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# CFD Modeling of Tapered Hole Microperforated Panels

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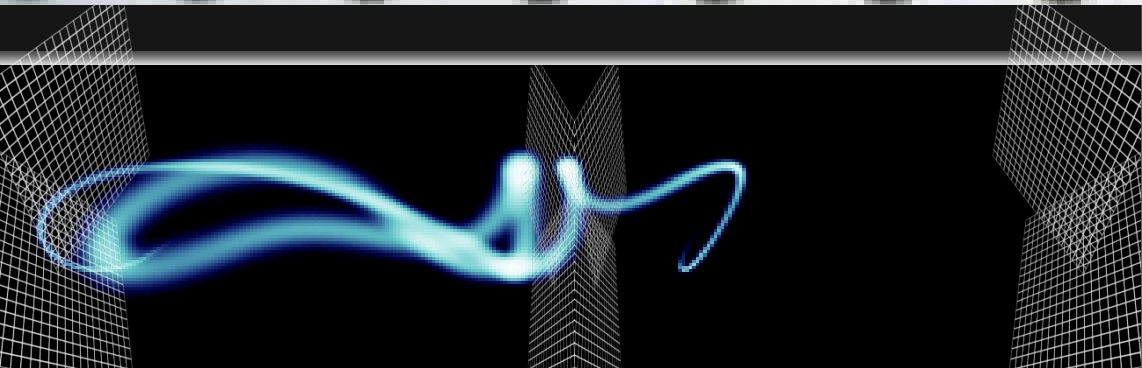
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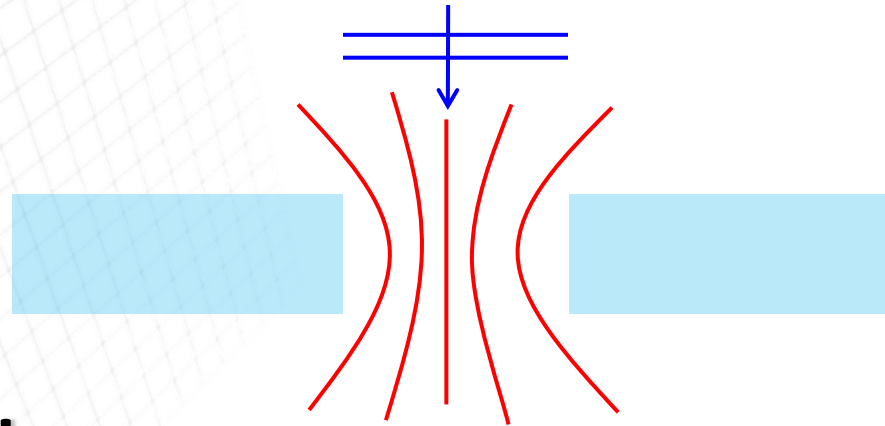
**CFD Modeling of  
Tapered Hole  
Microperforated Panels**

*Nicholas Kim and J. Stuart Bolton  
Ray W. Herrick Labs  
Purdue University*

*Internoise 2012  
Aug 21, 2012*



## ❖ Microperforated material

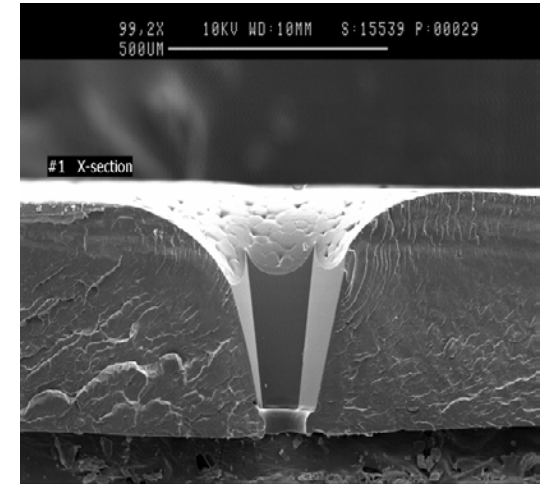
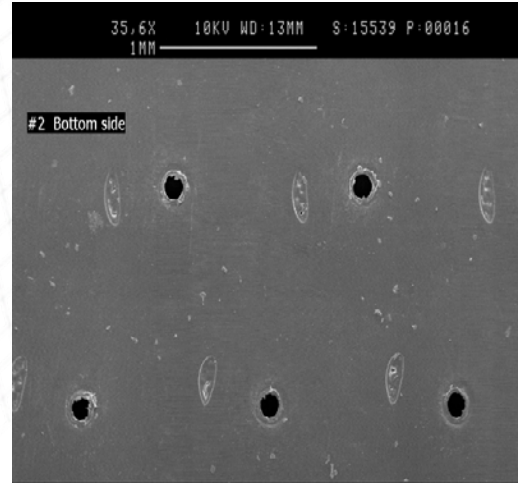
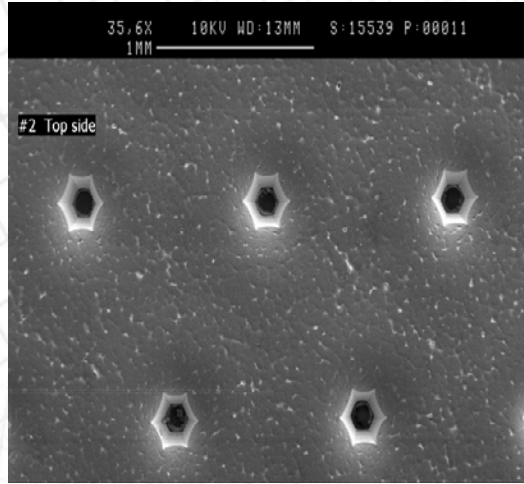


## ❖ Dissipation

- ❖ In hole
- ❖ Along outer surface
- ❖ Within shearing fluid

## ❖ Analytical models

- ❖ Maa (1975) and Guo *et al.* (2008) account for first two

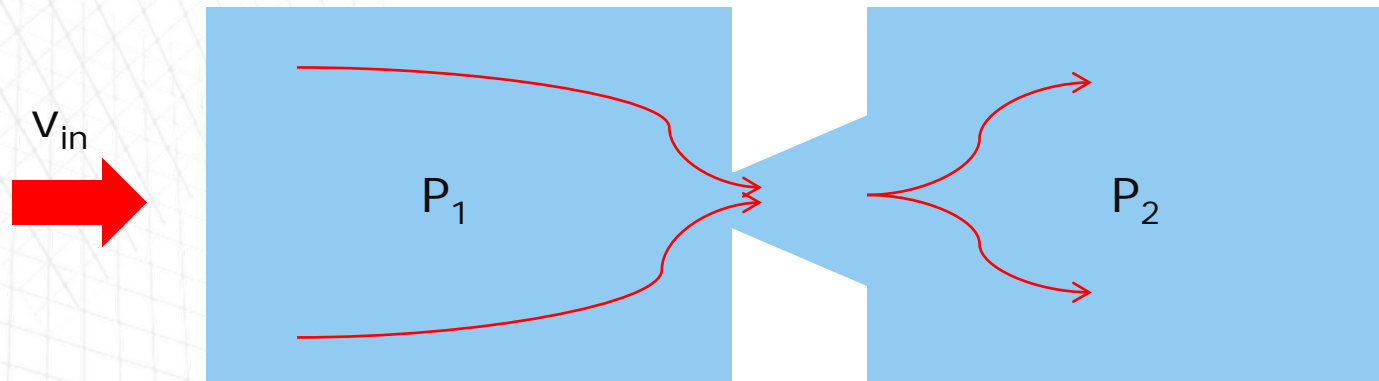


Microperforated panel

Real materials do not have regular hole shapes and so are not suitable for analytical treatment

## Objective

By using computational fluid dynamics approach, calculate dynamic flow resistance for tapered hole microperforated panel considering flow through one hole and compare with existing formulation



$$R_f = \frac{P_1 - P_2}{v_{in}}$$

## Guo Model



Cylinder

Surface

$$R = \left( \text{Re} \left\{ \frac{j\omega t}{\sigma c} \left[ 1 - \frac{2 J_1(k\sqrt{-j})}{k\sqrt{-j} J_0(k\sqrt{-j})} \right]^{-1} \right\} + \frac{\alpha 2 R_s}{\sigma \rho c} \right) \times \rho c$$

$$k = d \sqrt{\frac{\omega \rho_0}{4\eta}} \quad R_s = \frac{\sqrt{2\omega \rho_0 \eta}}{2} \quad \begin{array}{l} \alpha = 2 \quad \text{when smooth end} \\ \alpha = 4 \quad \text{when sharp end} \end{array}$$

