



# Analysis of energy requirements versus comfort levels for the integration of phase change materials in buildings



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## ABSTRACT

This paper investigates the importance of the design parameters when looking at possible energy savings and comfort enhancement in a building using Phase Change Materials (PCMs). Computer based simulations are performed using a simulation software for modelling a house and its thermal behaviour over a year. It is found that by varying the heating set point and the phase change (melting) temperature range of the PCM, significant changes can be observed. Some poor scenarios show that the integration of PCM can increase both the discomfort (up to 6% more discomfort hours) and the energy requirements (up to 25% more energy needed). On the other hand, appropriate scenarios bring significant energy savings (up to 33% less energy needed) and comfort enhancement (up to 31% less discomfort hours). This highlights the strong need for a clever design when integrating PCM into buildings. The goal is to find a trade-off between energy savings and comfort enhancement. The PCM with a phase change temperature range between 21 °C and 26 °C shows the best results. The study is based on climate conditions for Auckland City in New Zealand but most of the conclusions drawn can be applied to any climate.

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## 1. Introduction

The use of PCMs as a mean for thermal energy storage in buildings is complex but can provide multiple advantages. The interest of PCMs is that by going through a phase change, large amounts of heat can be stored and released. Building materials store thermal energy through sensible heat which requires a temperature difference while a PCM involves the latent heat at a selected temperature, therefore requiring lower temperature difference for the heat storage. These materials have been investigated and still are for their properties which can be compared using existing reviews [1–3].

A convenient integration of PCMs in buildings is through the use of impregnated gypsum boards, which are commonly added as an internal layer inside a building. This low-cost component provides a suitable structure for PCM containment which is great for both new constructions and retrofitting. One of the main interests of the gypsum board is that it is the innermost layer for most constructions. The replacement of an existing conventional gypsum board is therefore an easy task [4,5]. There are three methods of how PCM can be incorporated into the construction material:

direct incorporation, immersion and encapsulation. Micro-encapsulation prevents problems associated with PCM volume change and provides greater heat exchange area which increases heat transfer rate. Micro-encapsulation is the most appropriate when using gypsum boards, it prevents PCM leakage shows good cycling stability [6].

The main interest when using PCMs in buildings is that they should have the ability to reduce indoor temperature swings without any external help. To do so, the indoor temperature must vary across the phase change temperature range of the PCM. When the material goes from solid-phase to liquid-phase through a melting process, it absorbs large amounts of heat and therefore slows down the temperature rise that would otherwise occur inside the building. When the ambient temperature drops, the PCM goes through a solidification process and releases heat which has the effect of slowing down the decrease in temperature inside the building [7].

Not only can the integration of PCM in buildings reduce the energy requirements, but it can also enhance comfort. By reducing the indoor temperature variations, buildings rely less on heating, ventilation and air conditioning (HVAC) system. PCMs can also reduce the period of heating and cooling as temperature peaks can be avoided. At the moment the emphasis in most investigations on PCMs is mainly about the energy consumption and not about comfort levels. However the goal of an HVAC system is to provide comfortable conditions for people, therefore a better

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understanding on how these two parameters vary in the presence of PCMs is necessary since it has not been studied in the literature before. Also, the question of the optimum choice PCM phase change temperature range is not assessed in the literature and turns out to be of great importance as investigated in this paper.

The objectives of this work are: (1) to show how the HVAC set points influence both energy requirements and comfort; (2) to investigate how the phase change temperature range of PCM influences the energy consumption and the level of comfort; (3) to show that appropriate PCM designs depend on the trade-off between energy requirements and comfort levels.

## 2. Methodology of the investigation for a typical house using computer simulation

### 2.1. Development of a building simulation model

Computer-based simulations show several advantages when it can be validated against past experimental measurements. An important fact is that they save time and can be used to perform a study over a year-round within a few hours of simulation. It also allows great flexibility to show the influence of few parameters, while keeping all the other factors constant. To perform the calculations, the interface Design Builder which is based on the other software Energy Plus is used [8]. The latter allows the integration of PCMs and has been validated in previous studies [9–11].

The University of Auckland built two identical offices (with and without PCM) provided with a data collection system. The offices are built in Tamaki near Auckland where the climate is temperate. The two offices have been modelled and the results of the simulations are compared with the experimental data collected over a week as shown in Fig. 1. The thin blue dashed line shows the indoor air temperature as obtained from the simulation, the black solid bold line shows the measured indoor temperature, and the light green solid line shows the outside temperature.

The office with the PCM-impregnated gypsum boards is the one which was modelled and shown in Fig. 1. It can be observed that the gap between the experimental and simulated curves is reasonably small. This confirms the fact that simulation software can be trusted as it takes well into account the integration of PCM. The difference between the experimental measurements may have come from the integration of the properties of the construction materials

indicated to the software and from the difficulty of measuring some of the parameters such as infiltration rate.

### 2.2. Modelling of a typical house

#### 2.2.1. Geometry and materials

Following model validation, simulations were conducted for a typical two-story house in Auckland. The construction of the house is based on real construction plans and a typical materials structure. It is a two-story family house for five people. The geometry was therefore added in the simulation software and is shown in Fig. 2. A few simplifications have been done to speed up the simulations while keeping the results relevant. The two “Master bedrooms” are merged with their respective “En suite”. The “Living room” and the “Hall” of the first floor are merged together. Finally the three “Bedrooms” of the first floor are merged together. The black arrow in the figure points towards North. This North facing orientation is explained by the fact that the house is situated in the Southern Hemisphere. Every bedroom and living room has two windows to let as much light as possible in.

