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F	\eroe	lastic	ity	<sup>,</sup> and	aer	oac	oust	tics	of	wind	l tur	bine	es

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### AEROELASTICITY AND AEROACOUSTICS OF WIND TURBINES

Helge Aagaard Madsen

Risoe National Laboratory for Sustainable Energy Technical University of Denmark



#### **Outline**



- Introduction to Risø DTU
- **→** The typical wind turbine 2011
- Wind turbine loads and certification
- Aerodynamic and aeroelastic simulation tools
- > Aeroelastic stability
- Wind farms and wakes
- Aeroacoustics
- New technology outlook
- Summary

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### Risø's history in brief

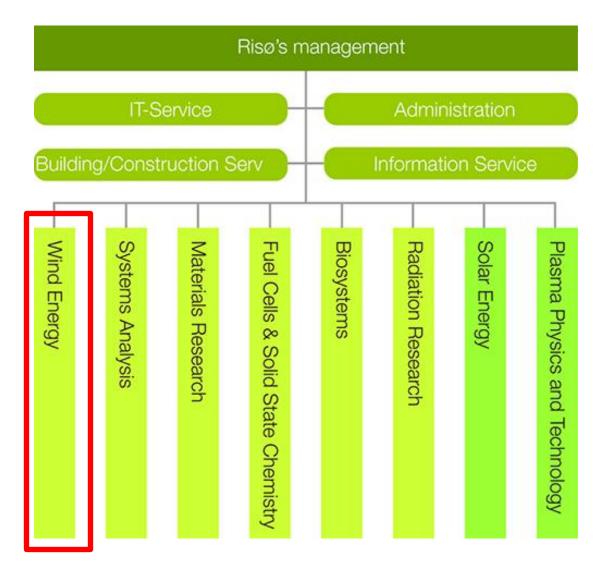




- 1956 Peaceful utilization of nuclear energy
- **1976** Nuclear energy and <u>other</u> energy sources
- 1986 Energy research in general
- 1990 R&D with energy as the primary area
- **1994** State-owned enterprise
- 2000 The last nuclear reactor is decommissioned
- 2005 Impact within
  - 1. Technology for greater competitiveness
  - 2. Sustainable energy supply
  - 3. Health technology
- **2007** Merged with DTU (The Technical University of Denmark)

#### Risø DTU





## **Wind Energy Division**



**Blade Test** Center Sparkær (Force+DNV) Risø **National Test Station** Høvsøre

150employees in5 researchprogrammes

#### **National Test Station for Large Wind Turbines - 2007**





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# DTU

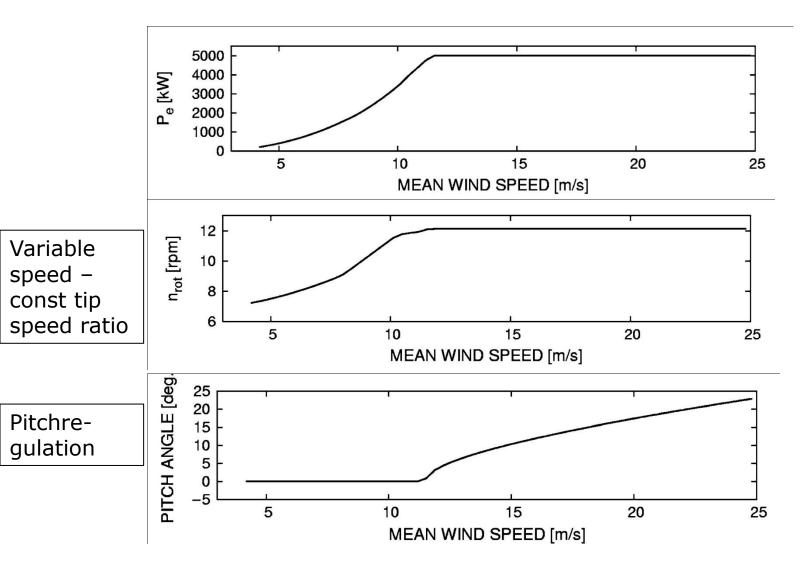
## The typical wind turbine design 2011



- > rated power 2-5 MW
- > 80-125 m rotor
- > pitchregulated
- > variable speed
- > steel, tubular tower
- gearbox or direct drive with multipole generator
- > load alleviation with cyclic pitch
- advanced control and monitoring system

