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# High Reynolds Channel Flows: Upstream interaction of various wall deformations

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**Summary.** The flow at high Reynolds number in the entrance of distorted channel is considered. We analyse the anticipated fluid responds to a downstream wall distortion, and we find that the non linear upstream length  $\Delta = O(R_e^{1/7})$ , using either a new asymptotic approach called Successive Complementary Expansions Method (SCEM) with generalized asymptotic expansions and a modal analysis of the perturbed flow. Comparisons with Navier-Stokes solutions show that the mathematical model is well founded.

# **1** Introduction

This paper considers the upstream interaction of flows in a two-dimensional channel at high Reynolds number with wall deformations. An asymptotic model using the successive complementary expansion method with generalized asymptotic expansions, called GIBL for Global Interactive Boundary Layer [1, 3], is used. The aim is to analyse the non linear asymptotic length  $\Delta$  of the upstream influence of an *accident* at  $x = x_0$  at the walls. As Smith [2] we found that  $\Delta = O(R_e^{1/7})$ , where  $R_e$  is the Reynolds number. The only hypothesis on the wall *accident* is that it is significant enough to perturbe the Poiseuille flow, so that the Poiseuille flow is no more a good approximation in the boundary layer.

Then by assuming an exponential variation in x of the perturbed flow, in order to obtain the Poiseuille flow as  $x \to -\infty$  (i.e. far upstream the wall deformations), we perform an eigenvalue analysis. We thus found that the first mode is related to non-symmetric wall deformations. Two kind of wall deformations are considered (local and global distortions) and comparisons between GIBL, Navier-Stokes solutions and eigenmodes show that the model is well founded.

# 2 Geometrical configuration

Two kind of geometrical configuration have been considered for the *accident*: (i) a local wall perturbation as in figure 1, or (ii) a global wall curvature as in figure 2.

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$$y_{l} = \frac{1}{2} - G(x, \varepsilon)$$

$$y_{l} = -\frac{1}{2} + F(x, \varepsilon)$$

**Fig. 1.** (i) case: Local wall perturbation; location of the accident at  $x = x_0$ .

In the test case (i), the walls are deformed in a domain  $x_0 \le x \le x_0 + L$  such as:

$$F = \frac{h_l}{2} \left( 1 + \cos \frac{2\pi x}{L} \right) ; G = -\frac{h_u}{2} \left( 1 + \cos \frac{2\pi x}{L} \right) . \tag{1}$$

where  $h_u$  and  $h_l$  are small parameters.

In the test case (ii), we use a generalized system of coordinates, where *X* and *Y* are distances along and perpendicular to the line H = 0. We call it the median line if the upper (or inner) and lower (or external) walls are respectively given by  $Y = \pm \frac{1}{2}$ .

For a point *M* with general coordinates *X* and *Y*, we can write  $\overrightarrow{OM} = \overrightarrow{OM_0}^2 + Y\mathbf{n}$ , where **n** is the unit normal vector. Then,  $\overrightarrow{dM} = dX(1 + KY)\tau + dY\mathbf{n}$ , where  $\tau$  is the unit vector tangent at  $M_0$  to the median line in such a way that  $(\tau, \mathbf{n})$  is direct; K(X) is the algebraic curvature of this line. Thus, K < 0 in the case of figure 2. The curvature for X > 0 by  $K = \delta k(X)$ , where  $\delta$  is a small positive parameter. Let *U* and *V* denote the velocity components parallel and perpendicular to the line H = 0, then, as  $\mathbf{V} = U\tau + V\mathbf{n}$ , the full equations of motion written in generalized coordinates are given in [4]. These equations must be solved with boundary conditions: U = V = 0 for  $Y = \pm \frac{1}{2}$ .



**Fig. 2.** (ii) case: Global wall curvature; location of the accident at  $X = X_0$ .

# 3 Fully established flow in a curved channel

For a channel of constant curvature  $\delta$ , the fully established flow  $U_0$  is solution of

$$(1+\delta Y)\frac{d^{2}U_{0}}{dY^{2}} + \delta \frac{dU_{0}}{dY} - \frac{\delta^{2}}{1+K_{0}Y}U_{0} = -GR_{e}$$
(2)

where  $G = -\frac{\partial P}{\partial X}$  is constant, and with  $U_0 = 0$  for  $Y = \pm \frac{1}{2}$ . Notice that for  $\delta = 0$  we retrieve the equation for the Poiseuille flow:  $\frac{d^2 U_0}{dY^2} = -2$ .