The characteristics of the materials provided to the software and every type of structure (roof, walls, partitions etc...) are given in Tables 1 and 2. It must be kept in mind that the PCM is added to the gypsum board, therefore it has a significant influence on the house's thermal mass. The total surface area of gypsum board is 810.5 m<sup>2</sup> for the whole house with a floor surface area of 256 m<sup>2</sup>.

#### 2.2.2. HVAC

In order to run a realistic study for the house, several assumptions had to be made to define the overall system. The simulations only give the energy loads needed and no HVAC system was defined. The goal of the study is to observe the heating and cooling loads needed, and not how to provide them. A zoning in the house is assumed so that every heated or cooled room receives the appropriate amount of heating or cooling needed to meet the set points. The loads are assumed to be variable so that the temperatures remain constant once the set points are reached, and the power adapts. In order to have a realistic design, all main rooms are assumed to have an HVAC system. Therefore only the bathroom, toilet and garage are left with no HVAC.

#### 2.2.3. Occupancy

Regarding the occupancy, the house is assumed to be occupied

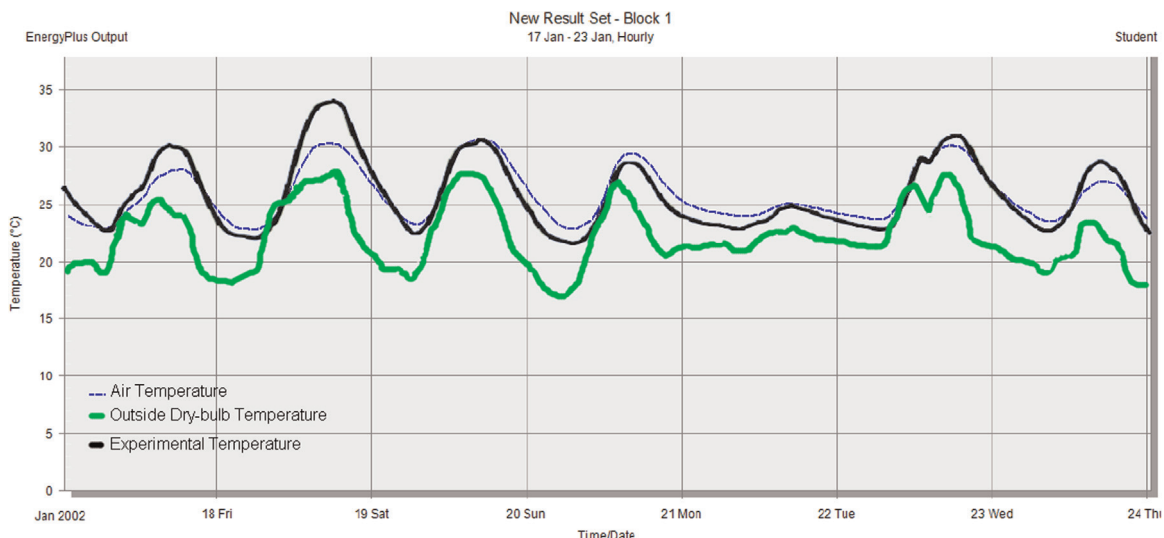


Fig. 1. Validation of the simulation software with experimental data over a week. (For interpretation of the references to colour in this figure the reader is referred to the web version of this article.)



**Table 2**  
Properties of the construction materials.

	Unit	Steel sheet	Carpet	Wood	Gypsum board	Wall insulation	Roof insulation	Concrete	Brick	Tiles
Thickness	[m]	0.002	0.001	0.035	0.013	0.09	0.17	0.085	0.07	0.014
Conductivity	[W/m K]	13.8	0.056	0.14	0.25	0.04	0.04	1.13	0.84	0.73
Density	[kg/m <sup>3</sup> ]	7817	213	650	600	12	12	2000	1700	2500
Specific heat	[J/kg K]	460	1400	1200	1089	840	840	1000	800	773

with a PCM having a phase change temperature range between 21 °C and 26 °C.

The total storage capacity of the PCM in the whole house is given in Eq. (1)

$$\dot{Q}_{storage} = A_{GB} \rho_{PCMB} wt\%_{PCM} h_{lat,PCM} = 227 \text{ MJ} \quad (1)$$

where  $\dot{Q}_{storage}$  in [MJ] is the storage capacity of PCM in the house;  $A_{GB} = 810.5 \text{ m}^2$  is the total surface area of gypsum board in the house;  $\rho_{PCMB} = 643 \text{ kg/m}^3$  is the density of the PCM-impregnated gypsum board;  $wt\%_{PCM} = 0.25$  is the weight ratio of PCM in the gypsum board;  $h_{lat,PCM} = 134 \text{ kJ/kg}$  is the latent heat of the PCM.

#### 2.4. Outputs of the model

The goal of this study is to observe two outputs: the comfort level and energy requirement. Both are factors of high importance and they go together creating a trade-off between energy savings and comfort. However, some scenarios show better combination and the target is to see what are these scenarios.

##### 2.4.1. Comfort level

The comfort level is measured by the number of discomfort hours, i.e. the number of hours during which the comfort conditions as defined by the standard ASHRAE 55-2004 are not met [15]. The standard states that if there is someone in the room and the conditions of temperature and humidity depicted in Fig. 3 are not satisfied, there is discomfort. It considers both the operative temperature and humidity ratio when deciding whether or not there is comfort. The humidity levels are taken from the weather file of the city and are assumed not to be controlled during the simulation.

