

6-1-2013

# The Use of Equivalent Source Models for Reduced Order Simulation in Room Acoustics

Yangfan Liu

*Purdue University*, liu278@purdue.edu

J Stuart Bolton

*Purdue University*, bolton@purdue.edu

Follow this and additional works at: <http://docs.lib.purdue.edu/herrick>

---

Liu, Yangfan and Bolton, J Stuart, "The Use of Equivalent Source Models for Reduced Order Simulation in Room Acoustics" (2013).  
*Publications of the Ray W. Herrick Laboratories*. Paper 80.  
<http://docs.lib.purdue.edu/herrick/80>

This document has been made available through Purdue e-Pubs, a service of the Purdue University Libraries. Please contact [epubs@purdue.edu](mailto:epubs@purdue.edu) for additional information.



# The Use of Equivalent Source Models for Reduced Order Simulation in Room Acoustics

Yangfan Liu  
Advisor: J. Stuart Bolton



# Motivation

## □ Room Acoustics with Source of Finite Size



Available techniques are not suitable for a fast and accurate room acoustics simulations of finite-size sources

Sound prediction of a flat screen TV:

- Not point sources - - - >
- Arbitrary room shapes - - - >
- Fast - - - >

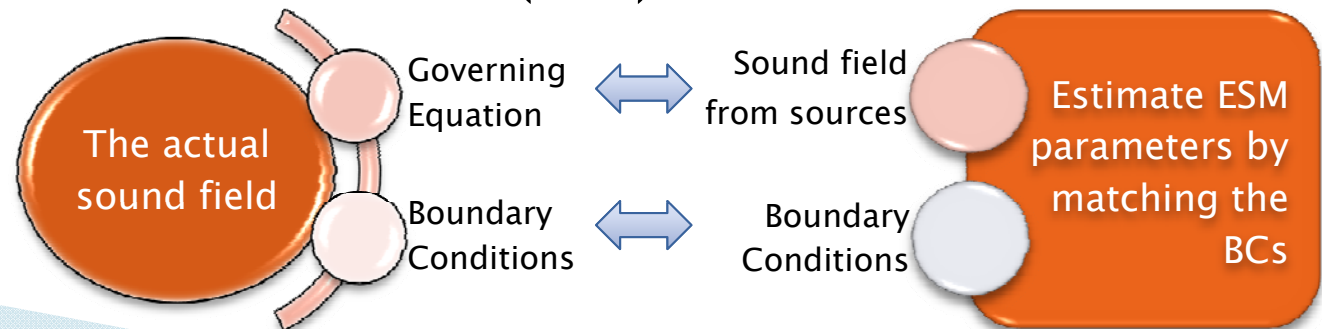
Traditional Methods:

- Ray Tracing
- Image Source Model (hybrid)
- BEM/FEM



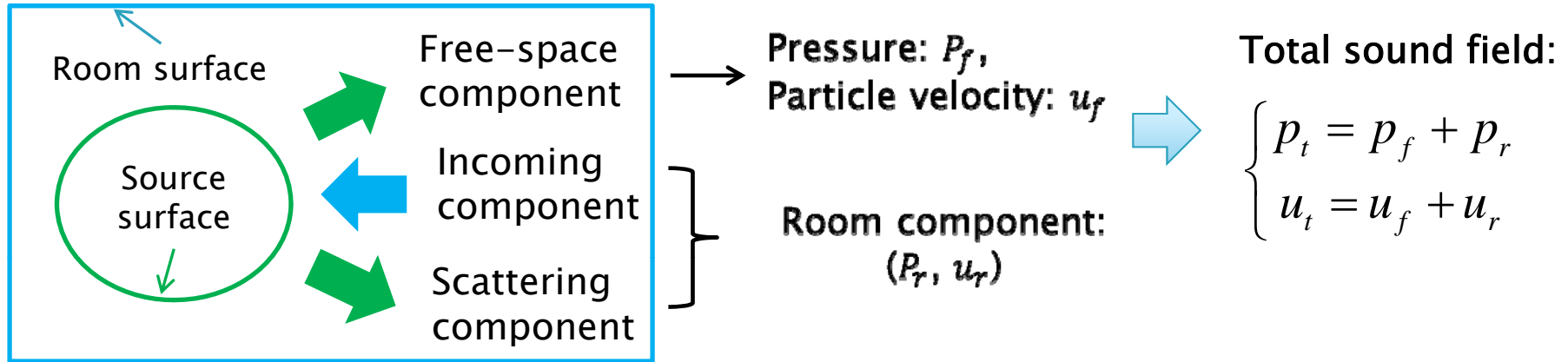
## □ The Use of Equivalent Source Models (ESM)