The exact solution is given by:

$$U_0(Y) = \frac{1}{64} GR_e \frac{f(\delta, Y)}{(\delta^2(1+\delta Y))}$$
(3)

where

$$\begin{split} f(\delta,Y) &= \left[ \delta^3 (1-4Y^2) + 8\delta^2 Y(2Y-1) + 4\delta(-4Y^2+8Y-3) + 16(1-2Y) \right] \ln\left(\frac{2-\delta}{2\delta}\right) + \\ &\left[ -\delta^3 (1-4Y^2) + 8\delta^2 Y(2Y+1) + 4\delta(4Y^2+8Y+3) + 16(1+2Y) \right] \ln\left(\frac{2+\delta}{2\delta}\right) - \\ &32 \left( 2\delta Y + \delta^2 Y^2 + 1 \right) \ln\left(\frac{1+\delta Y}{\delta}\right) \end{split}$$

As shown in figure 3, the corresponding exact solution  $U_0(y)$  bends towards the internal wall of the bend. Notice that, for a small constant curvature  $\delta$  and for a flow



**Fig. 3.** Velocity profile  $U_0(Y)$ ; Poiseuille flow (dashed line); profile for  $\delta = 1$  (straight line).

rate of 1/6, an approximate solution O( $\delta$ ) is  $U_0 = \left(\frac{1}{4} - Y^2\right) \left(1 - \frac{2\delta}{3}Y\right)$ , which implies a skin friction of  $C_f \frac{R_e}{2} = 1 \pm \frac{\delta}{3}$ .

#### 4 Global Interactive Boundary Layer (GIBL) model

According to the SCEM, a Uniformaly Valid Approximation (UVA) for the velocity and pressure fields (U, V, P) is obtained by complementing the core approximation  $(U_1 = u_0 + \delta u_1, V_1 = \delta v_1, P_1 = p_0 + \delta p_1)$  such as:

$$U = u_0(Y) + \delta[u_1(X, Y, \delta) + U_{BL}(X, \eta, \delta)]$$
  

$$V = \delta[v_1(X, Y, \delta) + \varepsilon V_{BL}(X, \eta, \delta)]$$
  

$$P = p_0(X) + \delta[p_1(X, Y, \delta, \varepsilon) + \Delta(\varepsilon) P_{BL}(X, \eta, \delta, \varepsilon)]$$
(4)

where  $\lim_{\eta\to\infty} U_{BL} = 0$ ,  $\lim_{\eta\to\infty} V_{BL} = 0$  and  $\lim_{\eta\to\infty} P_{BL} = 0$  (see [4] for more details). Thus, we obtain Uniformaly Valid Approximation (UVA) equations:

$$\frac{\partial U}{\partial X} + \frac{\partial V}{\partial Y} = 0$$
$$U\frac{\partial U}{\partial X} + V\frac{\partial U}{\partial Y} = -\frac{\partial P_1}{\partial X} + \frac{1}{R_e}\frac{\partial}{\partial Y}\left[(1+KY)\frac{\partial U}{\partial Y}\right]$$

with the following boundary conditions, U = V = 0, for  $Y = \pm \frac{1}{2}$ . The core equations being:

$$u_0 \frac{\partial V_1}{\partial X} - K u_0^2 = -\frac{\partial P_1}{\partial Y}$$
$$-u_0 \frac{\partial V_1}{\partial Y} + V_1 \frac{du_0}{dY} = -\frac{\partial (P_1 - p_0)}{\partial X}$$

A simplified model for the pressure gives  $\frac{\partial P_1}{\partial X} = \frac{dp_0}{dX} + \delta (A''' + k') \int_{\eta_c}^{\eta} u_0^2(\eta') d\eta' + \delta B'(X)$ . At the medline, i.e. for  $\eta = \eta_c$ , since the UVA *V* should match the core approximation  $V_1$ , we impose the coupling condition  $V = V_1 = -A'(X)u_0$ . For more details about GIBL, see the companion paper [5].