When measuring the comfort level based on the standard, it is the operative temperature which is considered and not the air temperature which is the target of the HVAC. The operative temperature is an average between the air and radiant temperature, the latter being dependant on the wall temperature. The radiant temperature has a strong influence on the feeling of warmth or coldness for a person inside a room. The addition of thermal mass

**Table 3**  
Total number of hours of occupancy per zone in the house over a year

	Occupancy [hr/year]
Ground floor: Master Bedroom 1	6150
Ground floor: Dining room	2918
Ground floor: Toilet	365
First floor: Master Bedroom 2	6150
First floor: Hall and Living room	2918
First floor: Bathroom	730
First floor: Other Bedrooms	6150
Total	25,381

with the PCM-impregnated gypsum board reduces the temperature swings of the building's envelope, and therefore influences the feeling of comfort. The simulation software provides the comfort level as the total number of discomfort hours for the rooms usage as depicted in Table 3. The garage is assumed to be unoccupied and rooms of the same type are merged together as explained before.

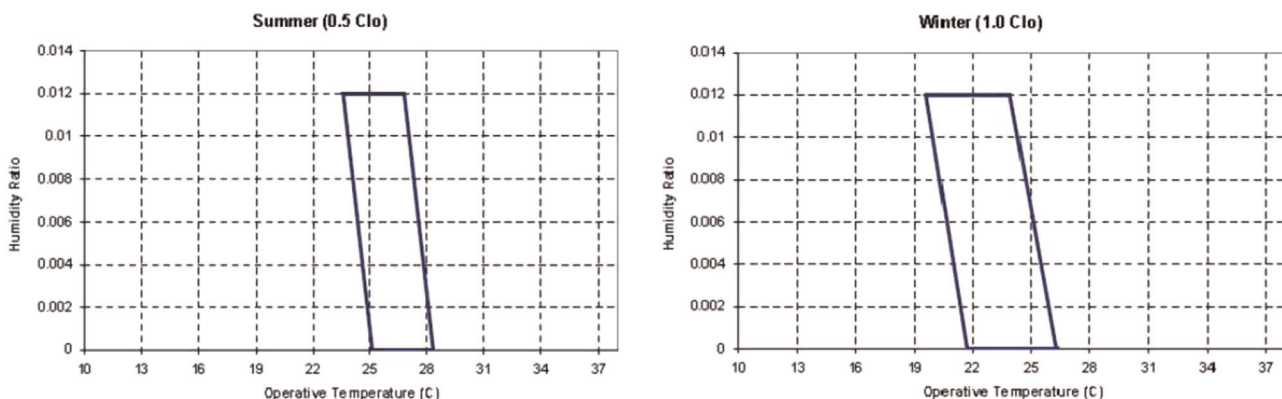
It has to be kept in mind however that the concept of comfort is a subjective one, and therefore may vary with external parameters such as culture, activity, age, gender, etc.

##### 2.4.2. Energy requirements

Energy requirement is the main focus of most studies involving the integration of PCMs and is the most critical parameter. The energy requirements define the consumption (varying with the HVAC system). From that, the annual costs of the consumed energy can be calculated. This decides if an investment in PCMs is economically viable and determines people's decisions to use PCM in their house. In this investigation however, only the heating and cooling ideal loads are given. A distinction is made between heating and cooling loads as it will be seen that they vary differently with the addition of PCM in the construction.

### 3. Results and discussion

Table 4 summarizes the outputs of the annual 30 year-round



**Fig. 3.** Comfort zones according to ASHRAE Standard 55-2004 used in the simulation software [15].



**Table 4**  
Results for the 30 year-round simulations investigating the comfort and energy requirements

Design	Heating load [kWh]	Cooling load [kWh]	Total load [kWh]	Discomfort hours [hours]	Energy savings [%]	Discomfort savings [%]
No PCM 20 °C	3171	1410	4581	15,822		
PCM 18–23 20 °C	2926	597	3523	16,790	23	–6
PCM 19–24 20 °C	2734	450	3184	16,236	31	–3
PCM 20–25 20 °C	2745	321	3066	14,953	33	5
PCM 21–26 20 °C	2815	251	3067	14,546	33	8
PCM 22–27 20 °C	2868	254	3122	14,921	32	6
No PCM 21 °C	4020	1432	5452	13,799		
PCM 18–23 21 °C	4407	604	5011	14,335	8	–4
PCM 19–24 21 °C	4038	454	4491	13,647	18	1
PCM 20–25 21 °C	3734	322	4057	12,573	26	9
PCM 21–26 21 °C	3680	253	3933	12,561	28	9
PCM 22–27 21 °C	3725	256	3981	12,932	27	6
No PCM 22 °C	5080	1466	6547	11,435		
PCM 18–23 22 °C	6284	628	6911	10,415	–6	9
PCM 19–24 22 °C	5859	468	6327	9900	3	13
PCM 20–25 22 °C	5332	331	5663	9642	13	16
PCM 21–26 22 °C	4920	257	5177	9977	21	13
PCM 22–27 22 °C	4820	259	5079	10,495	22	8
No PCM 23 °C	6338	1517	7855	10,715		
PCM 18–23 23 °C	8474	682	9155	8304	–17	23
PCM 19–24 23 °C	8055	501	8556	8280	–9	23
PCM 20–25 23 °C	7453	353	7806	8224	1	23
PCM 21–26 23 °C	6774	272	7046	8635	10	19
PCM 22–27 23 °C	6284	269	6553	9496	17	11
No PCM 24 °C	7786	1592	9378	10,287		
PCM 18–23 24 °C	10,895	799	11,694	7113	–25	31
PCM 19–24 24 °C	10,541	573	11,114	7116	–19	31
PCM 20–25 24 °C	9935	398	10,332	7054	–10	31
PCM 21–26 24 °C	9155	304	9460	7431	–1	28
PCM 22–27 24 °C	8357	296	8653	8512	8	17

simulations based on the same year's meteorological data. The results are sorted by constant heating set points to make relevant comparisons depending on the targeted temperature.