### Properties

$t$  : thickness of panel

$d$  : diameter of hole

$\sigma$  : porosity

$\omega$  : frequency

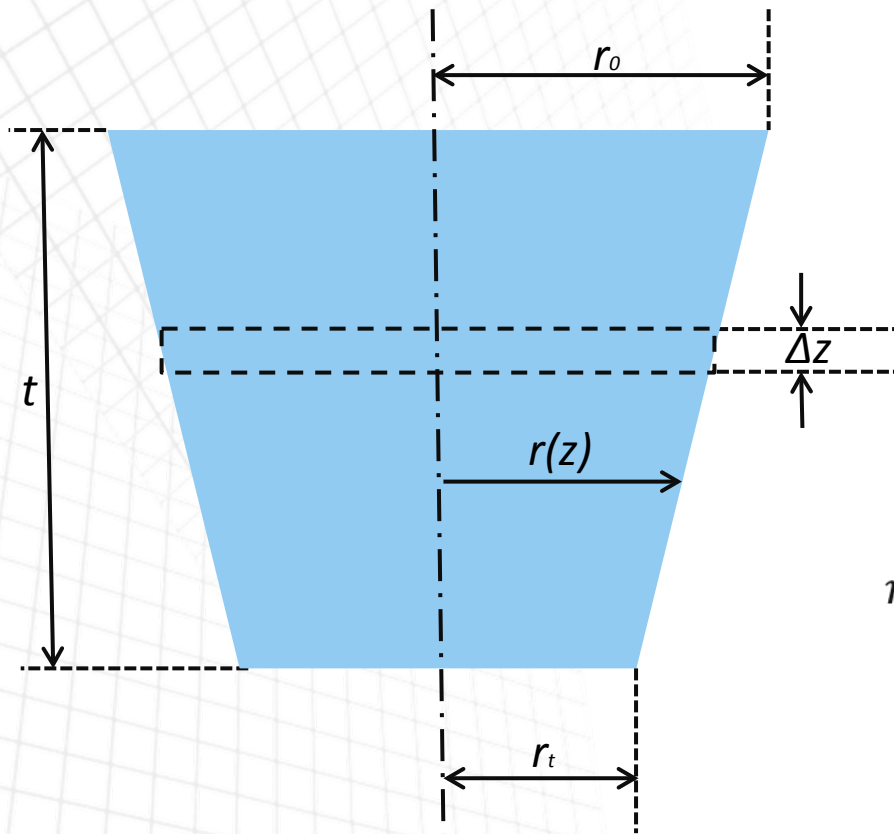
$R_s$  : surface resistance

➔ **Dynamic Flow resistance (R) is function of  $t$ ,  $d$ ,  $\sigma$**

➔ **Note that  $R_s \rightarrow 0$  as  $\omega \rightarrow 0$**



## Analytical Solution (Randeberg, 2000)

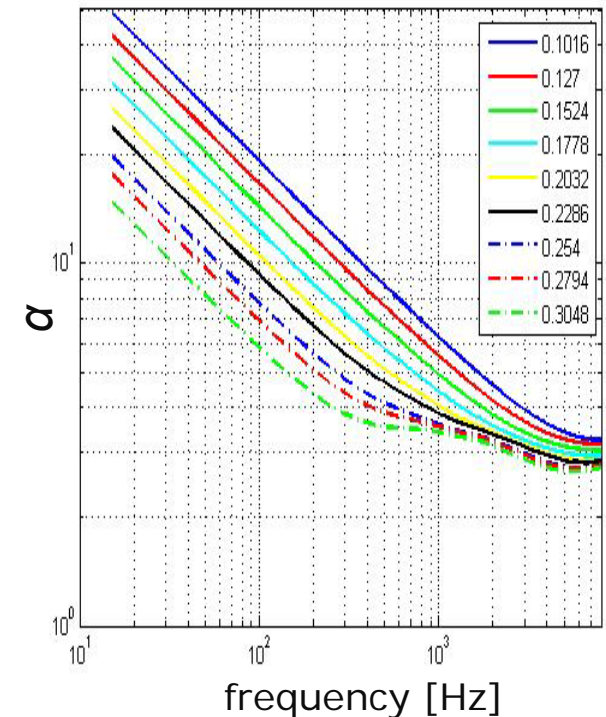
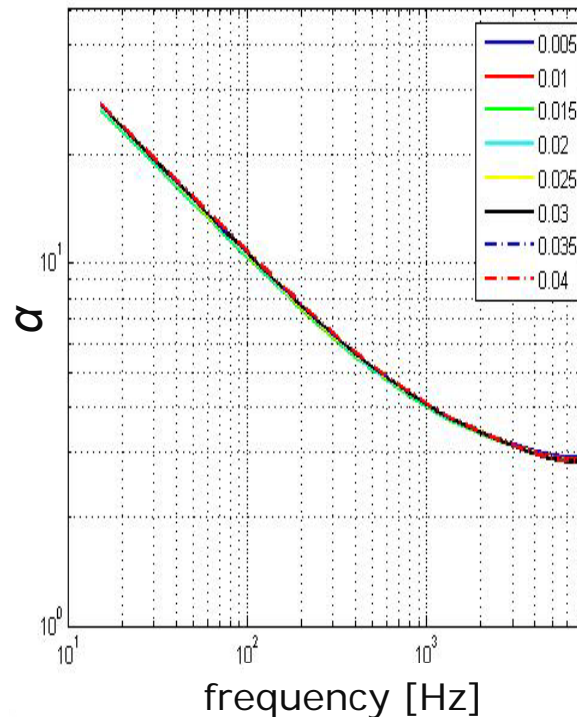
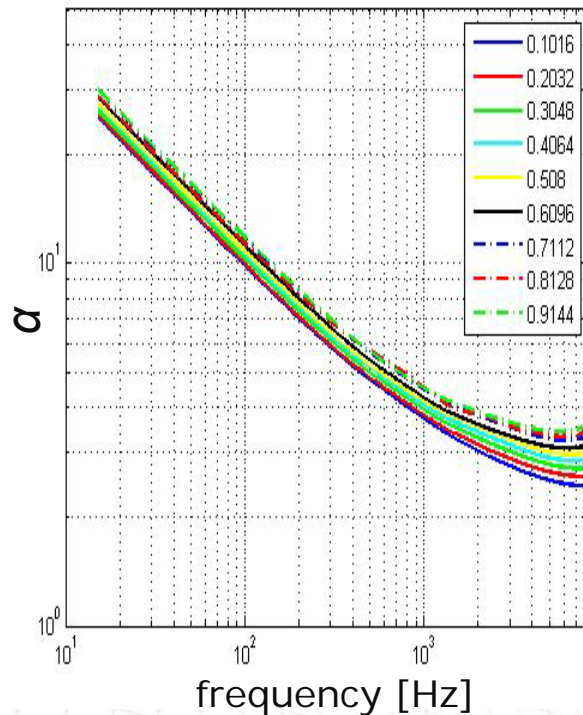


Based on Guo's model, Randeberg used integration method. (used  $\alpha = 4$  for sharp edged)

$$r = Re \left\{ \sum_{n=1}^N \frac{j\omega\Delta z}{\sigma_n c} \left[ 1 - \frac{2}{k_n \sqrt{-j}} \frac{J_1(k_n \sqrt{-j})}{J_0(k_n \sqrt{-j})} \right]^{-1} \right\} + \frac{\alpha R_s}{\sigma_1 \rho c} + \frac{\alpha R_s}{\sigma_N \rho c}$$

# Previous work (sharp-edged hole)

## The value of $\alpha$ vs. Frequency

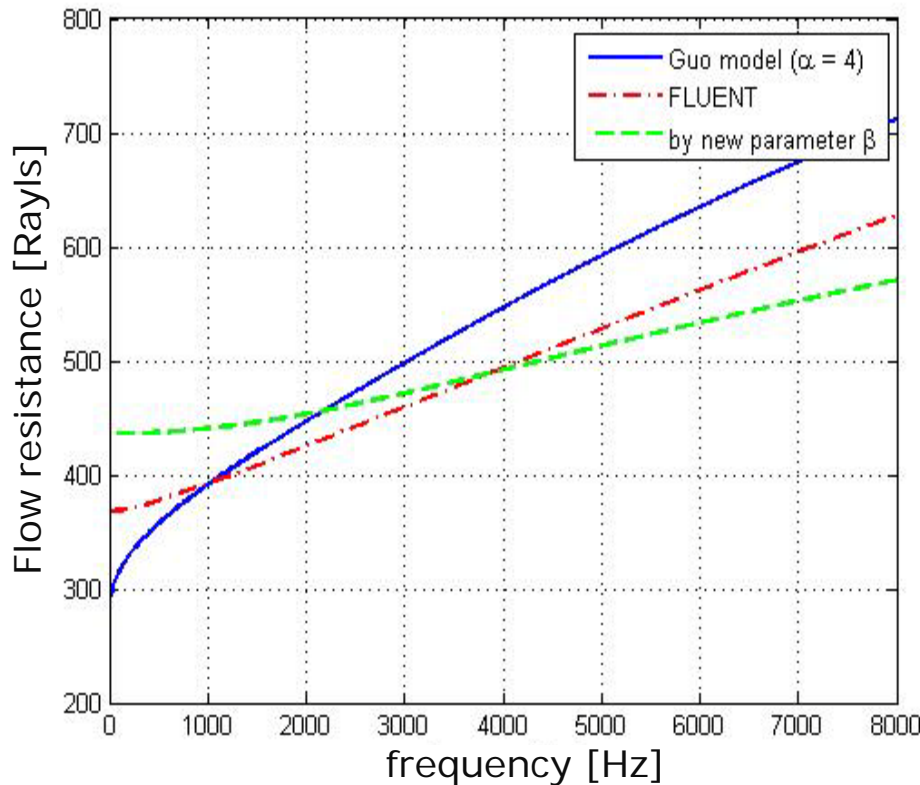


In these graphs, it is shown that  $\alpha$  is a function of frequency, thickness, and hole diameter