**ESM in General:**

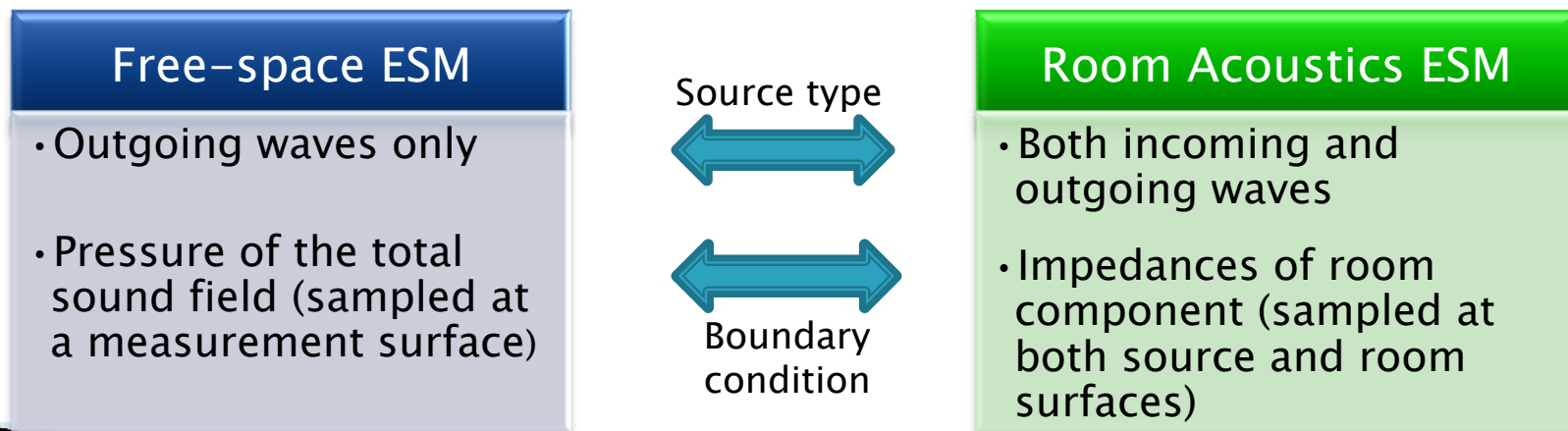


# ESM in Room Acoustics

## □ Sound Field Components in Room Acoustics



## □ Free-space ESM vs. Room Acoustics ESM



# Boundary Conditions

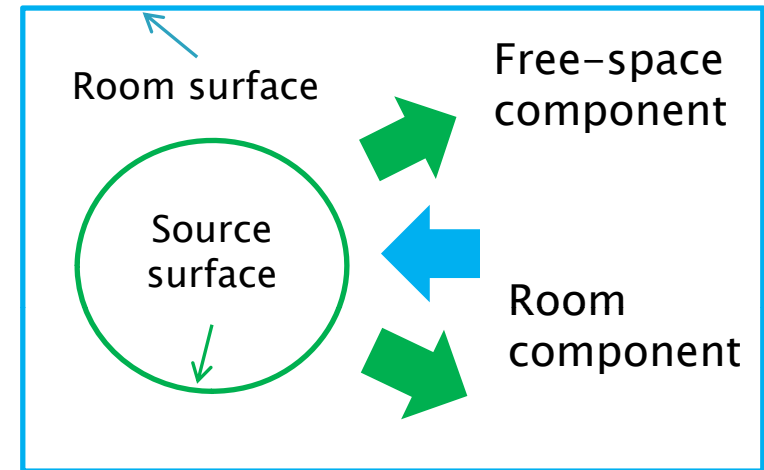
## □ BC of the Room Components in Room Acoustics

### ○ BC on source surface, $\Gamma_1$ , (admittance $\beta_1$ ):

Total:  $\beta_1(x)p_t(x) + u_0(x) = u_{nt}(x)$

Free-space:  $\beta_1(x)p_f(x) + u_0(x) = u_{nf}(x)$

In-vacuo driving velocity on the source surface. (source characteristics)



### ○ BC on room surface, $\Gamma_2$ , (admittance $\beta_2$ ):

Total:  $\beta_2(x)p_t(x) = u_{nt}(x)$

Total sound field:

$$\begin{cases} p_t = p_f + p_r \\ \vec{u}_t = \vec{u}_f + \vec{u}_r \end{cases}$$

Boundary conditions used in the room acoustics ESM:

$$\begin{cases} \beta_1(x)p_r(x) - u_{nr}(x) = 0 & x \in \Gamma_1 \\ \beta_2(x)p_r(x) - u_{nr}(x) = u_{rf}(x) - \beta_2(x)p_f(x) & x \in \Gamma_2 \end{cases}$$

**The problem of room acoustics:**

**Given:  $p_f, u_f$ ; Calculate:  $p_r, u_r$**

# Construction of ESM in General

## □ General Procedure of Constructing Room Acoustics ESM

After choose the type of equivalent sources:

$$\begin{cases} p(x) = \sum_{i=1}^N g_i(x, y_i) Q_i, \\ \vec{u}(x) = \frac{1}{j\omega\rho_0} \sum_{i=1}^N \vec{\nabla} g_i(x, y_i) Q_i, \end{cases}$$

- $g_i(x, y_i)$  - sound field of source with unit strength
- $Q_i$  - the source strength

Location in the sound field

Location of each source



Matrix form used to estimate the ESM parameters:

$$\begin{bmatrix} B_1 A_p^{(1)} - A_{u_n}^{(1)} \\ B_2 A_p^{(2)} - A_{u_n}^{(2)} \end{bmatrix} \vec{Q} = \begin{bmatrix} 0 \\ \vec{u}_{nf} - B_2 \vec{p}_f \end{bmatrix}$$

→ Boundary condition on source surfaces

→ Boundary condition on room surfaces

Evaluate  $P$  and  $u_n$  at a number of sampling locations on both source and room surfaces

$$\begin{cases} B_1 = \text{diag}(\beta_1(x_1), \beta_1(x_2), \dots, \beta_1(x_{M_1})), \\ B_2 = \text{diag}(\beta_2(x_{M_1+1}), \beta_2(x_{M_1+2}), \dots, \beta_2(x_M)) \\ (A_p^{(1)})_{ij} = g_j(x_i, y_j), & (A_{u_n}^{(1)})_{ij} = \frac{1}{j\omega\rho_0} \partial_n g_j(x_i, y_j), \\ (A_p^{(2)})_{ij} = g_j(x_{M_1+i}, y_j), & (A_{u_n}^{(2)})_{ij} = \frac{1}{j\omega\rho_0} \partial_n g_j(x_{M_1+i}, y_j), \end{cases}$$



