E-01	9.850E-01	-4.62E-01	-4.624E-01	-4.143E-01
1	-0.5	3	0.3	5.00E-01	5.464E-01	3.014E-01	-1.79E-01	-5.091E-02	-1.256E-01

Table A.1 (continued)

\bar{u}_m	Σ	\bar{p}'	n	\bar{Q}_{sim}	\bar{Q}_{Bair}	\bar{Q}_{new}	$\bar{\tau}_{m,sim}$	$\bar{\tau}_{m,Bair}$	$\bar{\tau}_{m,new}$
1	-0.5	27	0.3	-5.48E+02	-6.435E+02	-5.539E+02	-1.18E-03	-3.062E-04	-1.145E-03
-1	-0.5	0.111	0.3	-1.009E+00	-1.009E+00	-1.015E+00	4.62E-01	4.624E-01	4.143E-01
-1	-0.5	3	0.3	-1.50E+00	-1.454E+00	-1.699E+00	1.78E-01	5.091E-02	1.256E-01
-1	-0.5	27	0.3	-5.50E+02	-6.455E+02	-5.559E+02	1.18E-03	3.062E-04	1.145E-03
0.01	0	0.333	0.5	-1.79E-02	-1.847E-02	-1.798E-02	0	0	0
0.01	0	1	0.5	-7.65E-02	-8.276E-02	-7.933E-02	0	0	0
0.01	0	3	0.5	-3.33E-01	-3.623E-01	-3.699E-01	0	0	0
0.01	0	9	0.5	-2.54E+00	-2.328E+00	-2.640E+00	0	0	0
0.01	0	27	0.5	-2.28E+01	-2.285E+01	-2.290E+01	0	0	0
0.01	0	18	0.5	-1.01E+01	-9.493E+00	-1.024E+01	0	0	0
0.01	0.5	0.333	0.5	-1.79E-02	-1.847E-02	-1.798E-02	5.00E-03	4.992E-03	4.932E-03
0.01	0.5	1	0.5	-7.65E-02	-8.276E-02	-7.933E-02	4.70E-03	4.496E-03	4.472E-03
0.01	0.5	3	0.5	-3.33E-01	-3.623E-01	-3.699E-01	3.20E-03	1.942E-03	2.773E-03
0.01	0.5	9	0.5	-2.54E+00	-2.328E+00	-2.640E+00	1.10E-03	6.511E-04	1.085E-03
0.01	0.5	27	0.5	-2.28E+01	-2.285E+01	-2.290E+01	6.00E-04	2.170E-04	3.694E-04
0.01	0.5	18	0.5	-1.01E+01	-9.493E+00	-1.024E+01	6.40E-04	3.256E-04	5.522E-04
0.1	0	0.111	0.5	9.07E-02	9.069E-02	9.074E-02	0	0	0
0.1	0	0.333	0.5	7.20E-02	7.153E-02	7.202E-02	0	0	0
0.1	0	1	0.5	1.34E-02	7.241E-03	1.067E-02	0	0	0
0.1	0	3	0.5	-2.43E-01	-2.723E-01	-2.799E-01	0	0	0
0.1	0	9	0.5	-2.45E+00	-2.238E+00	-2.550E+00	0	0	0
0.1	0	27	0.5	-2.27E+01	-2.276E+01	-2.281E+01	0	0	0
0.1	0	18	0.5	-1.00E+01	-9.403E+00	-1.015E+01	0	0	0
0.1	0.5	0.111	0.5	9.09E-02	9.069E-02	9.071E-02	4.99E-02	4.997E-02	4.986E-02
0.1	0.5	0.333	0.5	7.23E-02	7.153E-02	7.192E-02	4.96E-02	4.989E-02	4.926E-02
0.1	0.5	1	0.5	1.36E-02	7.241E-03	1.045E-02	4.70E-02	4.493E-02	4.469E-02
0.1	0.5	3	0.5	-2.43E-01	-2.723E-01	-2.801E-01	3.09E-02	1.941E-02	2.773E-02
0.1	0.5	9	0.5	-2.45E+00	-2.238E+00	-2.550E+00	1.12E-02	6.507E-03	1.085E-02
0.1	0.5	27	0.5	-2.27E+01	-2.276E+01	-2.281E+01	4.00E-03	2.169E-03	3.694E-03
0.1	0.5	18	0.5	-1.00E+01	-9.403E+00	-1.015E+01	5.80E-03	3.254E-03	5.522E-03
0.1	1	0.111	0.5	9.12E-02	9.069E-02	9.061E-02	9.97E-02	9.975E102	9.936E-02
0.1	1	0.333	0.5	7.24E-02	7.153E-02	7.163E-02	9.90E-02	9.959E-02	9.819E-02
0.1	1	1	0.5	1.35E-02	7.241E-03	9.785E-03	9.39E-02	8.969E-02	8.918E-02
0.1	1	3	0.5	-2.43E-01	-2.723E-01	-2.806E-01	6.16E-02	3.875E-02	5.545E-02
0.1	1	9	0.5	-2.45E+00	-2.238E+00	-2.550E+00	2.20E-02	1.299E-02	2.169E-02
0.1	1	27	0.5	-2.27E+01	-2.276E+01	-2.281E+01	7.37E-03	4.330E-03	7.387E-03
0.1	1	18	0.5	-1.00E+01	-9.403E+00	-1.015E+01	1.12E-02	6.495E-03	1.104E-02
1	1	0.111	0.5	9.90E-01	9.907E-01	9.837E-01	8.40E-01	8.409E-01	7.857E-01
1	1	0.333	0.5	9.61E-01	9.715E-01	9.513E-01	8.37E-01	8.395E-01	7.819E-01
1	1	1	0.5	8.74E-01	9.072E-01	8.564E-01	8.05E-01	7.561E-01	7.487E-01
1	1	3	0.5	5.95E-01	6.277E-01	5.615E-01	5.88E-01	3.266E-01	5.365E-01
1	1	9	0.5	-1.58E+00	-1.338E+00	-1.662E+00	2.22E-01	1.095E-01	2.168E-01
1	1	27	0.5	-2.18E+01	-2.186E+01	-2.191E+01	7.40E-02	3.650E-02	7.387E-02
1	1	18	0.5	-9.10E+00	-8.503E+00	-9.251E+00	1.11E-01	5.475E-02	1.104E-01
0.1	0	0.111	0.75	9.07E-02	9.050E-02	9.075E-02	0	0	0
0.1	0	0.333	0.75	7.21E-02	7.071E-02	7.217E-02	0	0	0
0.1	0	3	0.75	-1.82E-01	-2.100E-01	-1.868E-01	0	0	0
0.1	0	9	0.75	-1.03E+00	-1.025E+00	-1.034E+00	0	0	0
0.1	0	27	0.75	-4.72E+00	-4.487E+00	-4.732E+00	0	0	0
0.1	1	0.111	0.75	9.12E-02	9.050E-02	9.070E-02	9.97E-02	9.988E-02	9.978E-02
0.1	1	0.333	0.75	7.26E-02	7.071E-02	7.204E-02	9.95E-02	9.982E-02	9.939E-02
0.1	1	3	0.75	-1.82E-01	-2.100E-01	-1.872E-01	8.36E-02	7.287E-02	8.215E-02
0.1	1	9	0.75	-1.02E+00	-1.025E+00	-1.034E+00	6.55E-02	5.062E-02	6.009E-02
0.1	1	27	0.75	-4.72E+00	-4.487E+00	-4.732E+00	4.28E-02	3.510E-02	4.196E-02

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