# Previous work (sharp-edged hole)

## Flow resistance computed by Fluent vs. $\beta$



$\alpha$ , end correction coefficient in Guo model, is dependent on frequency.

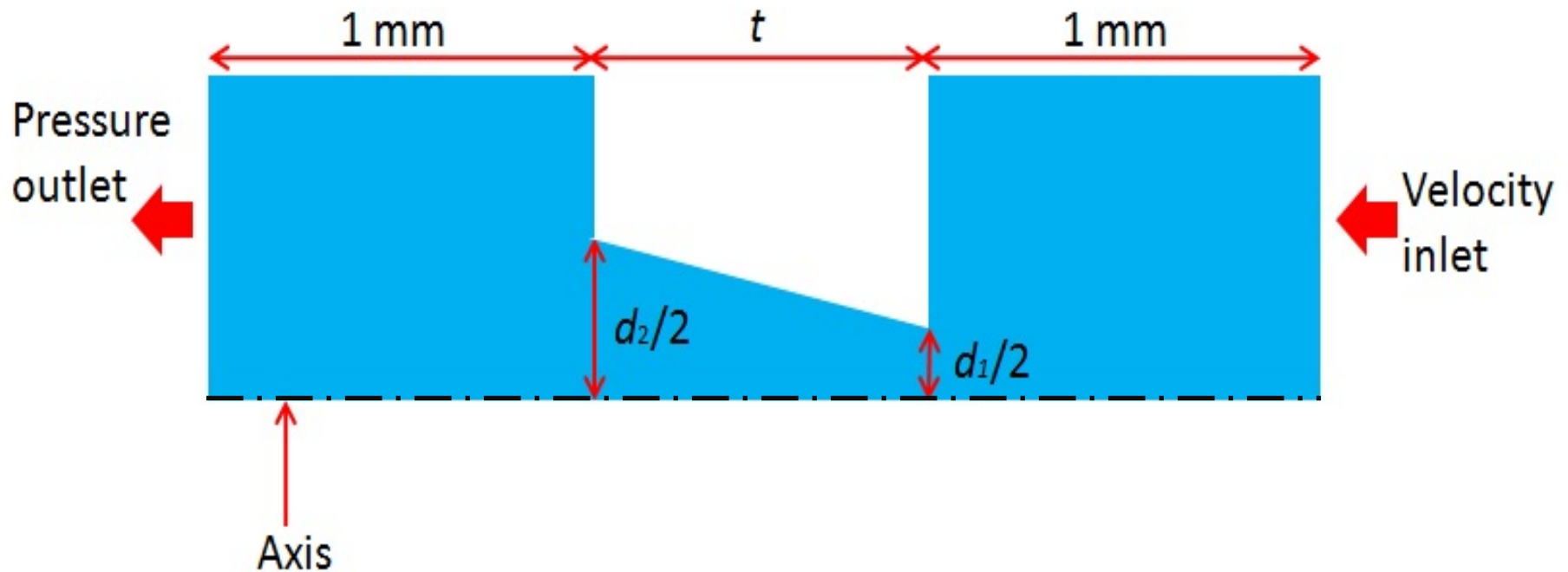
$$\alpha = \beta f^{-0.5}$$

$\beta$  is function of thickness, hole diameter, and porosity.

$$\beta = 16.9 \frac{t}{d} + 152.8$$

(at  $t = 0.4064$  mm,  $d = 0.2032$  mm,  $\sigma = 0.02$ )

## Geometry of CFD model



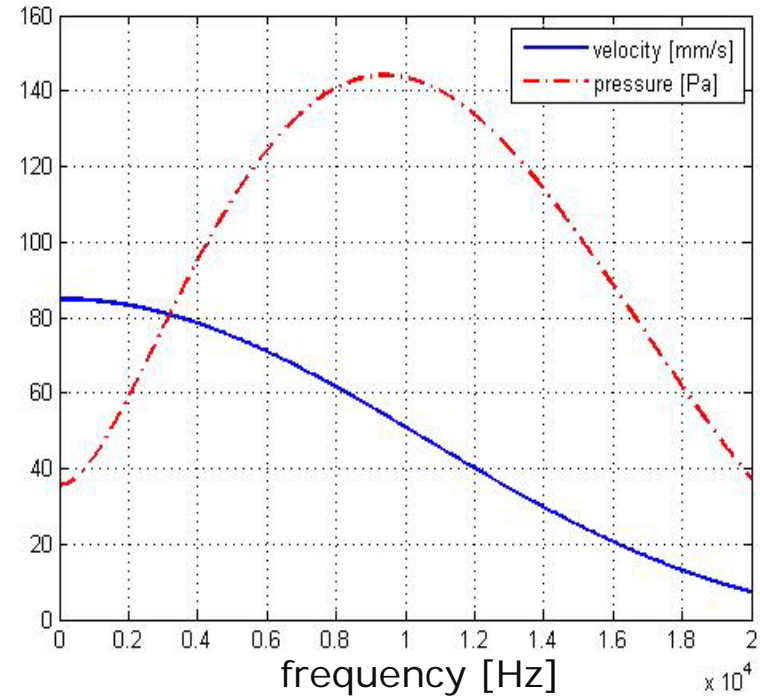
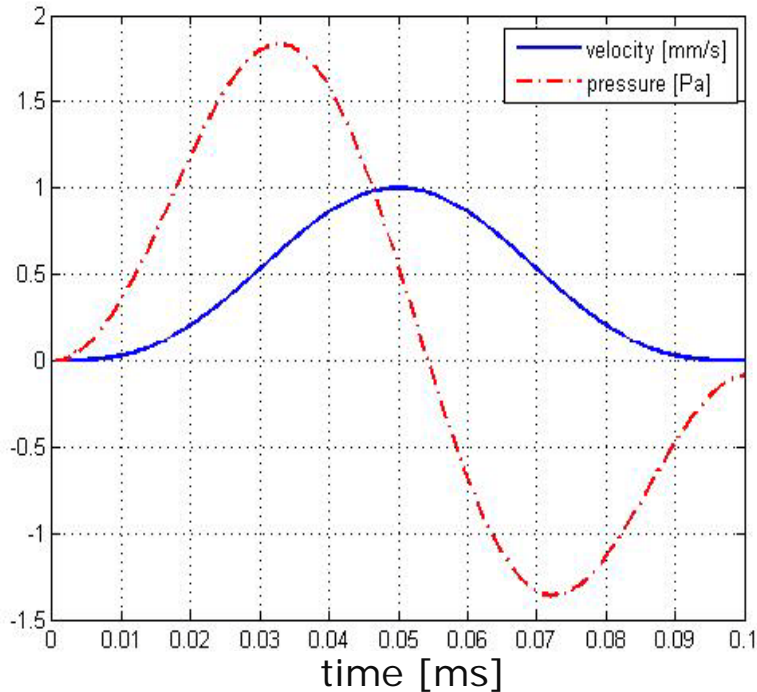
Mesh size : 0.005 mm, calculating time : 3-8 hours

## 36 Cases with 9 different thicknesses

( $t = 0.1016 \text{ mm} - 0.9144 \text{ mm}$ )

