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CMC Design Approach for Cryogenic Injector Heads of Rocket Thrust Chambers

M. Ortelt, H. Hald, A. Herbertz, W. Rotaermel DLR - German Aerospace Center Institute of Structures and Design, Stuttgart



Outline

- Motivation
- ➤ CMC Injector Concept Approach
 - ✓ Functional Aspects
 - → Design Features
 - → Structural Analysis
- - → Spray Evaluation
 - → Flow Calibration / Throttling Potential
 - → Laser Cut System
- ✓ Summary and Outlook

→ (References)



Motivation

Primary Idea: Effusion Cooled CMC-Combustion Chamber



- Inner liner made of porous carbon based CMC 7
- Effusion cooling through porous CMC-wall 7
- High system efficiency by achieving maximum reduced coolant mass flow ratio 7
- Prevention of thermo-chemical wall attack 7

\rightarrow Assumption: Highly optimized combustion



Motivation

CMC-Injector-Head in Conjunction with CMC Combustion Chamber



Technological Goals

- → Rapid spray forming
- Use of highly permeable and high temperature stable CMC
- Generation of homogeneous LOXdroplet-pattern
- Prevention of large LOXdroplets
- CMC-wall-protection against LOX-agression
- Prevention of Combustion instabilities
- ✓ Permit of cold injection



Motivation

Design Simplification



Design Principle of the Cone Injector. Stack of similar conical shells, developed at the Institute of Structures and Design.

- Easy segment stack by replication of similar injector elements
- Exploitation of advantageous thermomechanical CMC material properties
- Prevention of mechanical extremetolerance-requirement



Functional Aspects (1)

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	combustion chamber C with coaxial injector		CMC thrust chamber with 'cone-injector'		Co
d _{LOX} [m]	0.004		0.031		
T _g [K]	125		125		
p _C [MPa]	3	6	3	6	
$ ho_g(T_g, p_C) [kg/m^3]$	5.7338	11.235	5.7338	11.235	an
U _g [m/sec]	270	310	270	310	
T _f [K]	127		127		
$\sigma_{f}(T_{f}) [N/m]$	0.0046507		0.0046507		
					1
We _g []	3.57E+05	9.29E+05	2.77E+06	7.20E+06	1
	I	1	1	I	'[Rf. 22

Spray-Conditioning

Comparison between

- the resulting Weber numbers with coaxial injection as experienced from a DLR standard combustion chamber
- and the calculated Weber numbers for the 'coneinjector' as expected from the operation of the CMC thrust chamber at identical process parameters.

$$We_{g} = \frac{\rho_{g} * U_{g}^{2} * L}{\sigma_{f}}$$



CMC Injector Concept

Functional Aspects (2)

Hydrogen Injection

area,

In. 2

4,62

0.77

0

01-29

1,41

Δ

190

170

Hydrogen in jection temperature, ⁸ 130 130

70

50



Combustion Stability

200 [Rf. 1] Oxygen Stable injection, combustion Stable area, Investigation 180 combustion In.2 0 Unstable on co-axial 0 0.89 Unstable 1.33 combustion combustion 8 160 Solid symbols denote $\overline{\Delta}$ injector 2.17 0 stable combustion Hydrogen in Jection temperature, 90 0 00 elements đ 140 120 ETE: 100 Stable combustion Δ Unstable combustion 60 5 6 8 8 Oxidant-fuel ratio, O/F Oxidant-fuel ratio, O/F **LOX-Injection Area** LH2-Injection Area Stable Injection Temperature **Stable Injection Temperature**

 \rightarrow



Functional Aspects (3)



Permeable Materials



CMC-Sample of WPS-P50. Material matrix showing 50% porosity.

Grinding pattern. The medium pore based on Nitivy fibers and diluted oxidic diameter perpendicular to the fiber plies amounts about 200 µm.



Design Features



Longitudinal Section

First experimental investigations:

 → Use of H2O and GN2 for replacement of oxidizer LOX and fuel LH2



Structural Analysis (1) \rightarrow

Material Characteristics

	In Plane	Perpendicular	Steel 7,9 [g/cm ³]
Young's-Modulus	16 [kN/mm ²]	3 [kN/mm ²]	200 [kN/mm ²]
Schear-Modulus	1,3 [kN/mm ²]	6,2 [kN/mm ²]	77 [kN/mm ²]
Poisson-Ratio	0,02	0,03	0,3
Thermal Conductivity	0,7 – 0,8 [W/mK]	0,26 – 0,32 [W/mK]	15 [W/mK]
Thermal Expansion	1*10-5 [1/K]	2,4*10-5 [1/K]	1,8*10-5 [1/K]

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Structural Analysis (2)

Thermal Assumption





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Structural Analysis (3)

Thermal Simulation



Thermal boundary conditions during start-up phase at the overall system.

Thermal boundary conditions during steady-state phase at the faceplate.

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Structural Analysis (4)



Mechanical Simulation (1)



V. Mises stresses at 10 bar chamber pressure.

CMC-displacement at 10 bar chamber pressure non-critical.

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Structural Analysis (5)



Mechanical Simulation (2)



Displacement of inner CMC element. Critical at 60 bar pressure loss caused by low clamping angle. Approach for structural improvement. Design change.



Tests at M3 Test Bench (DLR-Lampoldshausen) → Spray Evaluation (1)



Qualitative view of spray formation using Water and Nitrogen.

Nitrogen sweeps over the readily available water and the sharp shear stream forms a finely mixed spray.



Tests at M3 Test Bench (DLR-Lampoldshausen) → Spray Evaluation (2)



Full flow *Water without Nitrogen*. Mass flow amounts about 900 g/sec.

Current design specifics even enable high mass flow rates to feed a 50 mm chamber (relevant DLR research level) running 60 bars with the 30 mm INJEX01.



Full flow Water + Nitrogen. Overall mass flow about 900 g/sec.



Partial flow Water + Nitrogen. Inner circle only in operation.



Spray Tests at M3 Test Bench



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Flow Calibration N2

Nitrogen flow calibration. Linear slopes.

Spray Tests at M3 Test Bench \rightarrow Flow Calibration H2O

H2O-Flow-Calibration INJEX01 - 20.04.2010 (PIT600-CMC)



in der Helmholtz-Gemeinschaft

Throttling Potential



Half section of demonstrator INJEX01.

Examples of Discrete Throttling Mechanisms

- ➤ Connection of dircrete flow circles (inner/outer)
- → Geometrical gap adjustment
- Discrete reduction of mass flow by valves

→ Expected throttling potential: Mass flow reduction down to 20 % conceivable, while retaining good spray-patterns!



Laser Measurement



Check runs of the laser cut measurement system.

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Summary and Outlook

Summary

- ➤ New injector concept + functional features presented
- → Structural integrity demonstrated
- ➤ Flow characteristics adjusted at conditions of 50 mm chamber (60 bars)
- ➤ Preliminary droplet investigations conducted

Outlook

- ➤ Further tests using cryogenic substitute media (LN2 + He)
- → Improved laser tests → droplet distribution + droplet speed
- ➤ First ignition tests + short hot runs at P6.1 test bench



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