By keeping the heating set point constant, energy savings can be compared for the designs with PCM-impregnated gypsum boards. Therefore a negative value in the savings means that instead of bringing better conditions, PCM makes it worse.

This way of presenting the data is to emphasise the fact that there are several acceptable solutions.

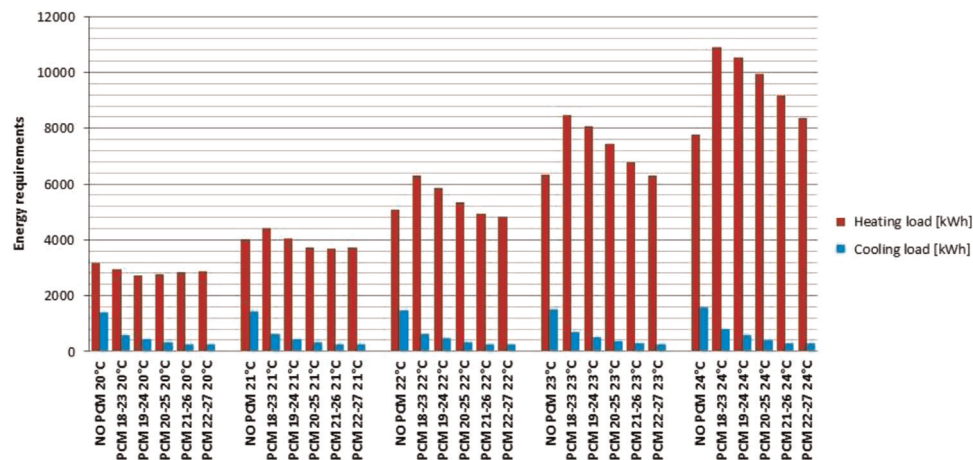
The discomfort savings index is defined as in Eq. (2) and is based on the number of hours when there are people in the rooms while the comfort conditions are not met.

$$\text{Discomfort savings} = \frac{(T_{\text{Discomfort, No PCM}} - T_{\text{Discomfort, PCM}})}{T_{\text{Discomfort, No PCM}}} 100 \text{ [\%]} \quad (2)$$

where *Discomfort savings* in [%] is the comfort improvement

from the integration of PCMs;  $T_{\text{discomfort, PCM}}$  and  $T_{\text{discomfort, No PCM}}$  are respectively the yearly number of hours of discomfort for a case with and without PCM, respectively at a specific heating set point.

The results shown in Table 4 can be used to assist a designer to select a suitable combination of heating set point and PCM melting range based on energy consumption and comfort. It also allows better understanding on whether the savings are from the heating or cooling load. This is depicted in Fig. 4 and clearly shows that most of the savings occur during cooling in summer. In summer, PCM stores large quantity of coolness at night, while in winter PCM stores only the limited excess energy coming from solar radiation. In summer, the PCM slows down any rise in temperature which otherwise would involve a need for cooling. The heat stored can be released freely later when the temperature drops at night. Regardless of the heating set point, the cooling load decreases sharply when the melting temperature range of the PCM increases.



**Fig. 4.** Energy requirements for the different scenarios for the house.

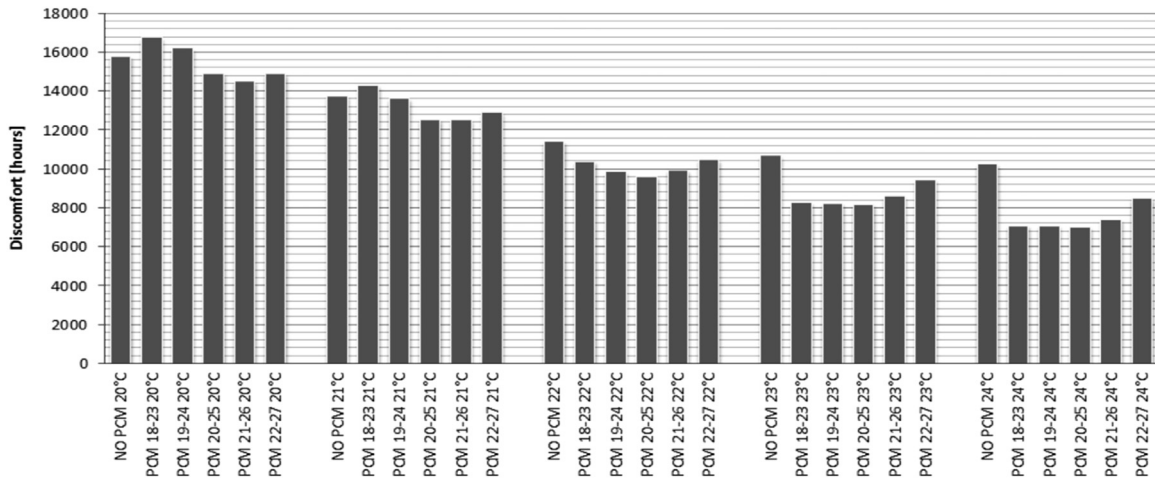


Fig. 5. Discomfort hours for the different scenarios for the house.

When looking at the heating loads, it is clear that the PCM brings little improvement and can also make it worse. The higher the heating set point, the more heating energy used and the less likely it is that the PCM integration will bring energy savings. This is explained by the fact that a heating set point situated within the phase change temperature range of the PCM will require extra energy to overcome the added thermal mass. For instance, a poor scenario “PCM 18–23 24 °C” will not save the energy needed for heating in comparison with a basic design without PCM as a lot of heat is required to go from 18 °C to the heating set point of 24 °C. The house is heated following a cyclic schedule, making it overcome the thermal mass every day to reheat it when people come home.