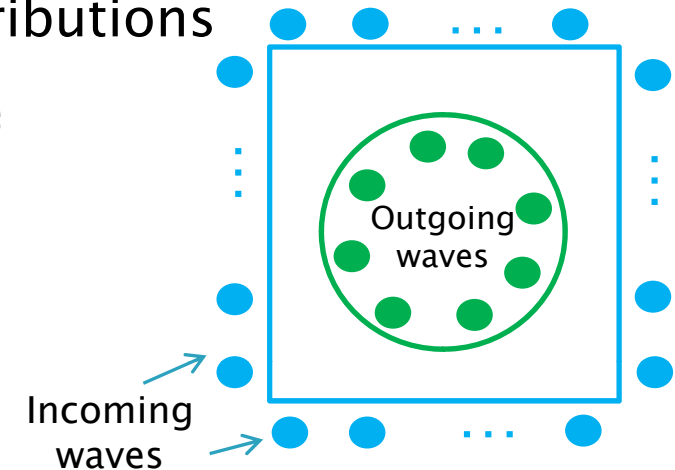
Construct a specific ESM  
(find the sound field expression)

# Two Types of ESMs

## □ Room Acoustics ESM Using Monopole Distributions

- Monopoles are distributed inside the source surface (outgoing wave) and outside the room surface (incoming wave)
- Sound field expression (2D):

$$g(x, y) = \frac{j}{4} H_0^{(1)}(kr)$$



## □ Room Acoustics ESM Using Multipole Distributions

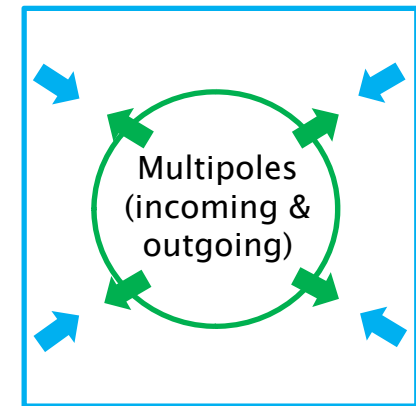
- Multipoles of monopole, dipole, quadrupole, ...  $n$ th order, ... (both incoming and outgoing)
- Sound field expression (2D):

$$n = 0: \quad P_0^{out}(x, y) = \frac{j}{4} H_0^{(1)}(kr), \quad P_0^{in}(x, y) = \frac{j}{4} H_0^{(2)}(kr)$$

$$n \text{ th order: } \begin{cases} P_{S_n}^{out} = S_n P_n^{out} = S_n R_n(P_0^{out}) \cdot \vec{v}_1 \cdot \vec{v}_2 \dots \vec{v}_n \\ P_{S_n}^{in} = S_n P_n^{in} = S_n R_n(P_0^{in}) \cdot \vec{v}_1 \cdot \vec{v}_2 \dots \vec{v}_n \end{cases},$$

$$R_n = \nabla^{\otimes n}$$

- ⊗ - tensor outer product
- - tensor inner product



Details on next slide


# Sound field of Multipole Sources

## □ Sound field expression for multipole sources (2D):

Sound field of order  $n + 1$  source and order  $n$  source can be related by directional derivative:

$$P_{S_{n+1}}(\vec{X}|\vec{X}_0, \omega) = d \langle \nabla P_{S_n}(\vec{X}|\vec{X}_0, \omega), \vec{v}_{n+1} \rangle$$

$$= d \left\langle \left[ \frac{\partial P_{S_n}(\vec{X}|\vec{X}_0, \omega)}{\partial x_0}, \frac{\partial P_{S_n}(\vec{X}|\vec{X}_0, \omega)}{\partial y_0} \right]^T, \vec{v}_{n+1} \right\rangle$$




**Dipole:**  $P_{S_1} = S d_1 \langle \nabla P_0, \vec{v}_1 \rangle,$

Dipole strength:  $S d_1$

**Quadrupole:**  $P_{S_2} = S d_1 d_2 \vec{v}_2^T \vec{R}_2 \vec{v}_1,$

Quadrupole strength:  $S d_1 d_2$



**Order n:**

$$P_{S_n} = Q_n g_n$$

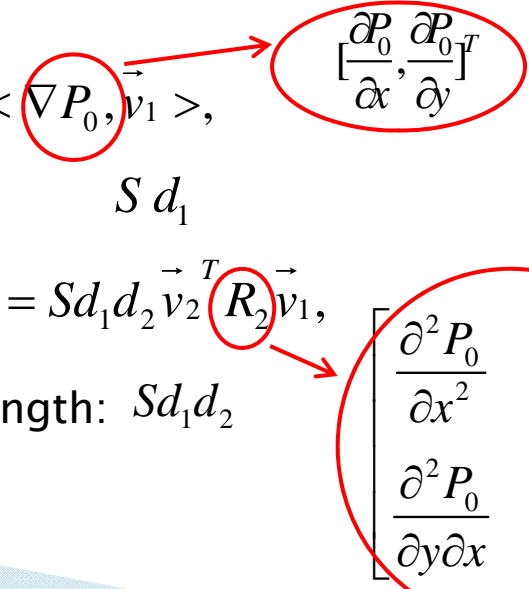
Source strength:

$$Q_n = S d_1 d_2 \dots d_n$$

$$g_n = R_n(P_0) \cdot \vec{v}_1 \cdot \vec{v}_2 \dots \vec{v}_n,$$

$$R_n = \nabla^{\otimes n}$$

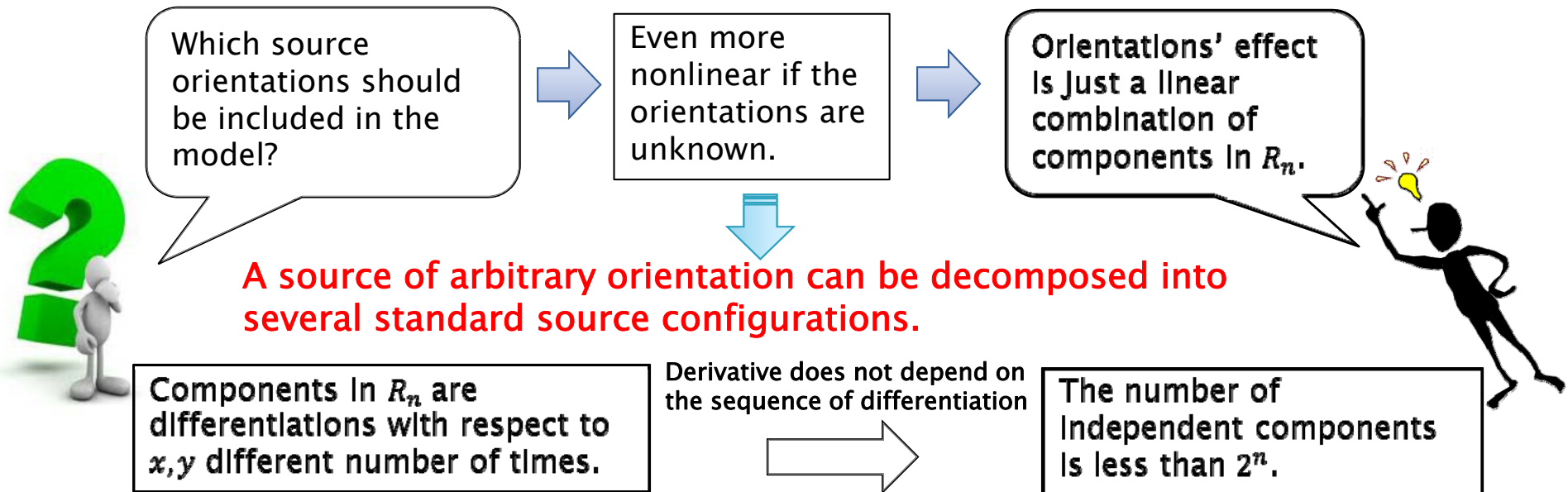
- $\otimes$  - tensor outer product
- $\cdot$  - tensor inner product





# Decomposition of the Multipoles

- For a source of order  $n$  ( $n > 1$ ), there could be infinitely many types because of the orientation vectors.



- How many independent elements?

$n$ th order source, in  $r$ -D space :

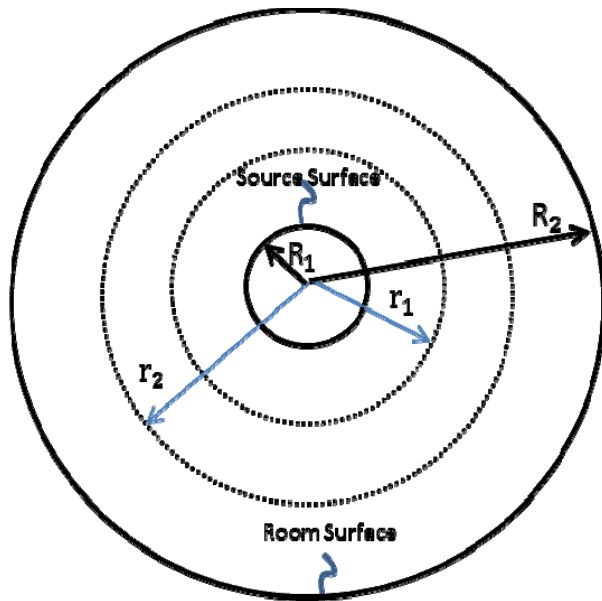
$$N(n, r) = \begin{cases} 1, & n = 0 \\ C_{n+r-1}^n, & n > 0 \end{cases}$$



- Can be numerically enumerated for an arbitrary  $n$ .

# Simulation Setup

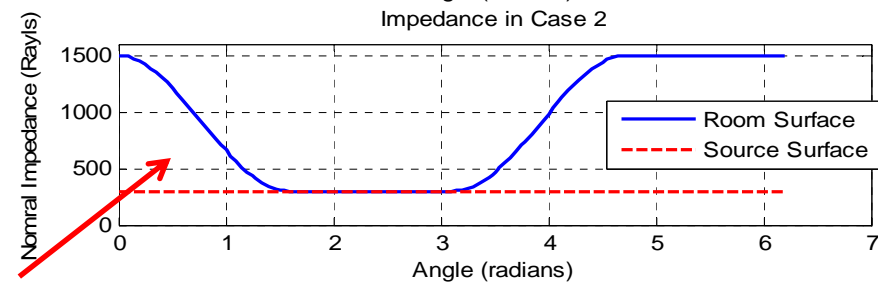
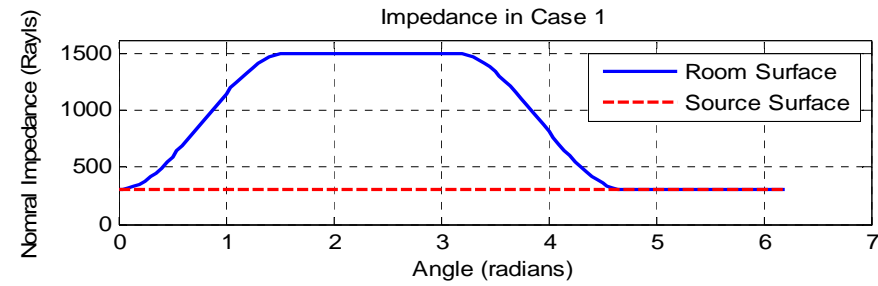
□ Simulation in a 2 dimensional room (circular geometry)



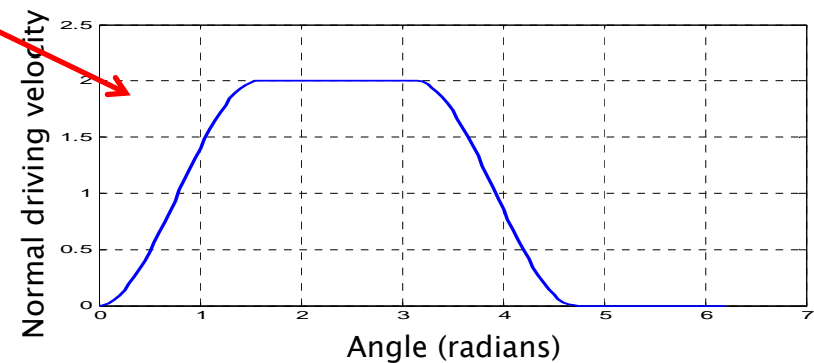
Geometry of the room

- $R_1 = 0.5$  m (source surface),
- $R_2 = 2$  m (room surface),
- $r_1 = 1$  m (100 microphones) ,
- $r_2 = 1.5$  m (100 microphones).