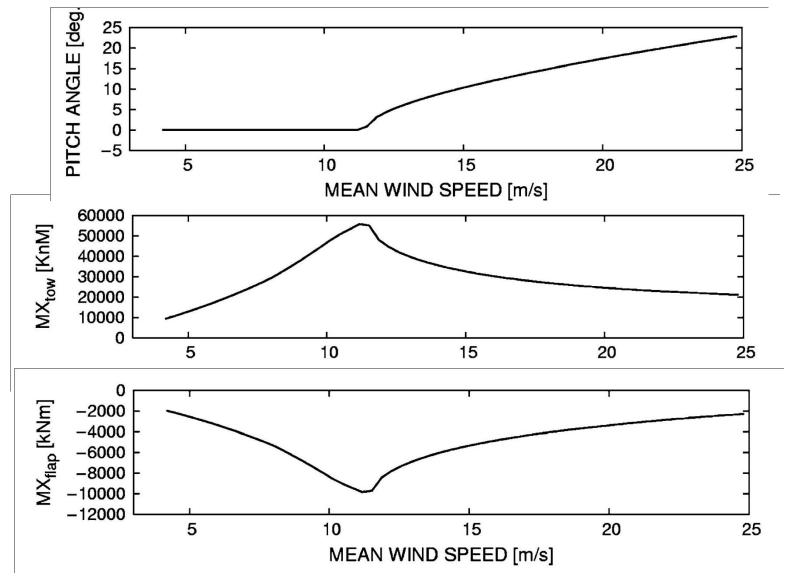
# The typical wind turbine design 2011





# The typical wind turbine design 2011

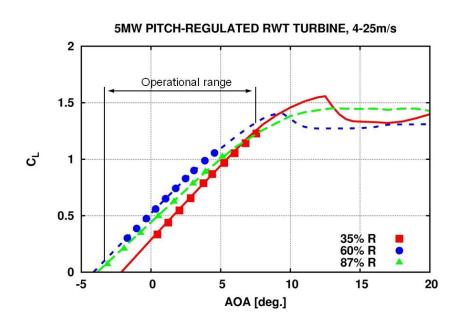


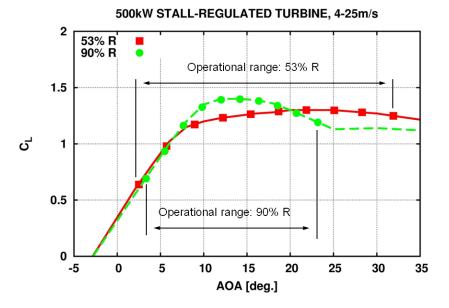


# The typical wind turbine design 2011



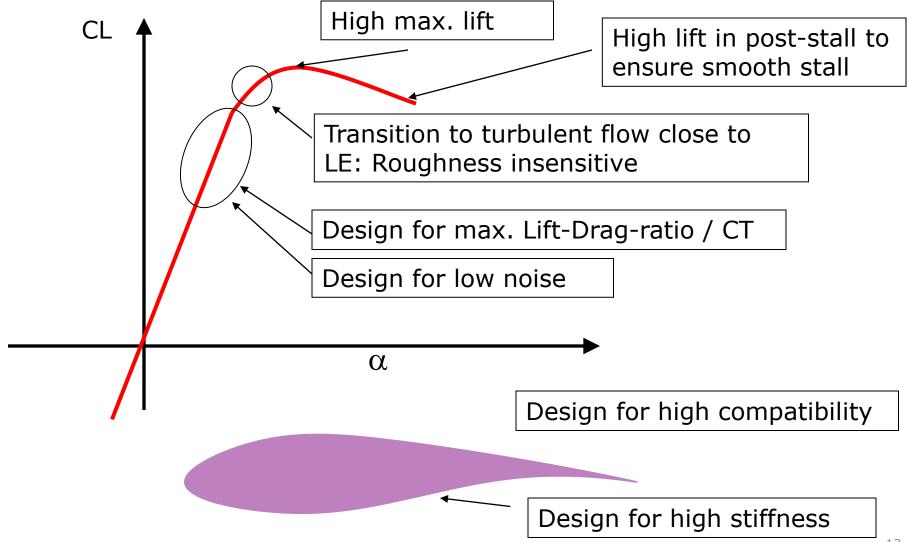
#### Old stall regulated turbine





## The typical wind turbine design 2011 -use of dedicated airfoil designs





# The typical wind turbine design 2011 -use of dedicated airfoil designs



#### Risø-A1 (15% to 30%)

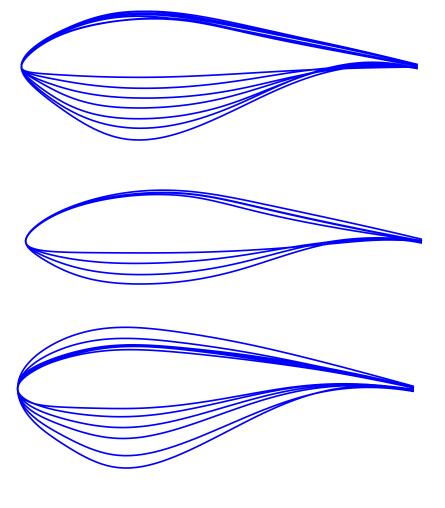
- Designed for stall, active stall and pitch
- Full scale tested on a 600 kW ASR wind turbine

#### Risø-P (12% to 24%)

- Designed to replace Risø-A1 for pitch control
- Used on 3 MW PRVS wind turbines

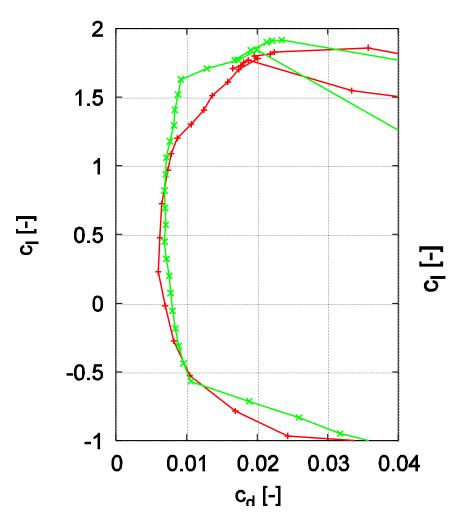
#### Risø-B1 (15% to 53%)

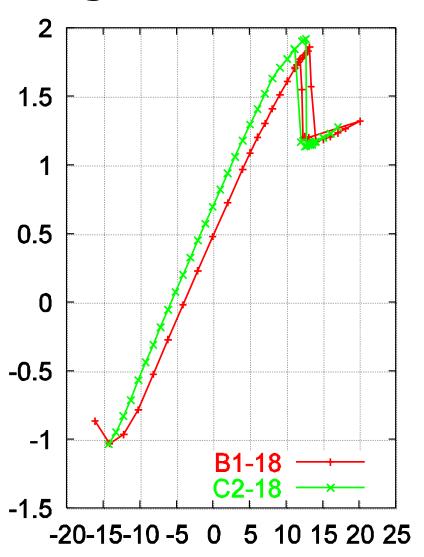
- Designed for pitch regulation variable speed control
- Used on several MW size PRVS wind turbines



# The typical wind turbine design 2011 -use of dedicated airfoil designs

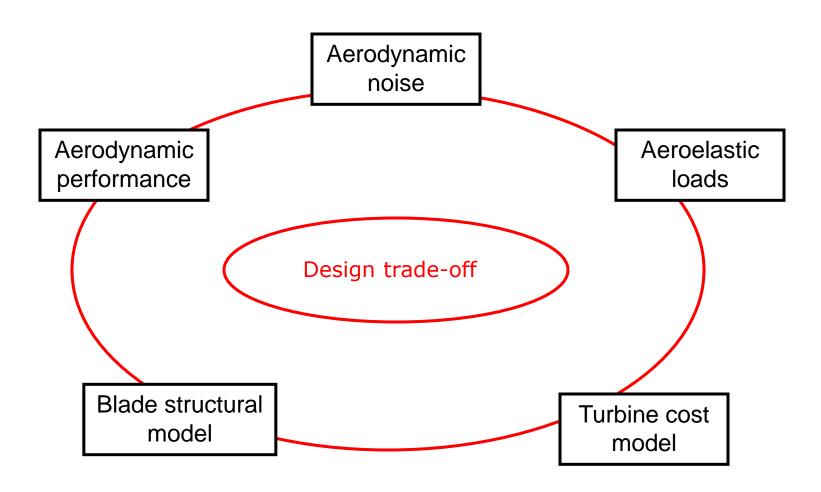








## Aeroelastic blade design



# Blade designed for maximum aerodynamic efficiency



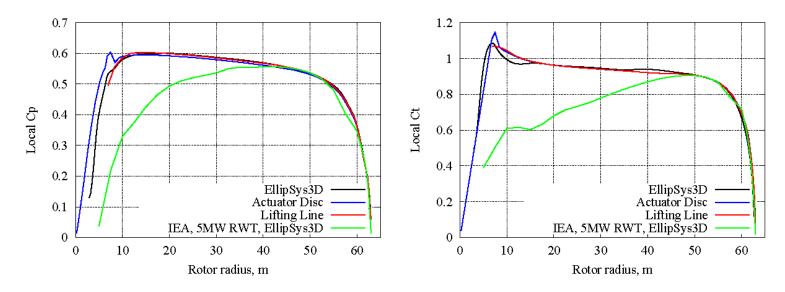


Table 1: Mechanical power and Thrust force for the present rotor. The IEA, 5MW RWT is included for comparison

	Mechanical power, P [MW]	Thrust force, T [kN]	$\bigcap CP$	CT
EllipSys3D	2.015	426	0.515	0.872
Lifting Line	2.011	424	0.514	0.868
Actuator Disc	1.995	425	0.510	0.870
IEA, 5MW RWT EllipSys3D	1.867	382	0.477	0.782

#### **Outline**

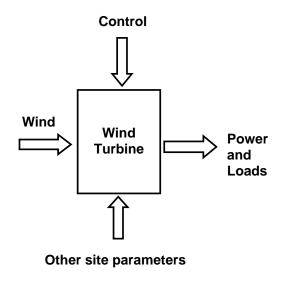


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#### Wind turbine loads and certification