A clever scenario such as a “PCM 20–25 20 °C” on the other hand appears to be a good choice. The house can reach the heating set point easily without having to go through the latent heat. Any free energy (mainly solar radiation) from this point is going to be stored in the PCM as it starts to change phase.

These observations about the energy requirements highlight the need for a good design in order to make the integration of PCMs economically viable. A look at Fig. 5 brings more understanding about the comfort levels.

The first observation that can be seen here is that the total of comfort hours increases when more energy is put into the HVAC

system which is expected. It also shows that the addition of PCM does not bring comfort enhancement in every case. This can be explained by the added thermal mass which can slow down the temperature rise and therefore prevent the rooms to reach the comfort zone in winter while it is the other way around in summer. One important point is that only some PCMs seem much more likely to bring comfort enhancement. For a fixed heating set point it is always either “PCM 20–25” or “PCM 21–26” which bring the more improvement compared to the case without PCM. This can be explained by the fact that both of these types of PCM match very well with the comfort zones depicted in Fig. 3 and therefore tend to keep longer the temperatures within these ranges for a longer time. One could say that the number of discomfort hours is high when assuming a total 25,381 h as given Table 3. However, it must be noted that the night temperature set point of 18 °C is not in the comfort zone from Fig. 3 even though it is a recommended temperature at night. This explains the relatively high level of discomfort from Fig. 6 even though in practice comfort is achieved.

As seen in Table 4, the scenarios providing more energy savings do not necessarily provide high comfort enhancement and vice versa. The idea is then to give for each heating set point an optimal design. A trade-off has to be made between energy and comfort. This trade-off is depicted in Fig. 6 and clearly shows the trend. As both outputs need to be minimised, the best designs are found on

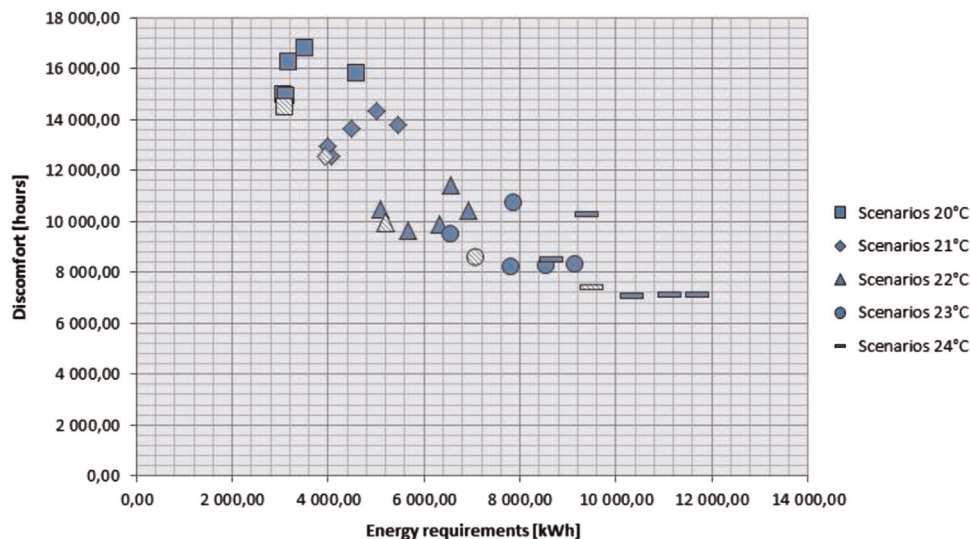


Fig. 6. Combination of the comfort levels with the energy requirements.

the lower part and the left of the graph in Fig. 6. Then depending on the priorities of the household, the best design can be selected. For each heating set point, the chosen design is indicated with the non-solid shape symbols.

The suggested optimal scenarios are then: PCM 21–26 20 °C/PCM 21–26 21 °C/PCM 21–26 22 °C/PCM 21–26 23 °C/PCM 21–26 24 °C. This graphical analysis exhibits the fact that the PCM changing phase within the range 21–26 °C brings the best trade-off for every heating set point.

In order to have a better understanding of these results, two days were investigated in more detail. A typical day in summer and a typical day in winter. The winter analysis depicts a scenario without PCM, and two scenarios using PCMs (good and bad). The summer analysis shows only one scenario without PCM and a good scenario with PCM as PCMs always bring improvement.

The “air temperature” in solid line is the indoor temperature targeted by the HVAC system. Therefore some plateaus are seen when the HVAC is required as seen on the graphs. The “operative temperature” in dashed line includes the effect of radiation and is the one which determines the feeling of comfort. The HVAC load is depicted on the right to show how the different scenarios induce different heating and cooling loads.

The winter day is a typical weekday and it is chosen to show the scenario “No PCM 20 °C” compared to the scenarios “PCM 20–25 20 °C” and “PCM 22–27 20 °C”. A constant heating set point and the analysis over the same room (Master bedroom on the first floor) give relevant results for a comparison of the influences of PCMs.

The results for this winter day are given in Figs. 7–9. As expected according to Table 4, the designs with PCM show lower heating requirements. The one with “No PCM” requires 3.61 kWh, the one with “PCM 20–25” requires 2.40 kWh and the one with “PCM 22–27” requires 2.90 kWh. This is only for a room over a day so the difference is small, but over the whole house and the whole winter it can bring significant differences.