	$d_1$ [mm]	$d_2$ [mm]		$d_1$ [mm]	$d_2$ [mm]		$d_1$ [mm]	$d_2$ [mm]
<b>Case1</b>	0.1016	0.127	<b>Case13</b>	0.127	0.254	<b>Case25</b>	0.1778	0.2794
<b>Case2</b>	0.1016	0.1524	<b>Case14</b>	0.127	0.2794	<b>Case26</b>	0.1778	0.3048
<b>Case3</b>	0.1016	0.1778	<b>Case15</b>	0.127	0.3048	<b>Case27</b>	0.2032	0.2286
<b>Case4</b>	0.1016	0.2032	<b>Case16</b>	0.1524	0.1778	<b>Case28</b>	0.2032	0.254
<b>Case5</b>	0.1016	0.2286	<b>Case17</b>	0.1524	0.2032	<b>Case29</b>	0.2032	0.2794
<b>Case6</b>	0.1016	0.254	<b>Case18</b>	0.1524	0.2286	<b>Case30</b>	0.2032	0.3048
<b>Case7</b>	0.1016	0.2794	<b>Case19</b>	0.1524	0.254	<b>Case31</b>	0.2286	0.254
<b>Case8</b>	0.1016	0.3048	<b>Case20</b>	0.1524	0.2794	<b>Case32</b>	0.2286	0.2794
<b>Case9</b>	0.127	0.1524	<b>Case21</b>	0.1524	0.3048	<b>Case33</b>	0.2286	0.3048
<b>Case10</b>	0.127	0.1778	<b>Case22</b>	0.1778	0.2032	<b>Case34</b>	0.254	0.2794
<b>Case11</b>	0.127	0.2032	<b>Case23</b>	0.1778	0.2286	<b>Case35</b>	0.254	0.3048
<b>Case12</b>	0.127	0.2286	<b>Case24</b>	0.1778	0.254	<b>Case36</b>	0.2794	0.3048

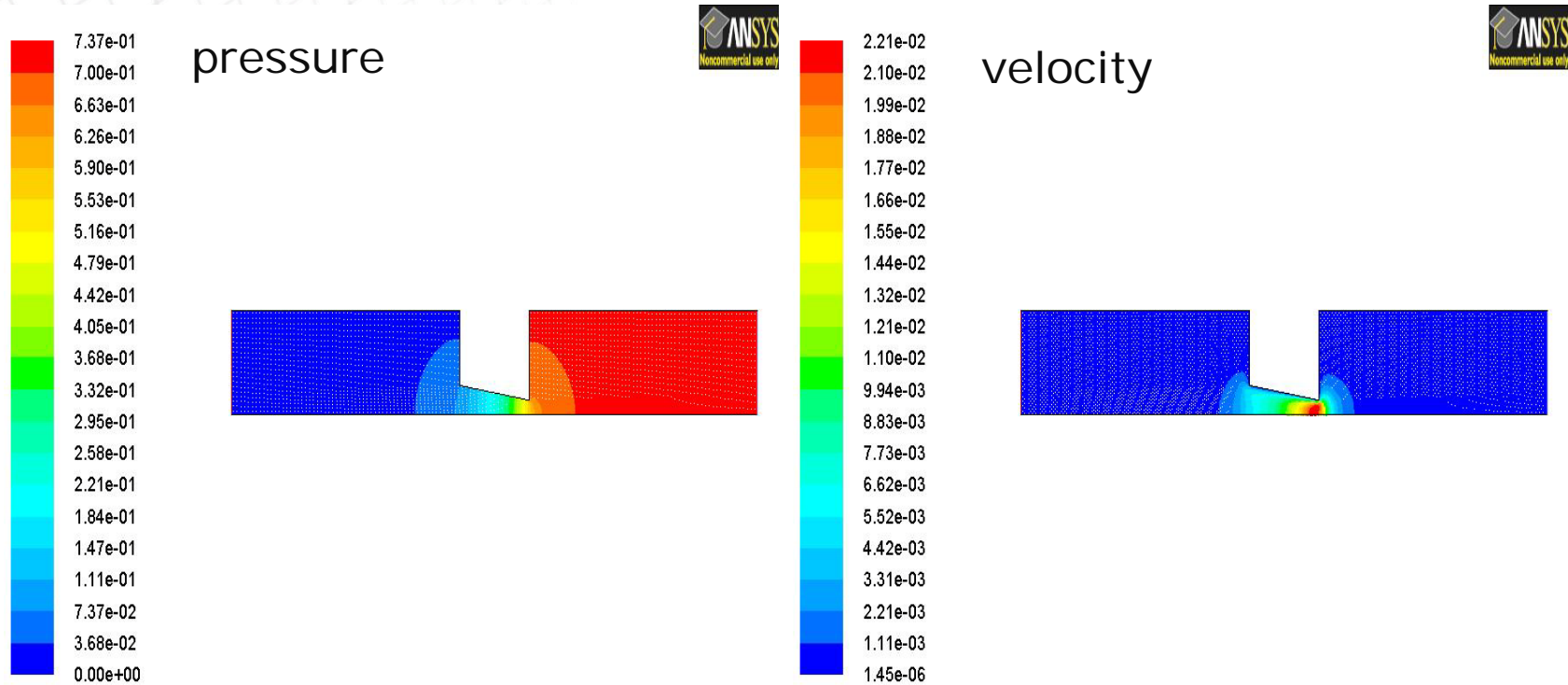
## Input velocity and pressure



**Inlet velocity was chosen to be a Hann windowed, 5 kHz half-sine wave having a maximum value of 1 mm/s in order to cover the frequency range up to 10 kHz**

# Pressure & velocity results from simulation

$t = 0.4064$  mm,  $d_1 = 0.1016$  mm,  $d_2 = 0.2032$  mm,  $\sigma = 0.02$



Contours of Static Pressure (pascal)

Aug 14, 2012  
ANSYS FLUENT 12.1 (2d, dp, pbns, lam)

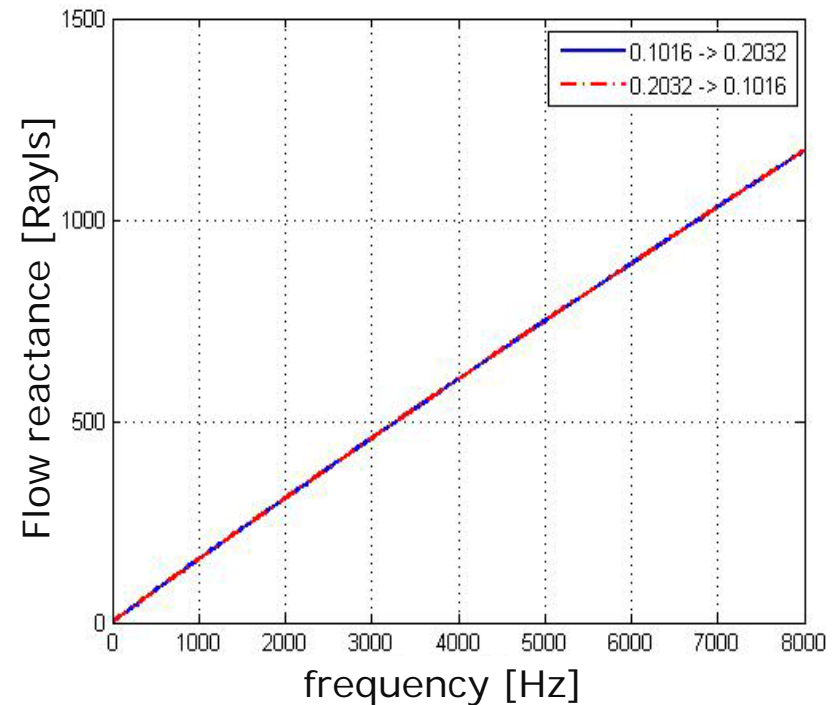
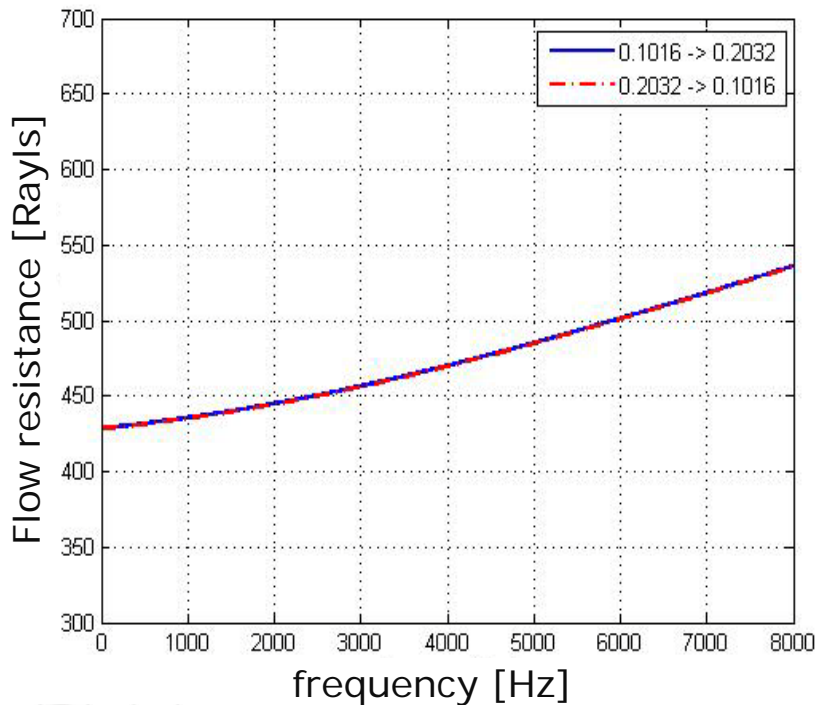
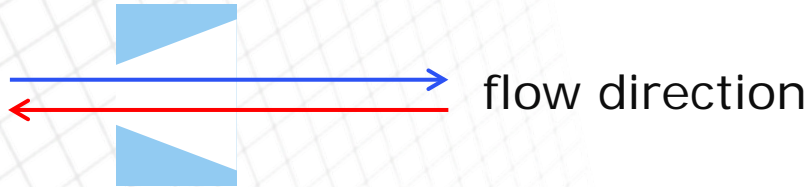
Velocity Vectors Colored By Velocity Magnitude (m/s)