## **Loading from:**



- turbulence and wind shear in the atmospheric inflow
- wakes from neighbouring turbines
- waves
- control action, e.g. an emergency stop

#### Wind turbine loads and certification



Design situation	DL C	Wind condition	Other conditions	Type of analysis	Partial safety factors	
1) Power production	1.1	${\rm NTM}  V_{\rm in} < V_{\rm hub} < V_{\rm out}$	For extrapolation of extreme events	U	N	list of load cases
	1.2	NTM $V_{\rm in} < V_{\rm hub} < V_{\rm out}$		F	*	List of load cases
	1.3	ETM $V_{\text{in}} < V_{\text{hub}} < V_{\text{out}}$		U	N	from TEC61400 1
	1.4	ECD $V_{\text{hub}} = V_{\text{r}} - 2 \text{ m/s}, V_{\text{r}},$ $V_{\text{r}} + 2 \text{ m/s}$		U	N	from IEC61400-1.
	1.5	EWS $V_{\rm in} < V_{\rm hub} < V_{\rm out}$		U	N	
Power production     plus occurrence of	2.1	NTM $V_{\text{in}} < V_{\text{hub}} < V_{\text{out}}$	Control system fault or loss of electrical network	U	N	
fault	2.2	NTM $V_{\rm in} < V_{\rm hub} < V_{\rm out}$	Protection system or preceding internal electrical fault	U	A	
	2.3	EOG $V_{\text{hub}} = V_{\text{r}} \pm 2 \text{ m/s and}$ $V_{\text{out}}$	External or internal electrical fault including loss of electrical network	U	А	
	2.4	${\rm NTM}  \mathcal{V}_{\rm in} < \mathcal{V}_{\rm hub} < \mathcal{V}_{\rm out}$	Control, protection, or electrical system faults including loss of electrical network	F	*	In total 1000-1500 loa cases to be simulated
3) Start up	3.1	NWP $V_{\rm in} < V_{\rm hub} < V_{\rm out}$		F	*	_
	3.2	EOG $V_{\text{hub}} = V_{\text{in}}, V_{\text{r}} \pm 2 \text{ m/s}$ and $V_{\text{out}}$		U	N	most 10 min.
	3.3	EDC $V_{\text{hub}} = V_{\text{in}}, V_{\text{r}} \pm 2 \text{ m/s}$ and $V_{\text{out}}$		U	N	simulations
4) Normal shut down	4.1	NWP $V_{\text{in}} < V_{\text{hub}} < V_{\text{out}}$		F	*	
	4.2	EOG $V_{\text{hub}} = V_{\text{r}} \pm 2 \text{ m/s and}$ $V_{\text{out}}$		U	N	
5) Emergency shut down	5.1	NTM $V_{\text{hub}} = V_{\text{r}} \pm 2 \text{ m/s and}$ $V_{\text{out}}$		U	N	
6) Parked (standing still or idling)	6.1	EWM 50-year recurrence period		U	N	🧖   f=fatigue
	6.2	EWM 50-year recurrence period	Loss of electrical network connection	U	A	
	6.3	EWM 1-year recurrence period	Extreme yaw misalignment	U	N	
	6.4	NTM $V_{\text{hub}} < 0.7 V_{\text{ref}}$		F	*	
Parked and fault conditions	7.1	EWM 1-year recurrence period		U	А	
8) Transport, assembly, maintenance and repair	8.1	NTM $V_{\rm maint}$ to be stated by the manufacturer		U	Т	u = ultimate load
	8.2	EWM 1-year recurrence period		U	А	

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# Numerical models/tools used for aerodynamic and aeroelastic analysys at the Aeroelastic Design Group (AED) at Risø DTU



- > EllipSys2D
  - 2D CFD code used mainly for computation on **2D airfoil sections**
- ➤ EllipSys3D
  - 3D CFD code used for **rotor computations** and flow over terrain
- **≻**Hawc2
  - Aeroelastic multibody code for aeroelastic time simulation of wind turbines
- **HAWCStab** 
  - code for computation of aeroelastic stability
- > HAWTopt
  - tool for design and optimization of rotors
- ➢ AirfoilOpt
  - tool for design and optimization of airfoils

#### Aeroelastic codes and simulations



#### **Engineering sub-models for simulation of:**

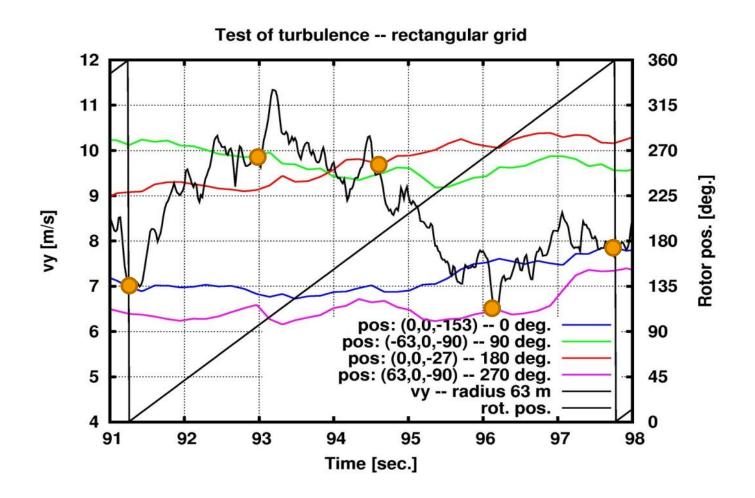
#### Aeroelastic codes for time simulations used by industry:

- FLEX5
- FAST
- BLADED
- HAWC2
- simulations in real time or faster

- > yawed flow
- > dynamic stall
- unsteady blade aerodynamics
- > unsteady inflow
- ➤ tip loss
- > tower shadow
- wakes from neighboring turbines
- > simulation of atmospheric inflow
- > hydrodynamics
- > wave loads
- > control

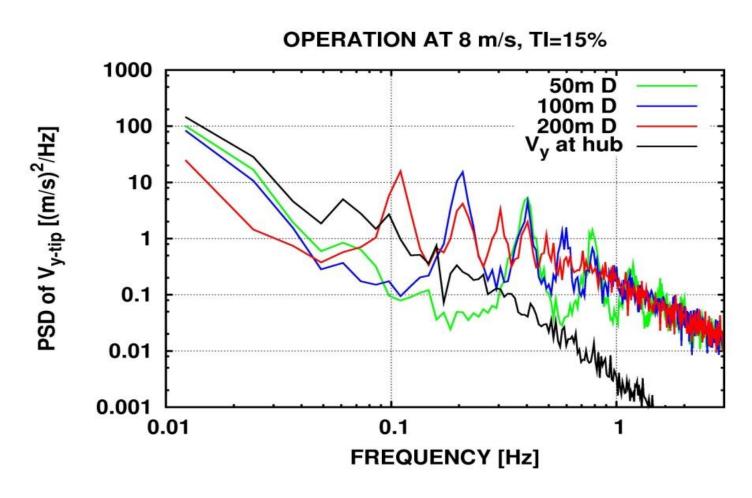
# Turbulence in atmospheric inflow is the main driver of loads - rotational sampling of turbulence