Non-uniform



Normal impedance on different surfaces



In-vacuo Driving velocity on the source surface

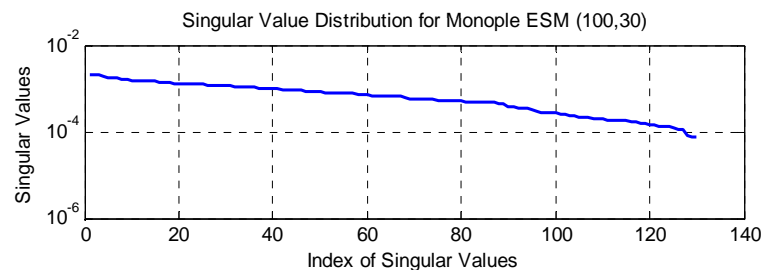
# Models Used in Simulations

## □ Different models used in the simulation

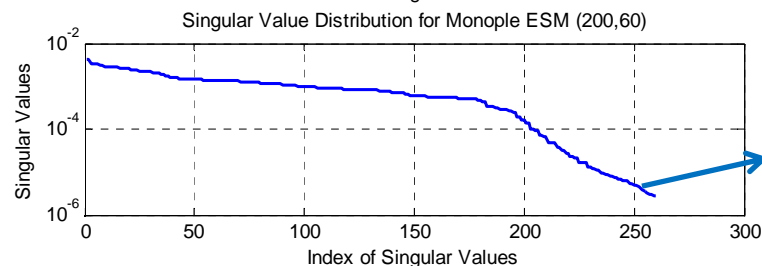
Type of ESM	Number of Parameters
Multipole ESM (order up to 3)	10
Multipole ESM (order up to 6)	28
Monopole ESM (outside: 100; inside: 30)	130
Monopole ESM (outside: 150; inside: 45)	195
Monopole ESM (outside: 200; inside: 60)	260
Monopole ESM (outside: 1000; inside: 300)	1300

- Result from a Boundary Element Model was used as the true sound field.

## □ Choice of regularization techniques

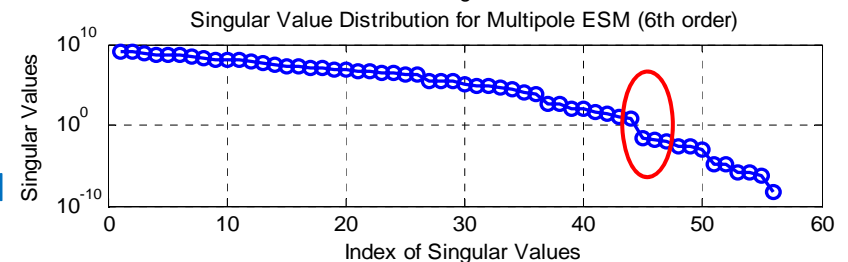
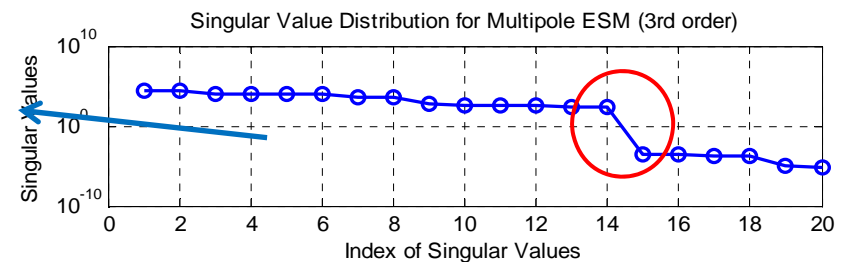