The goal of this comparison is to see how PCMs can save energy even in winter. This winter day is chosen because the temperature is cold outside, yet the solar radiation brings heat during the afternoon. Because on these graphs the emphasis is on the energy requirements, the “PCM 21–26” given as best trade-off energy versus comfort is not depicted here. It is indeed not the one bringing the more energy savings for a heating set point of 20 °C. The room was chosen due to its orientation (North facing). It receives solar radiation in the afternoon, and the heating set point of

20 °C is reached without any heating load at 5 p.m. when people come home. The heat is then released at a different pace in the evening, and at 10 pm the heating set point is lowered to 18 °C for the night temperature.

The scenario with “No PCM” shows a temperature rise to 24 °C and then a quick drop until it reaches 20 °C where the HVAC system activates. The high heating requirements come from the fact that the heat from solar radiation has not been stored in the building’s envelope and therefore it needs large loads in the evening and at night.

The design “PCM 20–25” is considered optimum and shows the lowest heating requirements for this heating set point. It is due to the fact that the room can easily reach 20 °C. From there, any solar gain is stored through the melting process and slows down the temperature rise which keeps a relatively constant temperature. The stored heat is then slowly released in the evening and at night, which lowers the heating requirements.

The scenario “PCM 22–27 20 °C” is an example of a non-optimum design for a set point of 20 °C. It shows that the temperature rises easily to 22 °C and then the melting process barely starts. This is due to the fact that in winter the temperature does not go high enough to melt as much PCM as in the previous scenario. Therefore, the temperature drop is slightly slowed down but less energy is stored. The consequence is that the heating requirements are higher than for the other PCM design but lower than the design without PCM.

These graphs show that designs with PCMs bring lower air temperature variations. However what is more interesting is what happens to the operative temperature. It clearly shows that the thermal inertia is significantly increased when a PCM with a melting temperature of 20–25 °C or 22–27 °C is selected. Therefore the temperatures take longer time to rise, but also to drop and hence it gives more constant temperature levels. Outside of the phase change temperature range, the designs show relatively similar temperature variations.

The summer day is a typical weekday and it is chosen to show the scenario “No PCM 21 °C” compared to the scenario “PCM 21–26 21 °C” with a constant cooling set point of 25 °C. The design with PCM, according to Table 3, gives lower energy requirements and also improves comfort. The results for this summer day are given in Fig. 10 and Fig. 11 for the same bedroom as for the winter study. This time, the effect of the PCM is very clear and shows much more constant temperatures over the day. It can be explained by the fact that the temperatures vary across the phase

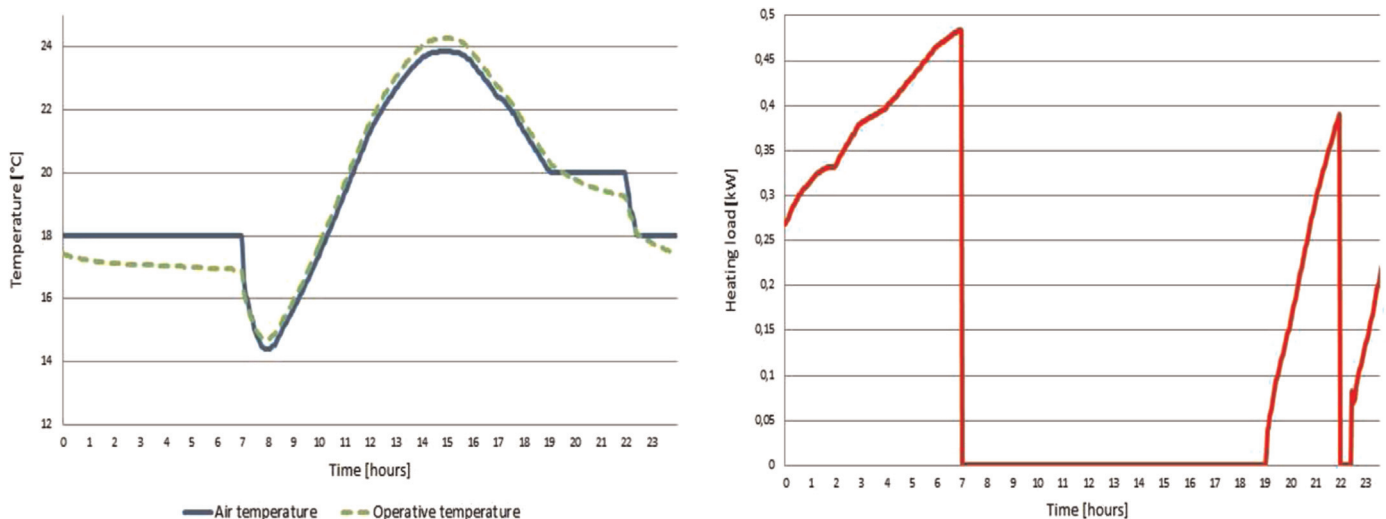


Fig. 7. Analysis over a winter day in a bedroom for the scenario “No PCM 20 °C”.