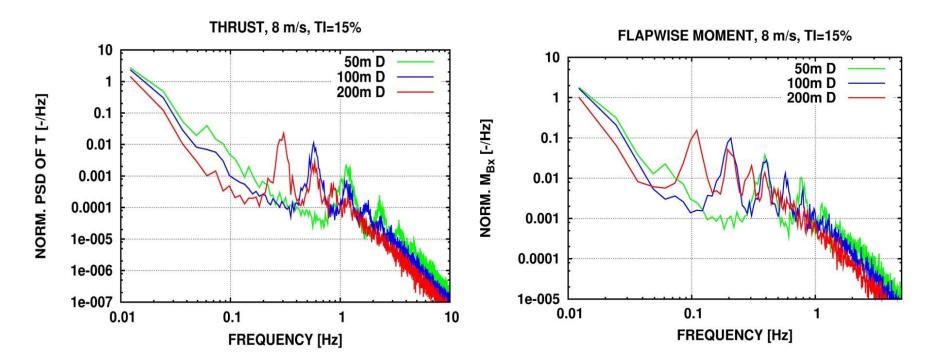
# Turbulence in atmospheric inflow main driver of loads - rotational sampling of turbulence





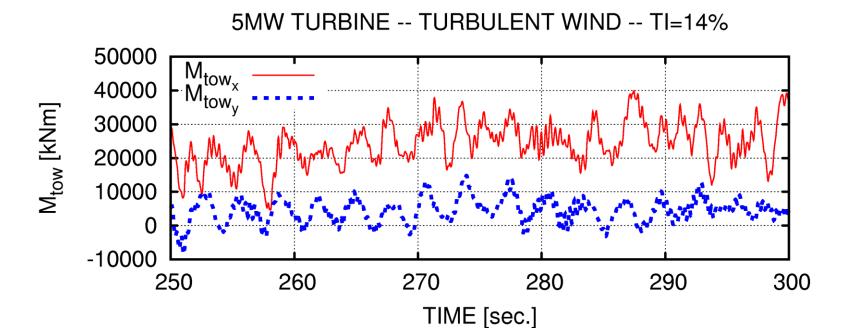
# Turbulence in atmospheric inflow main driver of loads - rotational sampling of turbulence





### Simulated tower loads

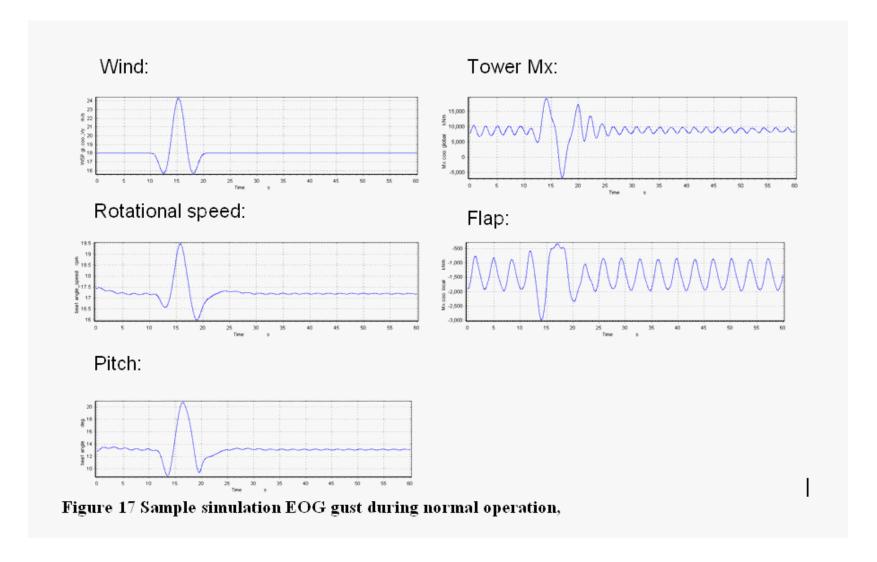




The bending moment in the tower base of a 5MW turbine at 18 m/s wind speed and 10% turbulence. Solid curve bending in main wind direction, dashed curve perpendicular.

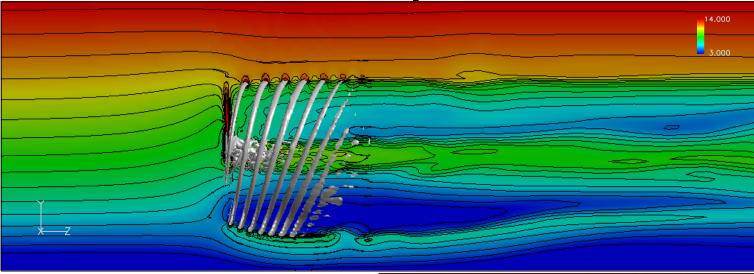
## Extreme load case - gust 18-24 m/s





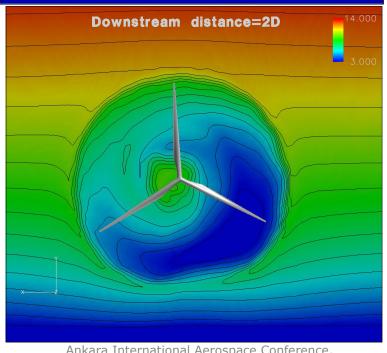
**CFD: Rotors in atmospheric shear** 





#### **Results from CFD-analysis:**

- •Shear causes aerodynamic hysteresis effects.
- Blade loads are different in horizontal position.
- Shear causes rotor yaw loads



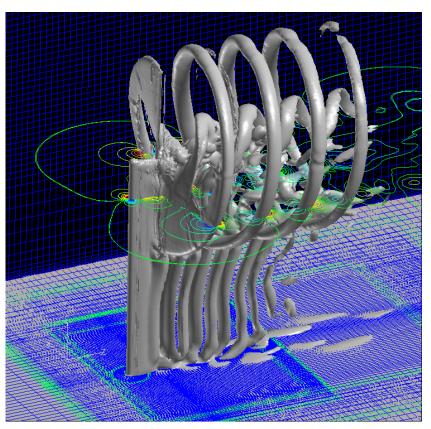
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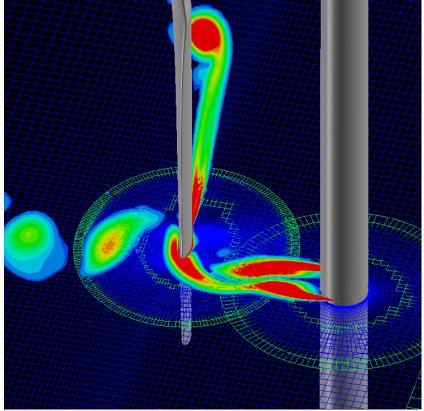
# CFD Wind turbine rotor-tower interaction



Details of blade-tower interaction investigated in order to:

- study lock-in phenomena
- develop semi-emperical tower shadow model and noise model