Aug 14, 2012  
ANSYS FLUENT 12.1 (2d, dp, pbns, lam)



## Flow resistance & reactance

( $t = 0.4064$  mm,  $\sigma = 0.02$ ,  $d_1 = 0.1016$  mm,  $d_2 = 0.2032$  mm)

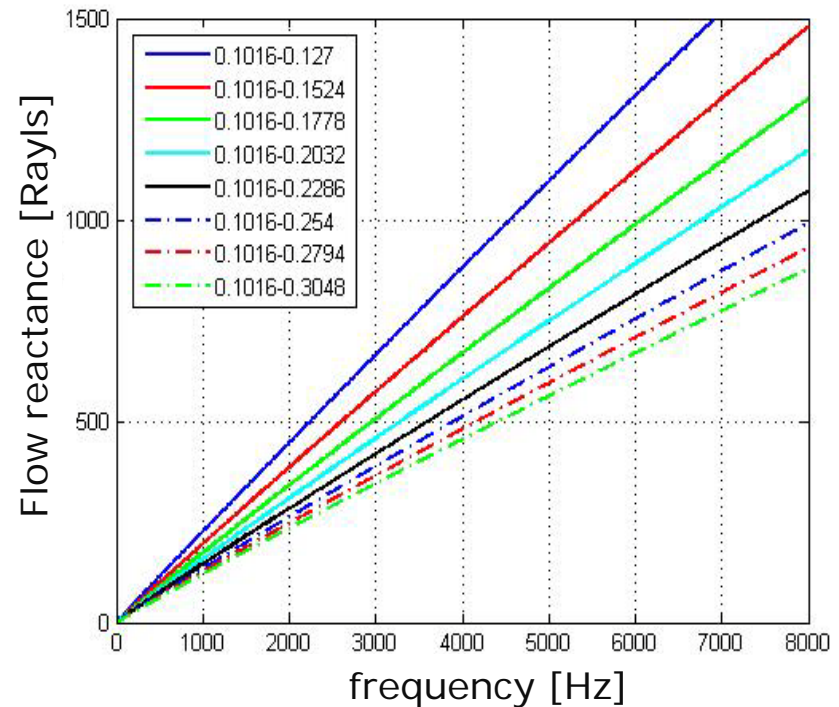
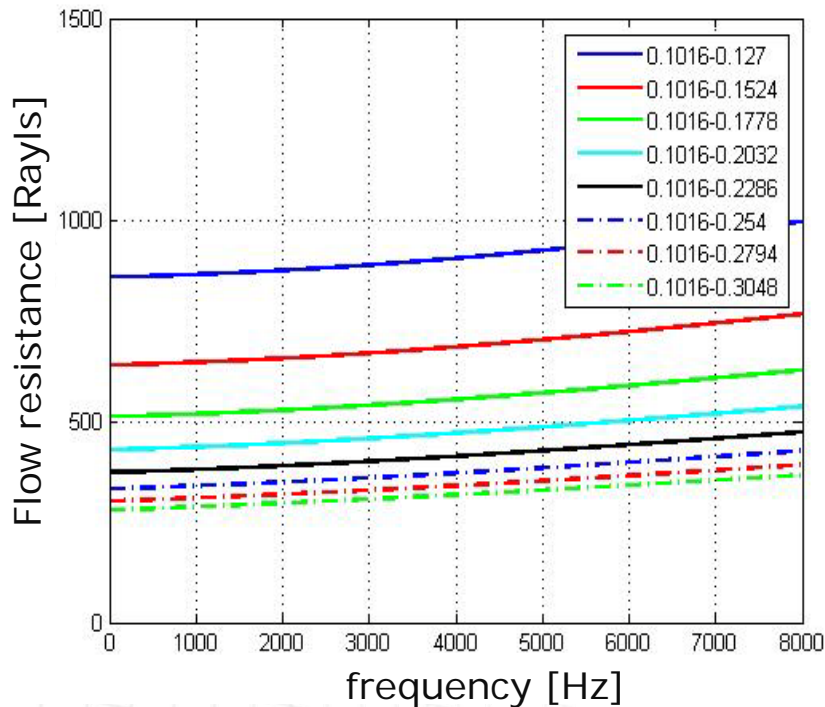


Flow direction does not affect resistance or reactance

# Dynamic flow resistance and reactance

## Fixed diameter of inlet hole

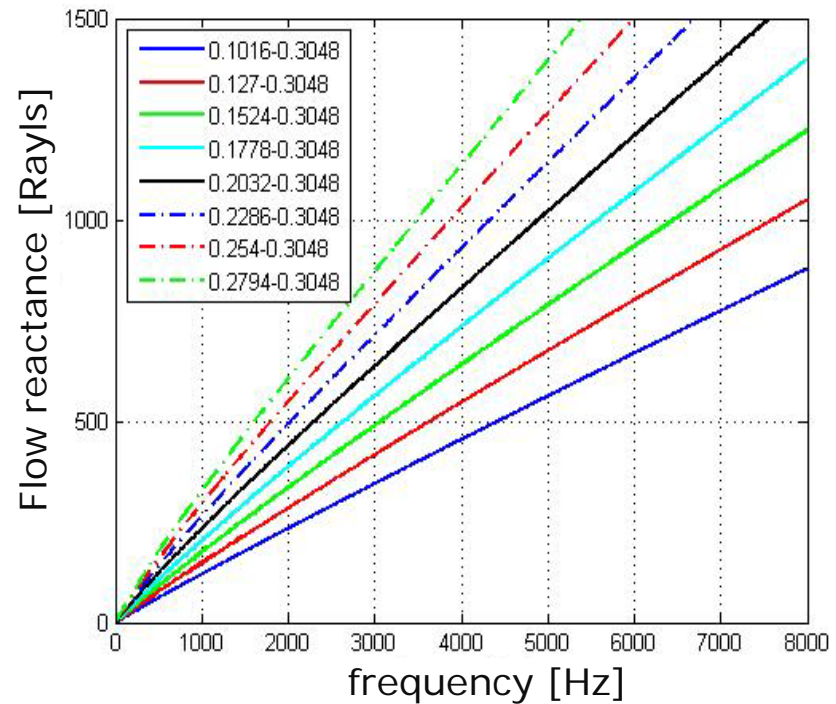
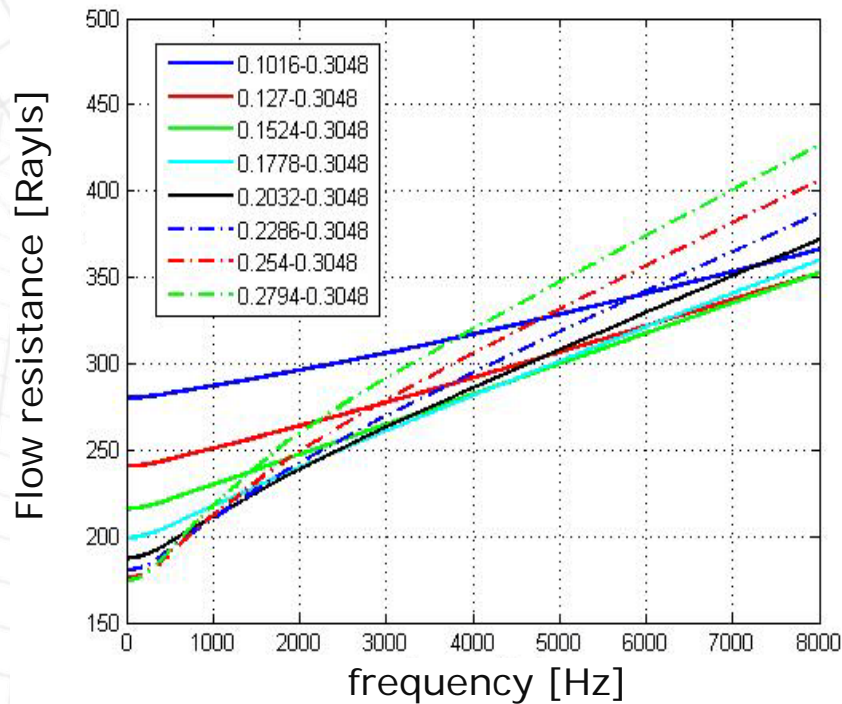
( $t = 0.4064$  mm,  $\sigma = 0.02$ ,  $d_1 = 0.1016$  mm)