Use TSVD for multipole ESM



Use GCV for monopole ESM

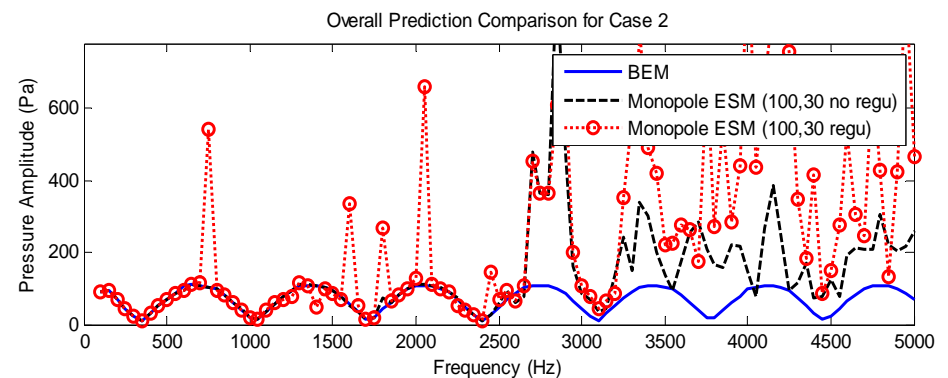
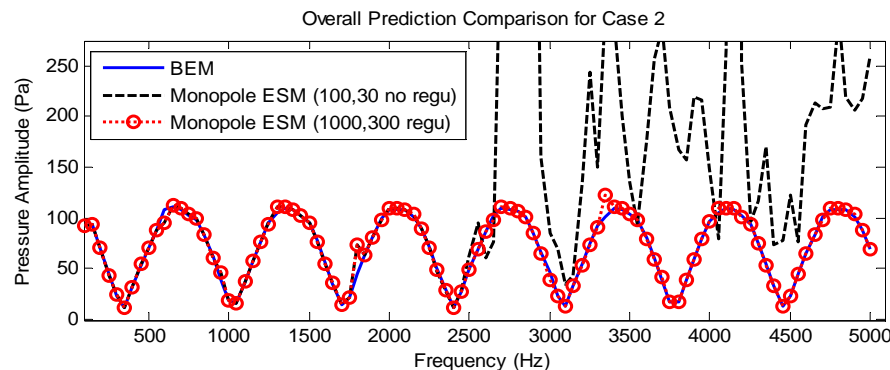
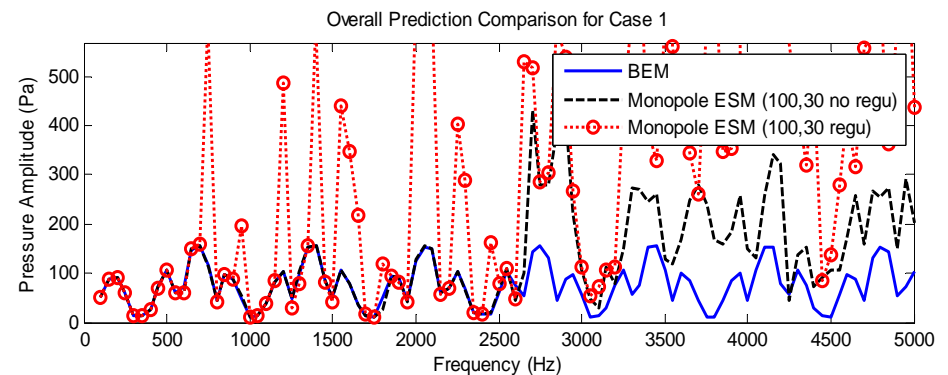
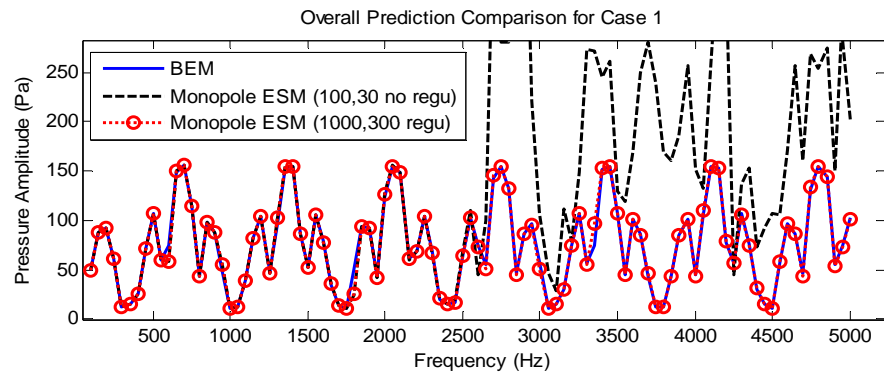
Singular value distribution in monopole ESMs



Singular value distribution in multipole ESMs

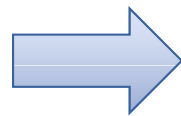
# Analysis of Results

## □ Results from monopole distribution ESMs



Spatially averaged prediction from monopole ESMs

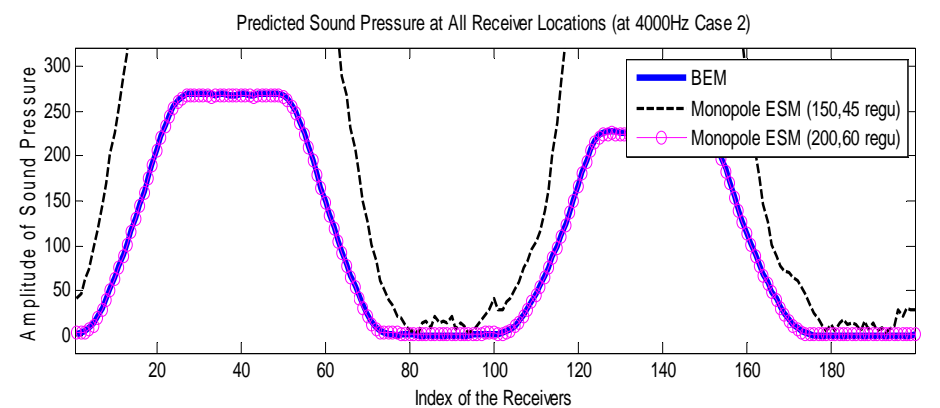
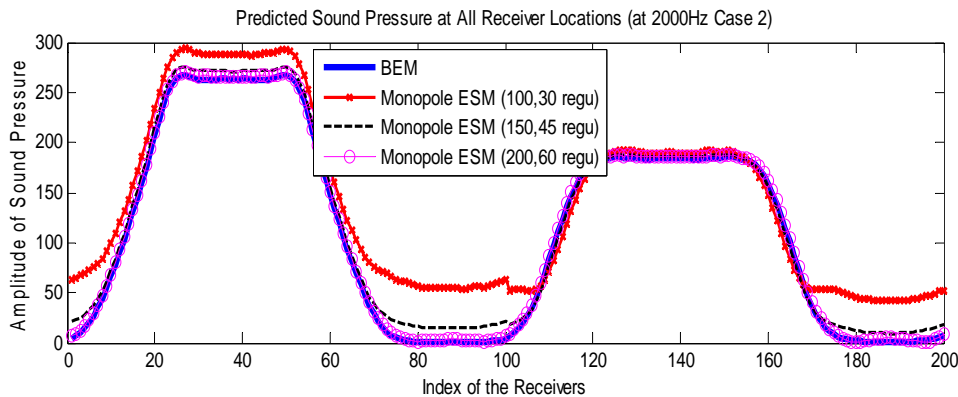
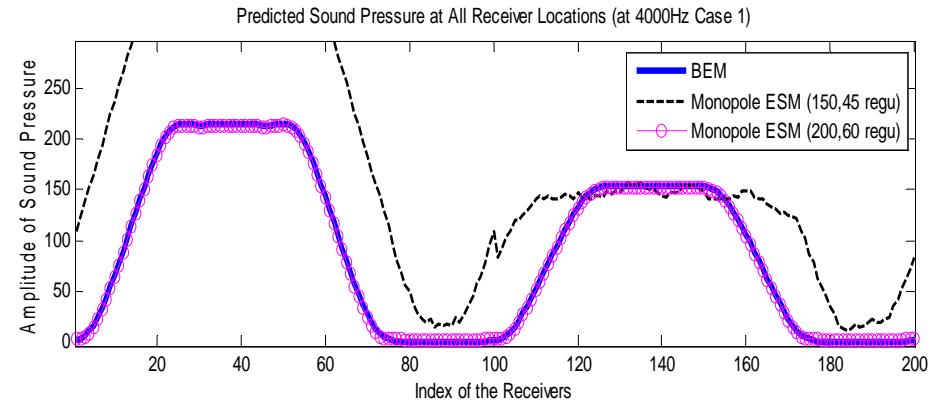
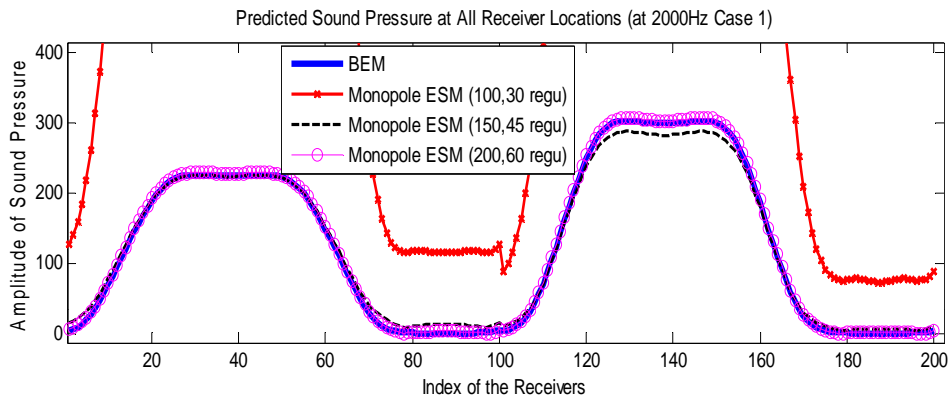
Spatially averaged prediction from monopole ESMs