#### **Outline**

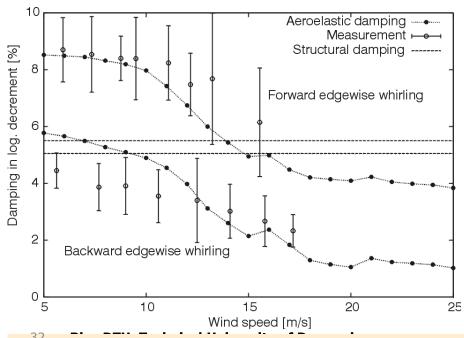


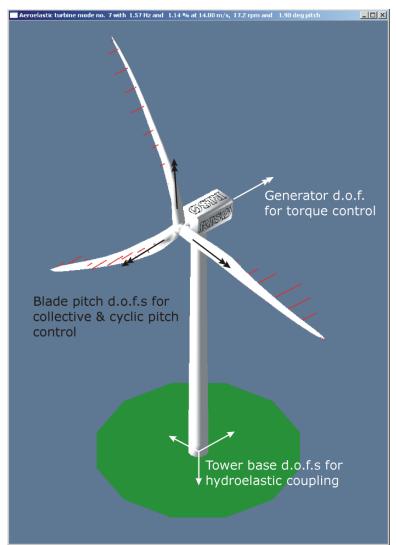
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# **HAWCStab2** – a linear aero(servo)elastic stability tool



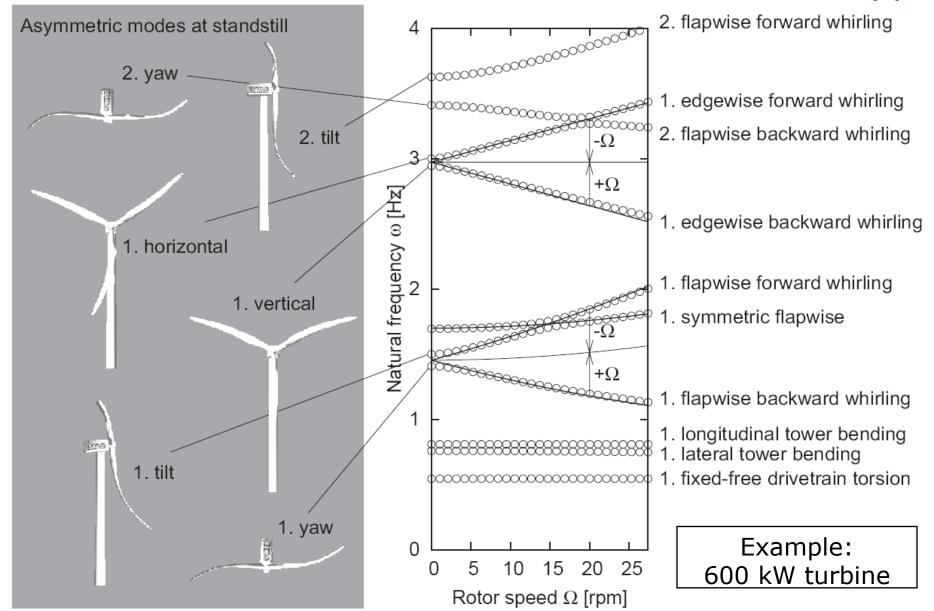
- Linearization of HAWC2 equations.
- Aeroelastic eigenvalue analysis
- Mode shape animation
- Present implementations
  - pitch and generator dof.s
  - controller model





### Typical modal dynamics of wind turbines







#### **Demonstration of the HAWCStab tool**



# Low damped modal shapes – can lead to instabilities

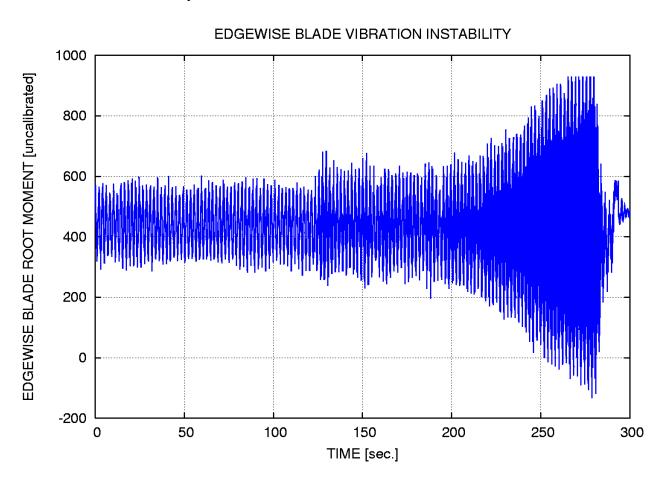


- modal shapes involving lateral tower top movement
- modal shapes involving blade edgewise tip motion
- flutter instability involving 2<sup>nd</sup> flapwise blade mode and 1<sup>st</sup> torsional mode

#### **Edgewise blade vibrations**



#### Measured instability



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#### Wind farms and wakes





#### **Wake operation**



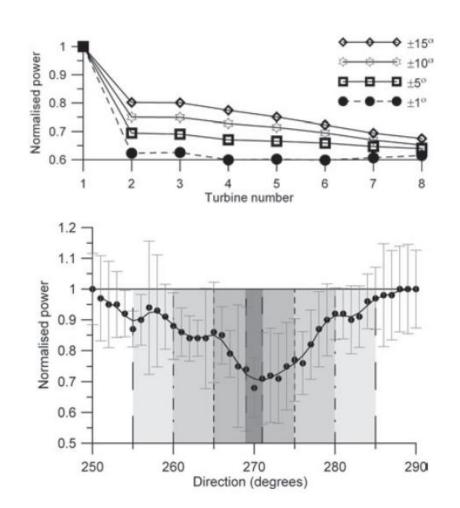
#### The presence of neighboring turbines causes:

- 1. Reductions in wind speed.
- 2. Increased turbulence turbine components fails (especially yaw system).



#### **Power reduction**

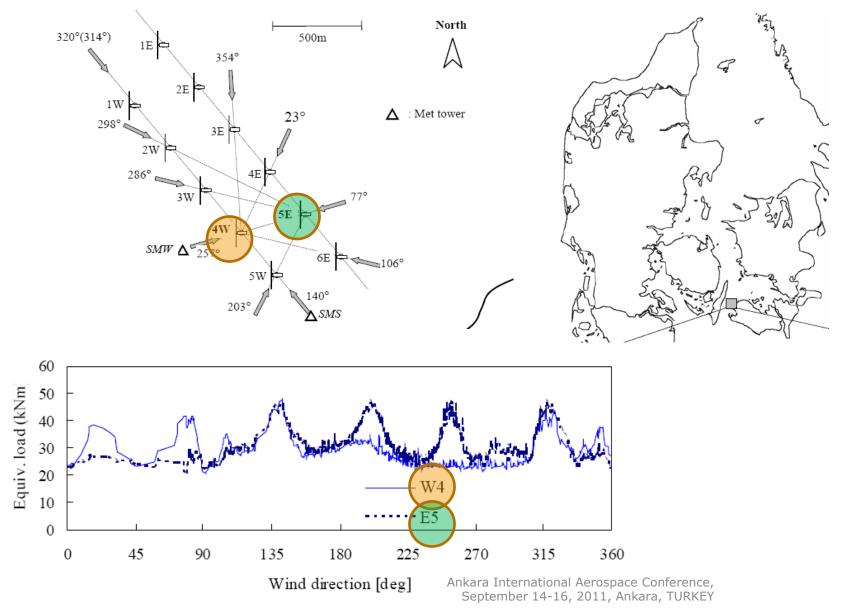




Models for power prediction exist but nearly all only depend on the upwind turbine thrust coefficient. Large uncertainty present.