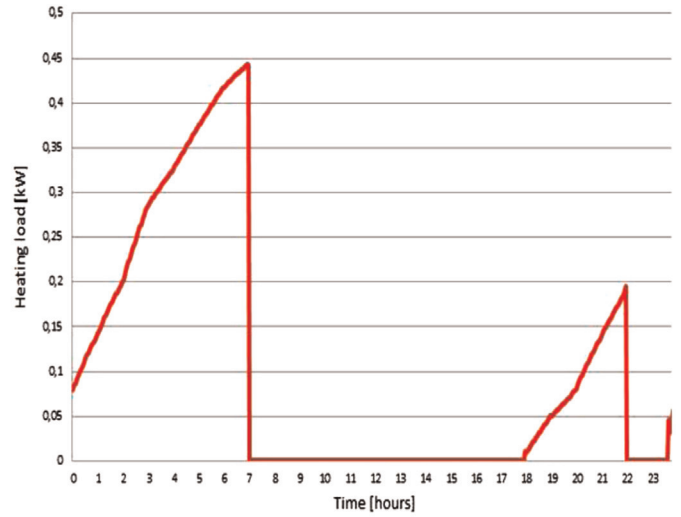
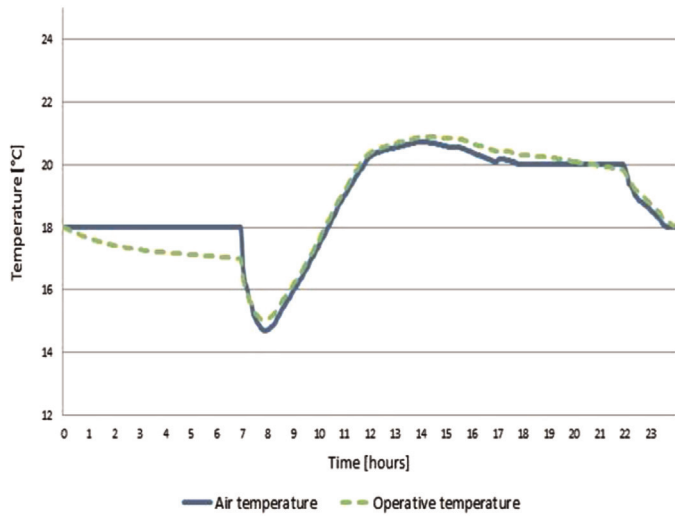


Fig. 8. Analysis over a winter day in a bedroom for the scenario “PCM 20–25 20 °C”.

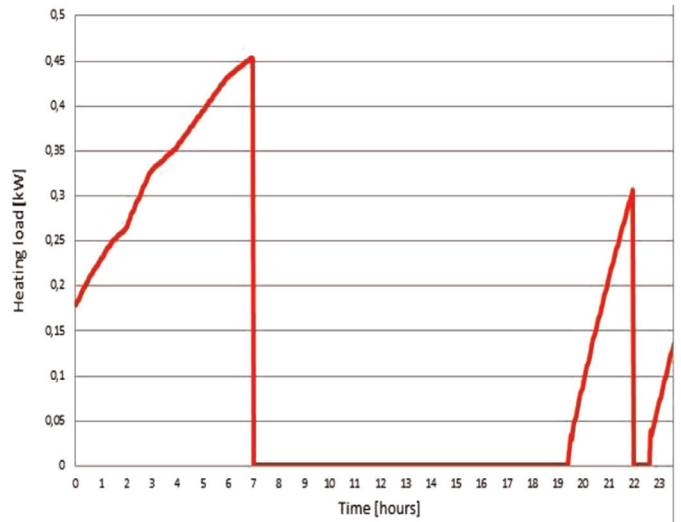
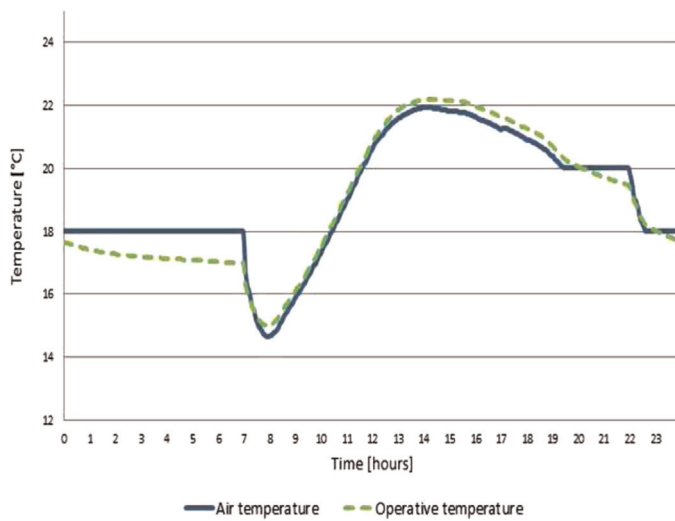


Fig. 9. Analysis over a winter day in a bedroom for the scenario “PCM 22–27 20 °C”.

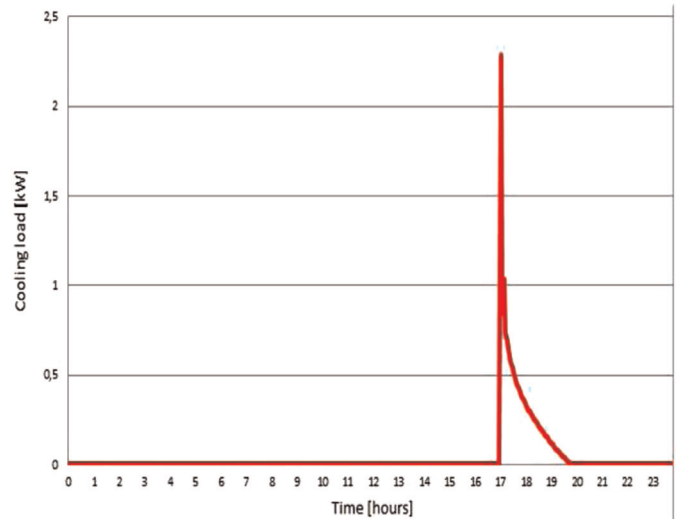
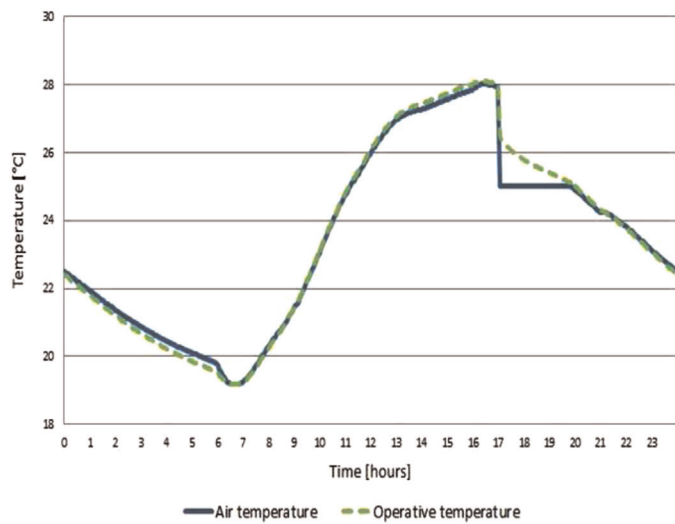