# Dynamic flow resistance and reactance

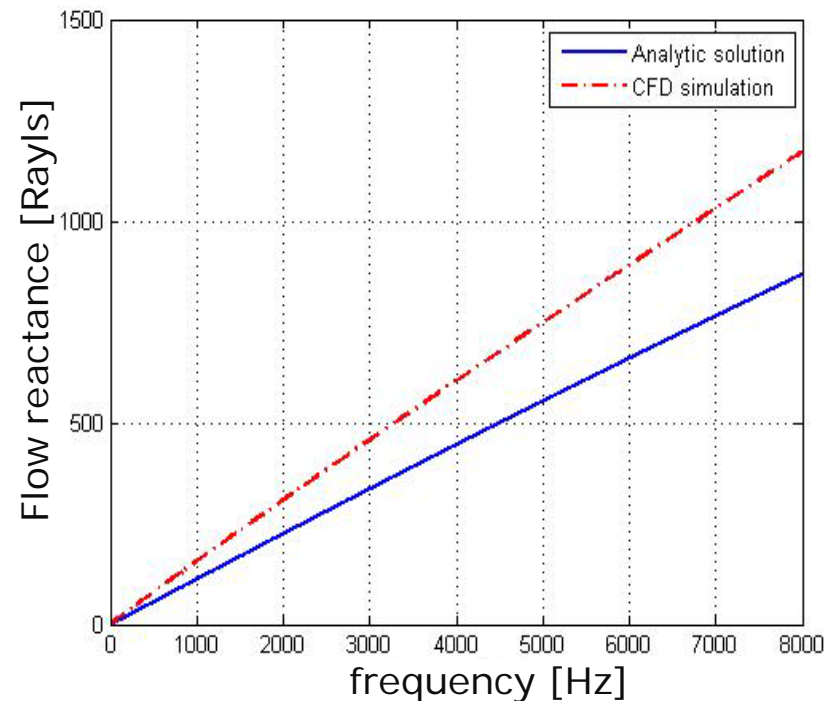
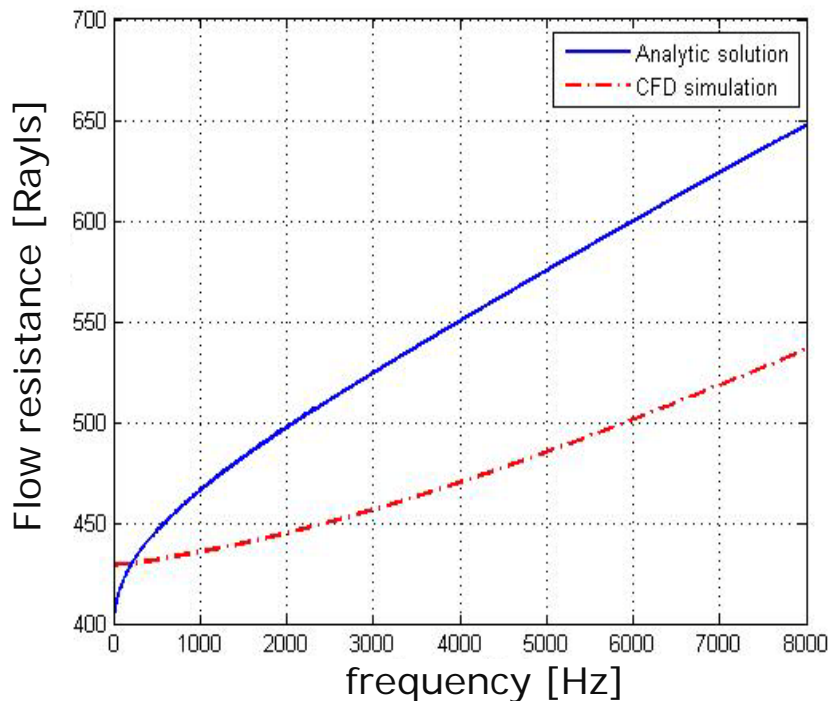
## Fixed diameter of outlet hole

( $t = 0.4064$  mm,  $\sigma = 0.02$ ,  $d_1 = 0.3.48$  mm)



## Flow resistance & reactance

( $t = 0.4064$  mm,  $\sigma = 0.02$ ,  $d_1 = 0.1016$  mm,  $d_2 = 0.2032$  mm)



Make  $\alpha$ , which is defined by Guo *et al.*, function of frequency to fit with CFD results



## Error correction factor $\alpha$

In the previous work (sharp-edged cylindrical hole)

$$\alpha = \left(16.9 \frac{t}{d} + 152.8\right) f^{-0.5}$$

$$\alpha = \beta f^{-0.5}$$

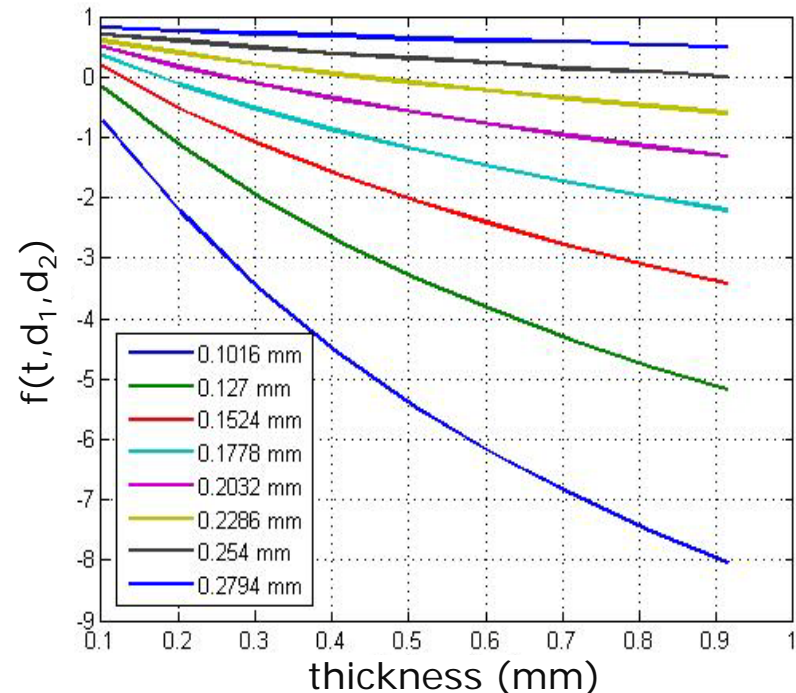
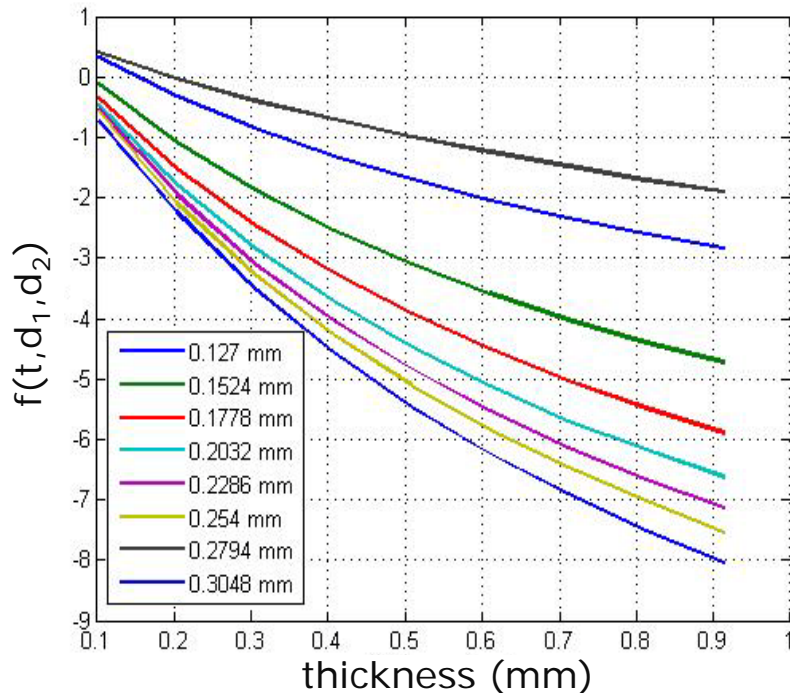
Make  $\beta$  a function as thickness, inlet diameter, and outlet diameter.