## **Example of increased loads Load measurements from Vindeby wind farm**





### Assessment of turbulence intensity IEC61400-1, Frandsen 2003



#### For fatigue loads:

$$I_{\text{eff}} = \frac{\hat{\sigma}_{\text{eff}}}{V_{\text{hub}}} = \frac{1}{V_{\text{hub}}} \left[ (1 - N p_{\text{w}}) \hat{\sigma}^{m} + p_{\text{w}} \sum_{i=1}^{N} \hat{\sigma}_{\text{T}}^{m} (d_{i}) \right]^{\frac{1}{m}}; p_{\text{w}} = 0,06$$

$$\hat{\sigma}_{T} = \sqrt{\frac{0.9V_{\text{hub}}^{2}}{(1.5 + 0.3d_{i}\sqrt{V_{\text{hub}}/c})^{2}} + \hat{\sigma}^{2}}$$

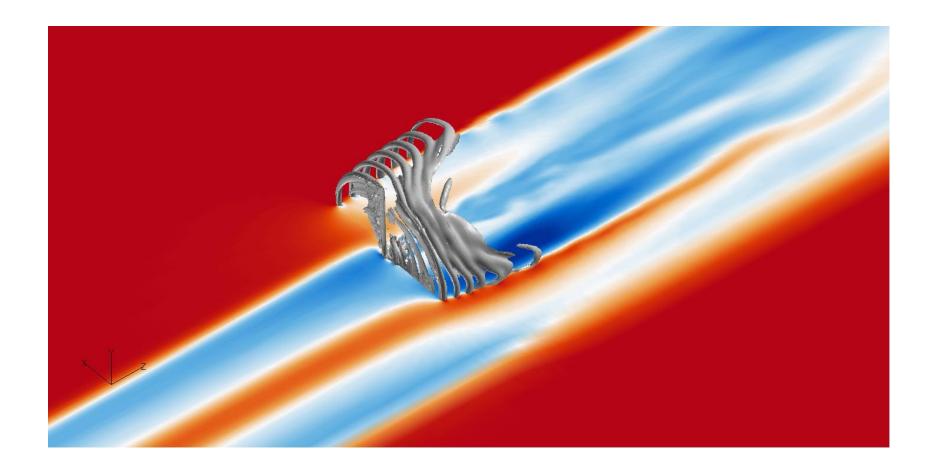
$$\sigma_1 \ge I_{\text{eff}} \cdot V_{\text{hub}} + 1,28 \,\hat{\sigma}_{\sigma}$$

#### For extreme loads:

$$I_{eff} = \frac{1}{V_{hub}} \max \left\{ \hat{\sigma}_T \right\}$$



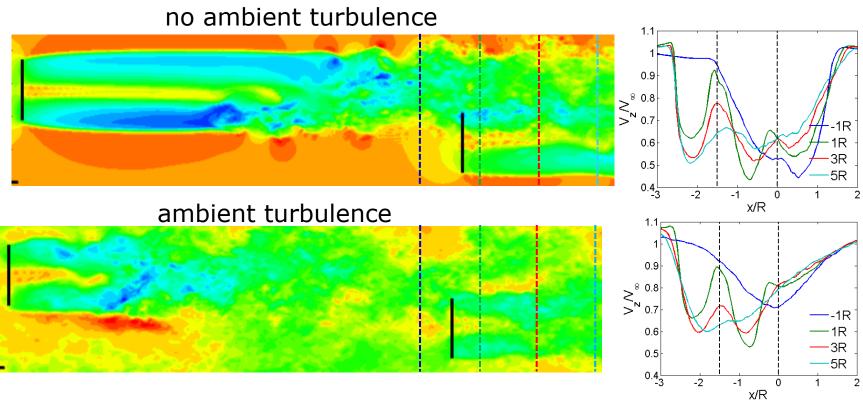
#### Computation of half wake with EllipSys3D



#### Actuator line CFD simulation Influence of Ambient Turbulence



- Upstream wake asymmetric due to inflow shear
- Ambient turbulence causes rapid vortex breakdown
- Fully turbulent wake more symmetric
- Rapid transition towards bell shaped deficit behind downstream turbine

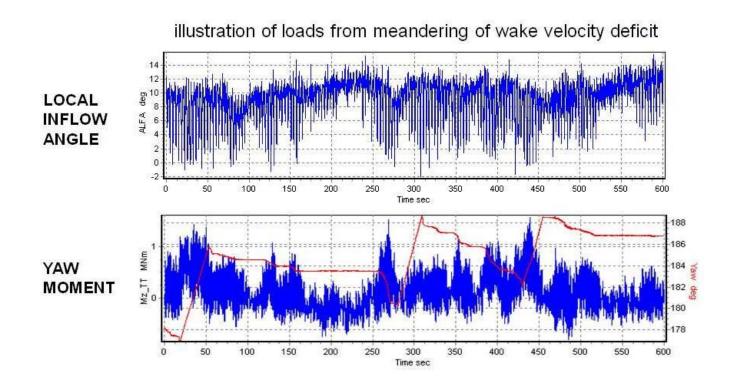




#### Measured influence of wake meandering

2002-2003

First version of model developed to investigate yaw loads in a wind farm

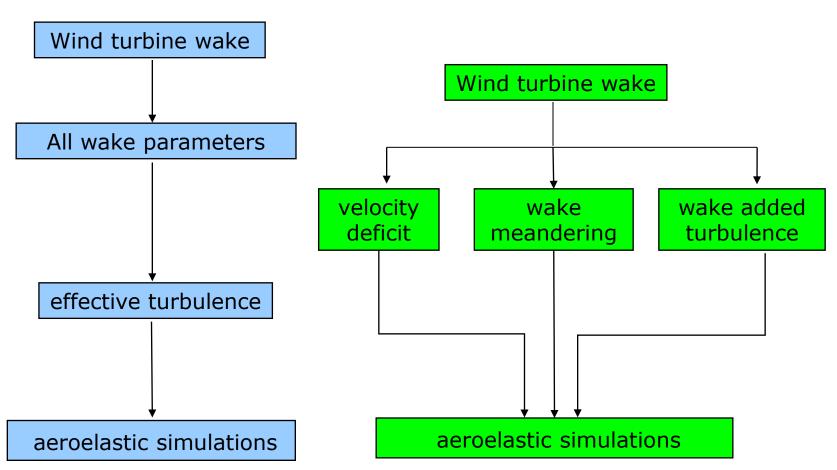


#### Different models for increased loading



Effective turbulence model

Dynamic wake meandering (DWM) model



## Load measurements on a NM80 2MW turbine in 3.3D wake



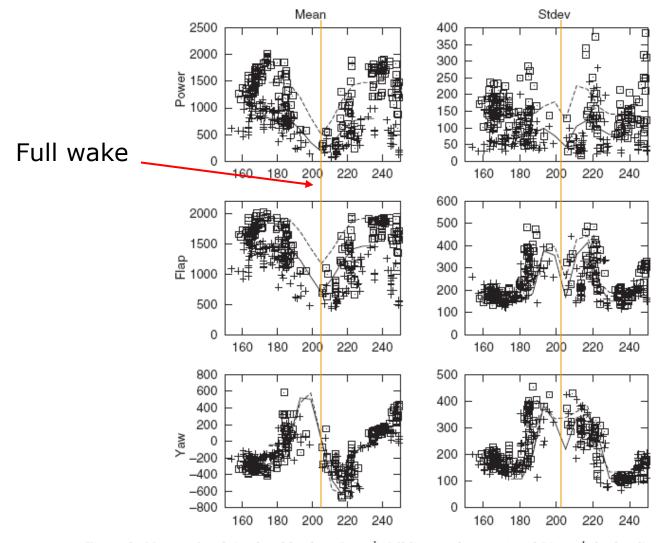


Figure 5. Measured and simulated loads at 8 m s<sup>-1</sup> (full lines and crosses) and 10 m s<sup>-1</sup> (broken lines and squares)