- Accurate (up to 5000 Hz) if there are a large number of monopoles and with regularization.
- Regularization may cause instabilities.

# Analysis of Results

## Results from monopole distribution ESMs



Prediction from monopole ESMs at 2000 Hz

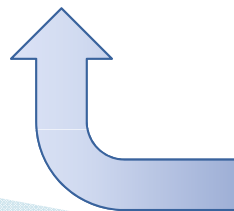
Prediction from monopole ESMs at 4000 Hz

- A monopole ESM requires at least 260 model parameters to achieve accurate predictions.

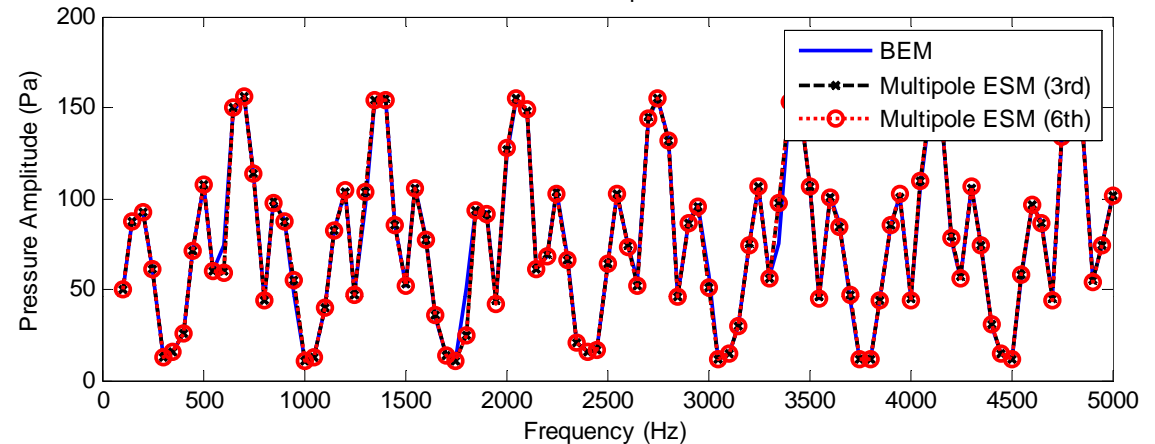
# Analysis of Results

## □ Results from multipole ESMs

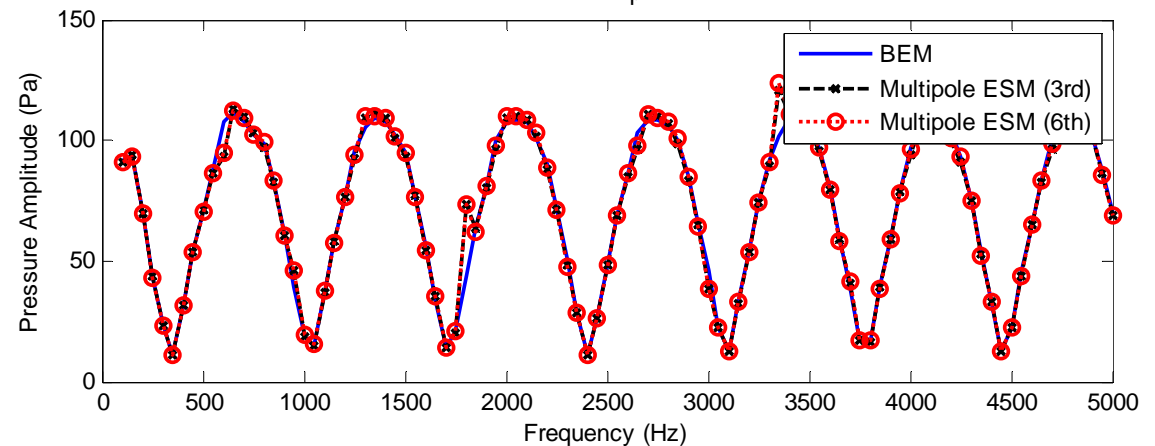
- Regularization does not cause instabilities (more robust).
- The spatially averaged predictions are similar with different multipole orders (accurate up to 5000 Hz).
- Multipole ESM requires much fewer number of model parameters than monopole ESM