$$\beta = \left(16.9 \frac{t}{d_1} + 153\right) f(t, d_1, d_2)$$



# Dynamic flow resistance and reactance

Define  $f(t, d_1, d_2)$



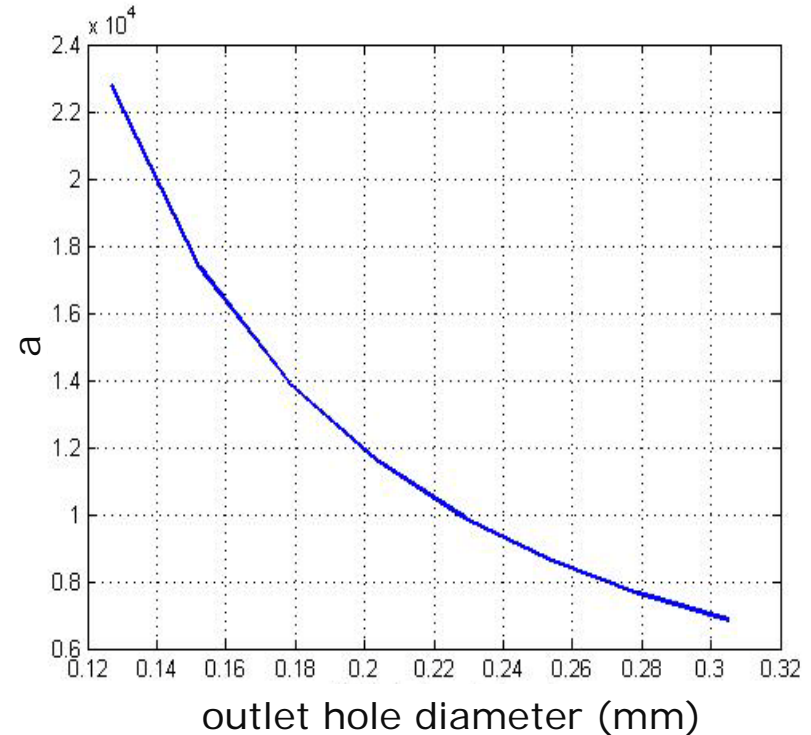
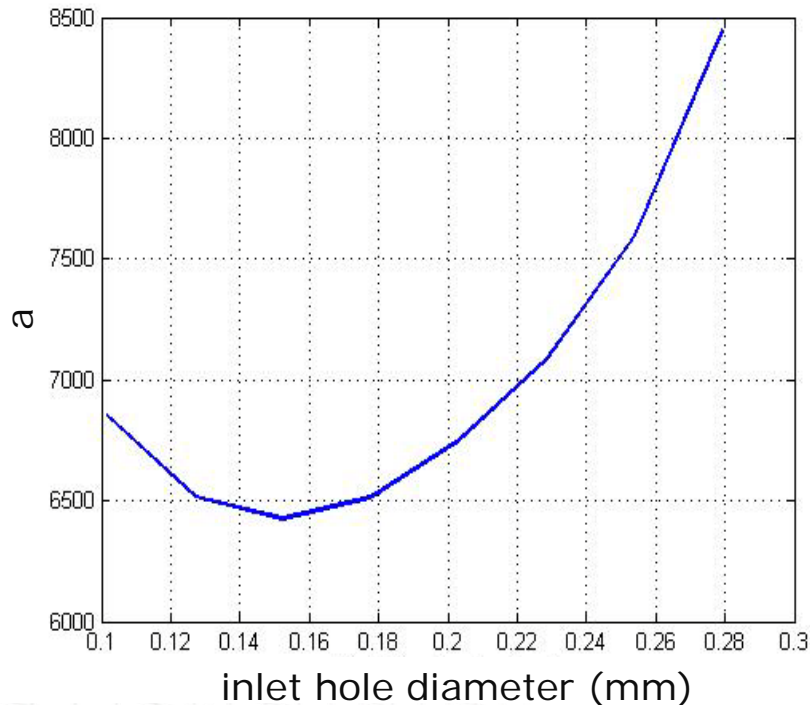
(left is fixed by  $d_1 = 0.1016$  mm, and right is fixed by  $d_2 = 0.3048$  mm)

Inversely proportional to thickness and almost linear

$$\rightarrow f(t, d_1, d_2) = a \left(1 - \frac{d_2}{d_1}\right) t + 1$$

# The value of $a$ vs. Frequency

Define slope  $a$

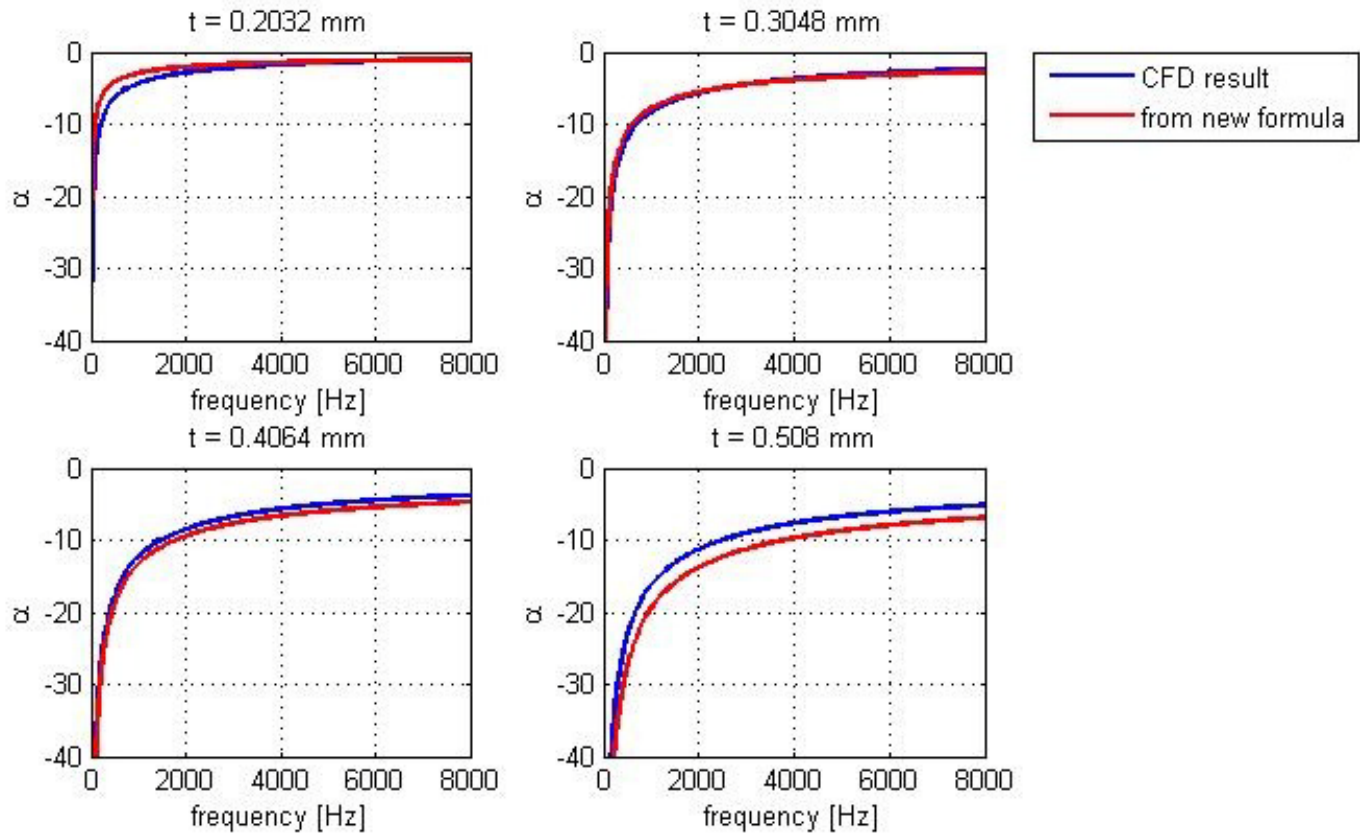


By second order Newton interpolation

$$a = \left( 6.66 \left( \frac{d_1}{d_2} \right)^2 - 7.07 \left( \frac{d_1}{d_2} \right) + 3.06 \right) \times 10^4$$