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#### **Aeroacoustics**

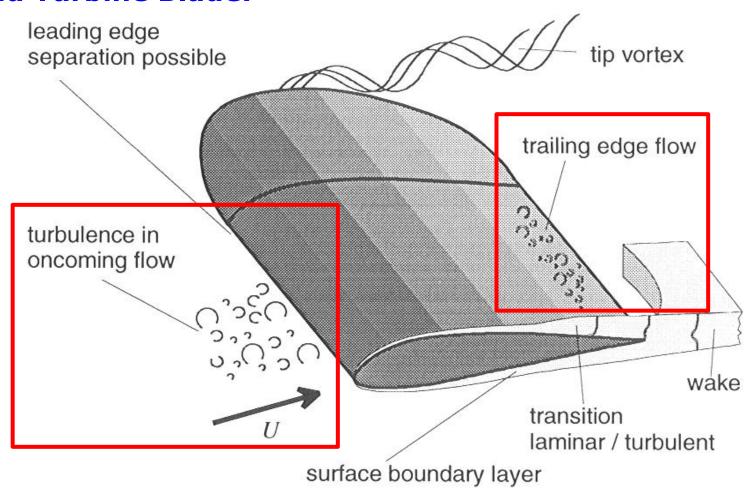


- broadband noise can often cause problems when siting turbines on land
- trailing edge noise and noise from inflow turbulence are the dominant sources
- max. blade tip speed ratio typically limited to 70 m/s due to noise constraints
- turbines have special low noise control
   modes by pitching more positive however
   production is reduced
- airfoils, blades and control are designed taking noise into account
- for turbines with a downwind rotor, low frequency noise can be a major problem

#### **Aerodynamic Noise**

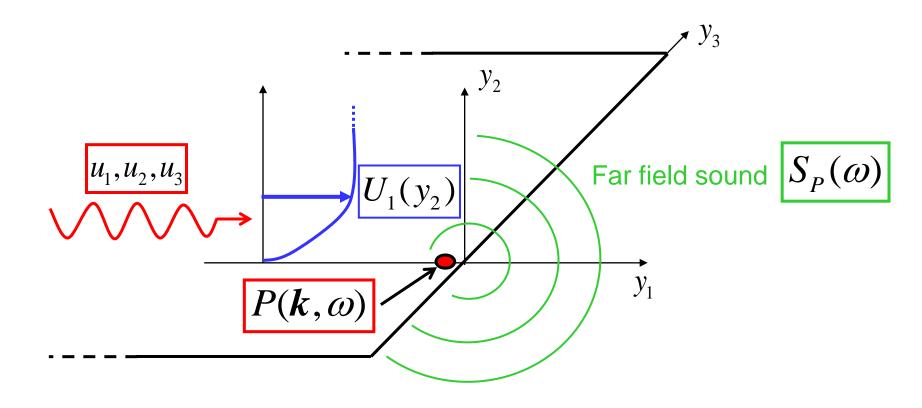


#### **Wind Turbine Blade:**



#### **Trailing Edge Noise**





#### **TNO Trailing Edge Noise Model**



Parchen (1998) combines a diffraction problem solution with knowledge of the turbulent fluctuations in the boundary layer

Airfoil Surface Pressure Spectrum (Blake, 1986)
 Lighthill analogy in spectral domain
 Solution for the Mean shear-Turbulence interaction:

$$P(\mathbf{k},\omega) = 4\rho_0^2 \frac{k_1^2}{k_1^2 + k_3^2} \int_0^{+\infty} L_2(y_2) \left(\frac{\partial U_1}{\partial y_2}\right) \overline{u_2}^2 \cdot \Phi_{22}(\mathbf{k},\omega) \cdot \Phi_m \left[\omega - U_c k_1\right] \cdot e^{-ky_2} dy_2$$

 <u>Far Field Noise</u> (Ffwocs Williams and Hall, 1970; Chandiramani, 1974; Chase, 1975; Howe, 1978; Brooks and Hodgson, 1981)

$$S_{P}(\omega) = \frac{L_{span}}{4\pi R^{2}} \int_{-\infty}^{+\infty} \frac{\omega}{c_{0}k_{1}} \cdot P(k_{1}, \omega) dk_{1}$$

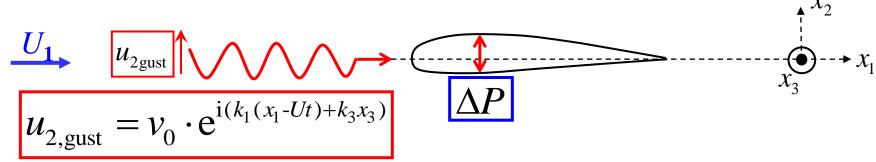
#### **Turbulent Inflow Noise Model**



#### Amiet's Theory (1976)

Linearized Inviscid Theory for flat plate with 0-mean loading

Inflow turbulence as a harmonic turbulent gust



Surface pressure response using Sears' theory:

$$\Delta P(x_1, x_3, t, k_1, k_3) = 2\pi \rho_0 v_0 g(x_1, k_1, k_3) \cdot e^{i(k_1 U t - k_3 x_3)}$$

where g is the transfer response function

#### **TE and TI Noise Characterization**



# **USING flush-mounted high-frequency MICROPHONES**



Trailing Edge Noise

Surface pressure spectrum near TE is correlated to TE far-field noise

Inflow Turbulence & Related Noise

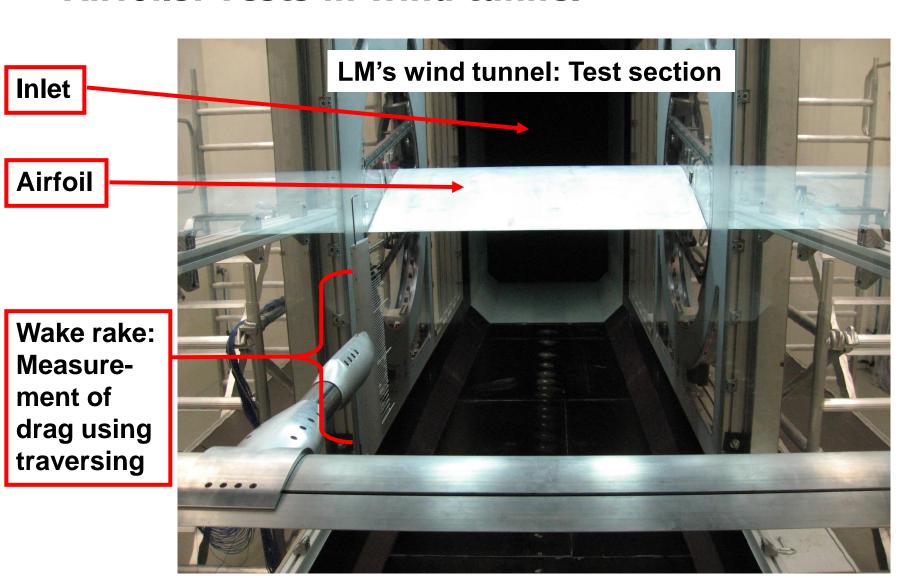
Surface pressure near LE characterizes the inflow turbulence