Fig. 10. Analysis over a summer day in a bedroom for the scenario “No PCM 21 °C”.



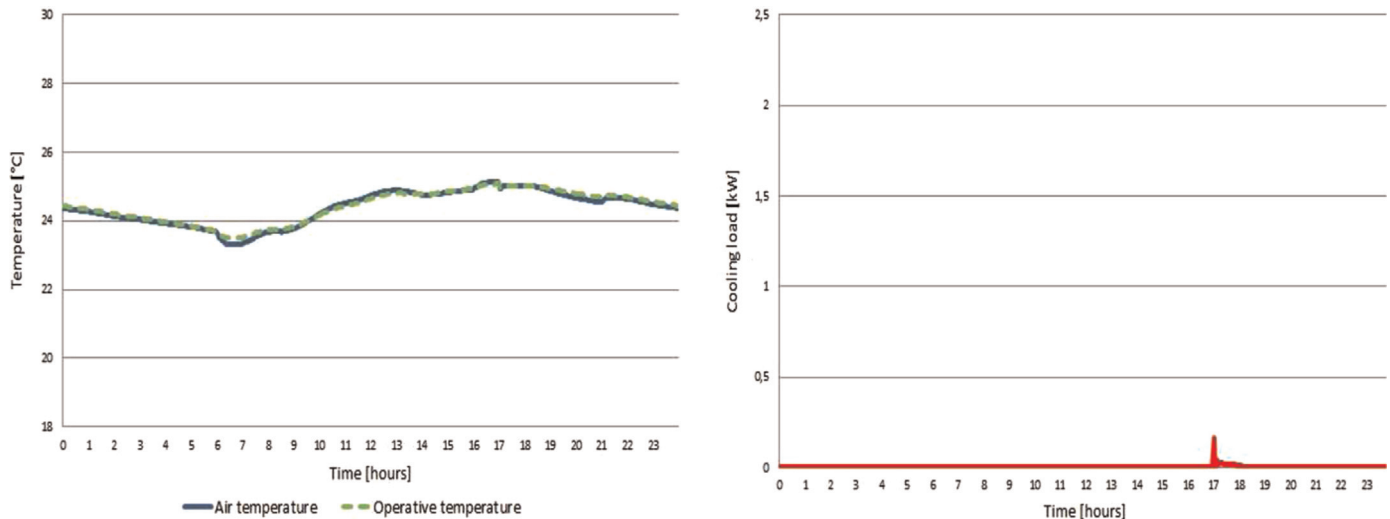


Fig. 11. Analysis over a summer day in a bedroom for the scenario “PCM 21–26 21 °C”.

change temperature range. The chosen design has the advantage that the PCM melts and freezes while being in the comfort zone. There are two benefits: first it almost eliminates the need for cooling (0.03 kWh versus 0.93 kWh), and second it significantly reduces the temperature swing which brings better comfort. The operative temperature varies only within a range of about 1.5 °C over the whole day versus about 9 °C for the design without PCM.

This shows that the integration of PCMs can be effective when done properly. Not only does it lower the cooling needs, but it also brings much more stable temperature and therefore enhances comfort.

#### 4. Conclusion

An approach targeting both comfort levels and energy requirements when integrating PCMs into a building was presented in this paper. It was shown how the performance depends on design and operation parameters. Simulation software was used to perform year-round simulations for 30 scenarios and calculate the performance of the building. Depending on the scenario, the results showed significant variations. It is demonstrated that the definition of the comfort (set point temperature) is strongly related to the PCM material used. This indicates that the choice of the right PCM has to be done for a given comfort temperature range.

Regarding the comfort levels, the two types of PCMs which bring the more improvement for any heating set point are the PCMs with a phase change temperature range of 20–25 °C and 21–26 °C. For energy savings, the results vary strongly depending on the heating set point. A general observation is that most of the savings occur from cooling and only little saving comes from heating. Combining efficiency and comfort indicators, it turns out to be that the PCM with a phase change temperature range of 21–26 °C is the best regardless of the heating set point.

This highlights the fact that integration of PCM brings significant benefits when the temperatures vary across the phase change temperature range. For better results, this phase change temperature range has to be in the comfort zone.

The integration of PCM into buildings is a complex non-linear problem which shows significant benefits only if well designed. Being weather dependant, the specific designs chosen as optimum for New Zealand might not be the same in other countries. Especially when PCMs are used, a fine tuning of the HVAC set points

has to be performed in order to avoid energy losses related to inconsistent design.

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