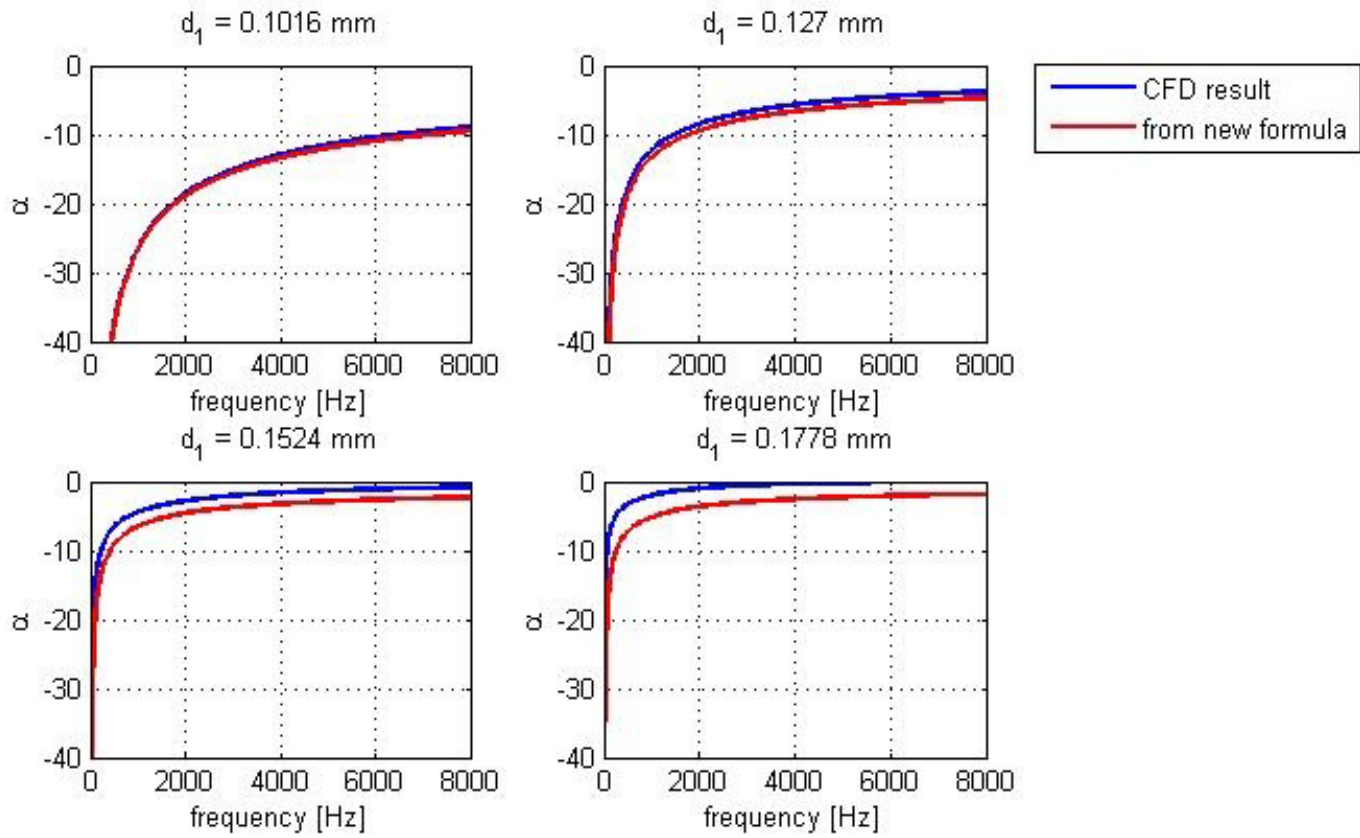
# The value of $\alpha$ vs. Frequency

$$d_1 = 0.1016 \text{ mm}, d_2 = 0.2032 \text{ mm}, \sigma = 0.02$$



# The value of $\alpha$ vs. Frequency

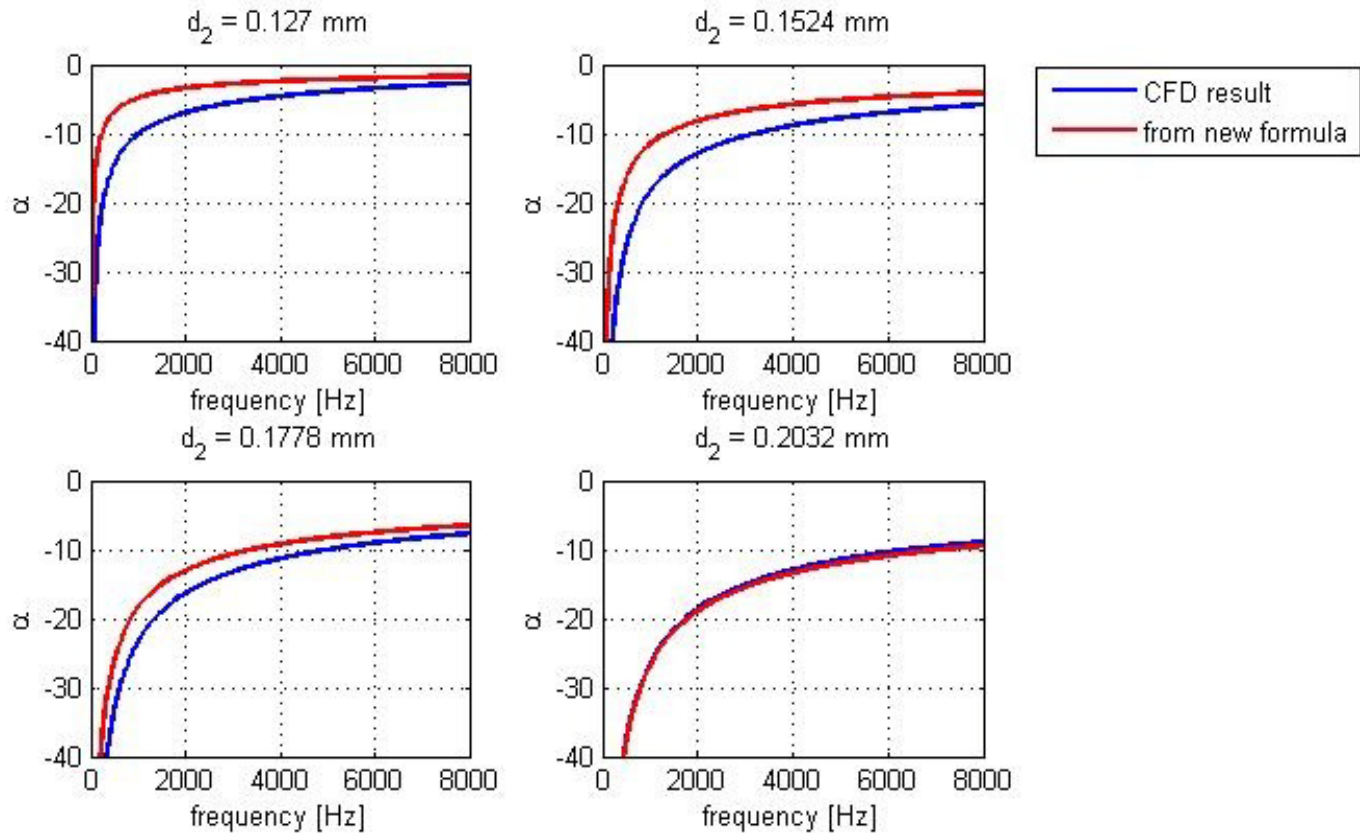
$t = 0.4064$  mm,  $d_2 = 0.2032$  mm,  $\sigma = 0.02$





# The value of $\alpha$ vs. Frequency

$t = 0.4064$  mm,  $d_1 = 0.1016$  mm,  $\sigma = 0.02$





- ❖ By changing the definition of  $\alpha$ , which is defined by Guo *et al.*, accuracy can be improved
- ❖ By making  $\beta$  a function of thickness, inlet hole diameter, and outlet hole diameter (as below), we can define dynamic flow resistance for any tapered hole.

$$\beta = \left\{ \left( 6.66 \left( \frac{d_1}{d_2} \right)^2 - 7.07 \left( \frac{d_1}{d_2} \right) + 3.06 \right) \times 10^4 \times \left( 1 - \frac{d_2}{d_1} \right) t + 1 \right\} \left( 16.9 \frac{t}{d_1} + 152.8 \right)$$

- ❖ Future : Make complete definition of  $\alpha$  and an examination of the effect of square or slit hole geometry

**Thanks to :**

- **Thomas Herdtle of 3M Corporation, St. Paul, Minnesota, for his useful, practical advice at an early stage of this work.**