### **5** Upstream interaction

#### 5.1 Upstream length

In a straight channel, upstream of the wall *accident*, for x < 0, the GIBL and core equations become:

$$U\frac{\partial U}{\partial x} + V\frac{\partial U}{\partial y} = -\frac{\partial P_1}{\partial x} + \frac{1}{Re}\frac{\partial^2 U}{\partial y^2}$$
(5)

$$\frac{\partial U}{\partial x} + \frac{\partial V}{\partial y} = 0 \tag{6}$$

$$-U_0 \frac{\partial V_1}{\partial y} + V_1 \frac{dU_0}{dy} = -\frac{\partial (P_1 - P_0)}{\partial x}$$
(7)

$$U_0 \frac{\partial V_1}{\partial x} = -\frac{\partial (P_1 - P_0)}{\partial y} \tag{8}$$

We now consider perturbations of the following form:  $U = U_0 + \varepsilon u$ ,  $V = \varepsilon v$  and  $P_1 = P_0 + \lambda p_1$ . If the critical unknown streamwise length scale is  $\Delta$ , then, with  $\overline{x} = \frac{x}{\Delta}$  and thus

 $\overline{V} = \Delta V$ , we obtain from (5,6,7,8) the following perturbation equations:

$$\frac{\partial u}{\partial \overline{x}} + \frac{\partial \overline{v}}{\partial y} = 0 \tag{9}$$

$$U_0 \frac{\partial u}{\partial \overline{x}} + \overline{v} \frac{\mathrm{d}U_0}{\mathrm{d}y} + \varepsilon \left( u \frac{\partial u}{\partial \overline{x}} + \overline{v} \frac{\partial u}{\partial y} \right) = -\frac{\lambda}{\varepsilon} \frac{\partial p_1}{\partial \overline{x}} + \frac{\Delta}{Re} \frac{\partial^2 u}{\partial y^2} \tag{10}$$

$$-U_0 \frac{\partial \overline{v_1}}{\partial y} + \overline{v_1} \frac{\mathrm{d}U_0}{\mathrm{d}y} = -\frac{\lambda}{\varepsilon} \frac{\partial p_1}{\partial \overline{x}}$$
(11)

$$U_0 \frac{\partial \overline{v_1}}{\partial \overline{x}} = -\frac{\lambda \Delta^2}{\varepsilon} \frac{\partial p_1}{\partial y}$$
(12)

If  $\varepsilon$  is the boundary layer thickness, the first significant perturbation is such as  $U_0 = O(\varepsilon)$ ,  $\overline{v} = O(\varepsilon)$  in the boundary layers, which implies from (10) that  $\varepsilon$ ,  $\frac{\lambda}{\varepsilon}$  and  $\frac{\Delta}{\varepsilon^2 R_e}$  are of same order.

An upstream interaction takes place if we have a generation of a significant transverse pressure gradient in the core flow, which implies from (12) that  $\frac{\lambda\Delta^2}{\epsilon} = O(1)$ . Thus, we easily obtain (as did Smith [2] by regular asymptotic expansions) the following crucial orders:

$$\Delta = O(R_e^{1/7}), \quad \epsilon = O(R_e^{-2/7}) \quad \text{and} \quad \lambda = O(R_e^{-4/7}).$$
 (13)

#### 5.2 Eigenmode analysis

For x < 0, the linearized UVA system of equations may be written as :

$$\begin{cases} U_0 \frac{\partial u}{\partial x} + U'_0 v = -\frac{\partial p_1}{\partial x} + \frac{1}{Re} \frac{\partial^2 u}{\partial y^2} \\ \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \\ U_0 \frac{\partial v_1}{\partial x} = -\frac{\partial p_1}{\partial y} \end{cases}$$
(14)

By replacing  $v_1$  by v in the transverse core momentum equation, and by assuming the following form for u, v and  $p_1$ :

$$u(x,y) = \hat{u}(y)e^{\theta x}, \quad v(x,y) = \hat{v}(y)e^{\theta x}, \quad p_1(x,y) = \hat{p}_1(y)e^{\theta x}$$
 (15)

we obtain for the perturbations:

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$$\theta \begin{pmatrix} U_0 & 0 & 1 \\ 1 & 0 & 0 \\ 0 & U_0 & 0 \end{pmatrix} \hat{q} = \begin{pmatrix} \frac{D^2}{Re} & -U_0' & 0 \\ 0 & -D^1 & 0 \\ 0 & 0 & -D^1 \end{pmatrix} \hat{q}$$
(16)

where 
$$\hat{q} = \begin{pmatrix} \hat{u} \\ \hat{v} \\ \hat{p}_1 \end{pmatrix}$$
,  $D^1 = \frac{\partial}{\partial y}$  and  $D^2 = \frac{\partial^2}{\partial y^2}$ .



**Fig. 4.** (a) Profiles of the first mode eigenfunctions  $\hat{u}$  (straight line),  $\hat{v}$  (dashed line) and  $\hat{p}_1$  (dotted line) for Re = 1000; (b) Upstream influence of the first mode for  $Re = 10^3$  (straight line),  $10^4$  (black circle),  $10^5$  (dashed line),  $10^6$  (white square).

We just have now to find the eigenvalues and eigenfunctions of the matrix  $B^{-1}A$ , where :

$$A = \begin{pmatrix} \frac{D^2}{Re} & -U'_0 & 0\\ 0 & -D^1 & 0\\ 0 & 0 & -D^1 \end{pmatrix} \text{ and } B = \begin{pmatrix} U_0 & 0 & 1\\ 1 & 0 & 0\\ 0 & U_0 & 0 \end{pmatrix}$$
(17)

For  $R_e = 1000$ , the first positive eigenvalue found is  $\theta_1 \simeq 2.0441$ . The figure 4(a) represents the eigenfunctions of this mode. As shown in figure 4(b), by computing this first positive eigenvalue for different Reynolds number ranging from  $10^3$  to  $10^6$ , we obtain that the corresponding upstream influence  $\Delta = O(R_e^{1/7})$  as in the analysis of the section 5.1.

# **6** Results

Both the order analysis of section 5.1 and the eigenmode analysis of section 5.2 show that  $\Delta = O(R_e^{1/7})$ . We now compute the flow field using the GIBL model described in section 4 for different accident types at x = 0.

First, we have considered a straight channel connected at x = 0 to a curved channel of constant curvature. The figures 5 (a) and (b) represent the median curved length evolution of  $V(X, \eta_c)$  for, respectively, a fixed  $\delta = 0.2$  at different Reynolds numbers, and a fixed  $R_e = 1000$  at different wall curvature. These two results confirm that  $\Delta = O(R_e^{1/7})$ .



**Fig. 5.** (ii) case: straight channel connected at x = 0 to a curved channel of constant curvature; (a)  $\delta = 0.2$ ,  $R_e$  from 100 to 10000; (b)  $R_e = 1000$ ,  $\delta$  from 0.1 to 1

Then, we have considered an asymmetrically perturbed straight channel at x = 0 with L = 4H and  $h_u = h_l = 0.3$ . The figure 6 represents the streamwise evolution of  $V(x, \eta_c)$ , where we recover as previously  $\Delta = O(R_e^{1/7})$ .



**Fig. 6.** (i) case: straight channel perturbed at x = 0 with L = 4H,  $h_u = h_l = 0.3$ ; *x*-evolution of the adimensionnalized  $V(x, \eta_c)$  for different values of  $R_e$  (from 100 to 10000).

Finally, we have compared the Navier-Stokes, GIBL and eigenmode analysis results. As shown in figure 7, all the results are very similar.



**Fig. 7.** NS, GIBL, first eigenmode comparaison;  $R_e = 1000$ ,  $\delta = 0.2$ ; left: *u* profile; middle: *v* profile; right:  $p_1$  profile.

# 7 Conclusion

The non linear upstream effect on a channel flow submitted to asymmetric disturbance has been studied. By using three differents tools, a new asymptotic approach called Successive Complementary Expansions Method (SCEM) with generalized asymptotic expansions, a modal analysis and direct Navier-Stokes computations, we found that the upstream influence length  $\Delta = O(R_e^{1/7})$ .

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