BL Transition

Surface pressure can be used to detect transition (Sudden increase of spectral intensity)

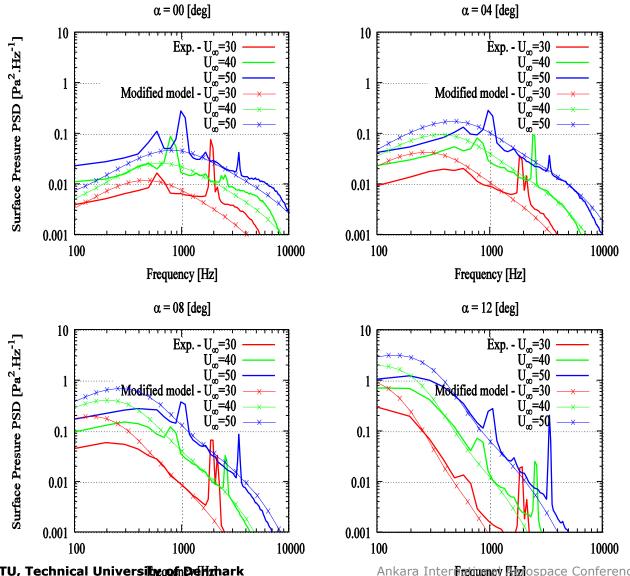


#### **Airfoils: Tests in wind tunnel**



#### Surf. Pres. measurements near TE measured in a wind tunnel

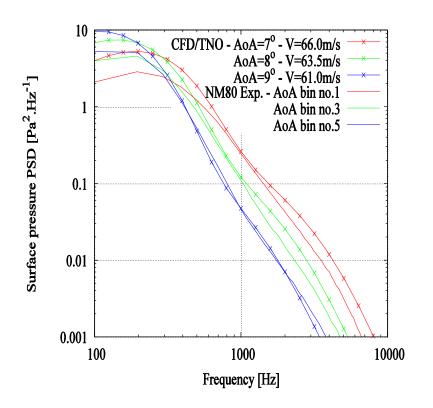




## Measurements on an 80 m diameter rotor – DANAERO project



# **Comparison Exp./Model**Surface Pressure near TE





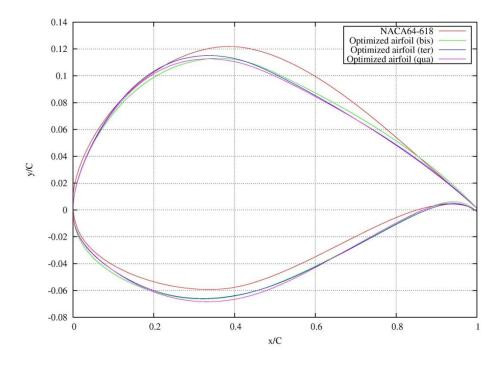


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> the models can now be used in a design optimization loop to design low noise airfoils



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#### **Floating turbines**



#### The HYWIND concept

#### HYMND concept by StatoilHydro

2-5MW pitch controlled wind turbine

Floating spar bouy attached to three mooring lines

Intended for water depths between 120 – 700m.

Demonstration project with Siemens 2.3MW 10km outside west coast of Norway.

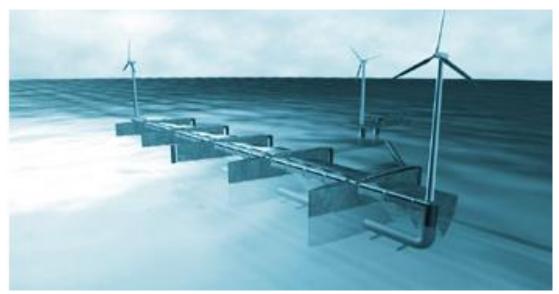






#### Combined wave and wind -- Poseidon





- Wave energy platform
- Dimensions are very large. Three turbines can produce extra power from wind – and contribute to the total damping of motion.





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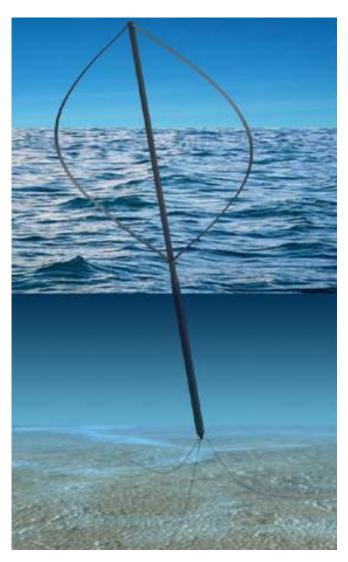
#### **Poseidon**



**Illustration** of the three 11 kW GAIA turbines mounted on the demonstration platform. The turbines are two-bladed fixed speed down-wind turbines with free yaw and a teeter mechanism.

## **DEEPWIND** – EU funded project on new floating wind turbine concept

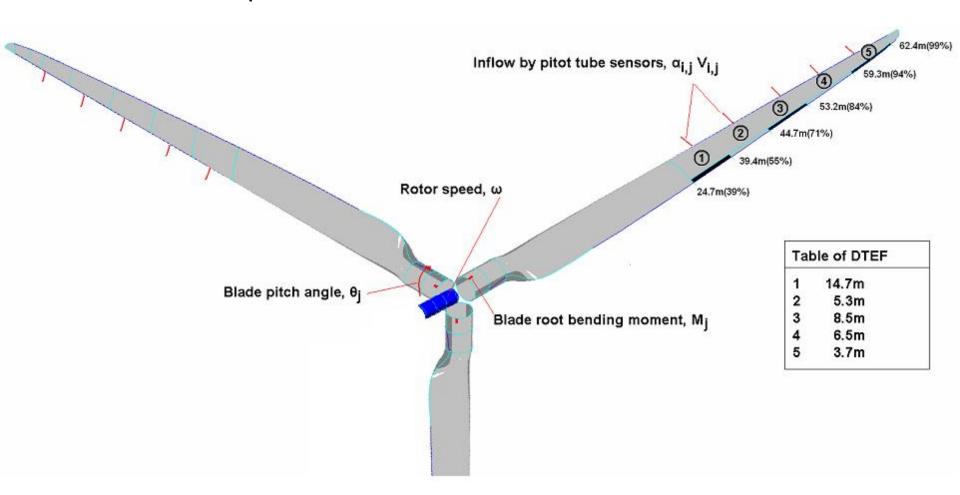




#### TRAILING EDGE FLAPS



#### Sensors and DTEG positions



#### **Outline**



- Introduction to Risø DTU
- **→** The typical wind turbine 2011
- Wind turbine loads and certification
- > Aerodynamic and aeroelastic tools
- > Aeroelastic stability
- Wind farms and wakes
- **Aeroacoustics**
- New technology outlook
- Summary

## Summary of key aeroelastic research issues 2011



- Modeling detailed influence of atmospheric inflow, turbulence and wind shear
- Wake modeling decreased power increased loading
- Vibrations at standstill
- Non-linear structural modelling of blades
- Dynamic effects in deep stall
- Structural damping enhancement
- Load alleviation using trailing-edge flaps or other devices
- Modeling floating design concepts



# THANK YOU for your attention