Overall Prediction Comparison for Case 1



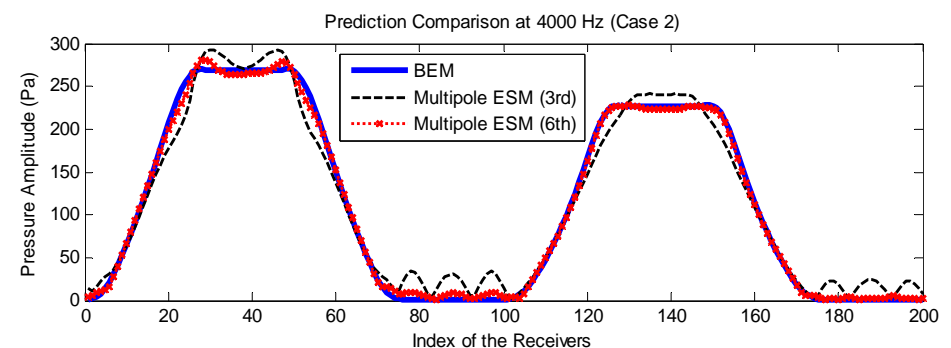
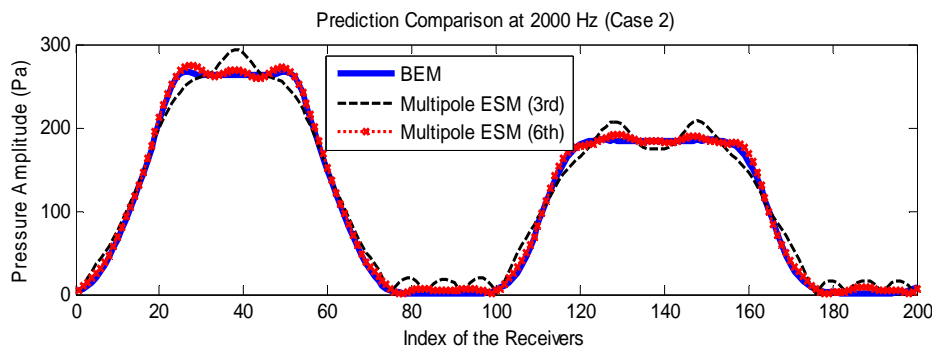
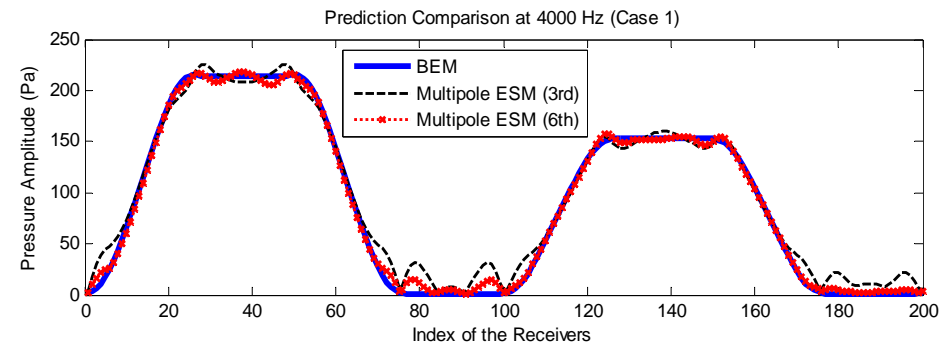
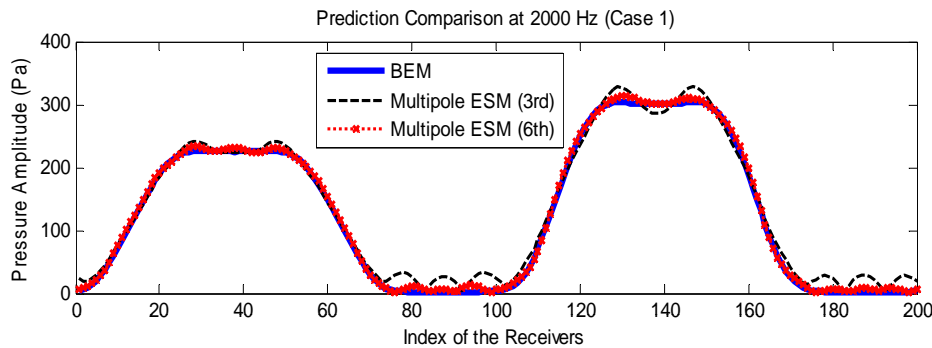
Overall Prediction Comparison for Case 2



Spatially averaged prediction from multipole ESMs

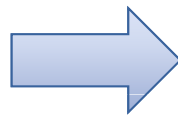
# Analysis of Results

## Results from multipole ESMs



Prediction from multipole ESMs at 2000 Hz

Prediction from multipole ESMs at 4000 Hz



- Increase of multipole order improves the prediction oscillation in space.
- It is flexible to balance the computational effort and the prediction accuracy. (choose an appropriate truncation order)

# Conclusions

- ❑ Equivalent source models are constructed for room acoustics simulations with finite-size source, arbitrary geometry and non-uniform surface normal impedances.
- ❑ In room acoustics ESMs, both outgoing and incoming waves should be included, and the impedance boundary conditions for the room component sound field are used for parameter estimation.
- ❑ Both monopole ESMs and multipole ESMs can achieve accurate performance up to 5000 Hz, but the multipole ESM requires fewer model parameters and is more robust.
- ❑ When using multipole ESM, there is a flexible balance between the computational effort and the prediction accuracy, controlled by choosing an appropriate truncation order.



*Thank You!*